



An improved coupled frequency domain model for FOWT

Gijs Bouman

17-01-2024

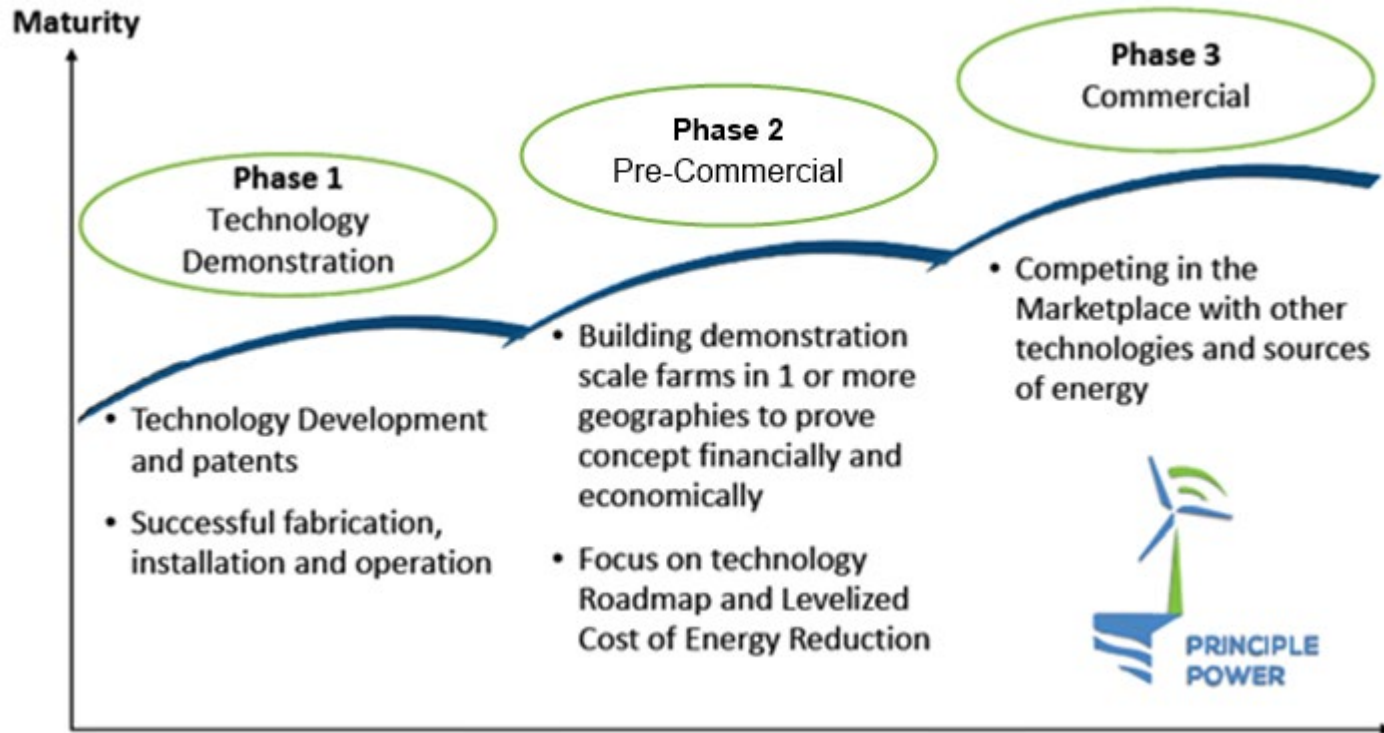


BETTER SHIPS, BLUE OCEANS

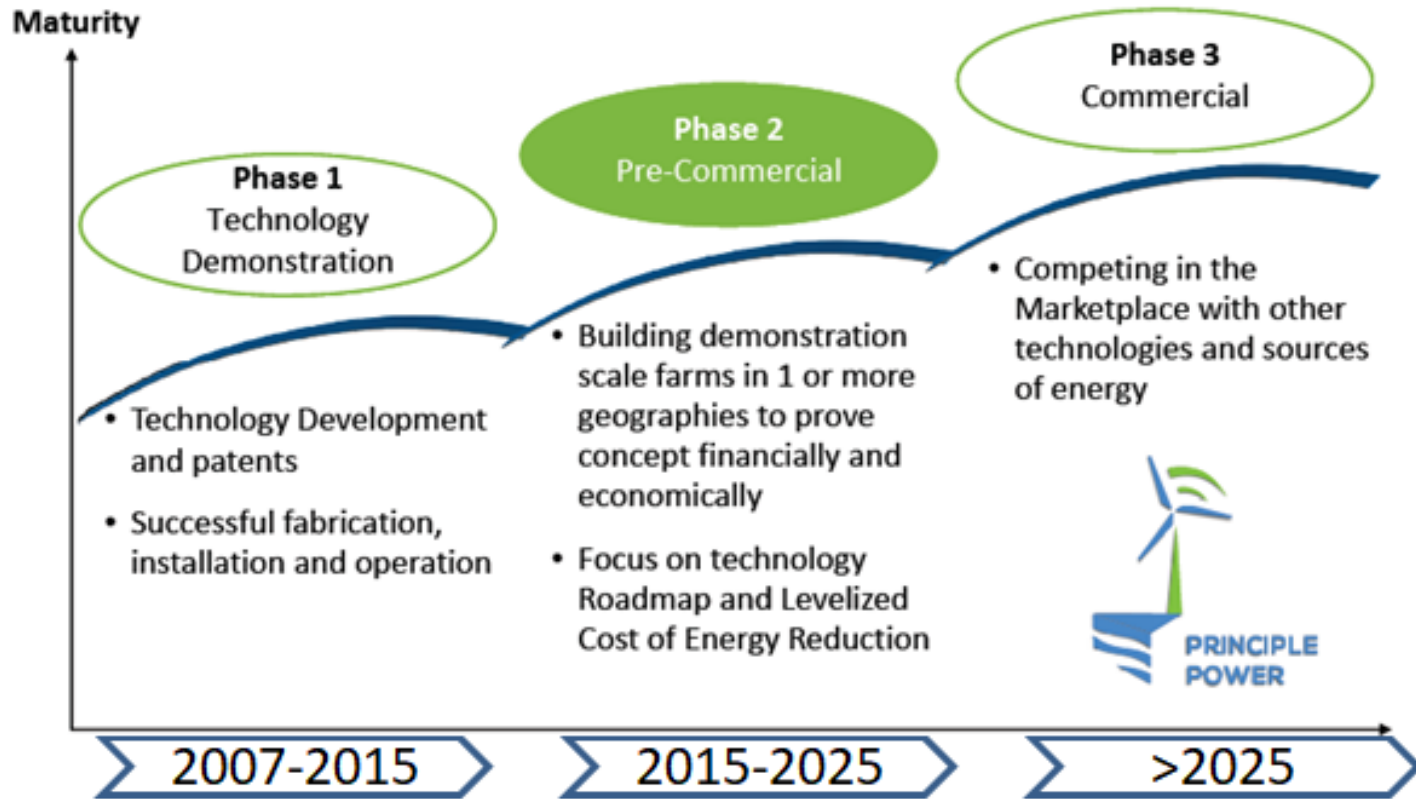


Introduction (and a question) —

How far is floating wind?



How far is floating wind?



- Optimization and push towards commercial phase results in many concepts in:
 - Floaters (type, size, material)
 - Turbines (size, type, blade pitch control)
 - Mooring systems (type, steel/synthetic)
 - Environments (mild/harsh, deep/shallow)



Sources: BW Ideol

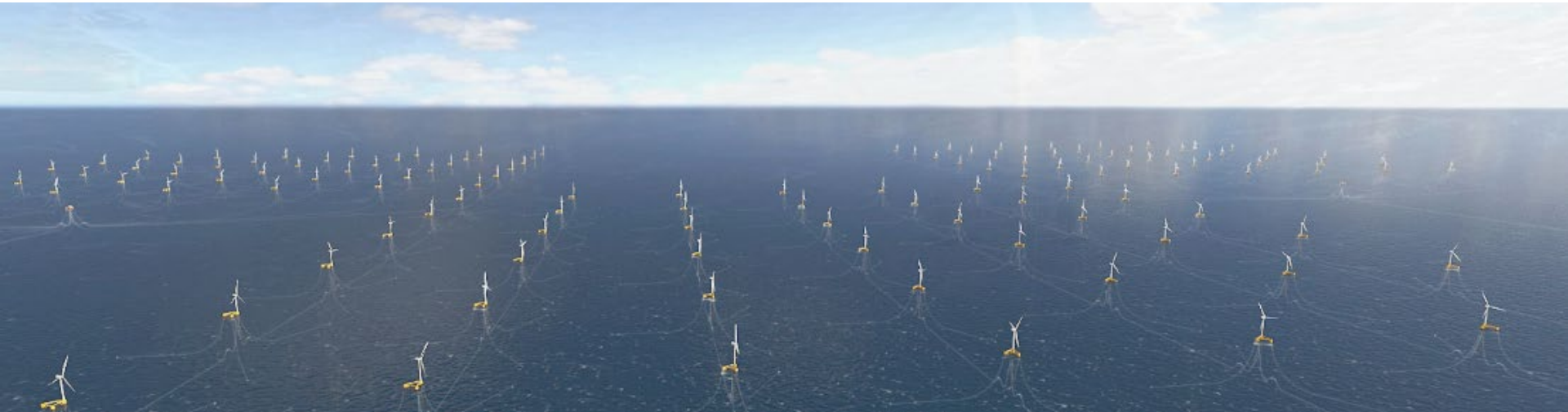
Principle power

SBM

Hexicon

X1 Wind

- In commercial phase, still work remaining to size designs for:
 - Environmental conditions (also within farm)
 - Water depth
 - Turbine size



Example: sizing of 22MW open-source reference platform

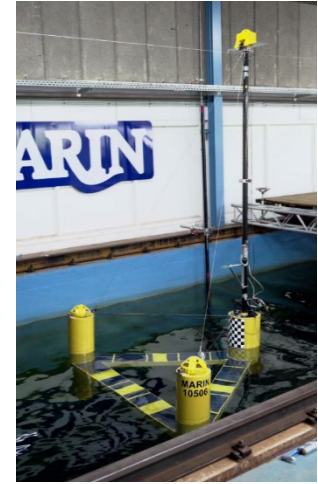
- Parametric design
- Design space screening in frequency domain
- Governing criteria found:
 - Platform tilt in operational sea (wind-dominated)
 - RNA accelerations in survival case (wave-dominated, parked rotor)



	MPM RNA accel. [m/s ²] Survival	MPM pitch angle [deg] Operational
Frequency domain	+2.74	-3.9

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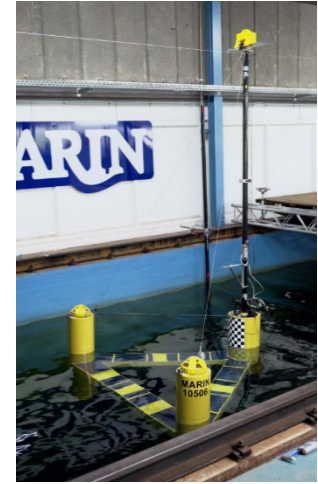
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Time domain/basin	+2.82	-5.0

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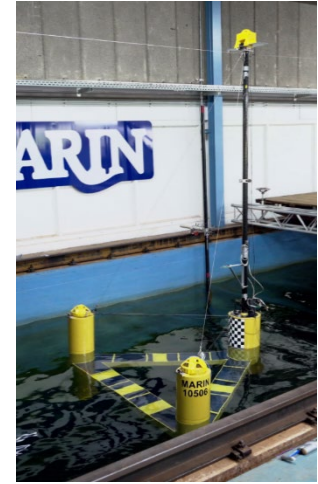
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- Validation in time domain and basin test
- **Missing dynamics from rotor and controller!**



	MPM RNA accel. [m/s ²] Survival	MPM pitch angle [deg] Operational	Aerodynamic modelling
Frequency domain	+2.74	-3.9	Constant force
Time domain/basin	+2.82	-5.0	BEM model, blade pitch controller

- ...an efficient evaluation tool
- ...to resolve the coupled aero-hydro-servodynamic response
- ...the floater motion spectra
- ...mooring line tension spectra

Methodology

- Published 2022 by NREL
- Open-source toolbox, available on GitHub repository
- Idea: find coupled system mass, stiffness and damping for use in frequency domain
- Aerodynamic added mass and damping found from linearization

The Science of Making Torque from Wind (TORQUE 2022) IOP Publishing
Journal of Physics: Conference Series 2265 (2022) 042020 doi:10.1088/1742-6596/2265/4/042020

An Open-Source Frequency-Domain Model for Floating Wind Turbine Design Optimization

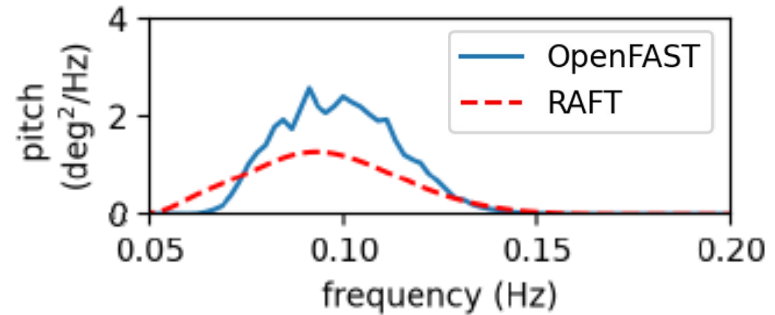
Matthew Hall, Stein Housner, Daniel Zalkind, Pietro Bortolotti,
David Ogden, Garrett Barter
National Renewable Energy Laboratory
15013 Denver West Parkway, Golden, CO, USA
E-mail: matthew.hall@nrel.gov

Abstract. A new frequency-domain dynamics model has been developed that uses open-source components to efficiently represent a complete floating wind turbine system. The model, called RAFT (Response Amplitudes of Floating Turbines), incorporates quasi-static mooring reactions, strip-theory and potential-flow hydrodynamics, blade-element-momentum aerodynamics, and linear turbine control. The formulation is compatible with a wide variety of support structure configurations and no manual or time-domain preprocessing steps are required, making RAFT very practical in design and optimization workflows. The model is applied to three reference floating wind turbine designs and its predictions are compared with results from time-domain OpenFAST simulations. There is good agreement in mean offsets as well the statistics and spectra of the dynamic response, verifying RAFT's general suitability for floating wind analysis. Follow-on work will include verification of potential-flow and turbine-control features and application to optimization problems.

1. Introduction

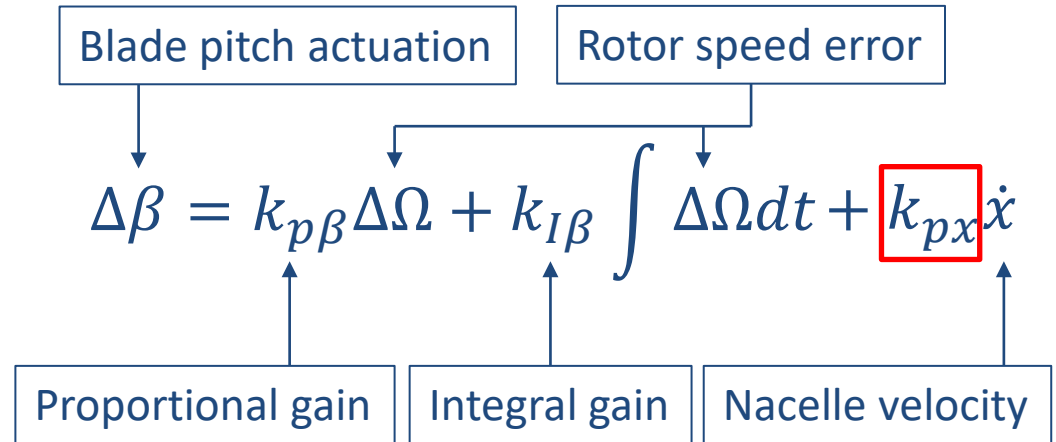
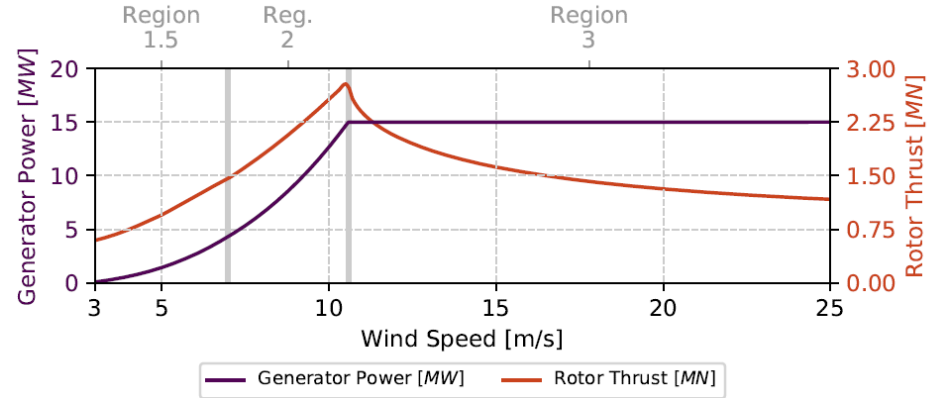
Frequency-domain models are an important tool for designing floating structures because they

- Overall, good results obtained when compared to time-domain
- Mismatch in pitch and mooring tension response for semi-submersible (15MW VoltturnUS-S)
- Differences attributed to:
 - Hydrodynamic modelling (strip theory)
 - Mooring system modelling (quasi static)
- Objective: improve pitch response prediction by:
 - Improving blade pitch control implementation
 - Coupling RAFT to MARIN wave diffraction code

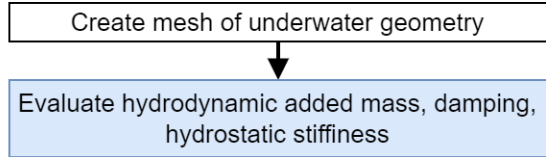


Blade pitch control: velocity feedback

- PI controller, with setpoint on RPM
- Negative slope in thrust curve above rated wind
- Negative floater pitch damping for above-rated wind speeds
- Solution: nacelle **velocity feedback**
- Implemented with low-pass filter



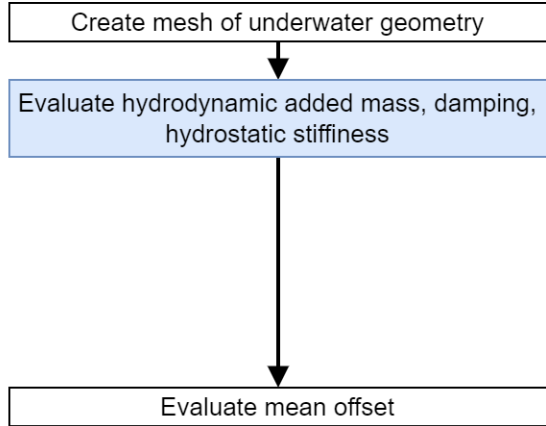
DIFFRAC



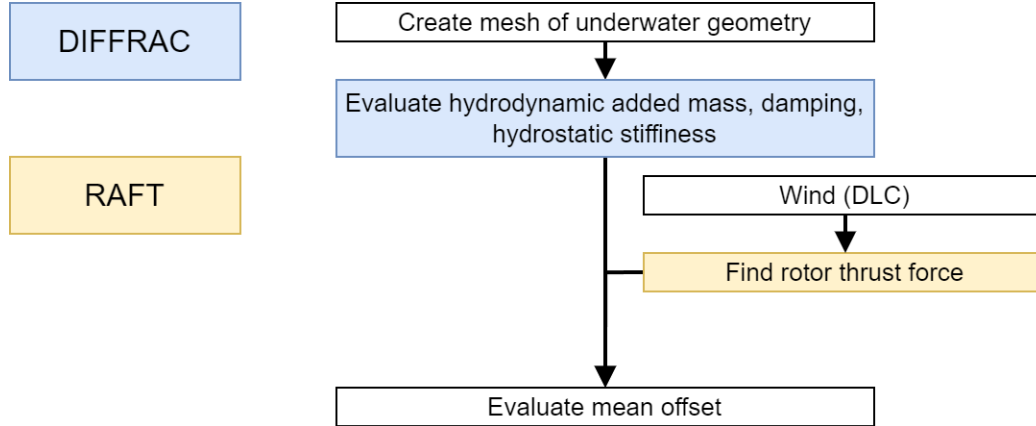
Hydrodynamic look-up table (RAO)

- Most expensive step
- Done for every geometry

DIFFRAC



All linearization done around mean state



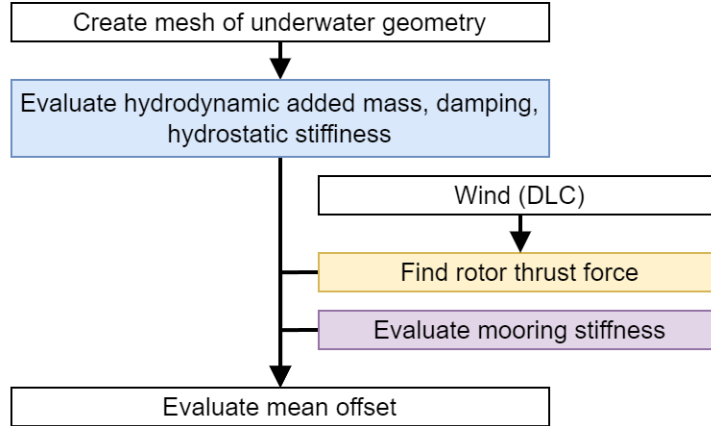
Done for each:

- Turbine
- Blade pitch controller
- Wind condition

DIFFRAC

RAFT

aNyMOOR

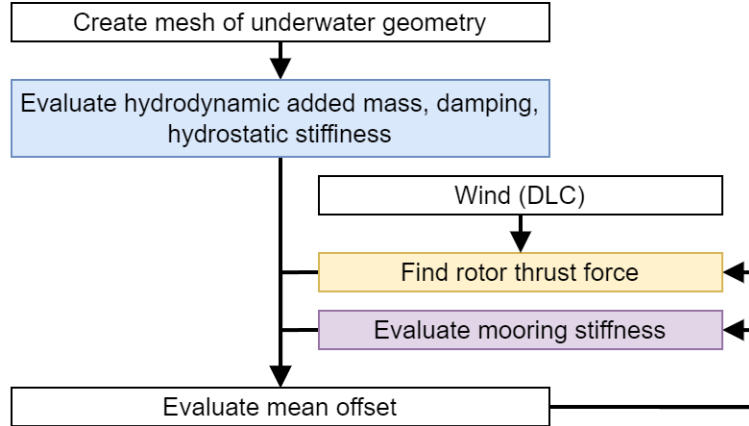


Static mooring system

DIFFRAC

RAFT

aNyMOOR



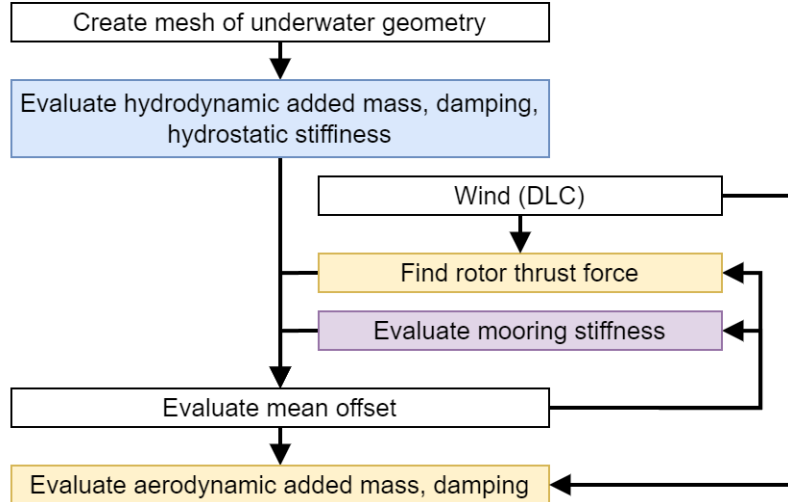
Iterations needed:

- Account for rotor tilt
- Mooring stiffness linearized at offset

DIFFRAC

RAFT

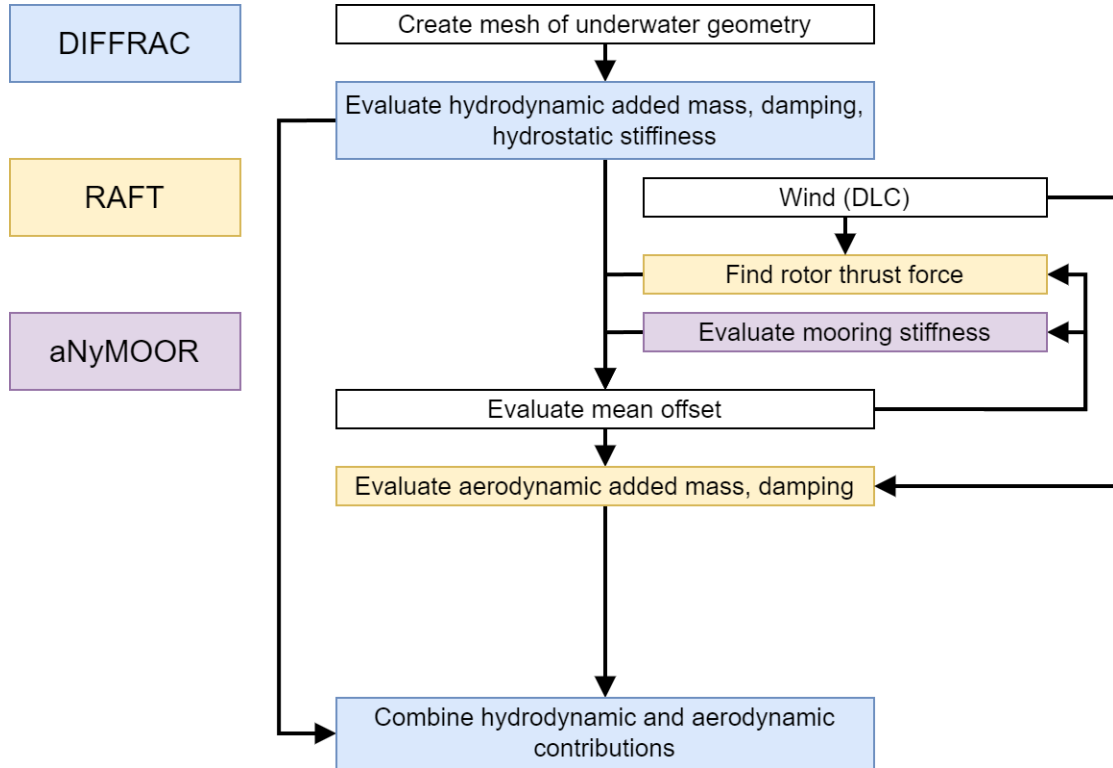
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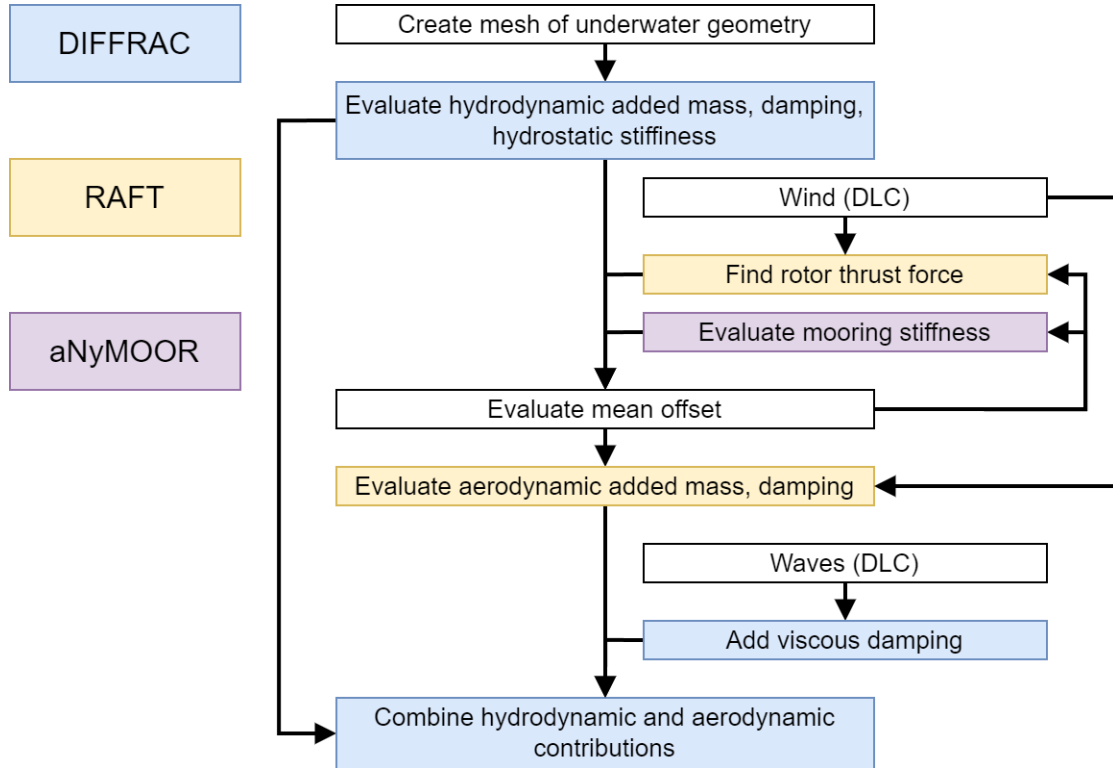
RAFT used to evaluate aerodynamic mass/damping matrices around mean

Includes blade pitch controller!

Coupling to MARIN hydrodynamic tools

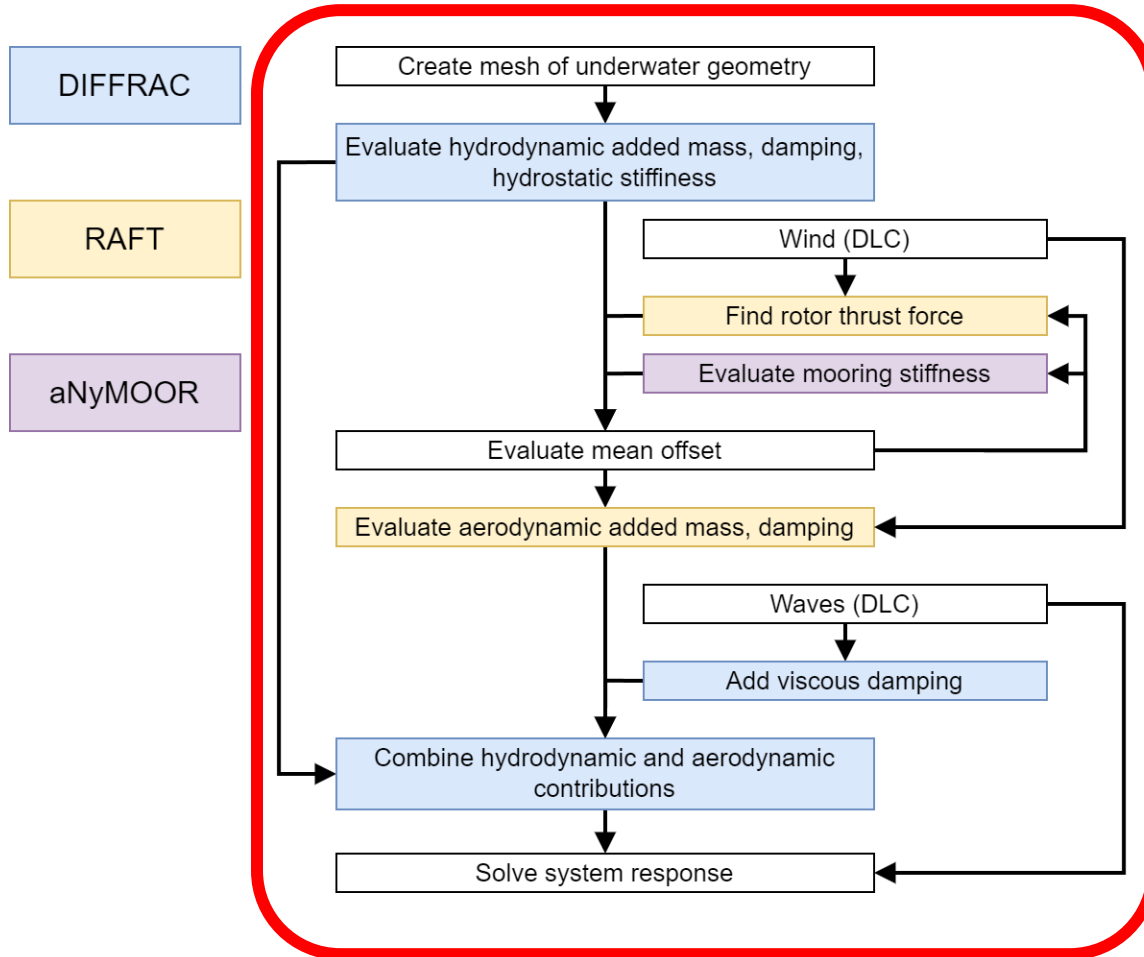


Coupling to MARIN hydrodynamic tools



Viscous damping
linearized for sea state

Coupling to MARIN hydrodynamic tools



Combined system subjected to wave excitation

“RAFT extended”



BETTER SHIPS, BLUE OCEANS



Results

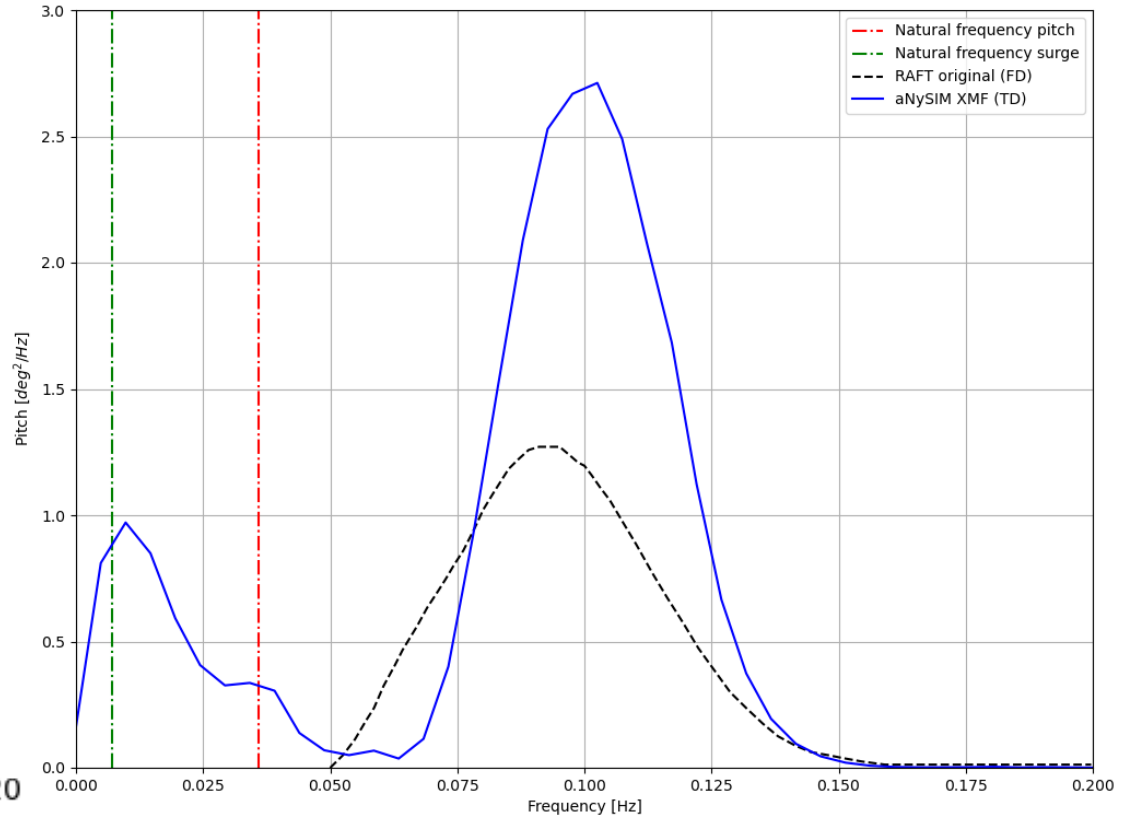
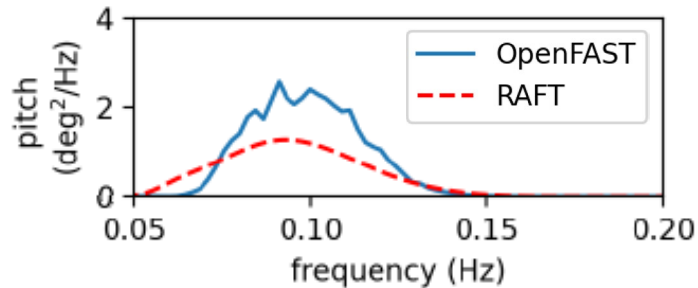
- VoltornUS-S platform
 - 15MW
 - Semi-submersible
 - Catenary mooring
- Natural frequencies
 - Surge: 0.007 Hz (142.9 s)
 - Pitch: 0.036 Hz (27.8 s)
- ROSCO controller
 - Nacelle acceleration feedback to actively dampen floater pitch motion



Source: Allen, Christopher, Anthony Viselli, Habib Dagher, Andrew Goupee, Evan Gaertner, Nikhar Abbas, Matthew Hall, and Garrett Barter. Definition of the UMaine VoltornUS-S Reference Platform Developed for the IEA Wind 15-Megawatt Offshore Reference Wind Turbine. Golden, CO: National Renewable Energy Laboratory. NREL/TP-5000-76773. <https://www.nrel.gov/docs/fy20osti/76773.pdf>.

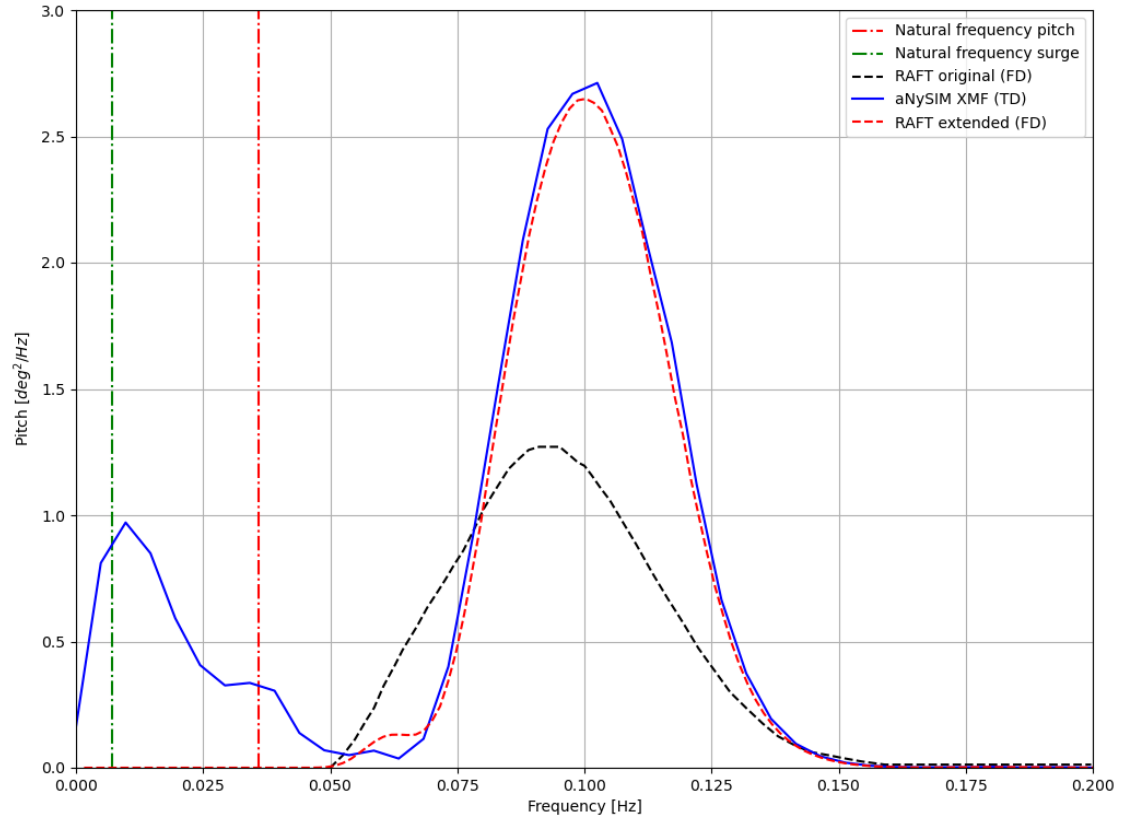
Wave-dominated, below-rated case

- $H_s = 6.0$ m
- $T_p = 12.0$ s
- $u_{wind} = 8$ m/s
- Constant wind



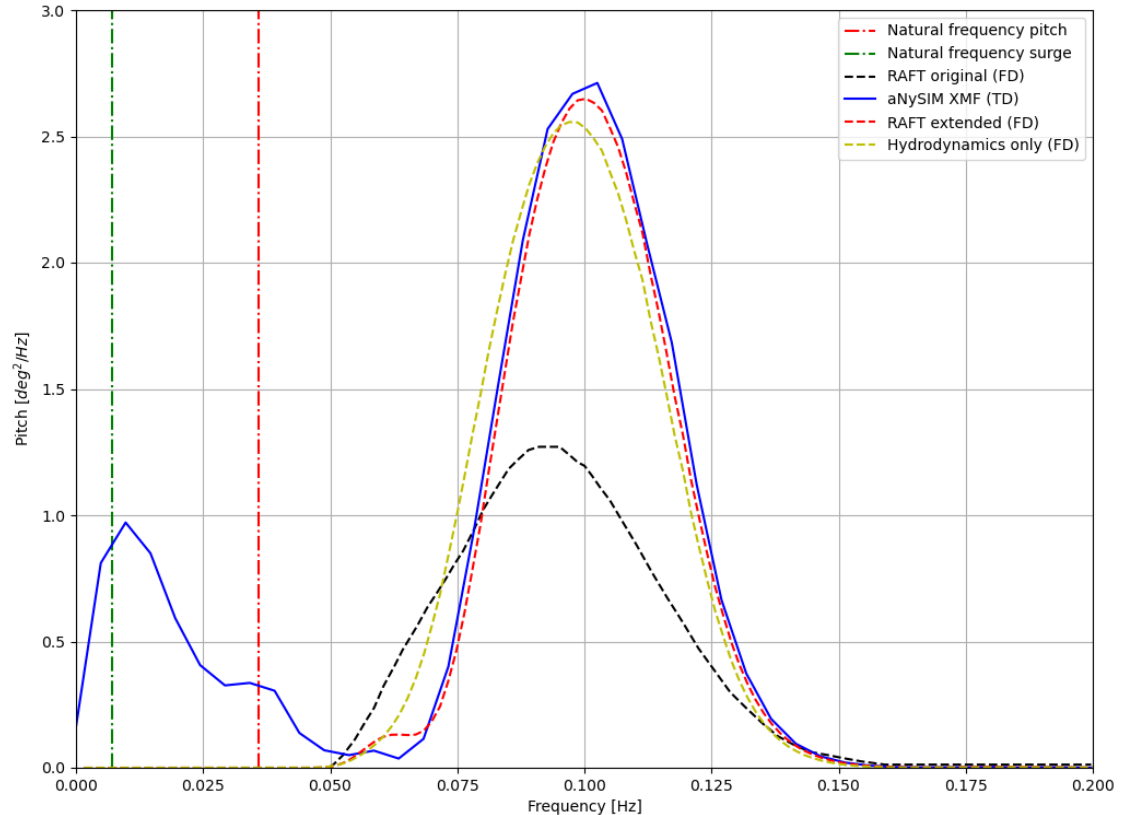
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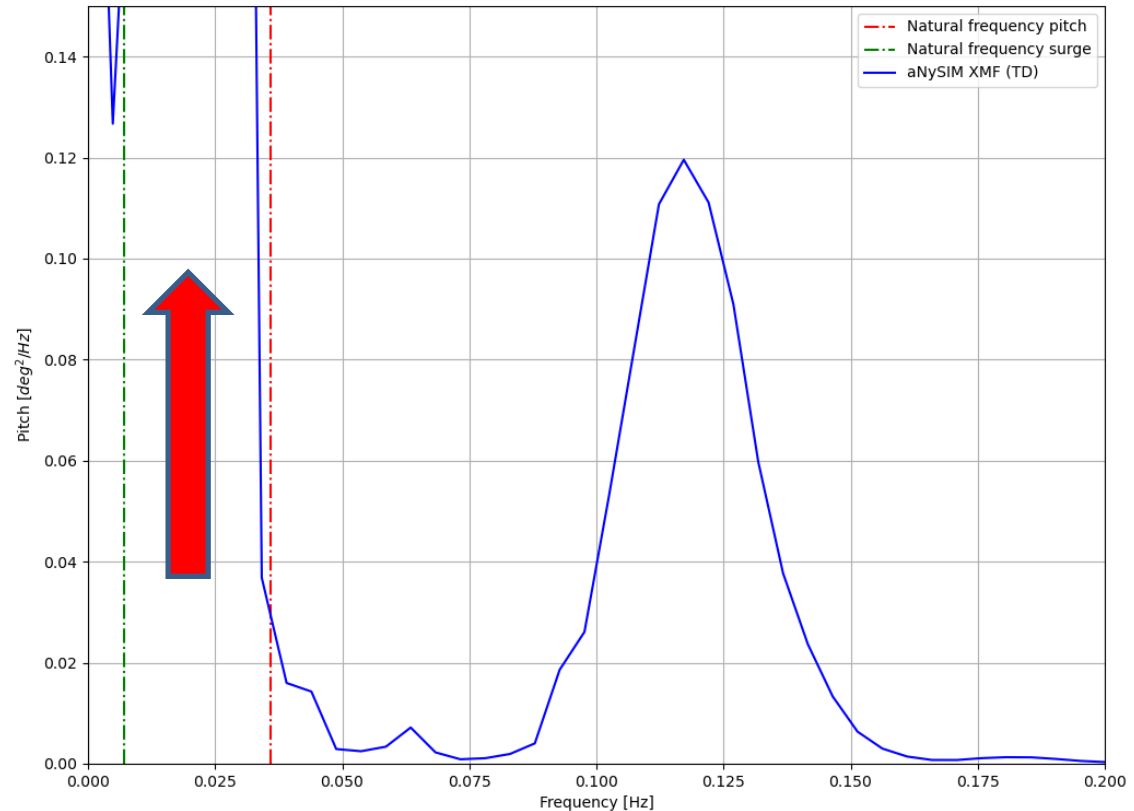


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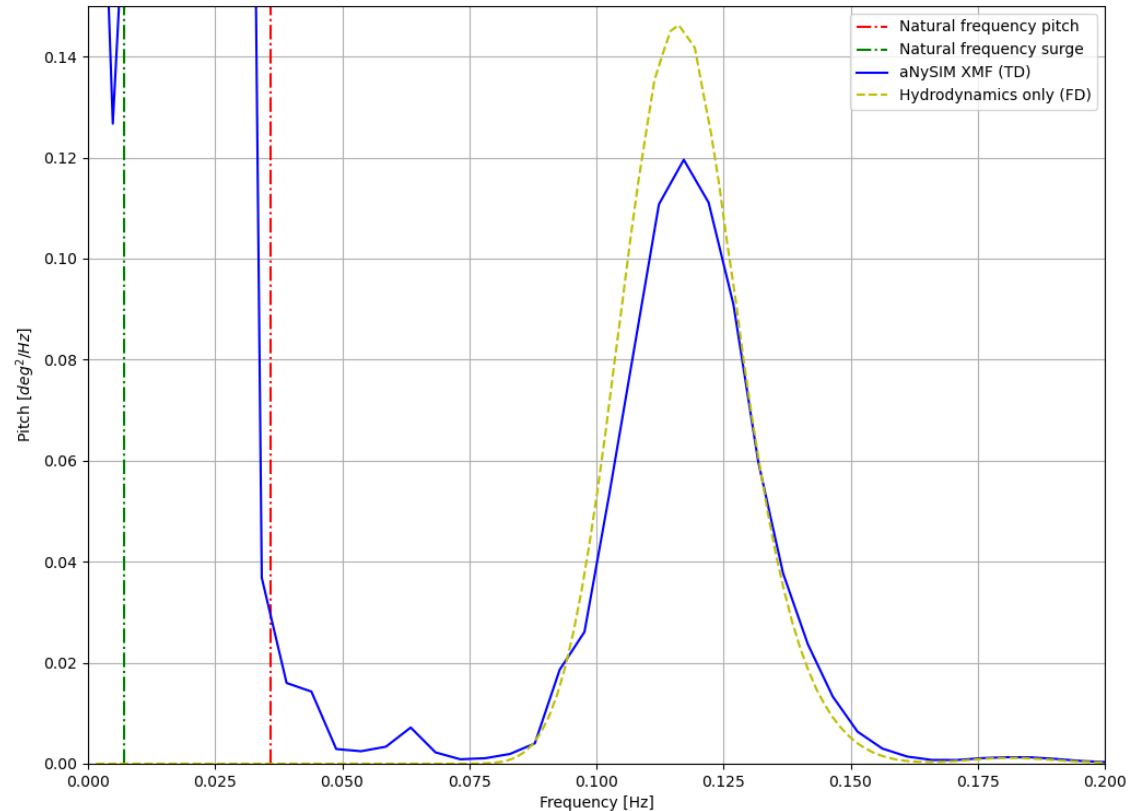


- $H_s = 1.84$ m
- $T_p = 7.44$ s
- $u_{wind} = 12.0$ m/s
- Steady wind
- Strong blade pitch actuation seen near natural frequencies



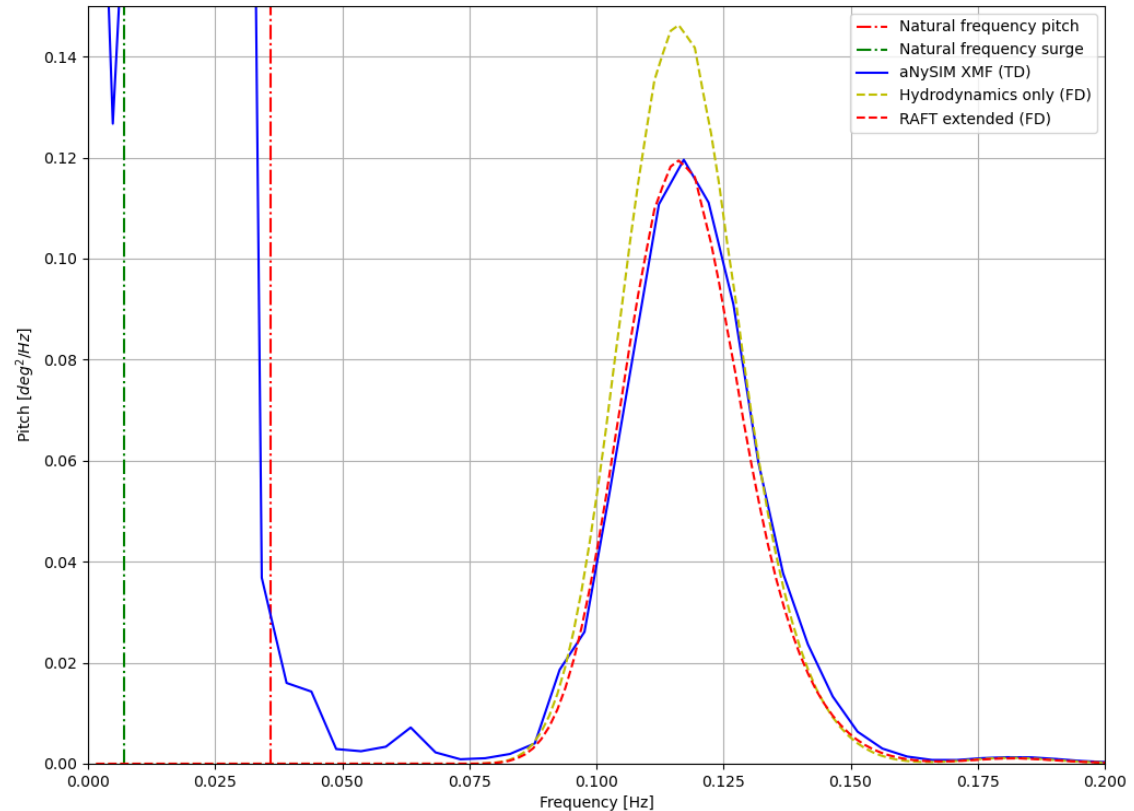
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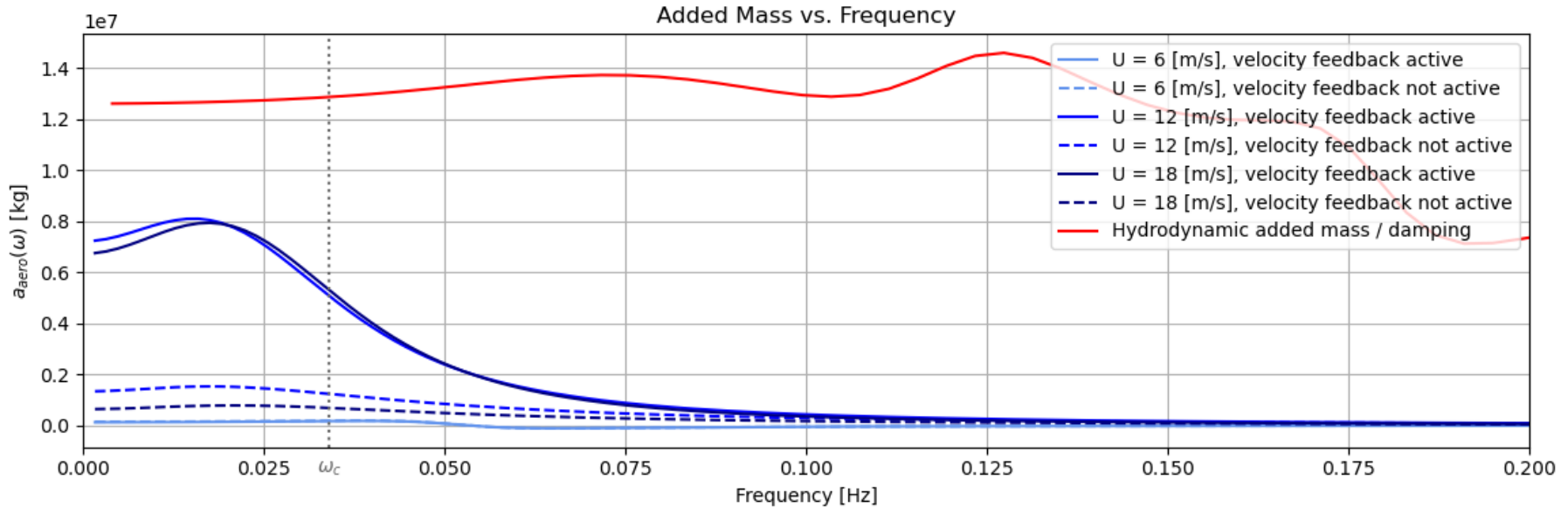


Wind-dominated, above-rated case

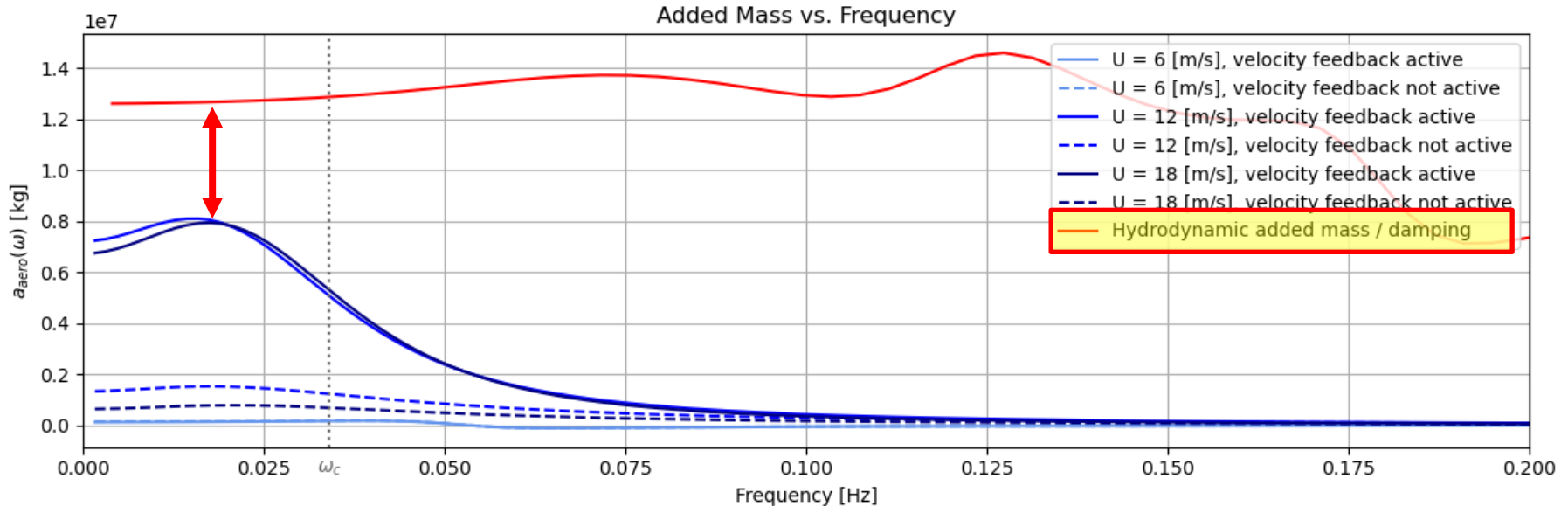
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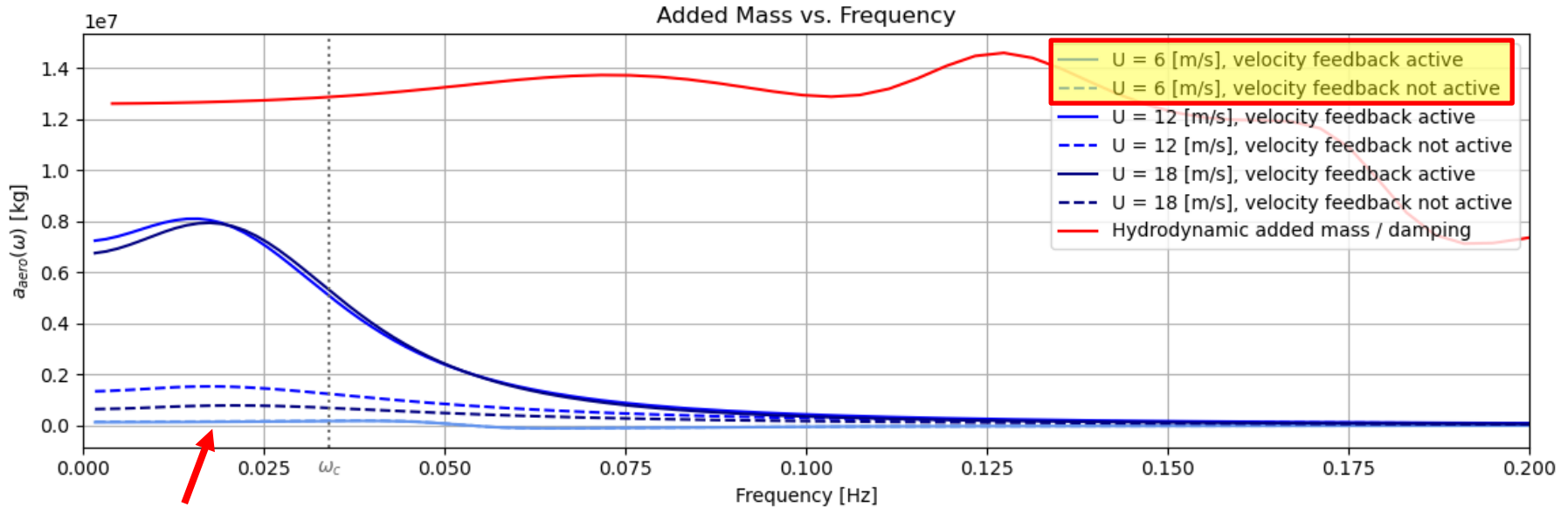
- So, what happens here?
- Rotor and blade pitch controller create a response that depends on the platform motions
 - Acceleration-dependent forces: added mass
 - Velocity-dependent forces: damping



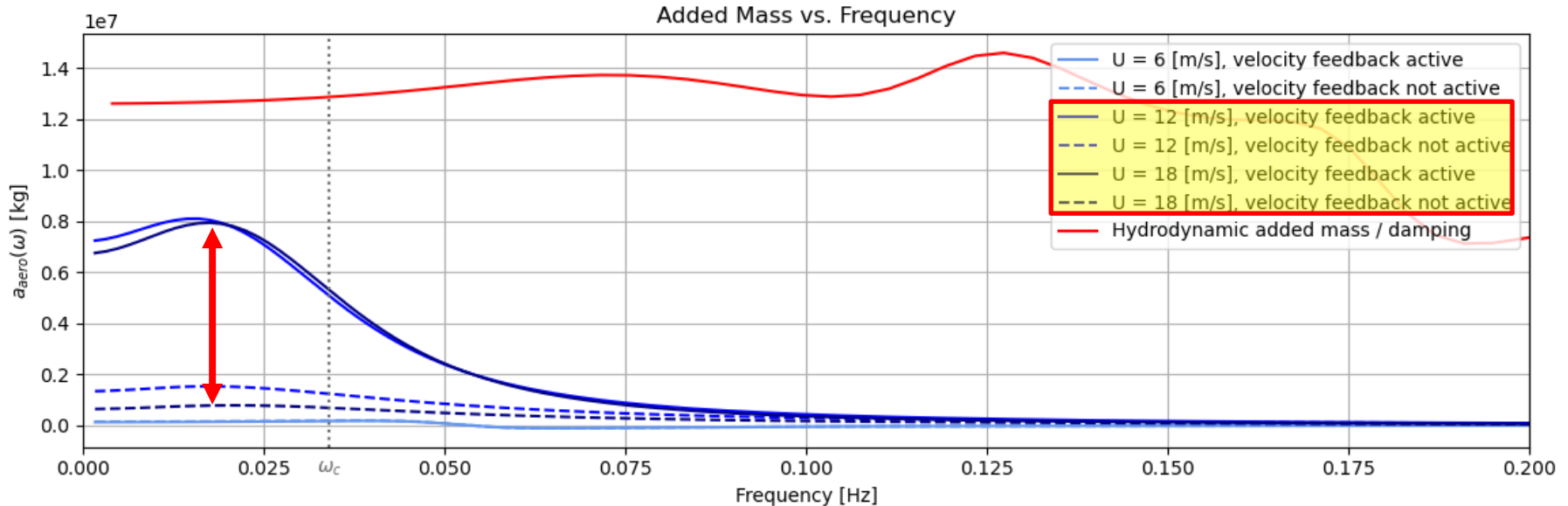
- For displacement in x-direction
- Total system mass: $2e7$ kg



- For displacement in x-direction
- Hydrodynamic added mass dominates over aerodynamic added mass

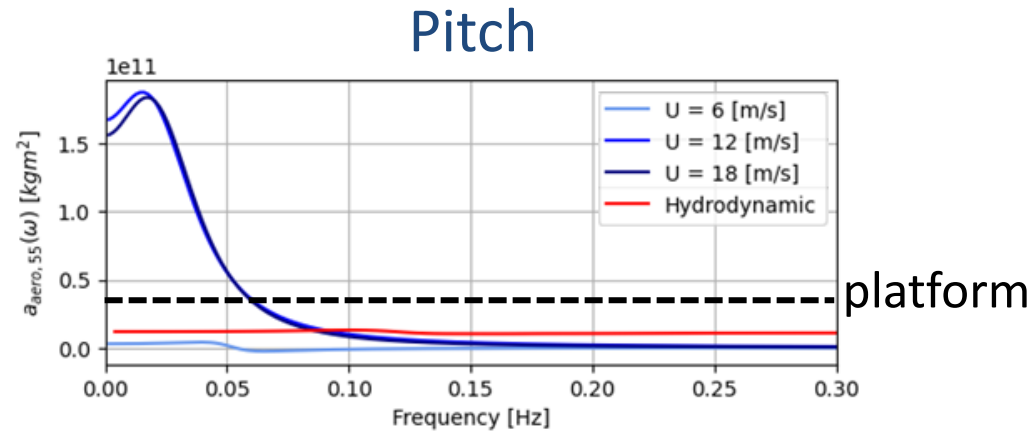
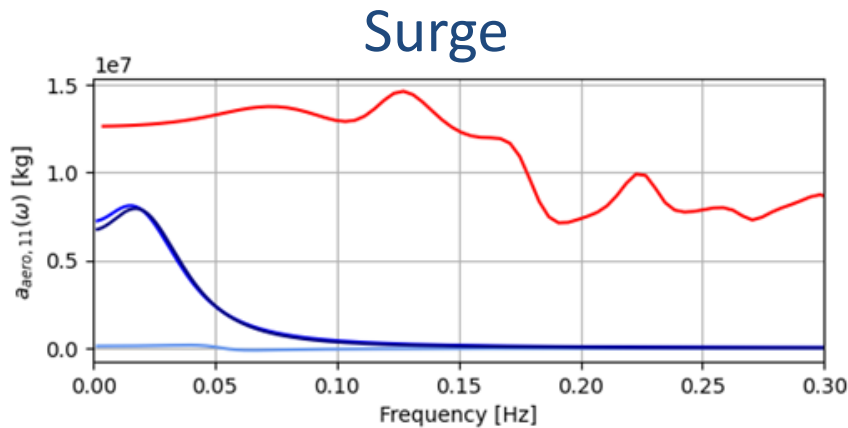


- For displacement in x-direction
- Below rated: small change made by controller feedback

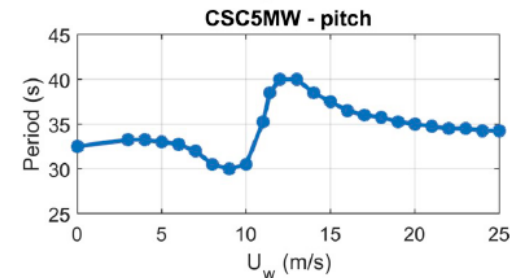
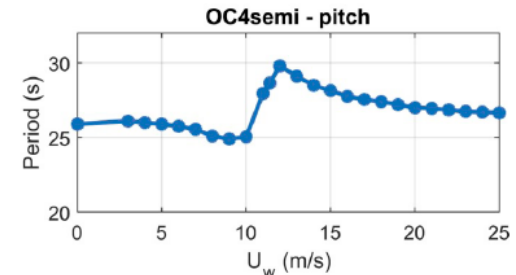
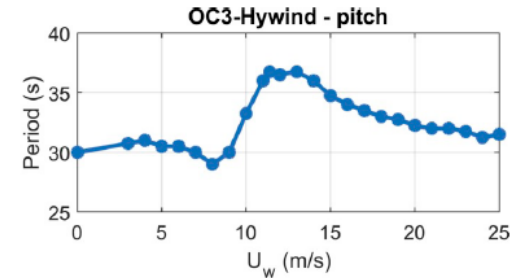


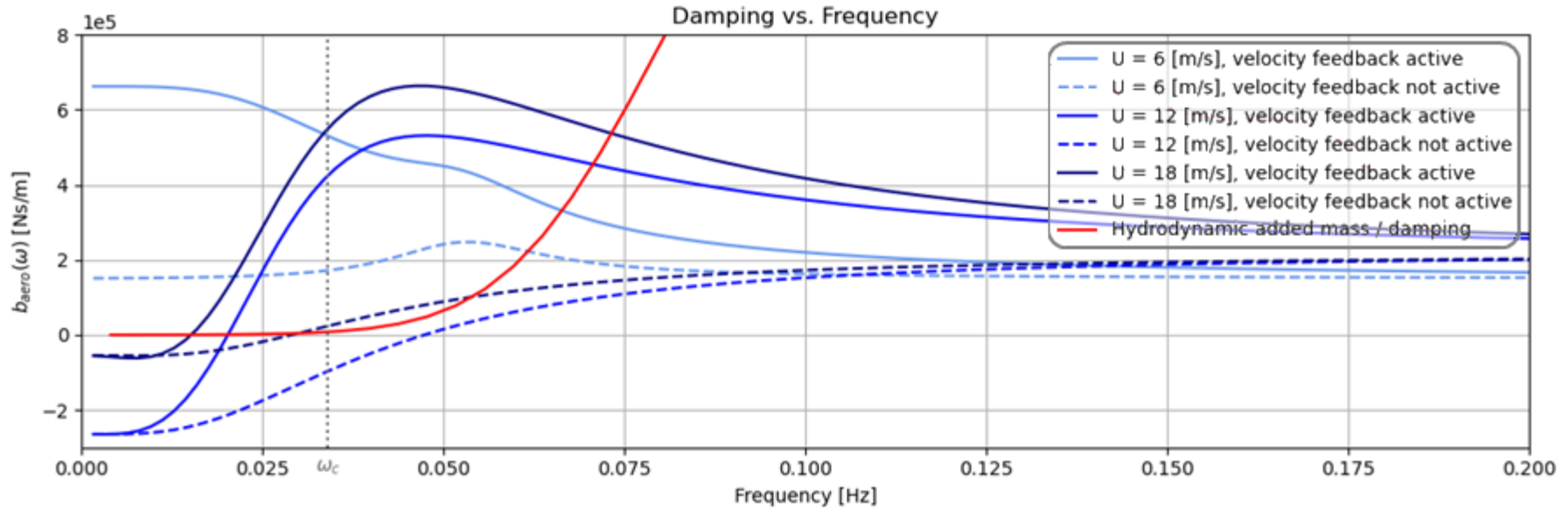
- For displacement in x-direction
- Above rated: large change made by controller feedback, strong dependence on wind speed and frequency!

- Aerodynamic effects occur at hub height: 150m
- Large effect on pitch mode due to large arm
- Platform pitch inertia: $4.2e10 \text{ kg} * \text{m}^2$

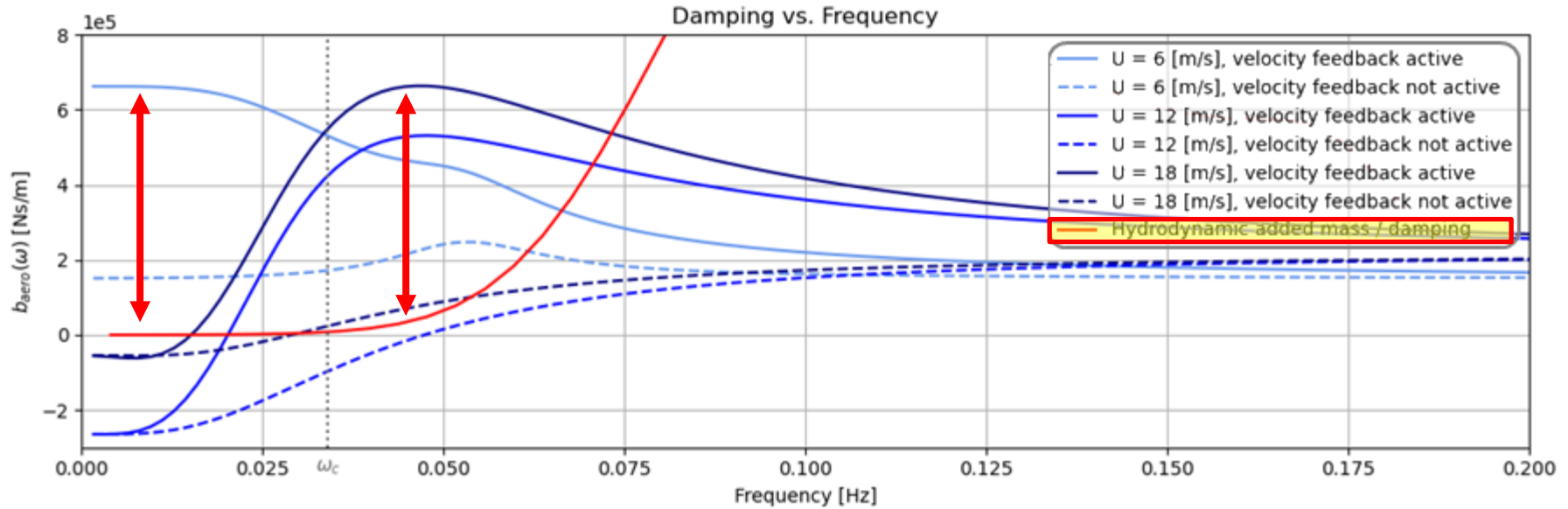


- Aerodynamic added mass dominant in pitch, depends on:
 - Wind speed
 - Frequency
 - Blade pitch controller strategy
- Seen in system natural period $\omega_n = \sqrt{k/m}$
 - Also for surge, but less extreme

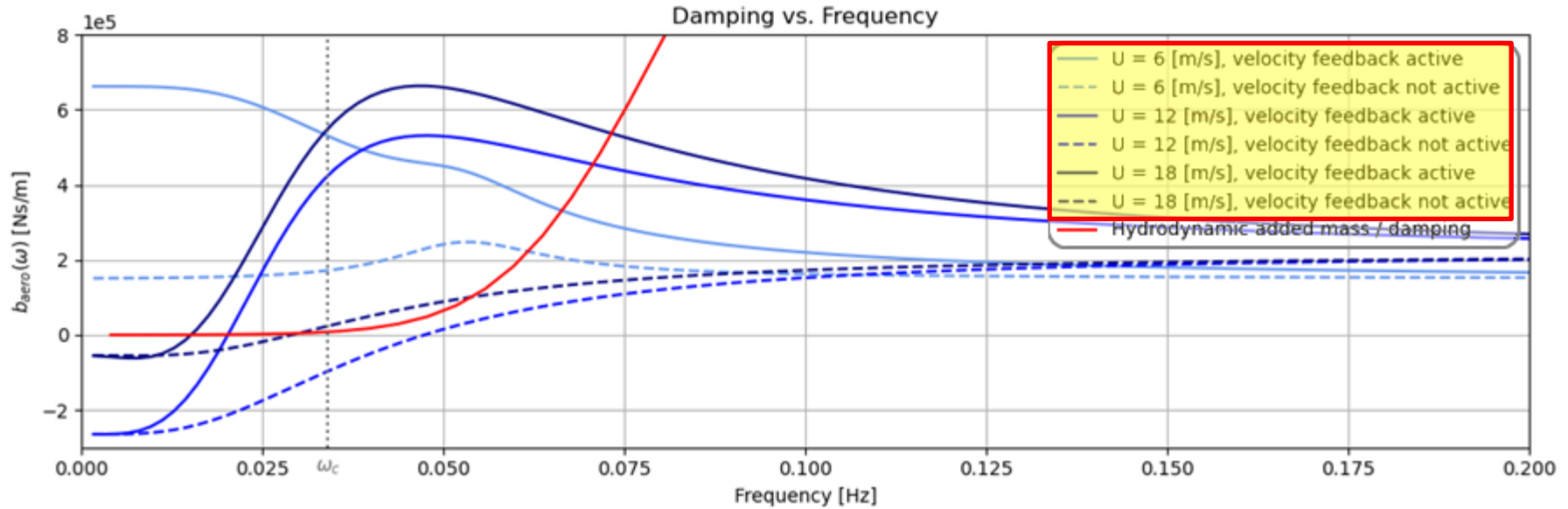




- For displacement in x-direction



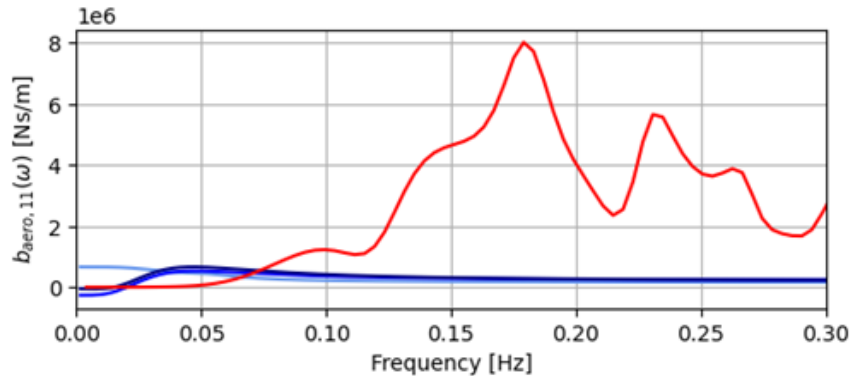
- For displacement in x-direction
- Aerodynamic damping dominates over hydrodynamic damping at lower frequencies



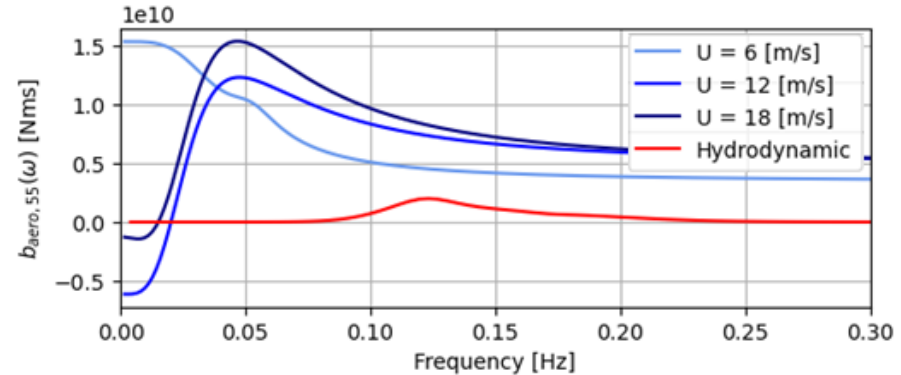
- Large dependence seen on:
 - Control strategy
 - Wind speed
 - Frequency

- Aerodynamic effects occur at hub height: 150m
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Surge

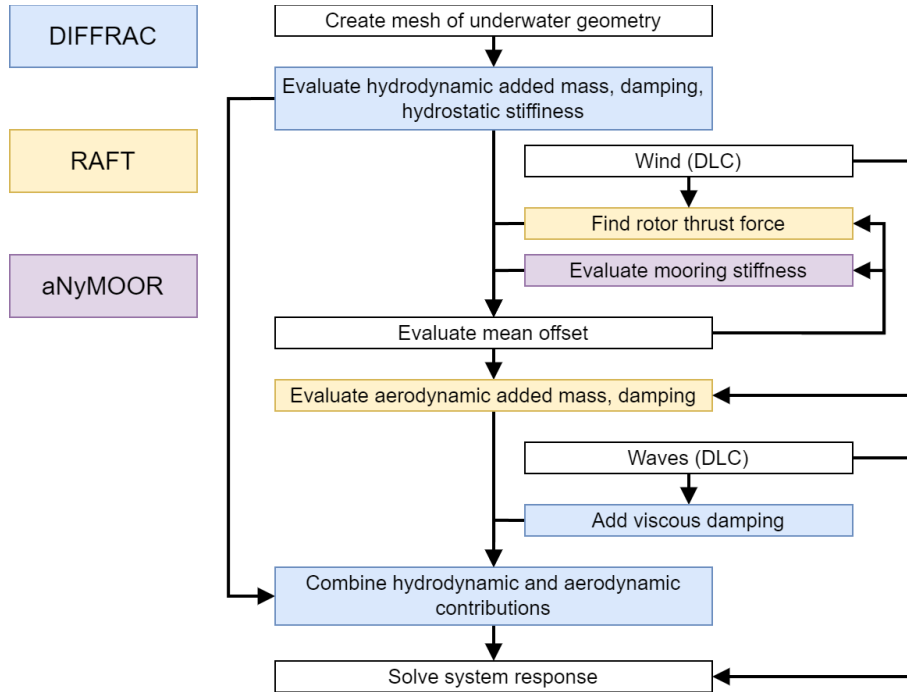


Pitch

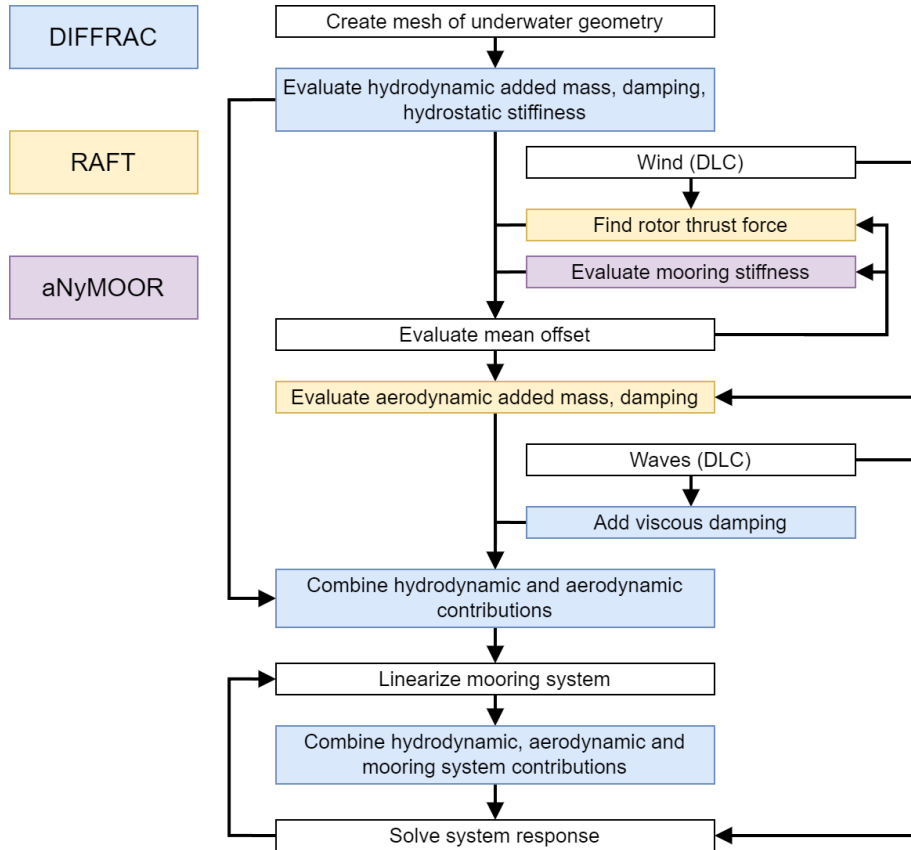


Conclusions and outlook

- Good match in motions in wave-frequent range
- Effects taken into account:
 - Aerodynamics**
 - Hydrodynamics
 - Mooring system dynamics → Mooring line tension
 - Turbulent wind excitation → Low-frequent motions
 - Controller (PI with feedback)
 - Structural flexibility

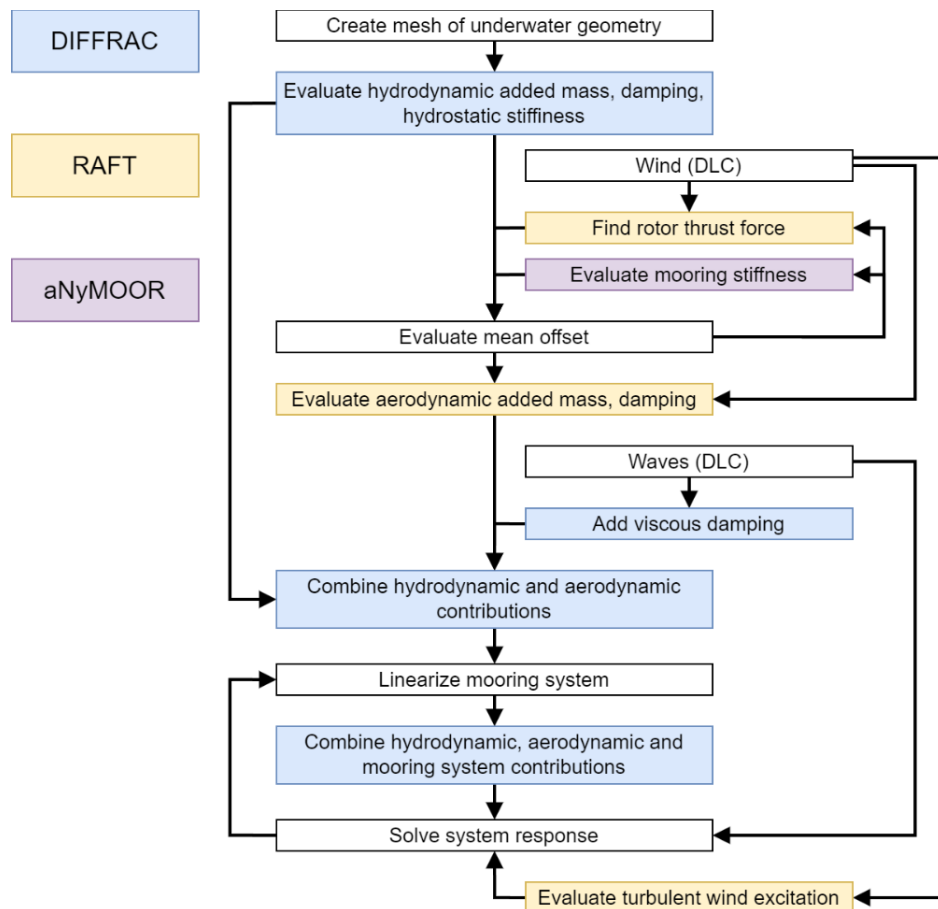


Current implementation



Mooring line dynamics

- Linearize inertia and drag forces
- Add to (frequency-dependent) mass, stiffness, damping matrices



Turbulent wind excitation

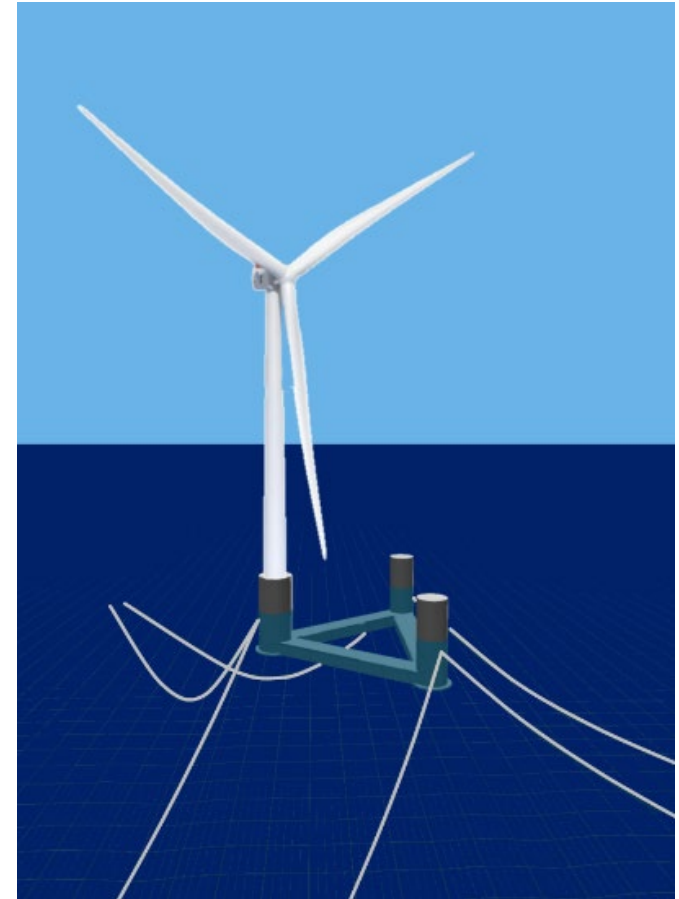
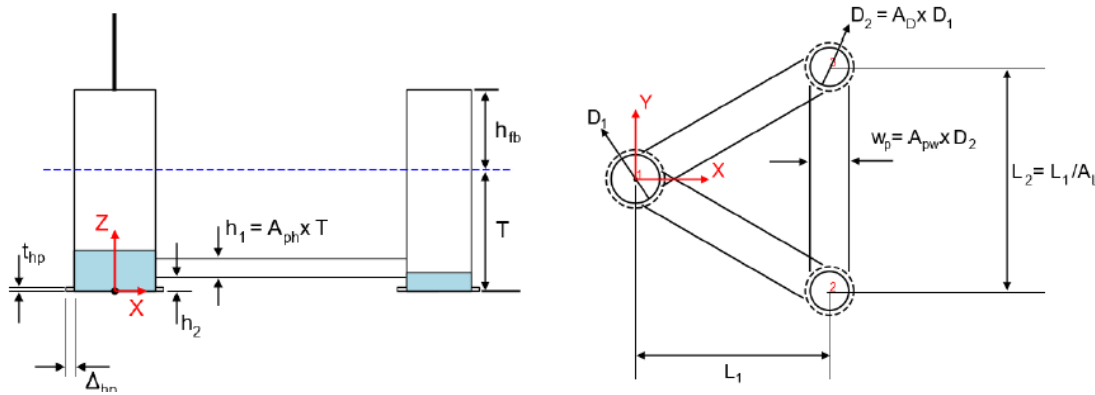
- RAFT describes transfer function H_{UF} to find the turbulent aerodynamic excitation force:

$$\hat{f}_{aero}(\omega) = H_{UF}(\omega)U(\omega)$$

- Wind and wave spectra disconnected \rightarrow superimpose
- Introduce $\hat{f}_{aero}(\omega)$ as an external, right-hand-side-force

Example: sizing of 22MW open-source reference platform

- Parametric design
- Coupled motion spectra resolved \rightarrow find correct MPM platform pitch angle
- Allows to evaluate hundreds of designs in a matter of hours-days.



- Resolve FOWT motions and mooring line tensions in early stage
 - Design screening
 - Optimization of floater geometry
 - Optimization of blade pitch controller
 - Response-based scaling for turbine and/or environment
- Resolve FOWT loads
- Spectra: both for ultimate (MPM) and fatigue evaluation!
- It may not be possible to linearize different control strategies

Thank you!



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- Linearize...
 - Thrust
 - Torque
 - Rotor dynamics
 - Controller action
- Rewrite into...
 - Added mass $a_{aero}(\omega)$
 - Damping $b_{aero}(\omega)$
 - Excitation force $\hat{f}_{aero}(\omega)$

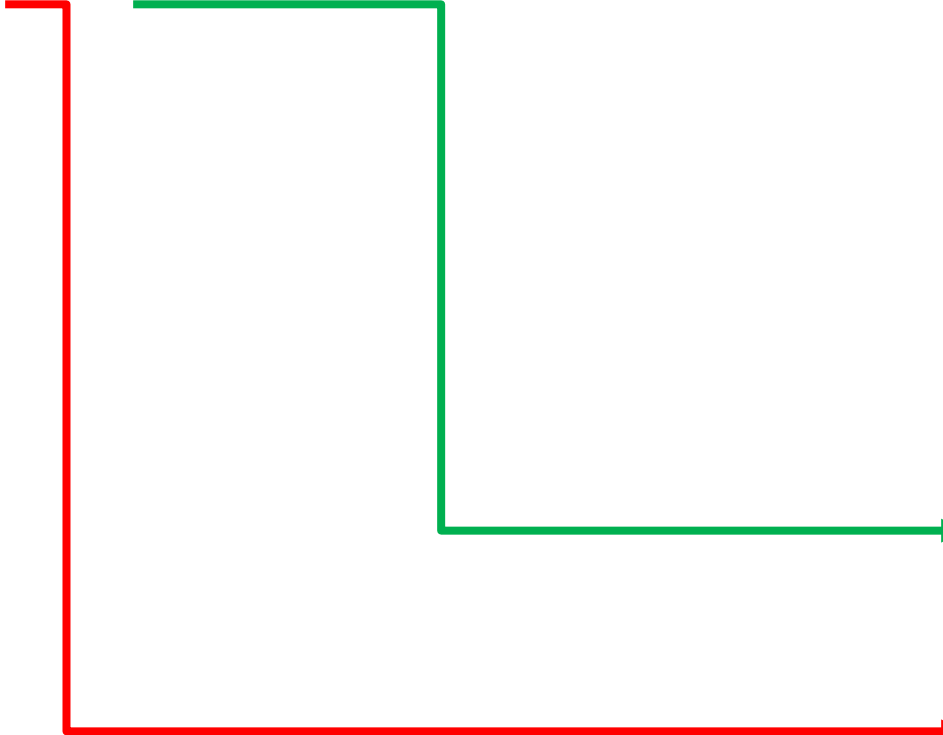
$$T = \bar{T} + T_U \Delta(U - \dot{x}) + T_\Omega \Delta\Omega + T_\beta \Delta\beta$$

$$T = \bar{T} + T_U \Delta(U - \dot{x}) + T_\Omega \Delta\Omega + T_\beta \Delta\beta$$



Mean thrust

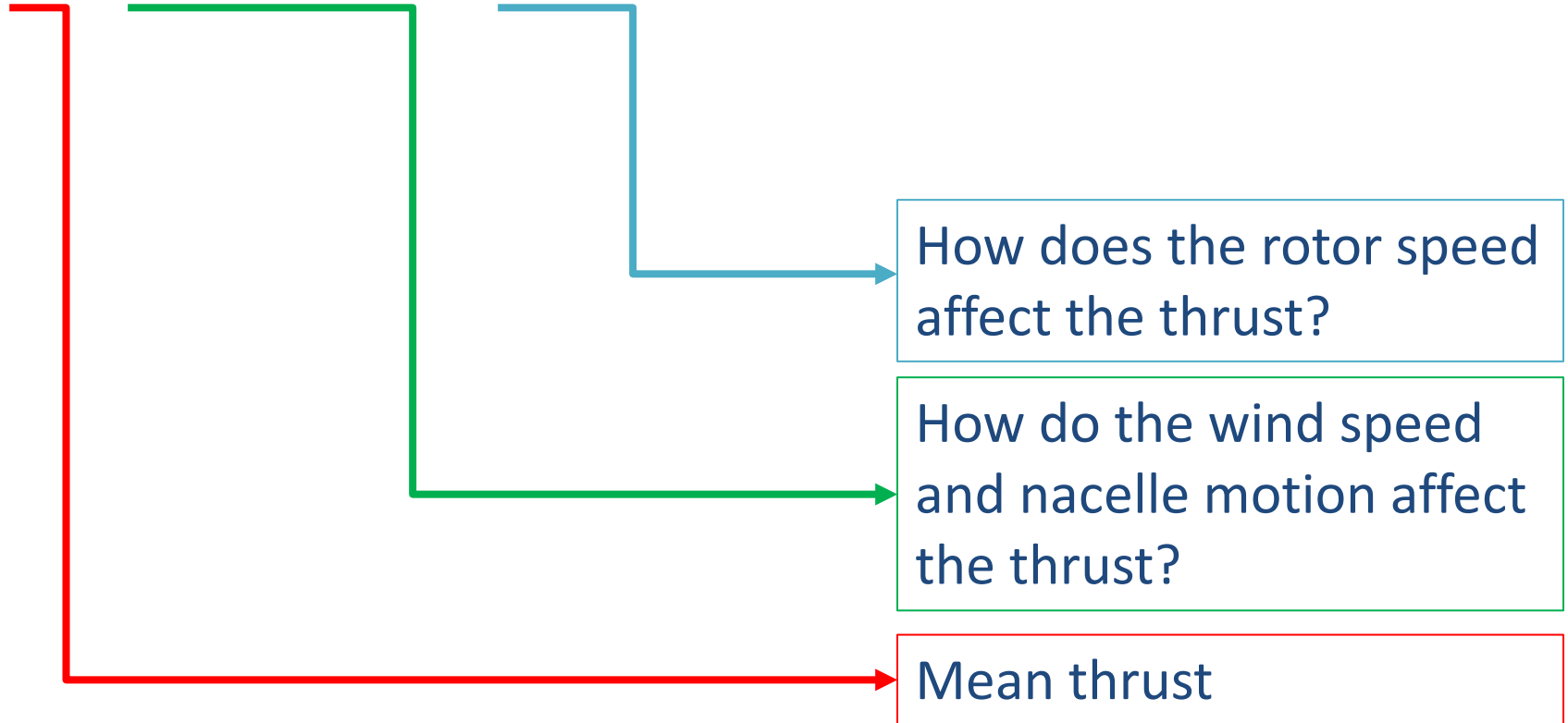
$$T = \bar{T} + T_U \Delta(U - \dot{x}) + T_\Omega \Delta\Omega + T_\beta \Delta\beta$$



How do the wind speed and nacelle motion affect the thrust?

Mean thrust

$$T = \bar{T} + T_U \Delta(U - \dot{x}) + T_\Omega \Delta\Omega + T_\beta \Delta\beta$$



$$T = \bar{T} + T_U \Delta(U - \dot{x}) + T_\Omega \Delta\Omega + T_\beta \Delta\beta$$

How does the blade pitch angle affect the thrust?

How does the rotor speed affect the thrust?

How do the wind speed and nacelle motion affect the thrust?

Mean thrust

Zeroth and second moment of response spectrum:

$$m_0 = \int S_w(\omega) * RAO(\omega)^2 d\omega$$

$$m_2 = \int S_w(\omega) * RAO(\omega)^2 * \omega^2 d\omega$$

Zero crossing period:

$$T_Z = 2\pi\sqrt{m_0/m_2}$$

Root-mean-square of response:

$$RMS = \sqrt{m_0}$$

Most probable maximum:

$$MPM = \sqrt{2 \ln N} * RMS$$

$$N = T_{eval}/T_Z$$