

# **A model study of wind turbine interference**

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- ◆ **Background**
- ◆ **Model and measurements**
- ◆ **Effect of turbine operating condition**
- ◆ **Yaw effects**
- ◆ **Conclusions**

- Turbine interaction reduces power output and increases dynamic loads
- Wake structure depends on turbine operating conditions. Is it always best to operate at turbine peak performance?
- Wake may be deflected by yawing the turbine. How much power is gained or lost by yawing?



*In wind parks, turbines interact!*

### Main purpose of investigation:

Measure turbine interaction under controlled laboratory conditions

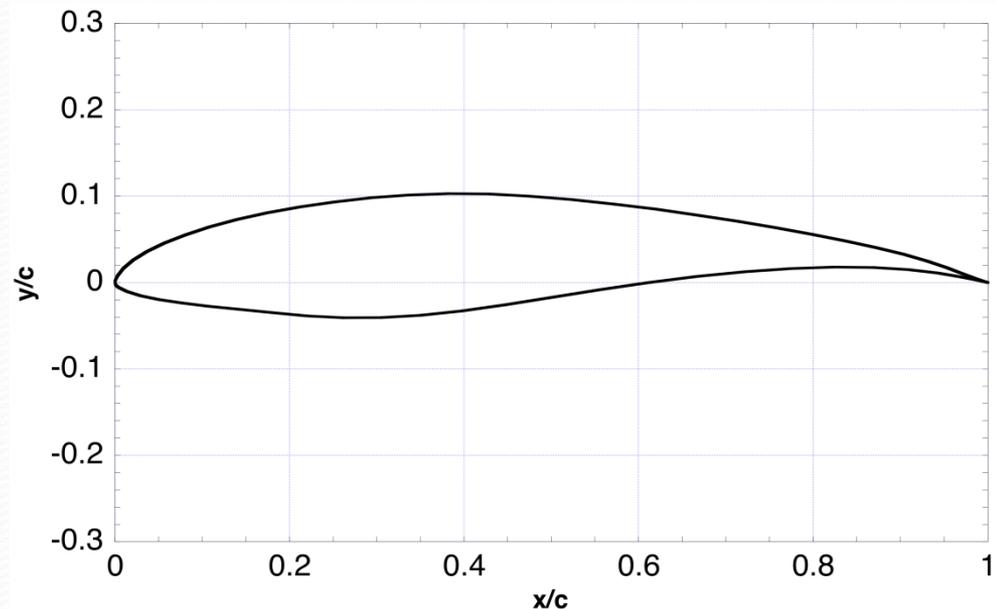
- Model turbine designed using standard Blade Element Momentum theory
- Rotor diameter  $D=0.9\text{m}$ . Design tip speed ratio,  $\lambda=6$
- Wind tunnel test section: Crosssection= $2 \times 2.7\text{m}$ , total length= $12\text{m}$
- Power predictions performed with BEM and CFD (Fluent) software



## Airfoil: NREL S826 14% thickness

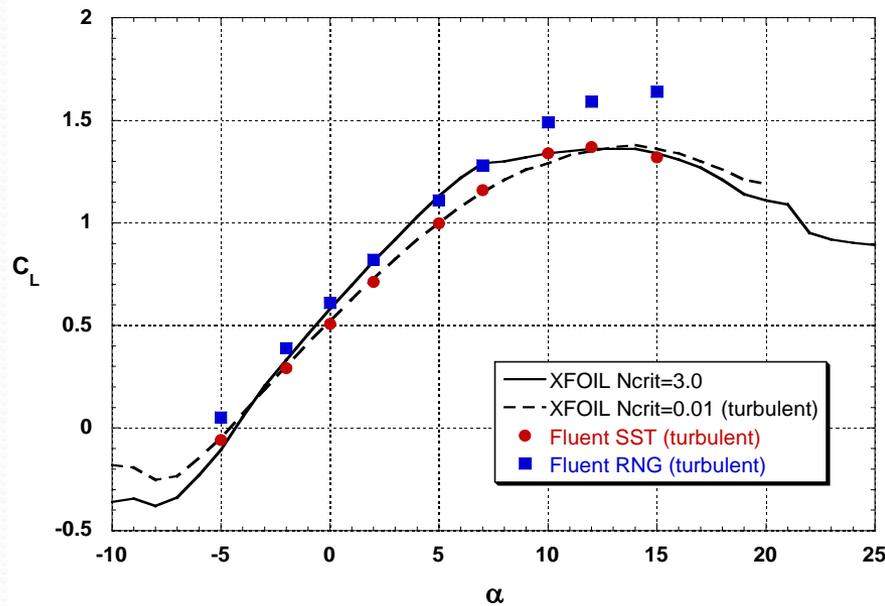
### Characteristics:

- Gentle separation due to trailing edge ramp
- Rapid transition on suction side due to small radius of curvature
- Low sensitivity to surface roughness
- Strong separation on lower side at negative angles of attack

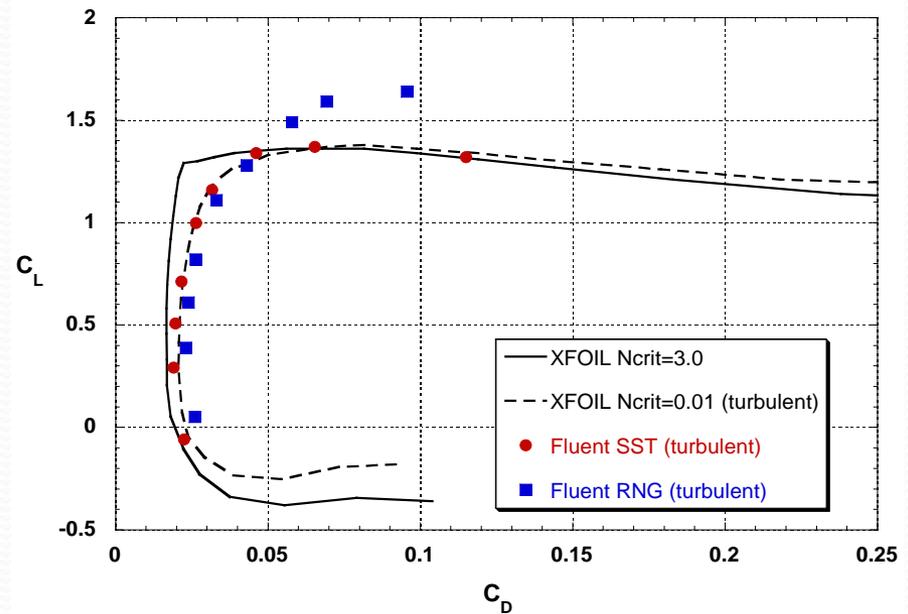


## 2D predictions of S826 performance

Fully turbulent XFOIL predictions agree well with  $k-\omega$  SST



$C_L$  vs  $\alpha$

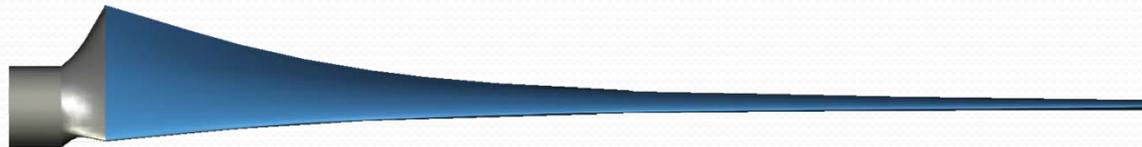


$C_L$  vs  $C_D$

## Standard Blade Element Momentum theory gives blade geometry

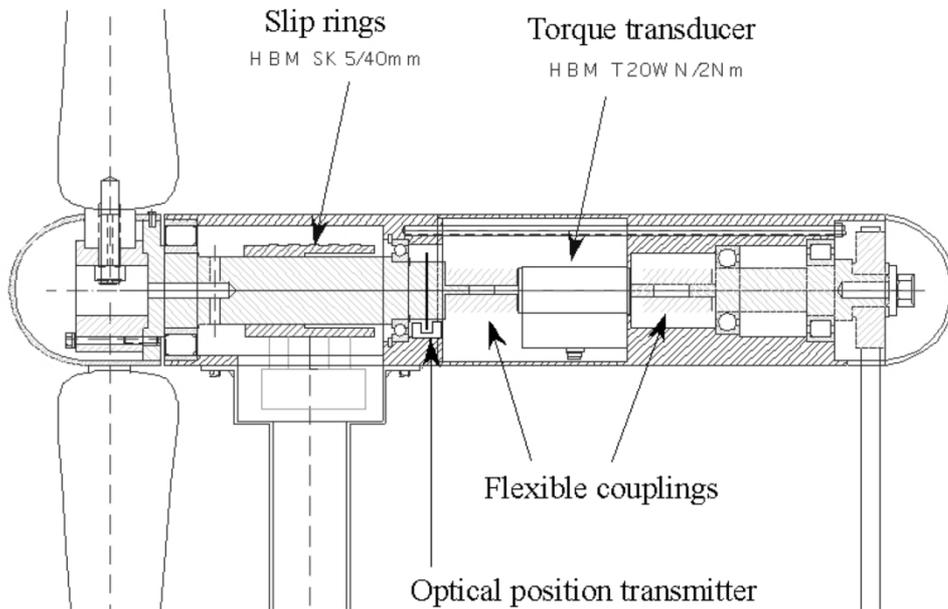


**View in streamwise direction**

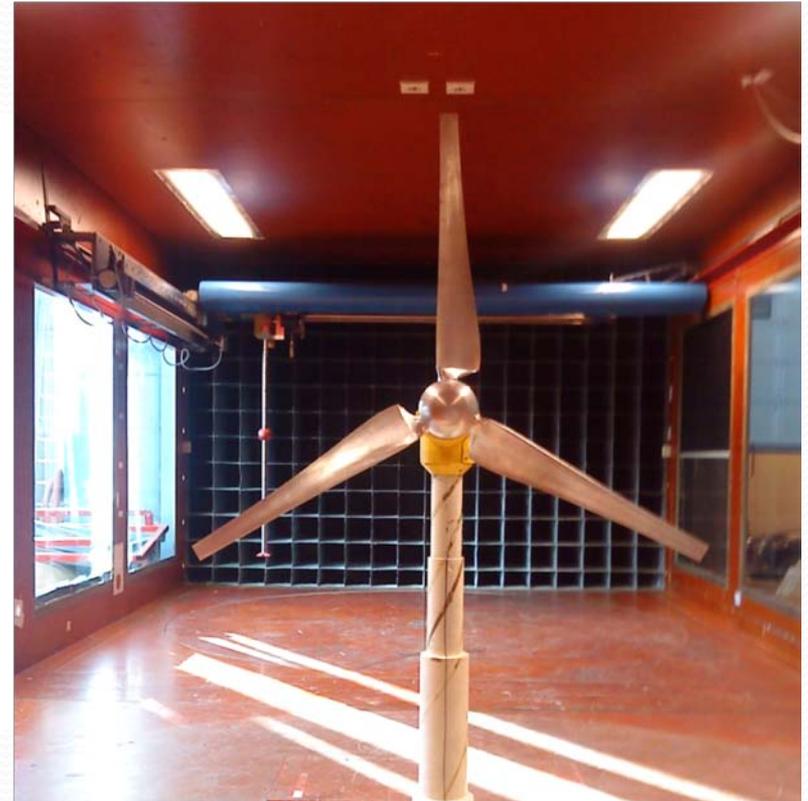


**View in plane of rotation**

# Model and measurement systems

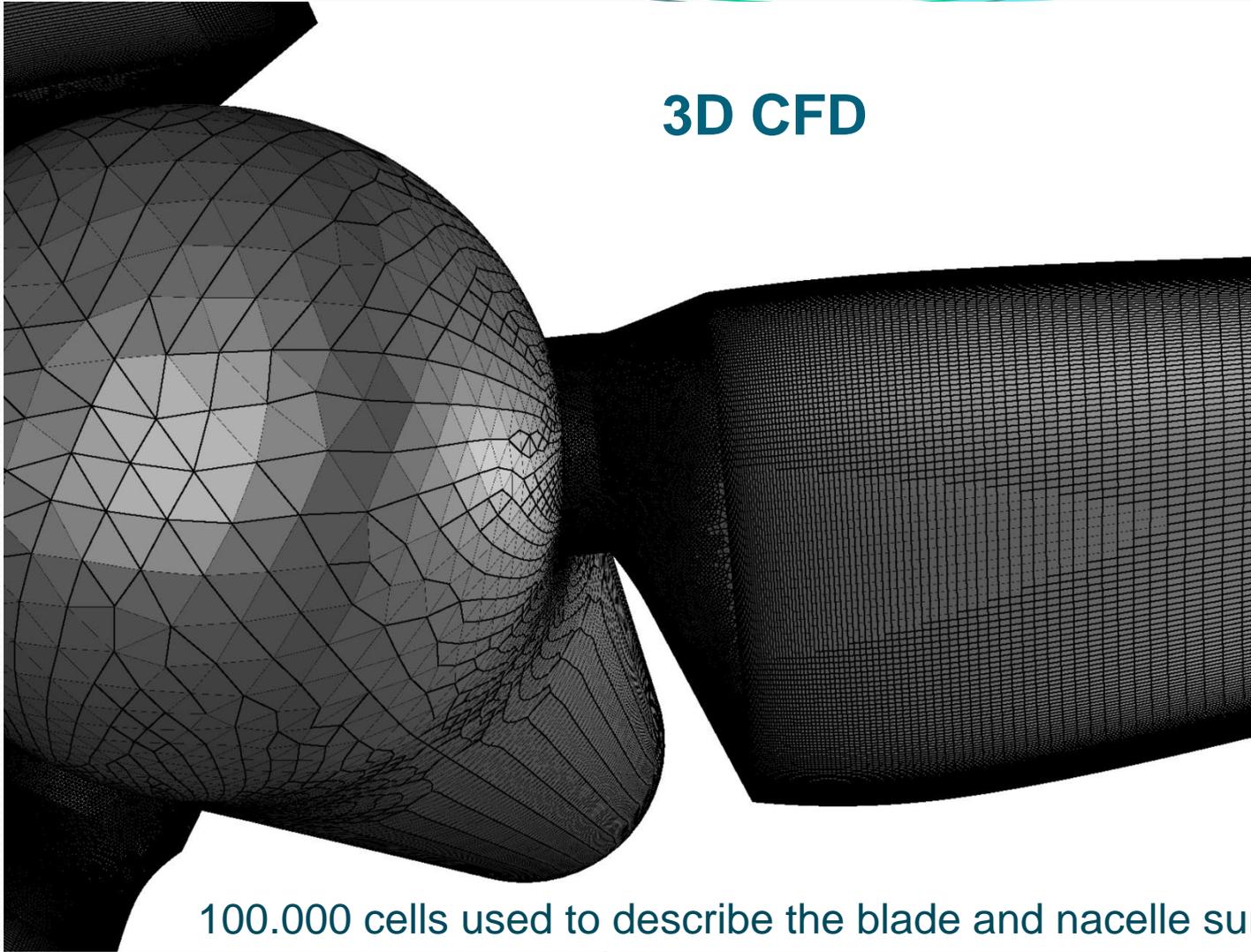


**Model instrumentation**



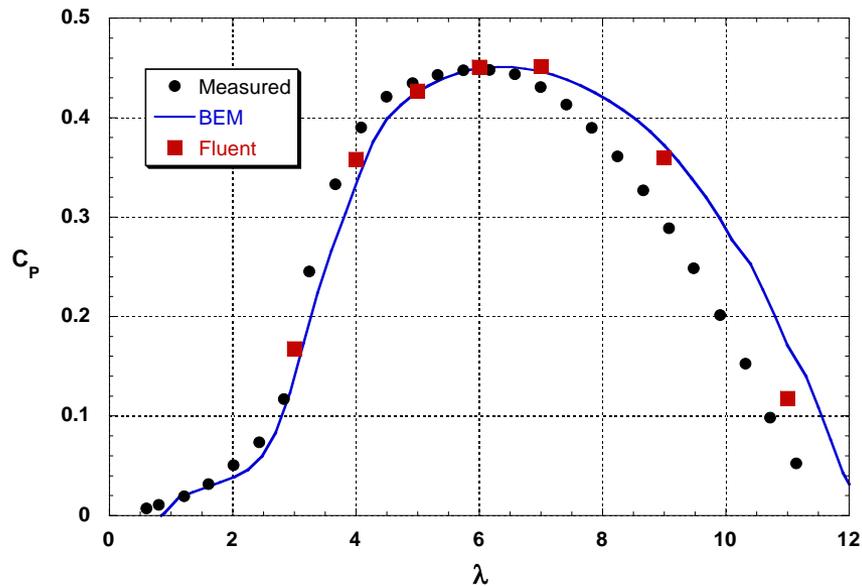
**Model in wind tunnel**

## 3D CFD

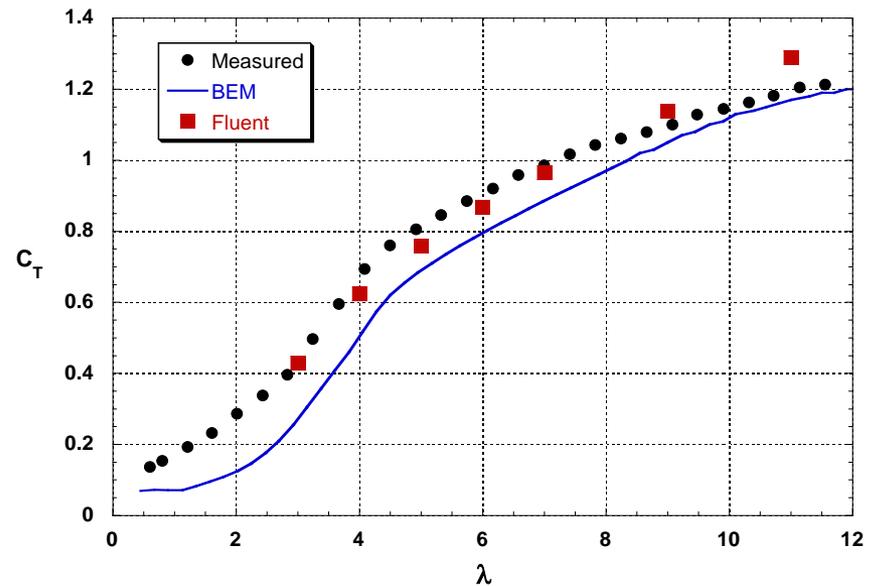


100.000 cells used to describe the blade and nacelle surfaces  
 $3.5 \cdot 10^6$  grid points in 1/3 volume

## Comparisons between predictions and measurements

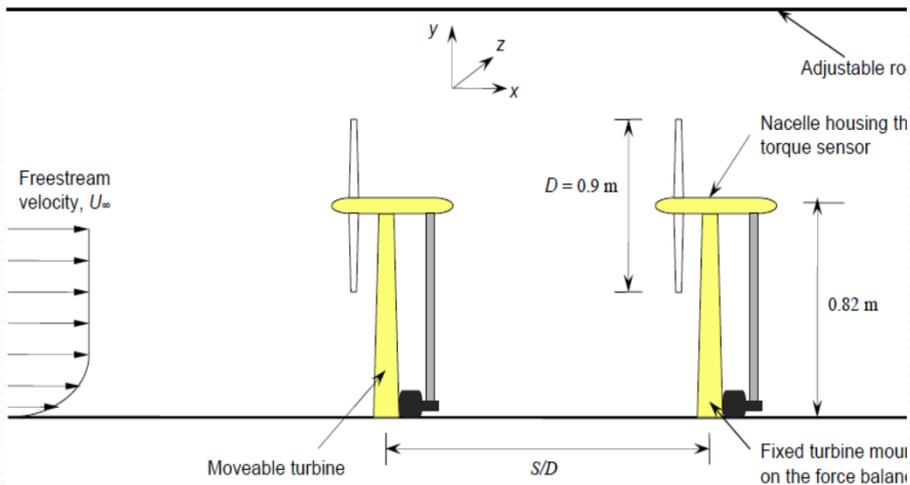


Power coefficient vs tip speed ratio

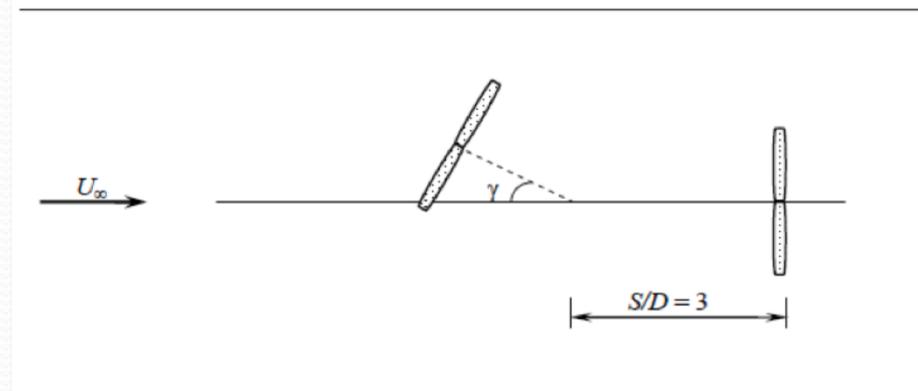


Thrust coefficient vs tip speed ratio

## Measurements for 2 similar turbines (Simplified wind farm experiment)

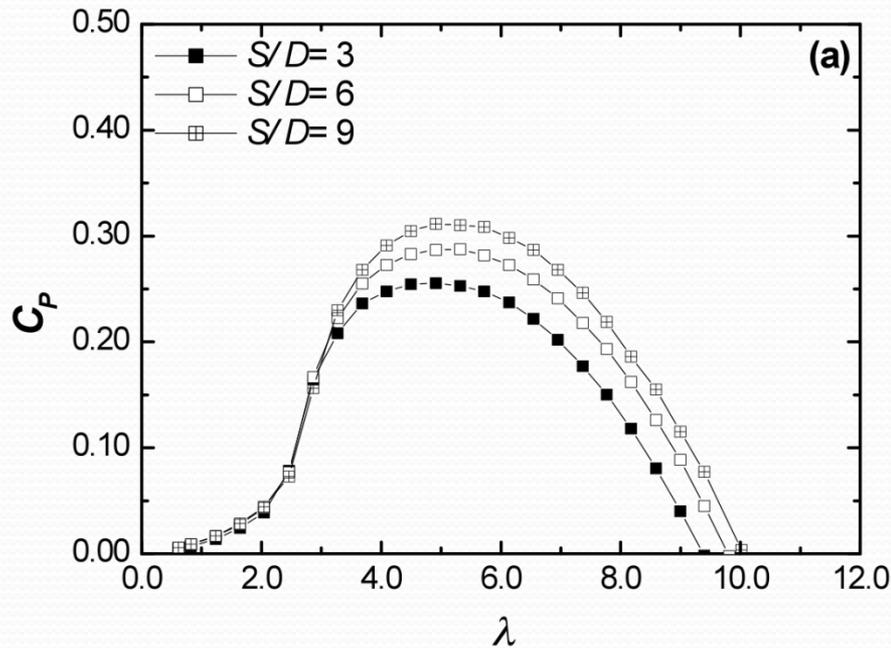


Two in-line turbines

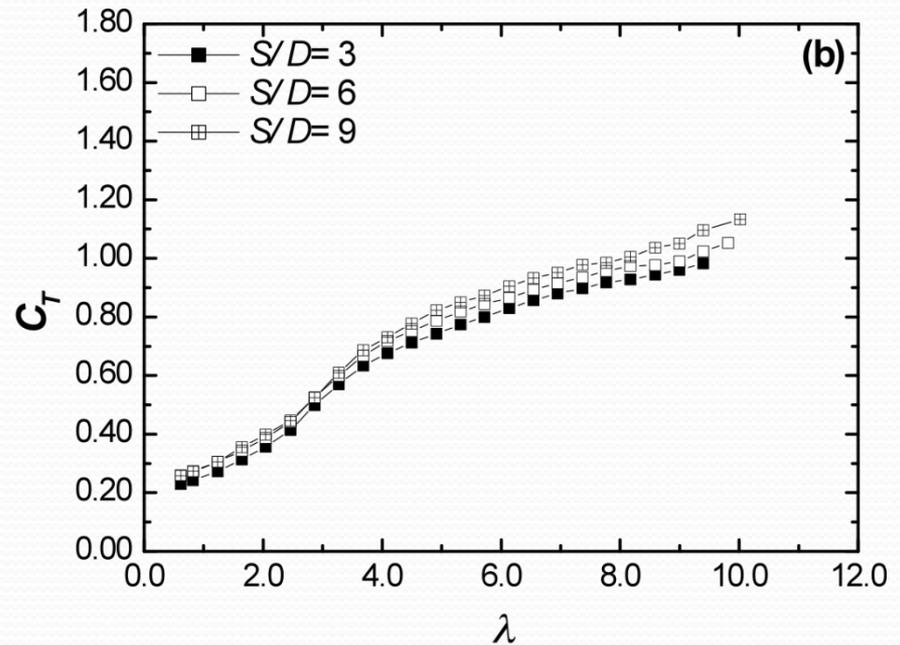


Yawed upstream turbine

## Effect of distance between turbines Upstream turbine operating at peak efficiency

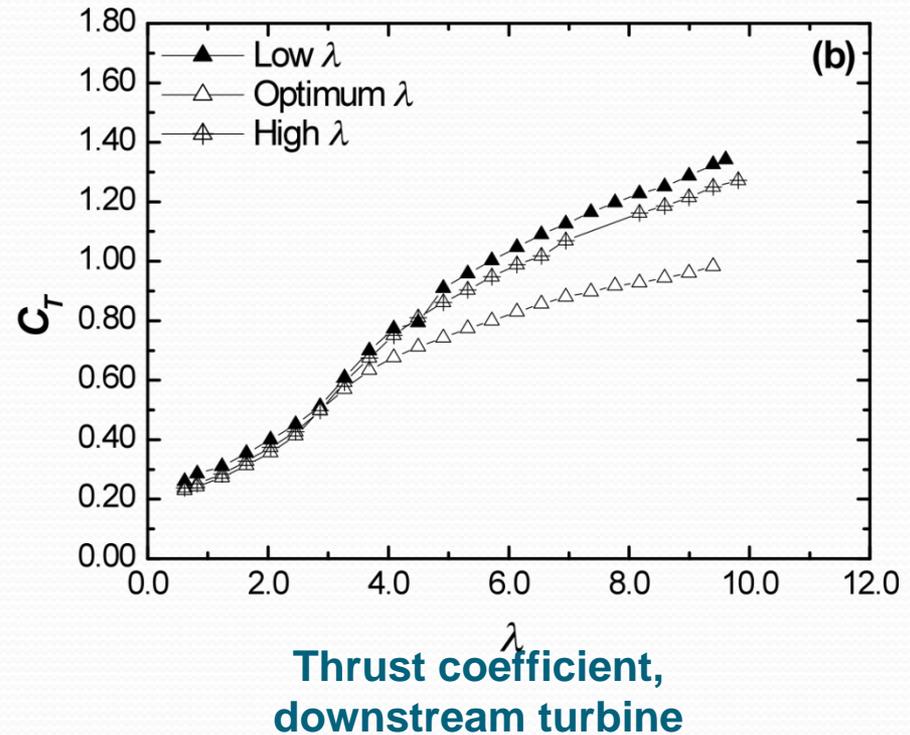
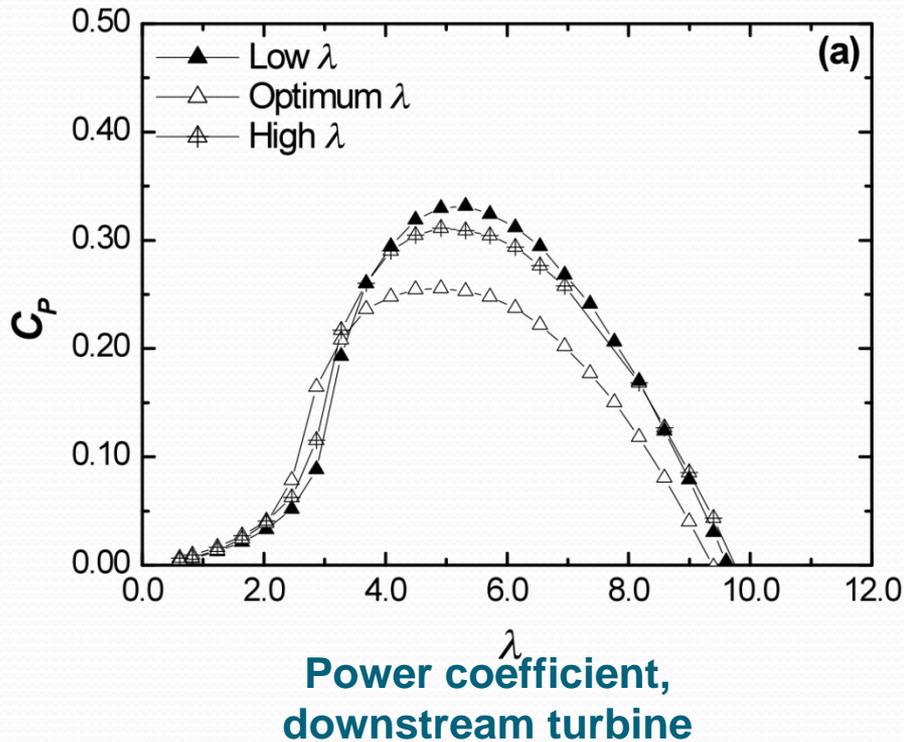


Power coefficient,  
downstream turbine



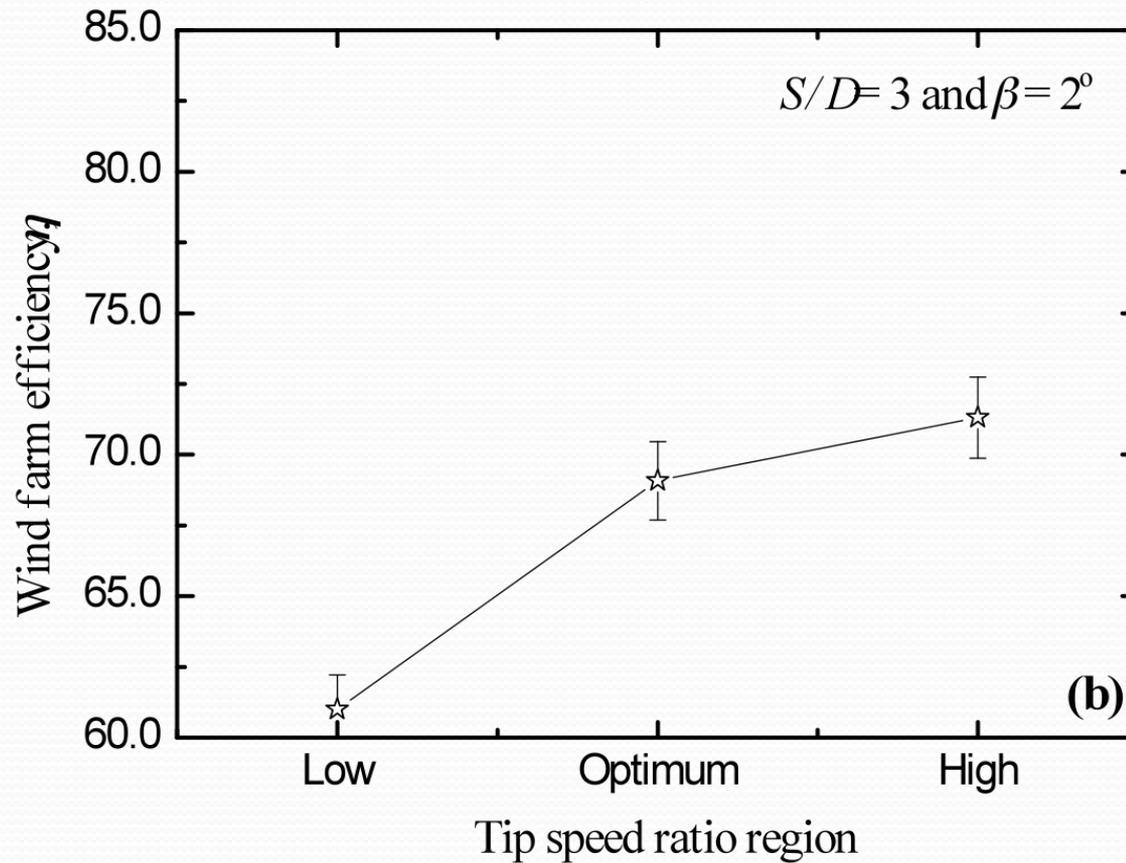
Thrust coefficient,  
downstream turbine

At a given distance, the output from the downstream turbine depends on the operating point of the first turbine  
**S/D=3**



# Total output compared to two unobstructed turbines

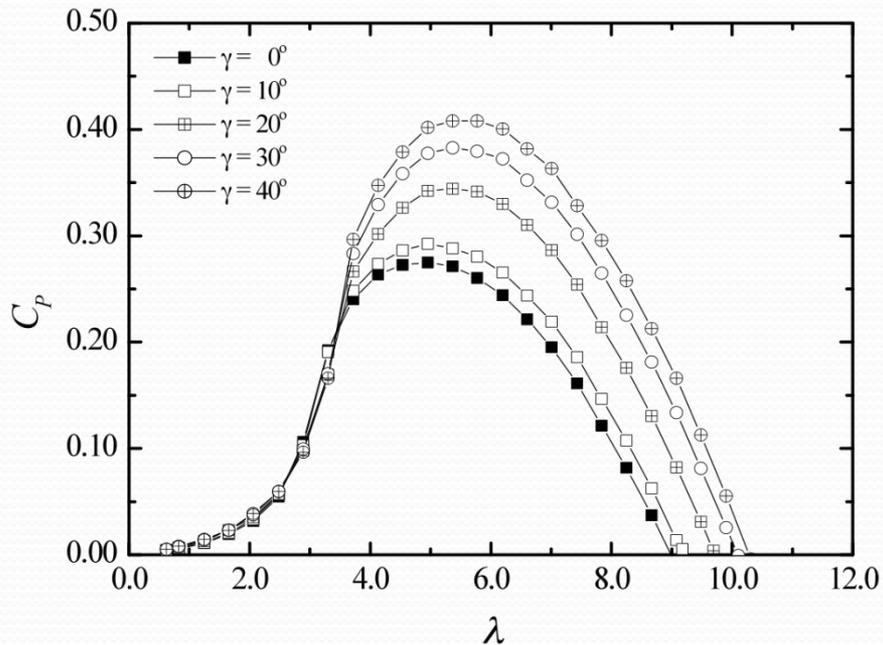
## $S/D=3$



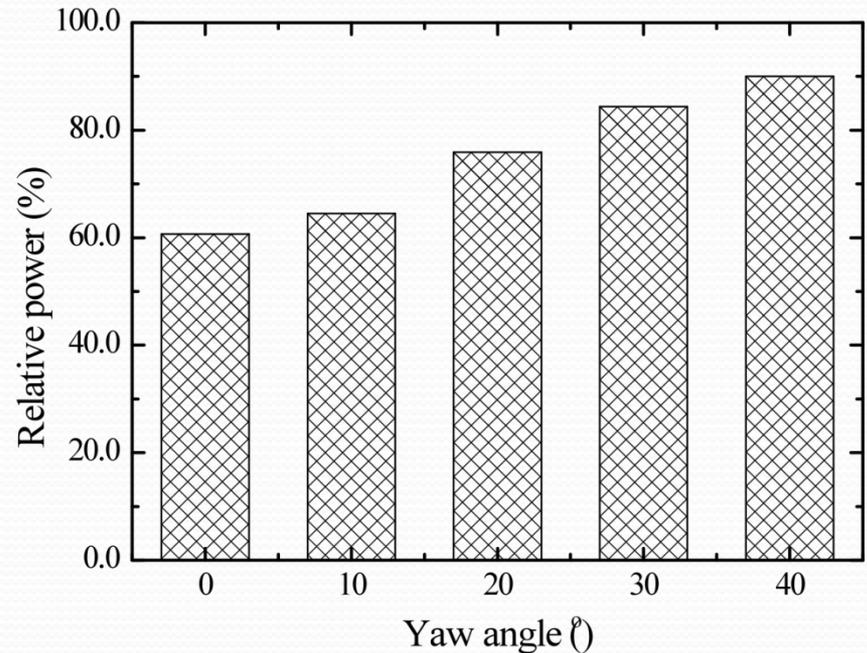
# Effect of yawing upstream turbine

## Upstream turbine operating at peak efficiency

### S/D=3

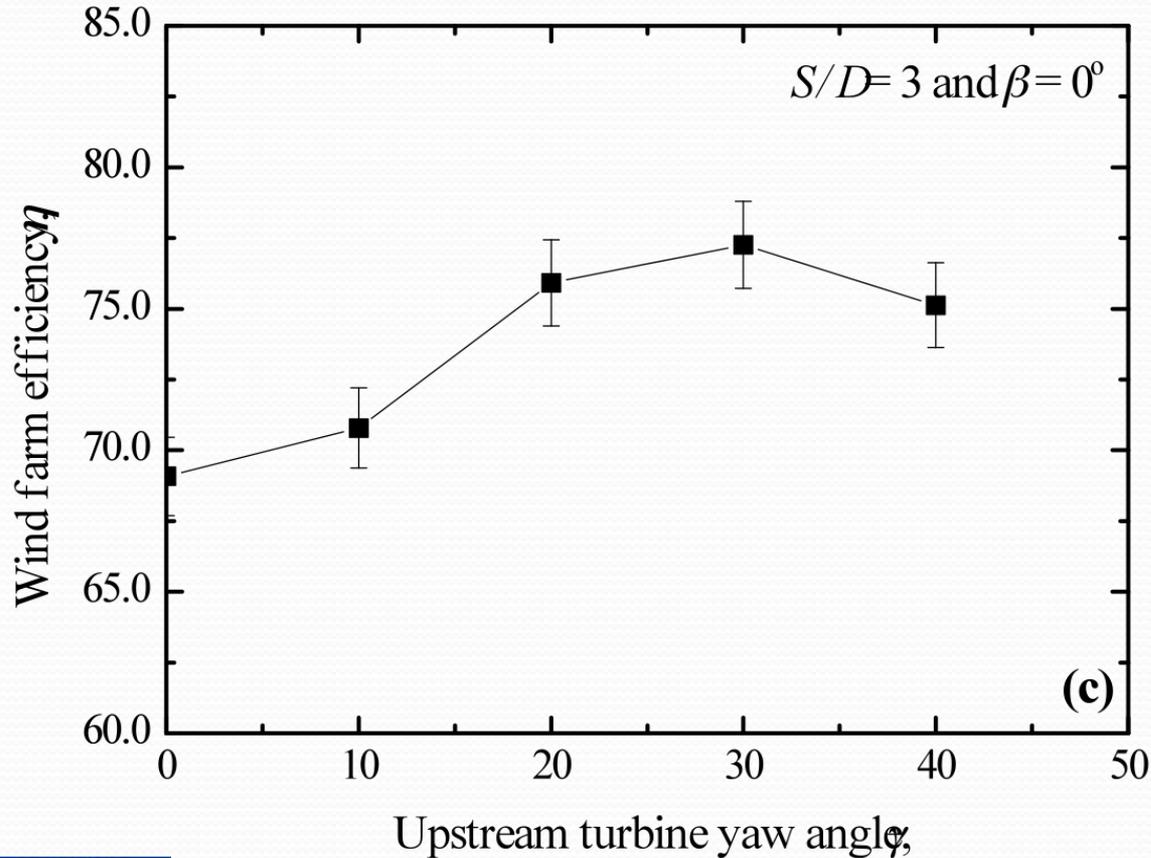


Power coefficient, downstream turbine



Power coefficient, downstream turbine, compared to single, non-yawed turbine

Total output compared to two unobstructed turbines  
 Upstream turbine operating at peak efficiency  
**S/D=3**

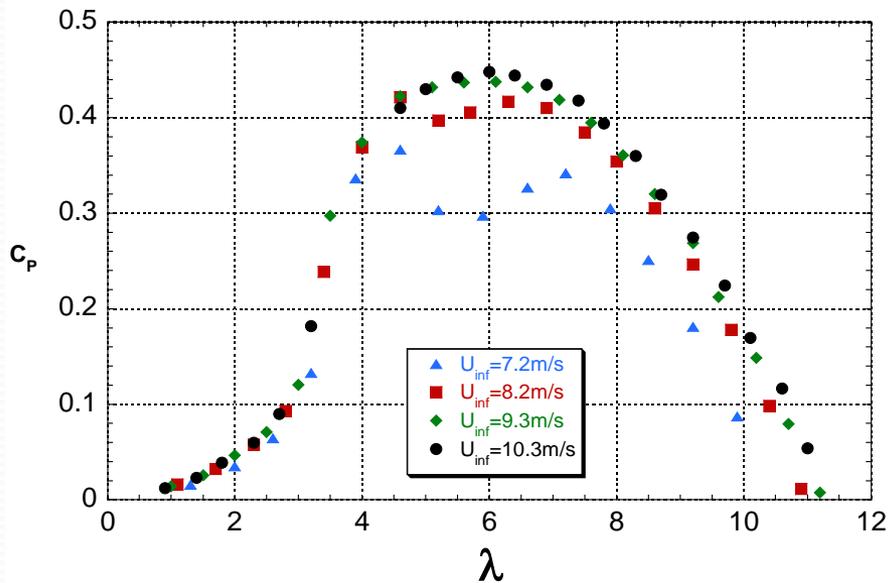


# Conclusions

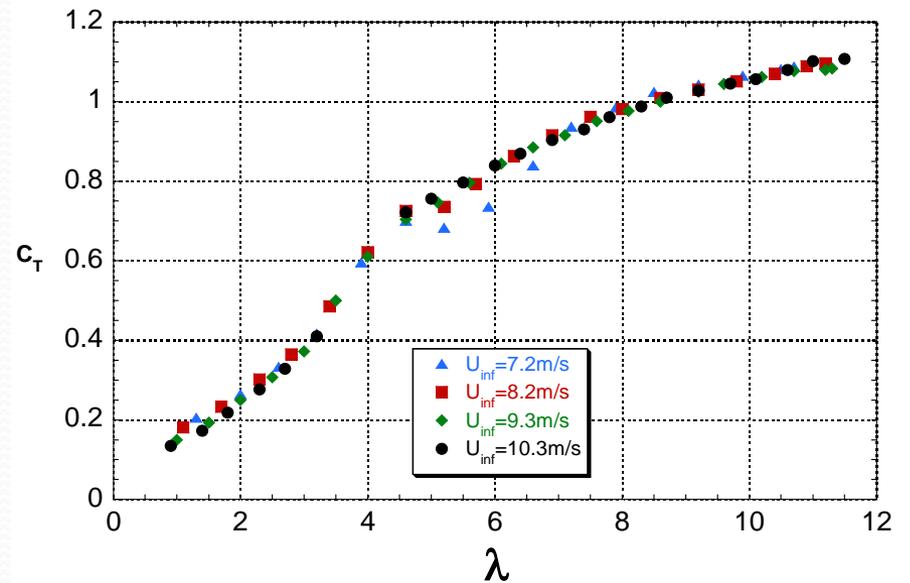
- When two wind turbines are placed in-line and both operated at best efficiency, the output of a turbine at  $S=3D$  is less than 60% of that upstream
- The power reduction is influenced by the wake characteristics from the turbine upstream and therefore by its operating point
- By reducing the power extracted from the first, the TOTAL output may be increased
- Yawing a turbine reduces its power by  $\cos^3\gamma$ . But it also deflects the wake which increases the output further downstream
- Two turbines operating in-line at best efficiency may increase the total output from about 69% of two unobstructed turbines at zero yaw, to 78% when the first is yawed 30 degrees. (Figures taken for  $S/D=3$ .)

# Reynolds number dependence

Turbine was designed for  $\lambda = 6$  and  $U_{ref} = 10\text{m/s}$



Power coefficient vs tip speed ratio



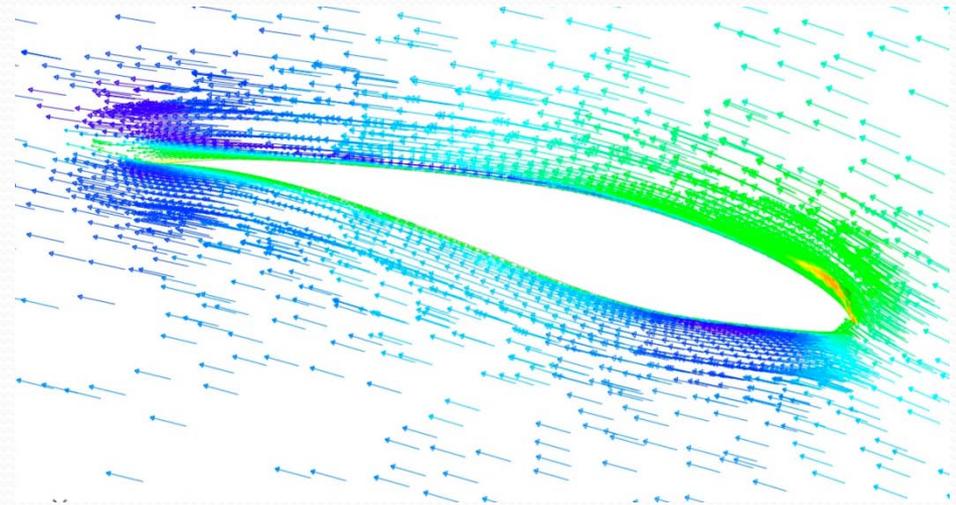
Thrust coefficient vs tip speed ratio

## 3D CFD details

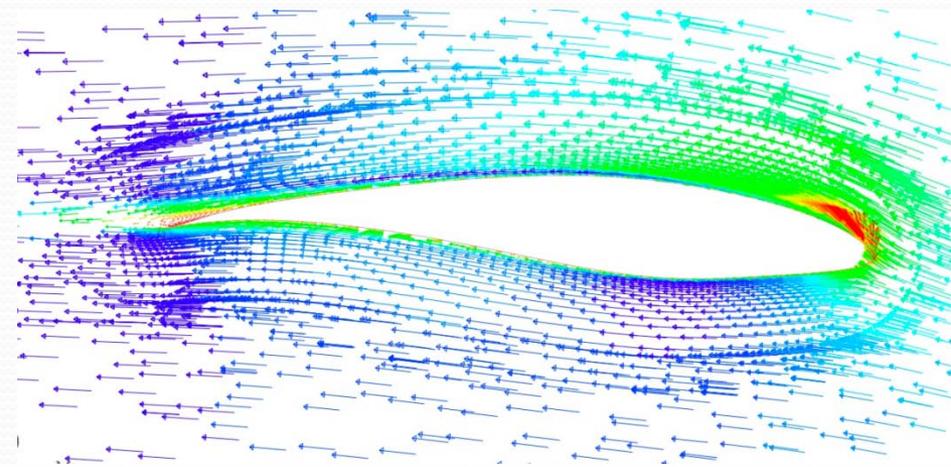
- 1/3 of the rotor including the nacelle was simulated.
- CFD domain same as wind tunnel test section (-4.5D to 7.8D in streamwise direction, 2.9D in spanwise direction).
- $k-\omega$  SST turbulence model with  $y^+ < 5$  for first grid point.
- Structured boundary layer grid around blade up to  $0.1c$ , tetrahedral grids used further out.
- QUICK and SIMPLEC schemes used for convective and pressure terms.
- 100.000 cells used to describe the blade and nacelle surfaces,  $3.5 \cdot 10^6$  grid points used.
- 4CPU PC parallel processing,  $\approx 24$  hours computing time per case

## At design tip speed ratio ( $\lambda = 6$ ) Flow almost two-dimensional

- Flow mostly attached except at the trailing edge separation ramp
- Angle of attack close to  $7^\circ$  over most of the blade
- $C_L \sim 1.2$



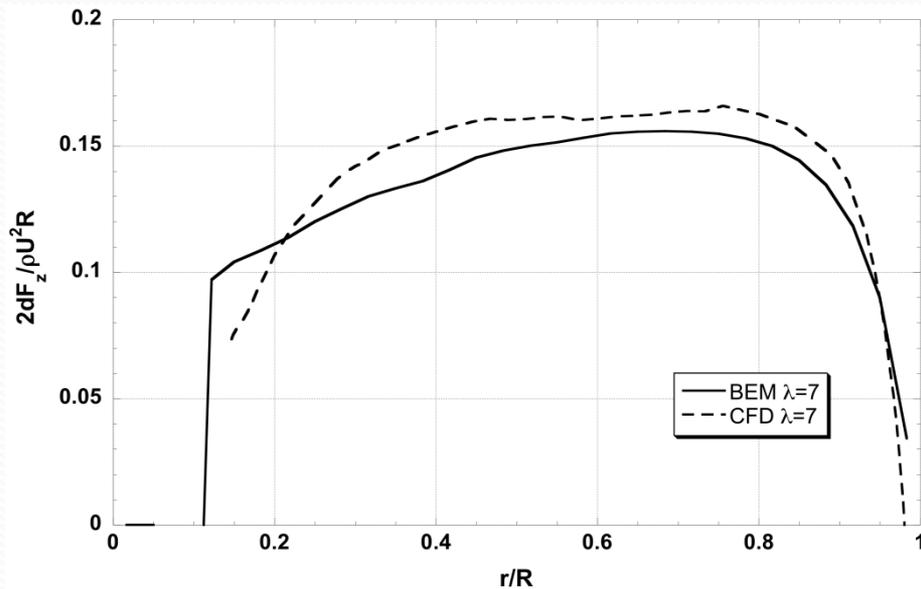
$r/R=0.44$



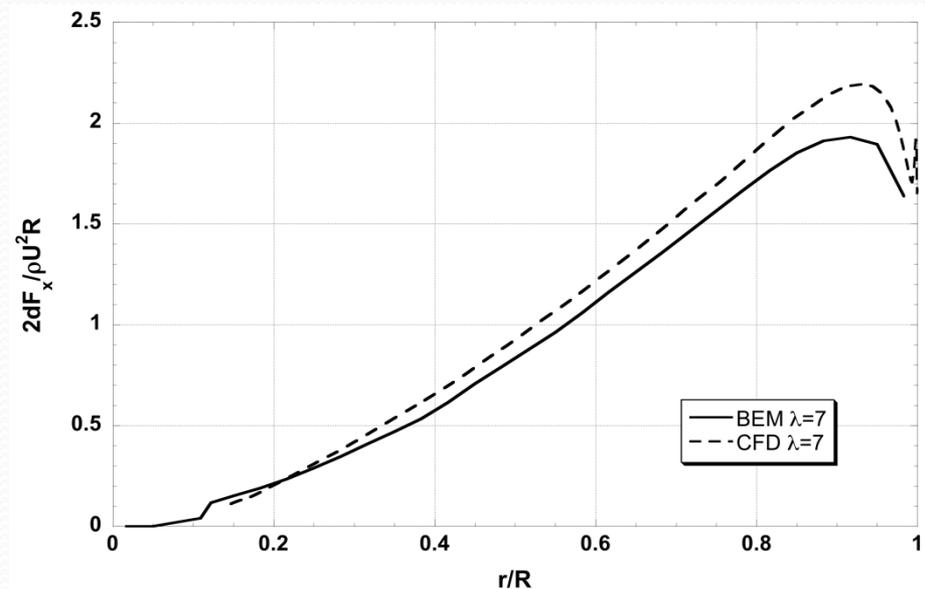
$r/R=0.89$

# Force distributions near design tip speed ratio

Good agreement between BEM and CFD

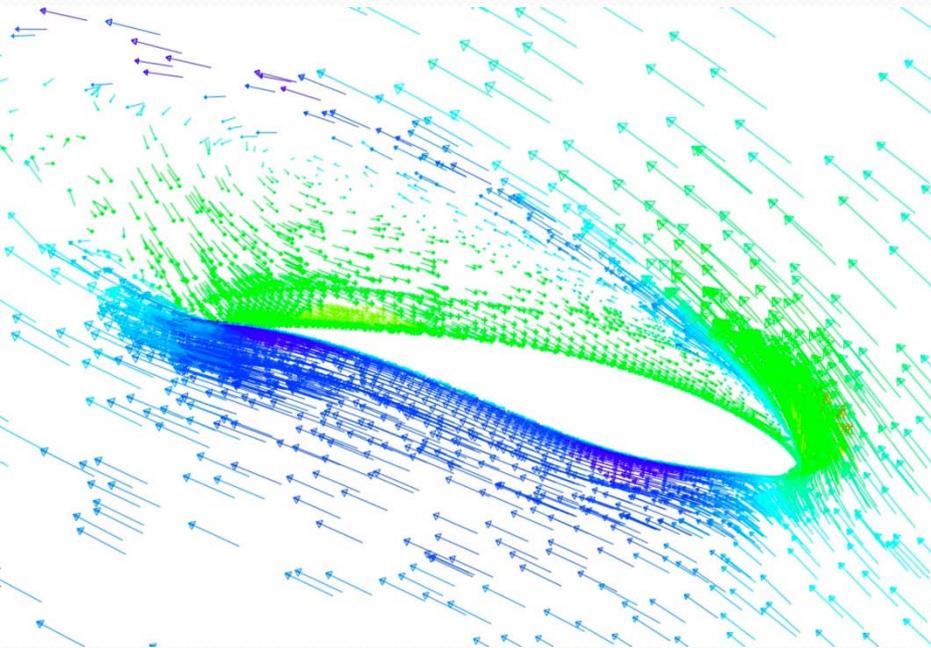


Tangential force

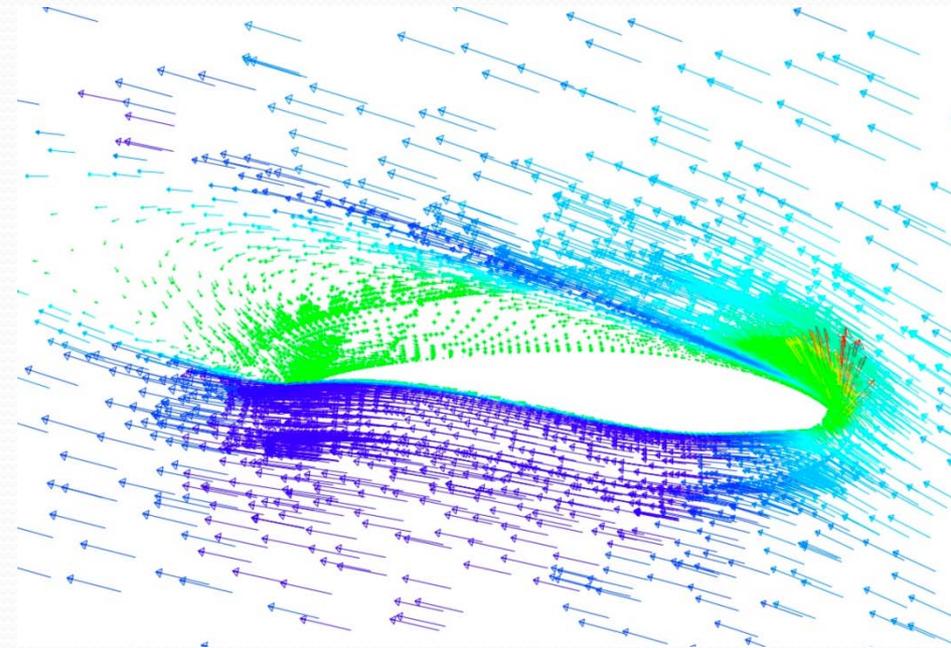


Streamwise force

At low tip speed ratio ( $\lambda = 3$ ) the blade operates in deep stall mode and the flow is highly three-dimensional.  
BEM expected to fail severely



$r/R=0.44$

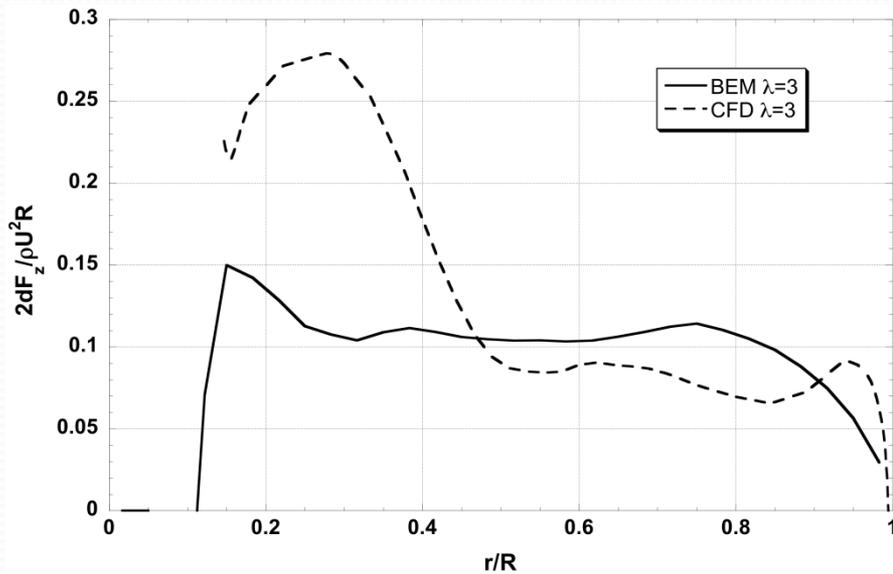


$r/R=0.89$

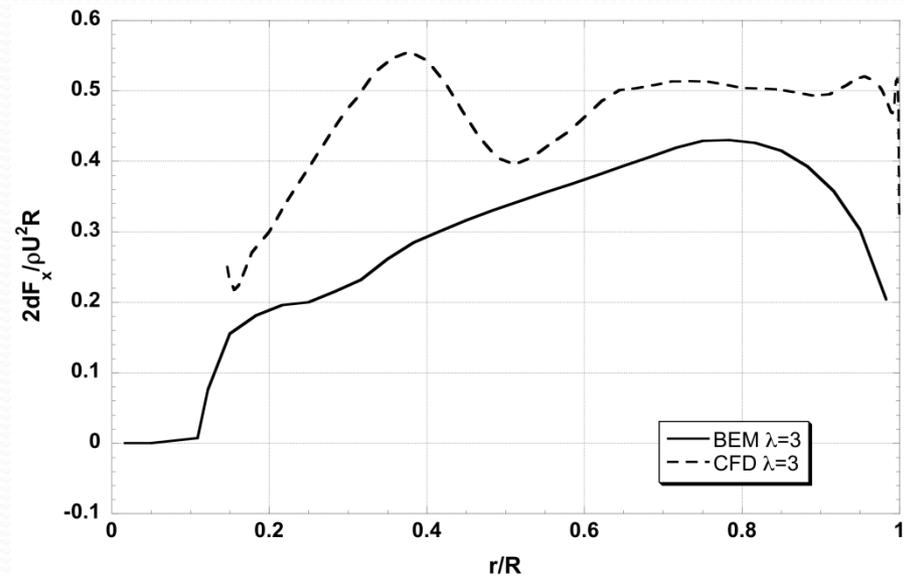
# Force distributions for $\lambda = 3$

Significant differences between BEM and CFD distributions.

(Still  $C_p$  predictions virtually identical, but BEM  $C_T$  severely under-estimated)



Tangential force



Streamwise force