

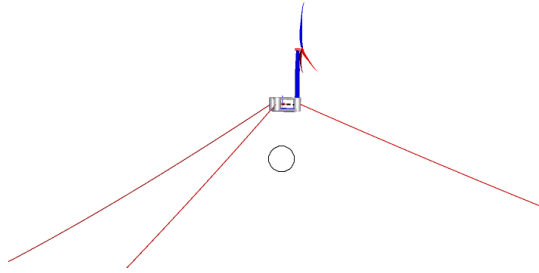
Comparison of quasi-static and dynamic mooring line models for shared mooring floating wind farms

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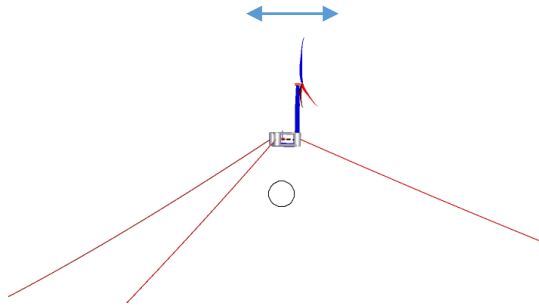
Motivation



Damping sources for floater motion :

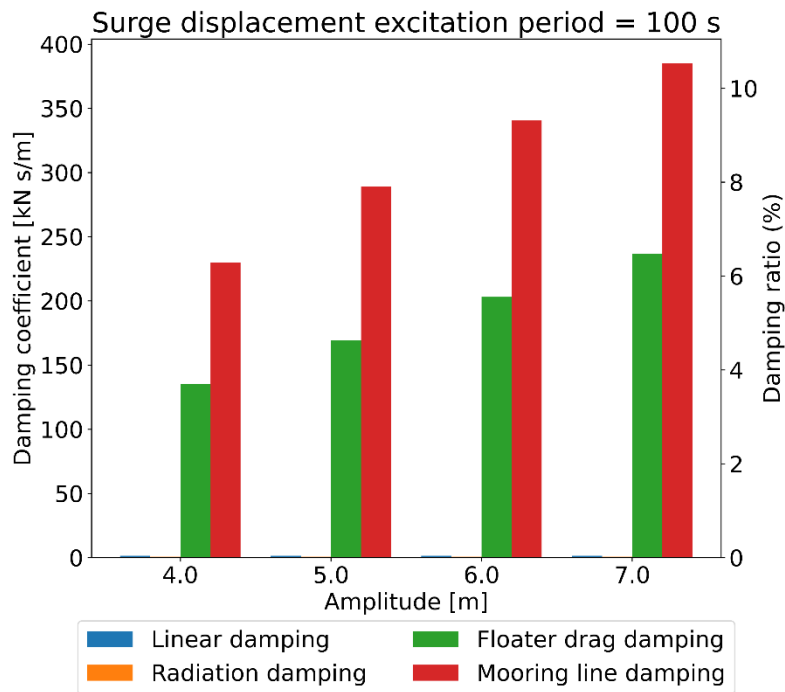
- Radiation damping
- Aerodynamic damping
- Damping from drag loads on the floater
- Damping from inertial and drag loads acting on the mooring lines

Motivation



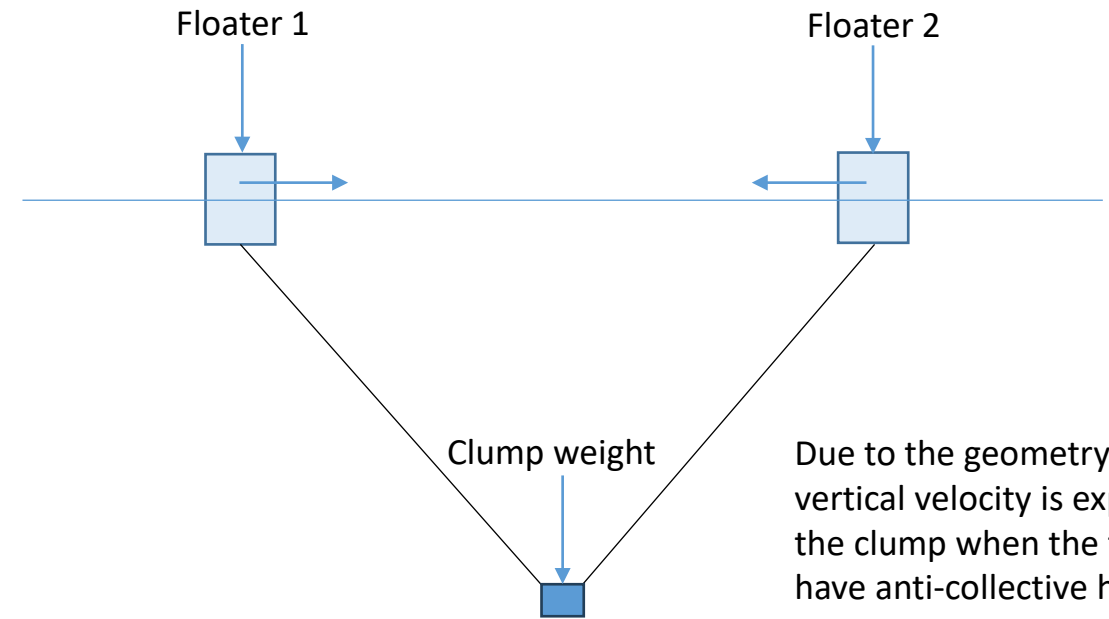
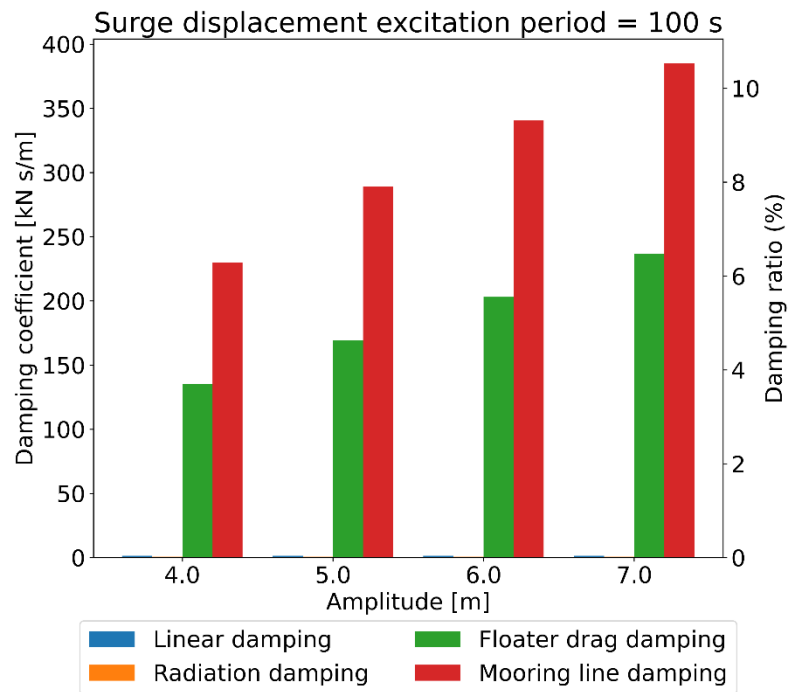
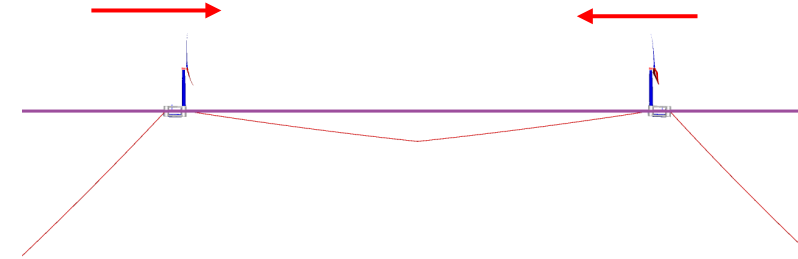
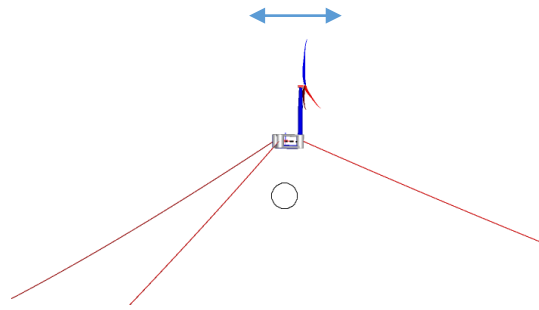
A prescribed motion test on an individually moored turbine produces the following damping coefficients at low frequencies.

-> **At lower frequencies the damping from the floater and mooring line becomes comparable.**



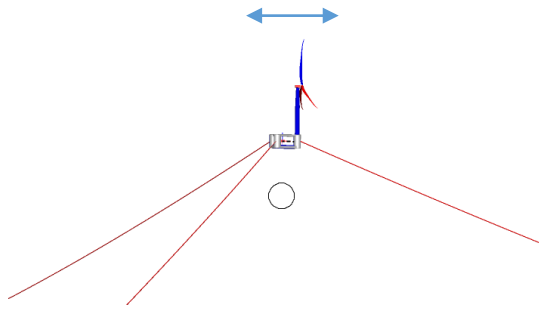
Anchor line		
Line material	Polyester	
Line diameter	264	mm
Minimum breaking load	18.64	MN
Line stiffness (EA)	466	MN
Dry mass coefficient	44.7	kg/m
Wet mass coefficient	9.5	kg/m
Horizontal footprint	1300	m
Line pretension	1.8	MN
Water depth	500	m

Motivation

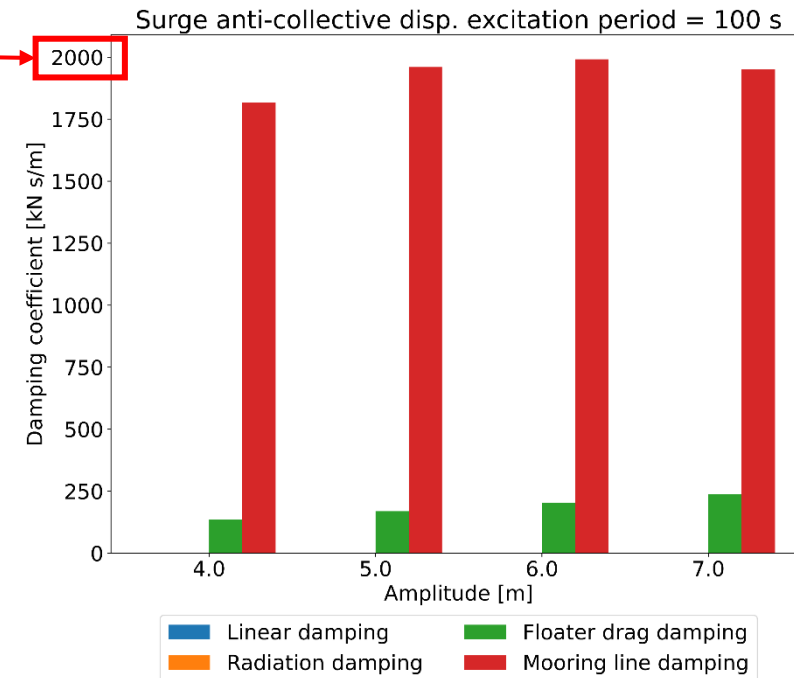
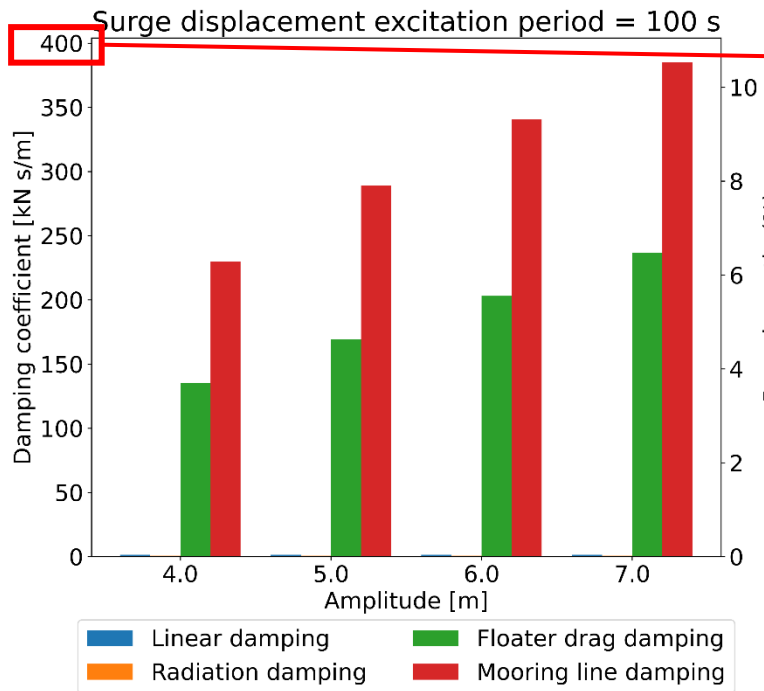
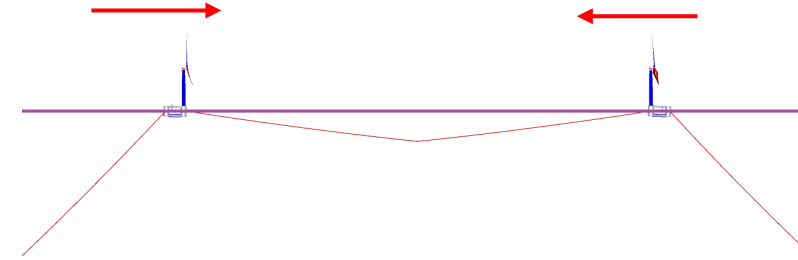


Due to the geometry – large vertical velocity is expected at the clump when the floaters have anti-collective horizontal motions.

Motivation



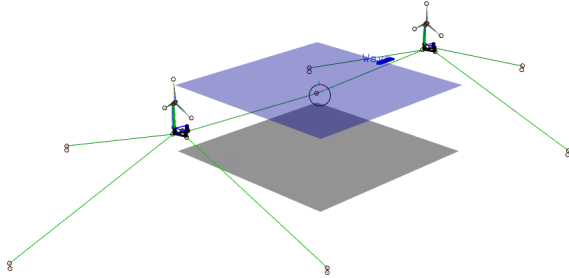
Ascertain how this affects the platform motions by comparing simulations using quasi-static and dynamic line models.



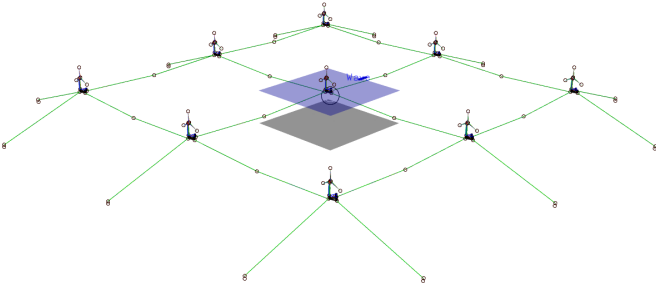
Test cases

Lattice variants

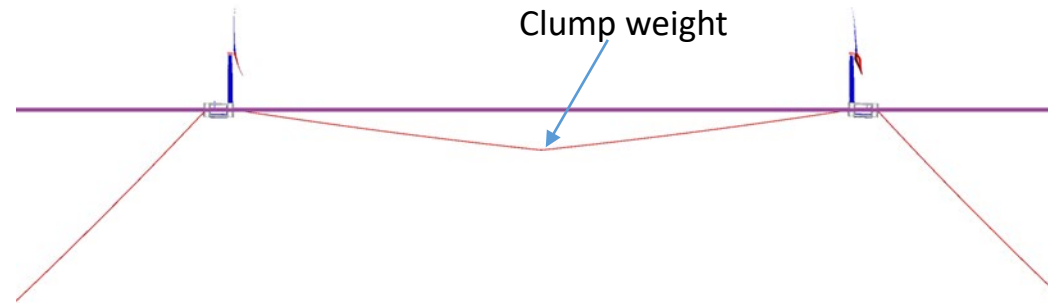
Grid21



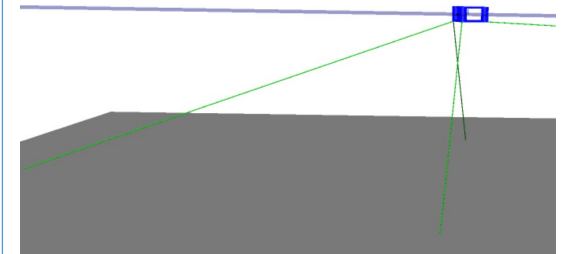
Grid33



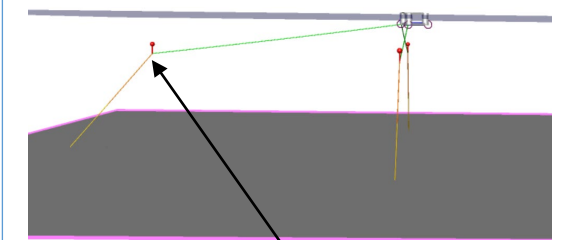
Shared line design



Anchor line design



Taut anchor lines



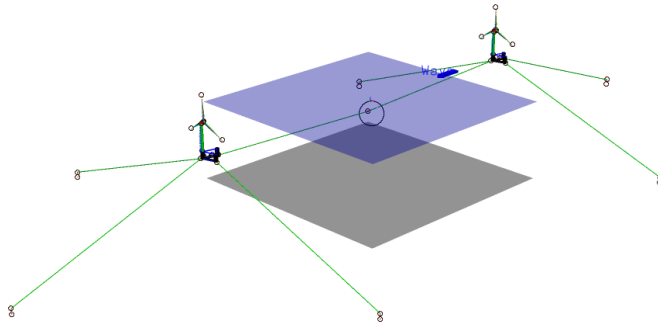
Buoyancy module on anchor lines

Line/clump property variants

Variant	Polyester line dia [mm]	Clump weight [t]	MBL (MN)	Pretension (MN)
D264_clmp50	264	50	18.640	3.754
D264_clmp30	264	30	18.640	3.201
D241_clmp30	241	30	15.696	2.860
D213_clmp30	213	30	12.263	2.414

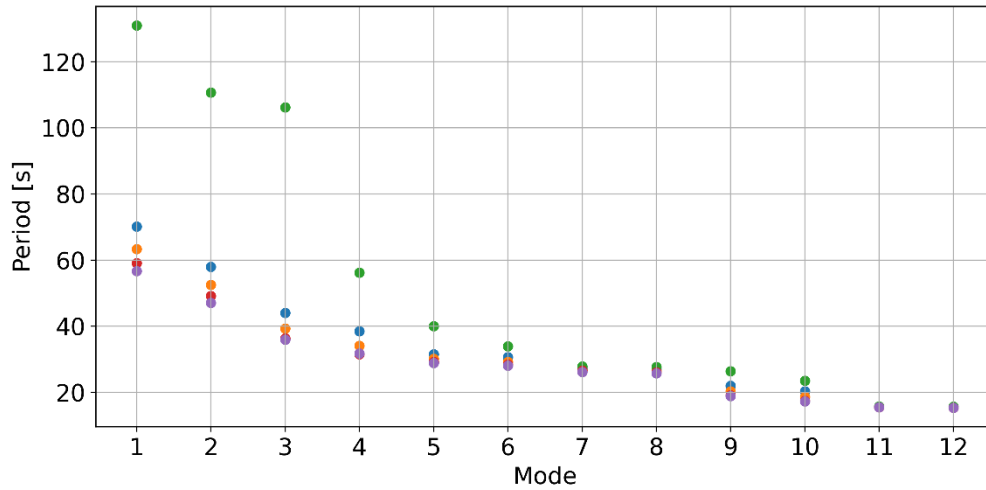
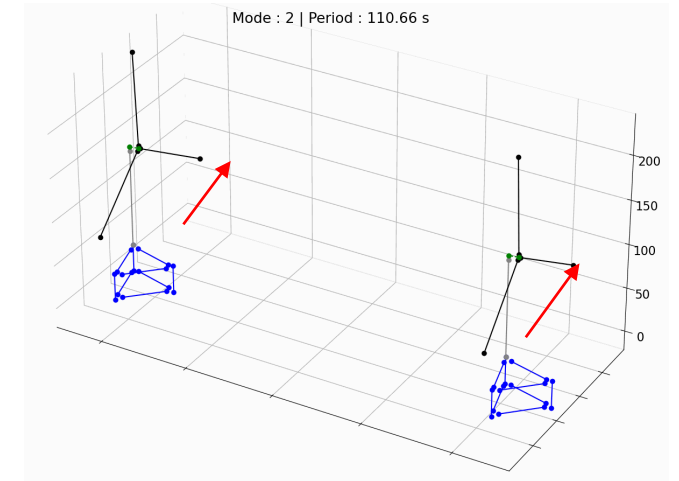
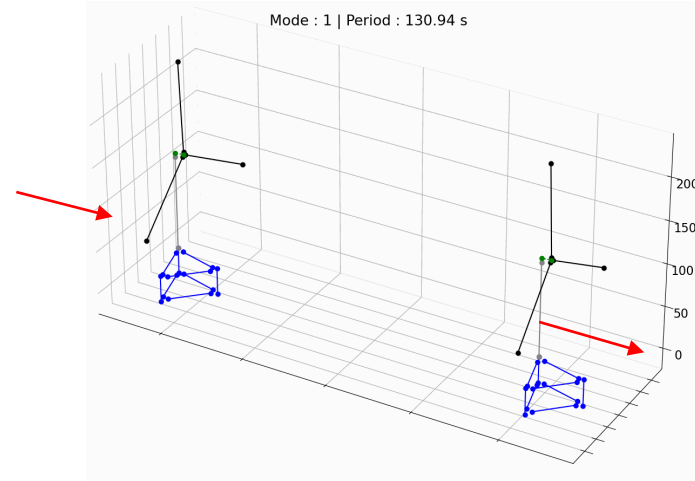
Eigen value analysis – Grid21

Grid21

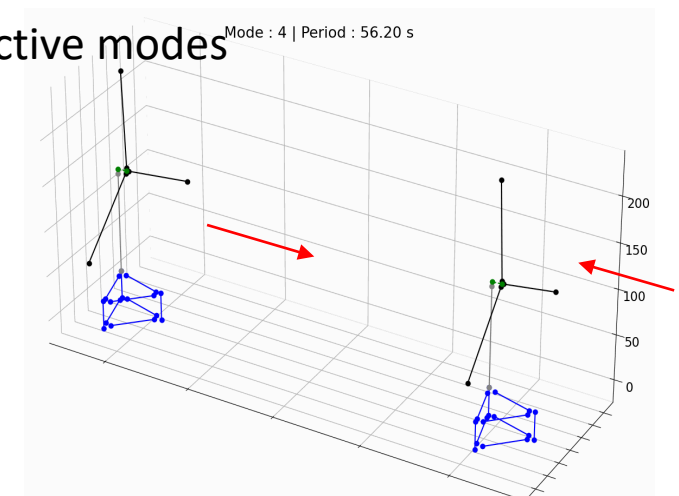
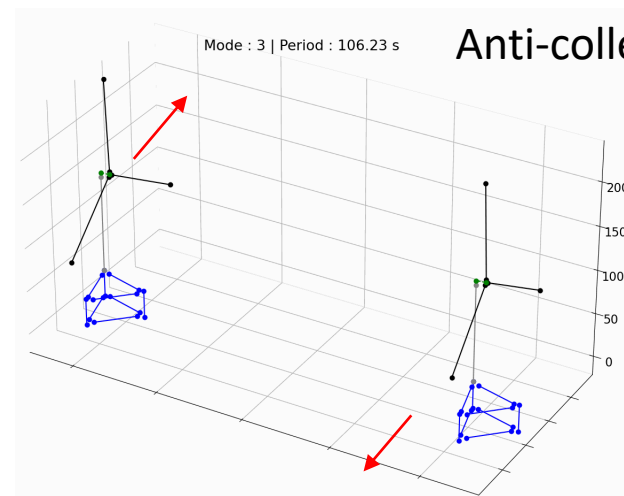


First 4 modes for D241_clmp30_B300

Collective modes



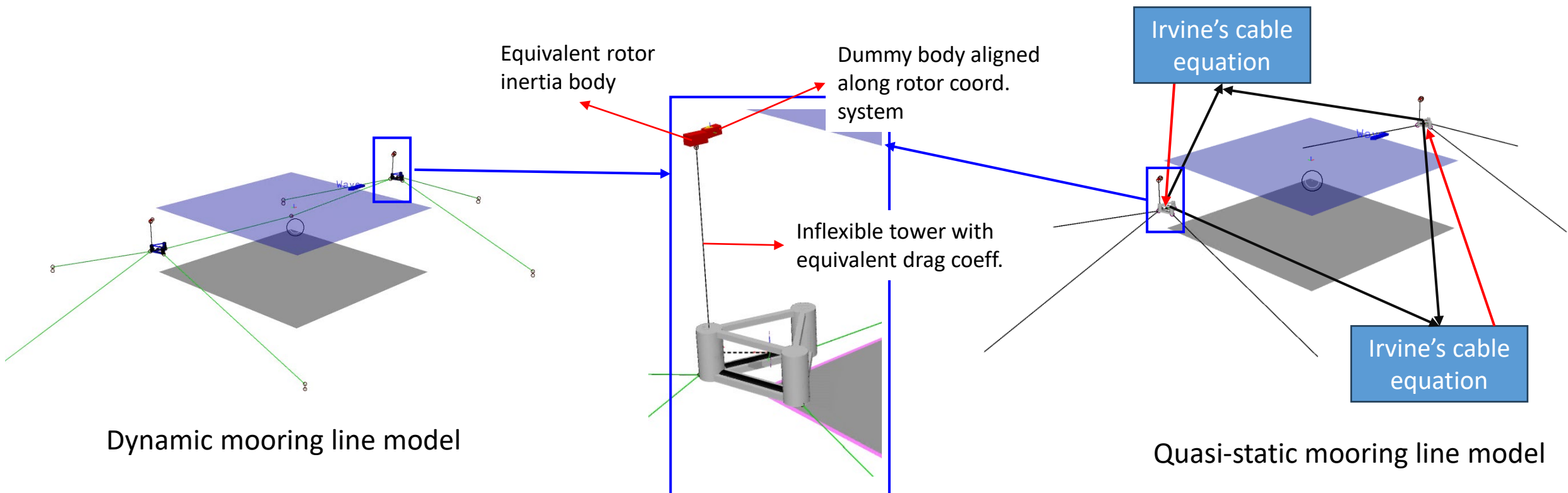
Anti-collective modes



- D213_clmp30 ● D241_clmp30_B300 ● D264_clmp50
- D241_clmp30 ● D264_clmp30

Methodology

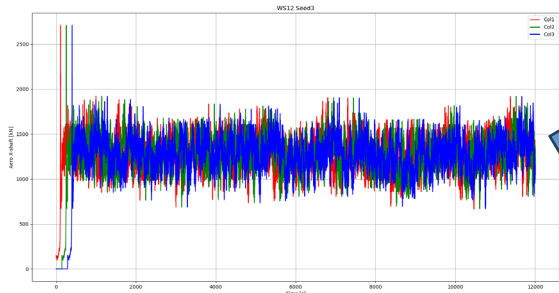
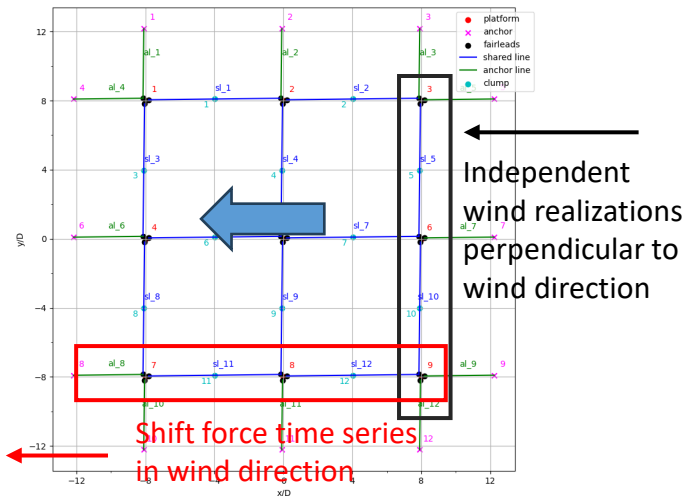
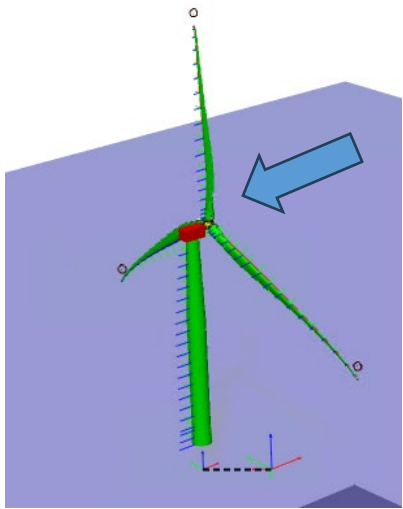
- Environment condition : $H_s - 2m$, $T_p - 7s$, Wind speed = 12 m/s, Turbulent wind
- Focus is to conclude on the importance of damping -> two statically equivalent models are built :
 - Quasi-static mooring line model -> Catenary/Irvine's cable equations
 - Dynamic mooring line model -> RIFLEX FEM model



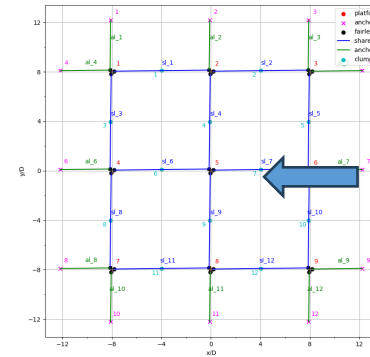
Methodology

- 2 step simplified simulation approach

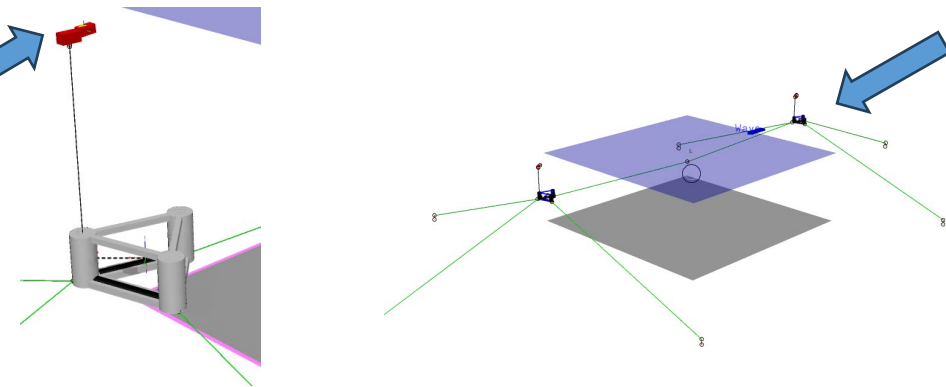
Step 1 - Turbulent wind fixed tower + rotor test -> obtain time series of 6 dof aero force



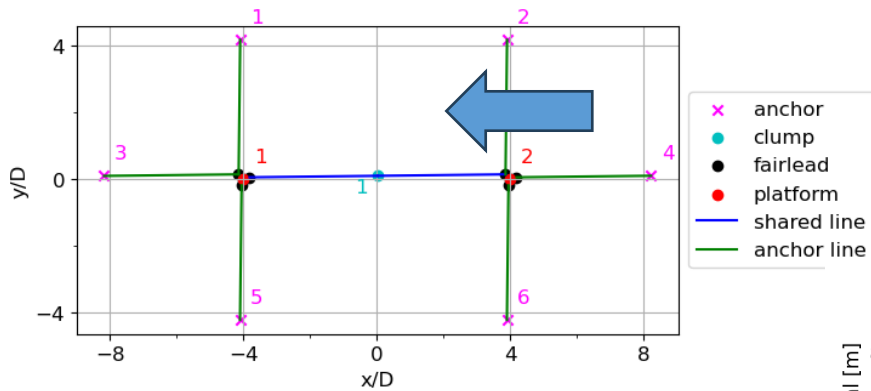
Step 2 – Prescribed 6 dof aero force + constant wind (to get the tower drag) + wave simulation



- ☺ Computationally efficient approach (10x faster)
- ☹ No effect of platform motions on aerodynamics

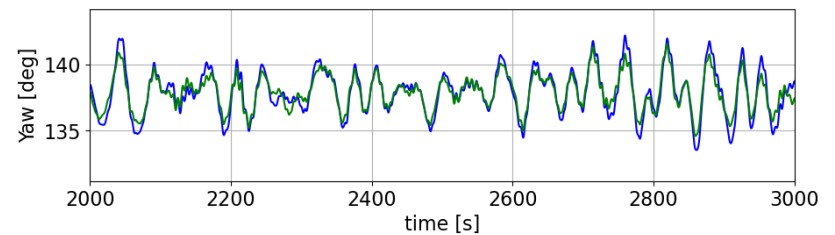
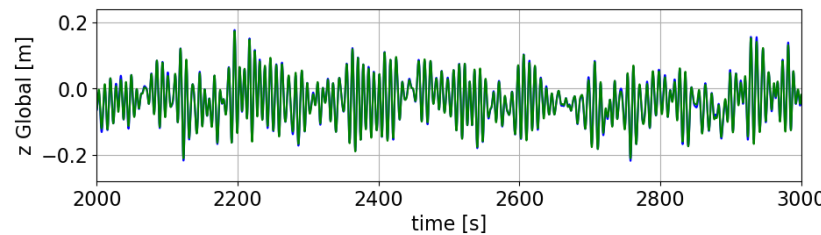
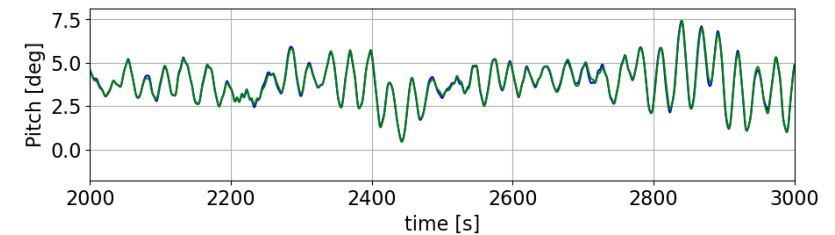
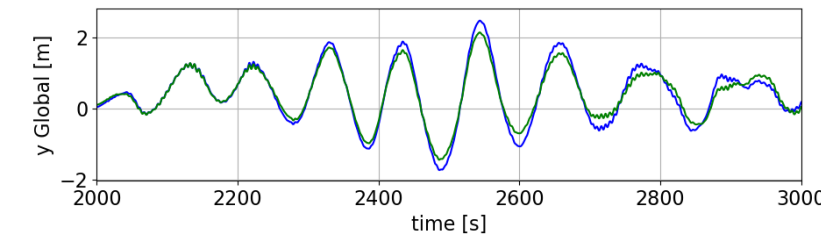
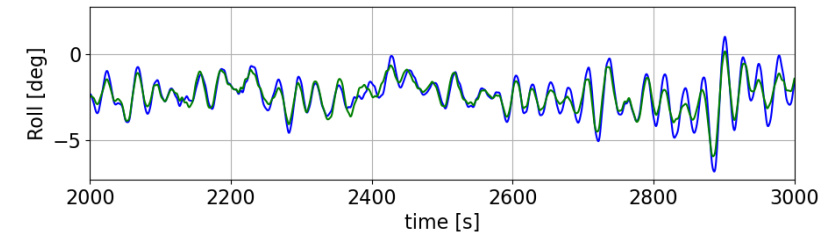
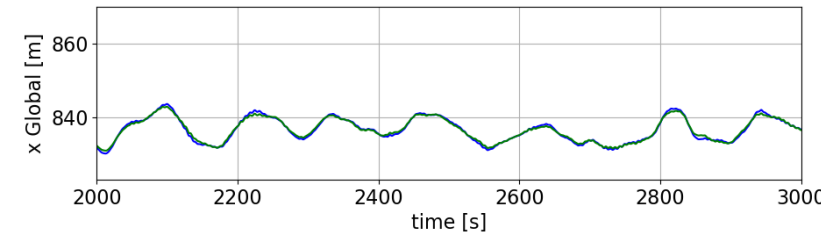


Results – Grid21 Platform motions

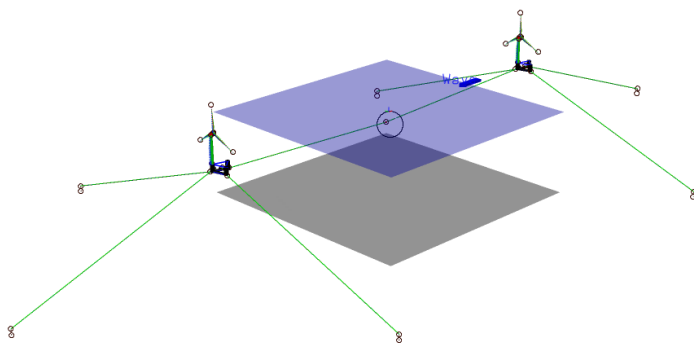


General observation over all tested cases -> A portion of motion time series of the least stiff system is shown below -> **The dynamic and quasi-static line models lead to nearly same motion responses for all the cases (small low frequency variations observed)**

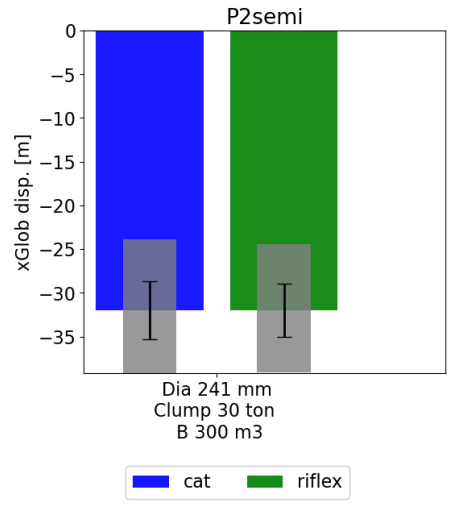
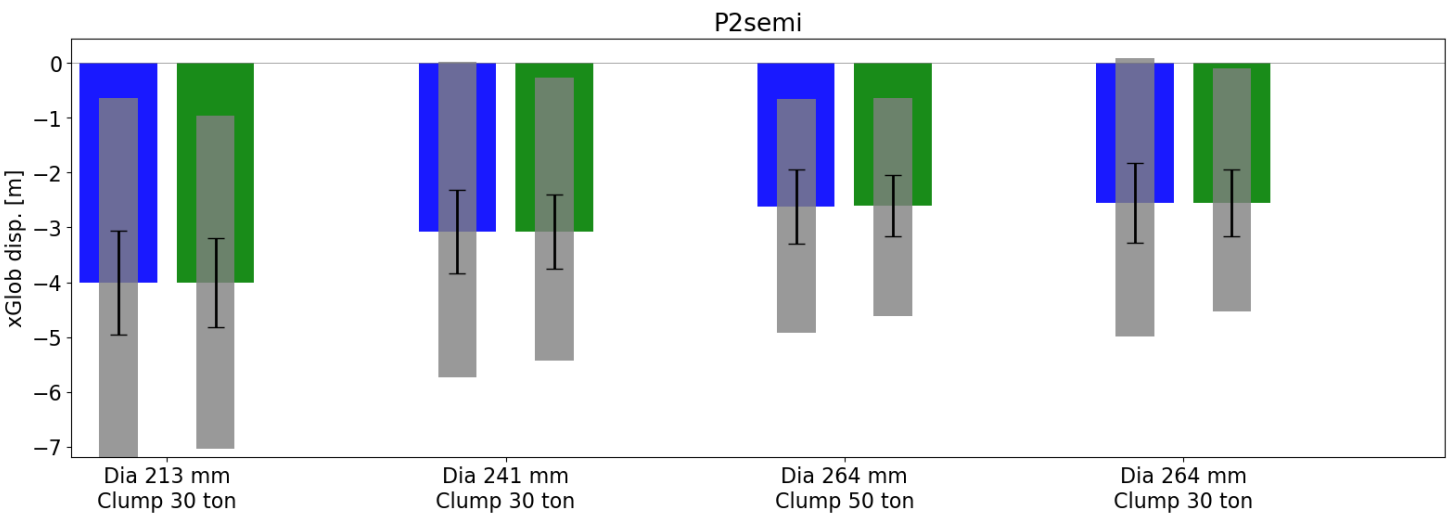
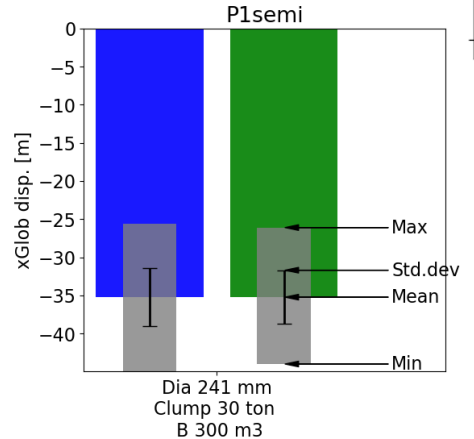
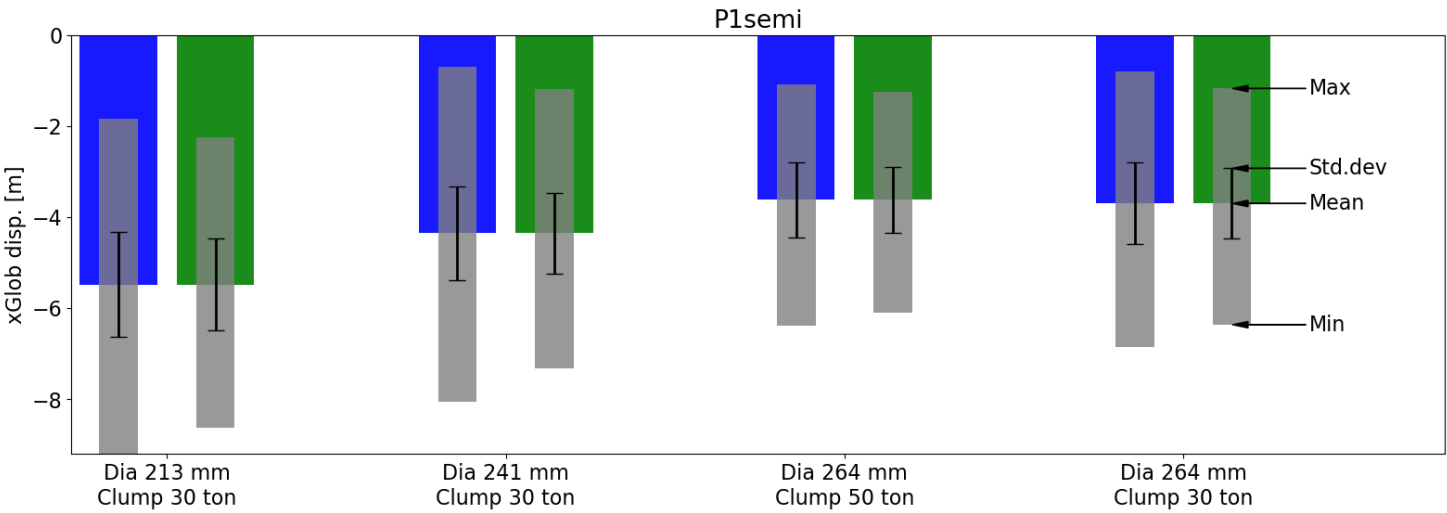
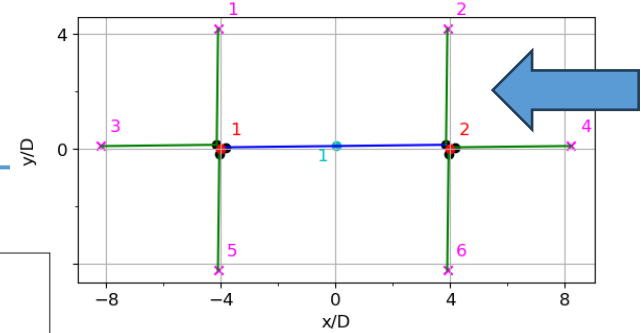
grid_2_1 D241_clmp30_B300 P2semi



— cat — reflex



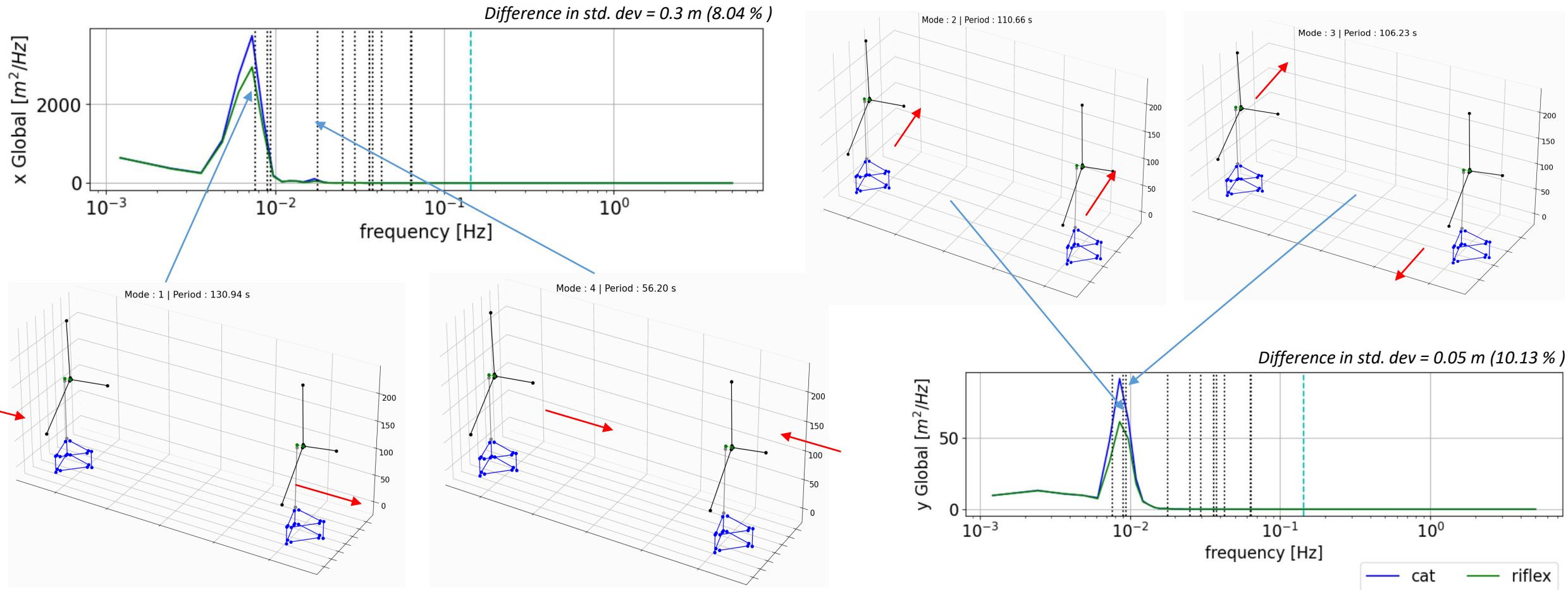
Results – Grid21 Platform motion statistics



- Statistically the motions are similar between catenary and RIFLEX models.
- For all the cases std. dev are comparable between RIFLEX and catenary -> slightly smaller in case of RIFLEX due to damping.
- The max. difference in std. deviation is 0.3 m (Buoy case)

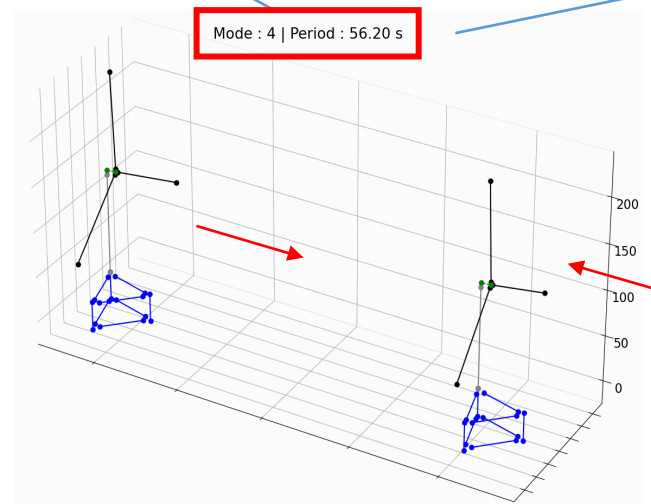
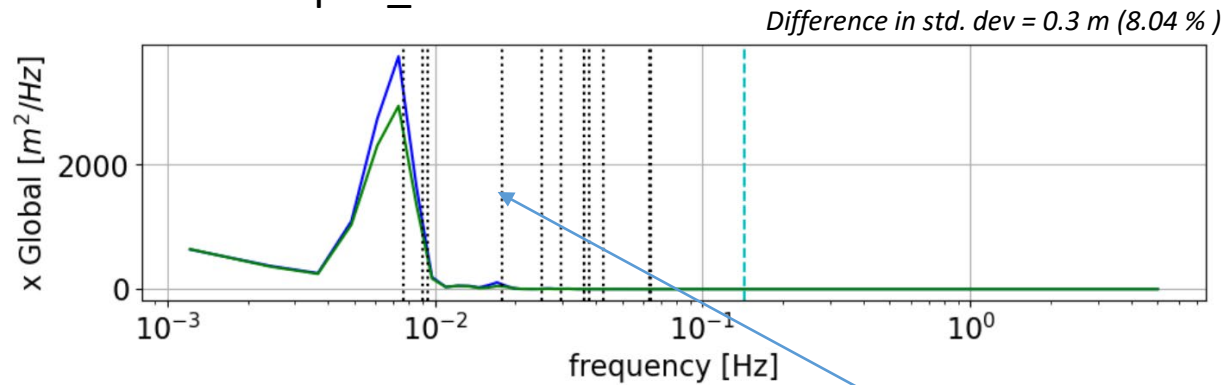
Results – Grid21 Platform motion spectrum

The differences in motions of the platform are seen at specific eigen frequencies – motion spectrum for D241CImp30_B300 P2semi is shown as an example.



Results – Grid21 Platform motion spectrum

D241CImp30_B300 P2semi

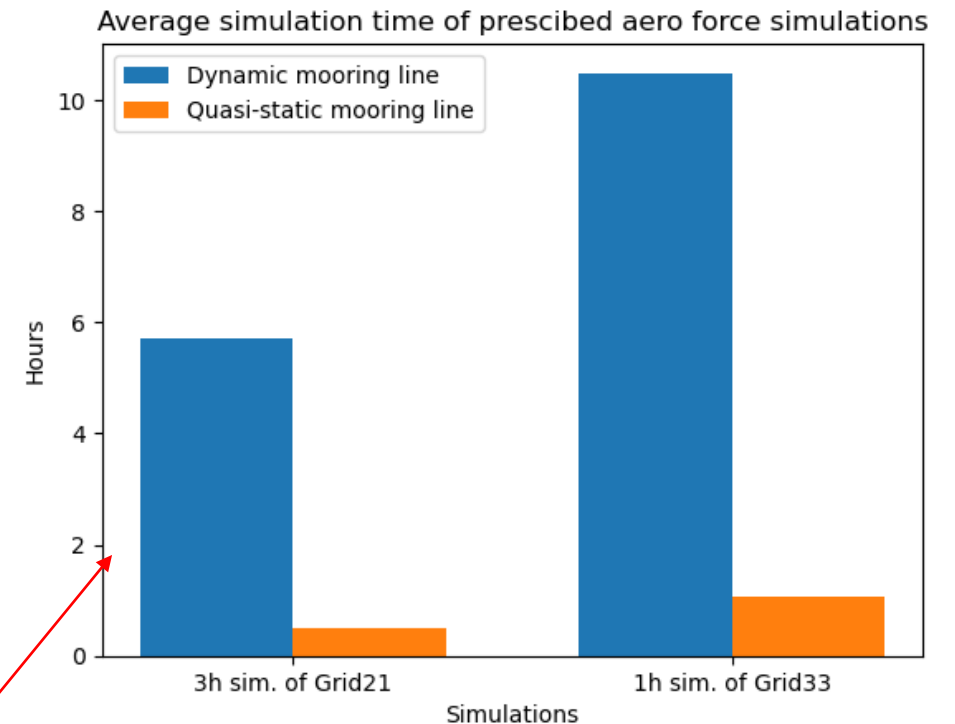


Eigen period of the anti-collective mode not low enough for the line damping to be dominant.

Similar observations were made for Grid33 variant of the lattice.

Conclusion

- Compared the motions of platforms the in 2 shared mooring lattice configurations with 5 different variants of the mooring system design with dynamic and quasi-static mooring line models.
- For near rated condition (and extreme wave condition ($H_s = 11$ m, $T_p = 12$ s – not shown in the presentation) no significant difference in platform motions is seen between using a dynamic mooring line model and quasi-static mooring line model.
- The small differences occur at specific eigen frequencies of the lattice.
- Comparing the dynamic and quasi-static mooring line models (Grid21) :
 - Max. difference in std. dev in displacement in x global direction – 0.3 m (Deviation of approx. 8.04%)
- Significant time saving in simulation can be obtained by using a quasi-static mooring line model for lattice simulations.

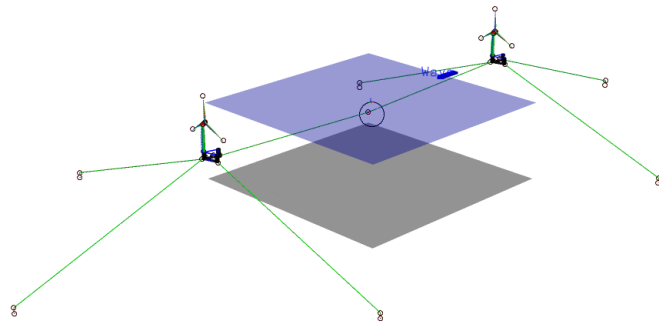
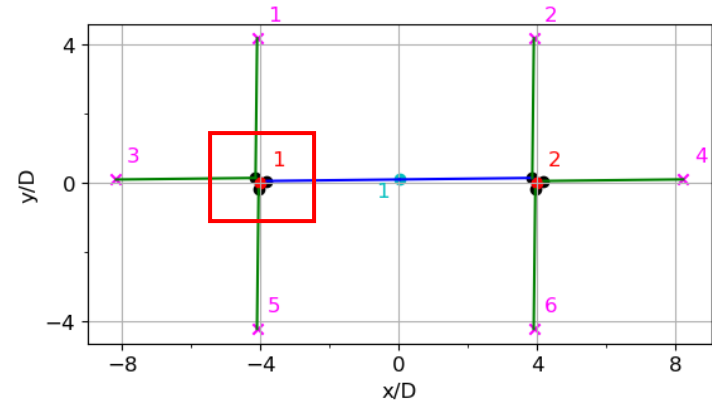


Thank you

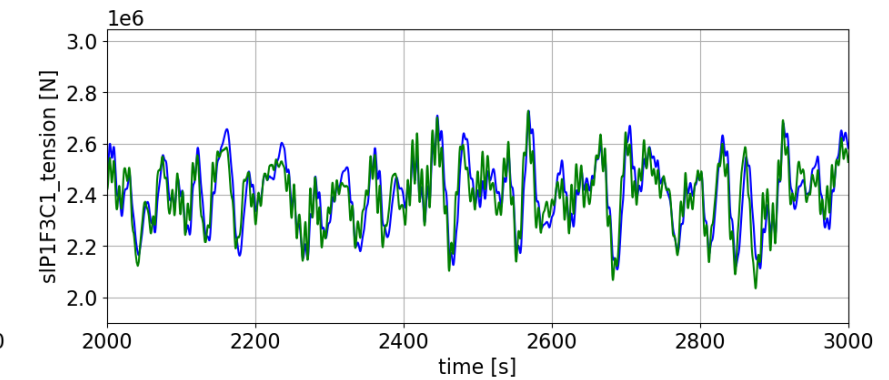
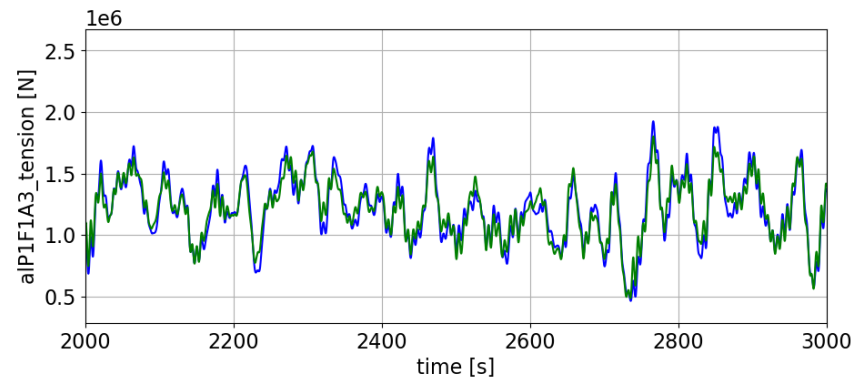
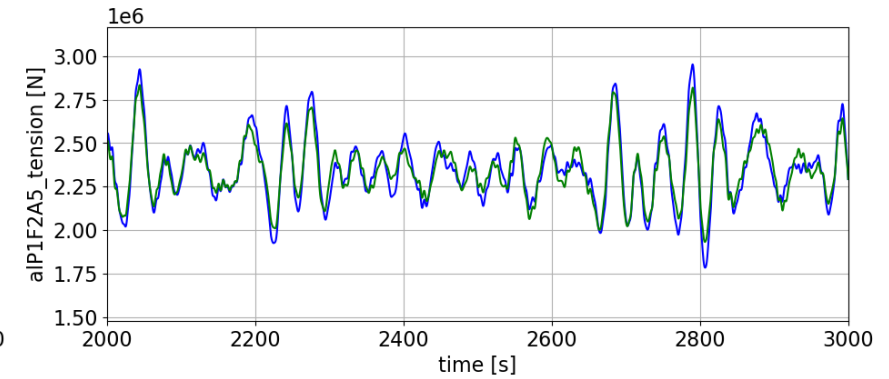
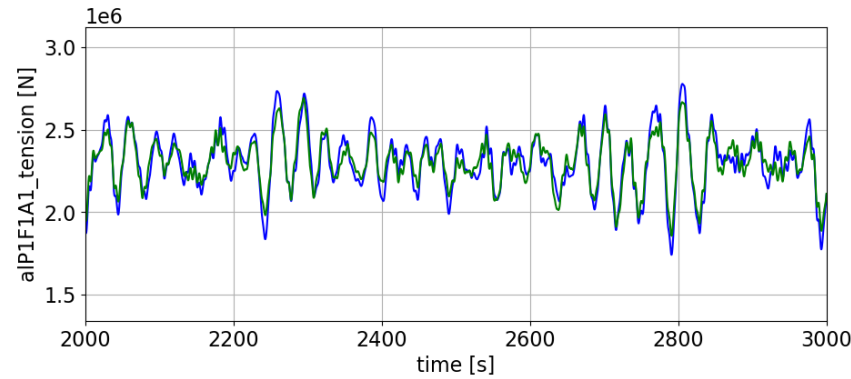
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Grid21 – Tension time series

Small motions of the platform \Rightarrow good agreement between dynamic and quasi-static tensions

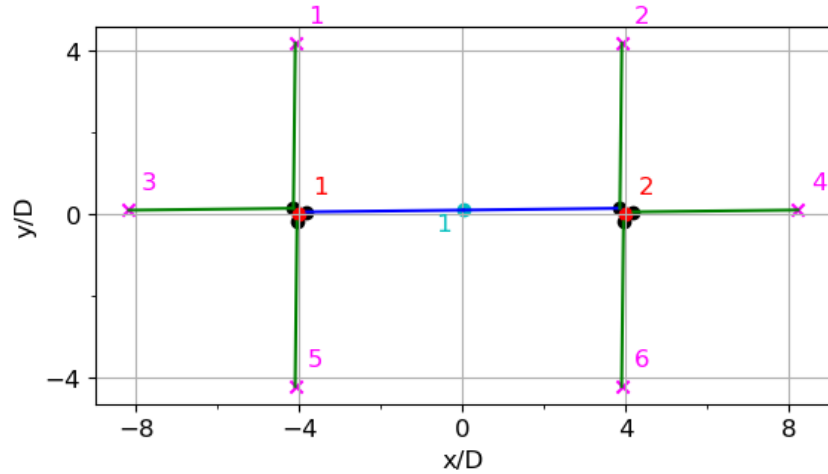


Comparison of tension time series for P1semi in grid_2_1 D213_clmp30



— cat — riflex

Grid21 – Tension statistics



- Max tension occurs in P2 al_4 as expected due to thrust accumulation.
- Similar tensions from both dynamic as well as quasi-static approaches.
- No slacking of the lines in any of the designs for this sea state.
- Max tensions much less than MBL → this may not be the driving condition.

