

Increased tower eigenfrequencies on floating foundations and their implications for large two- and three-bladed turbines

How a disadvantage turns
into an advantage

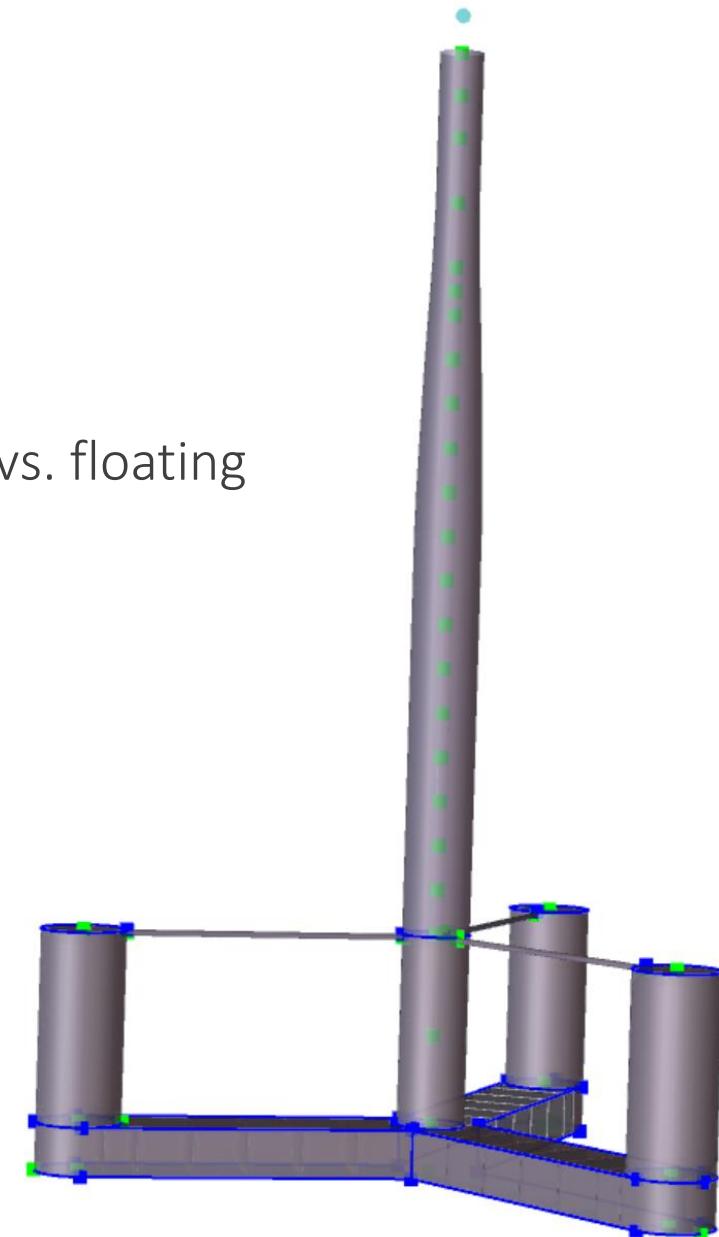
-- Fabian Anstock, EERA DeepWind 18.01.23



Quelle: WES (Wind Energy Solutions)

Agenda

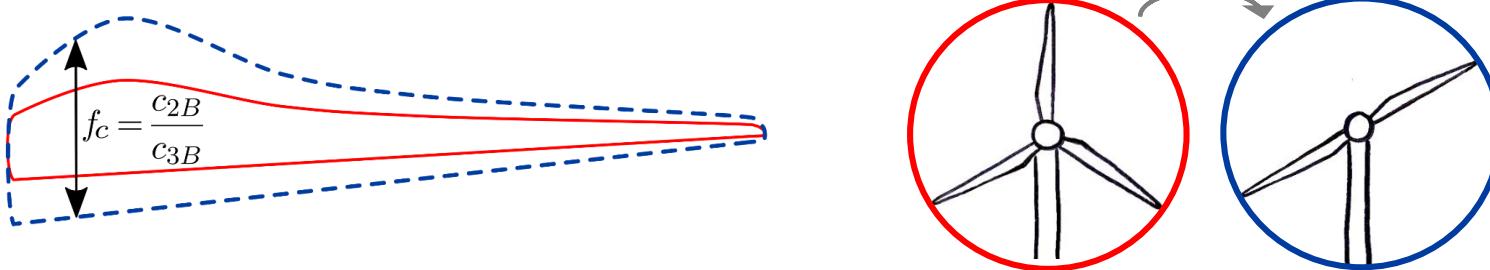
1. Two-bladed reference turbine
2. Load comparison 2B vs. 3B for bottom-fixed
3. Campbell-Diagram for tower eigenfrequency: bottom-fixed vs. floating
4. Summary



1) Two-bladed turbine from a three-bladed reference (20MW INNWIND)

Most comparable two-bladed turbine:

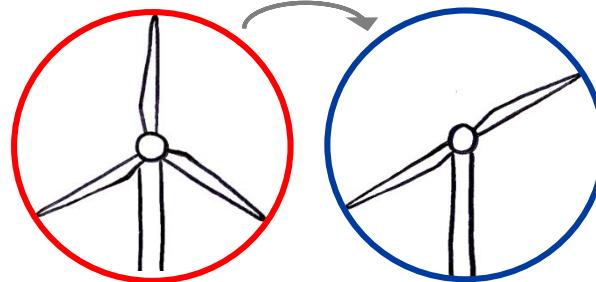
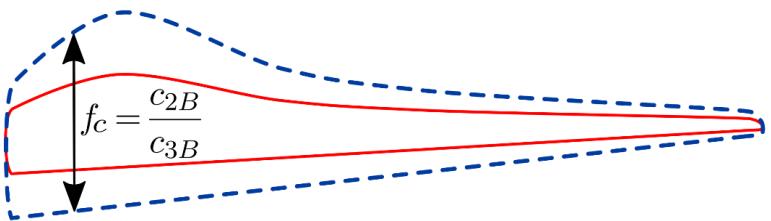
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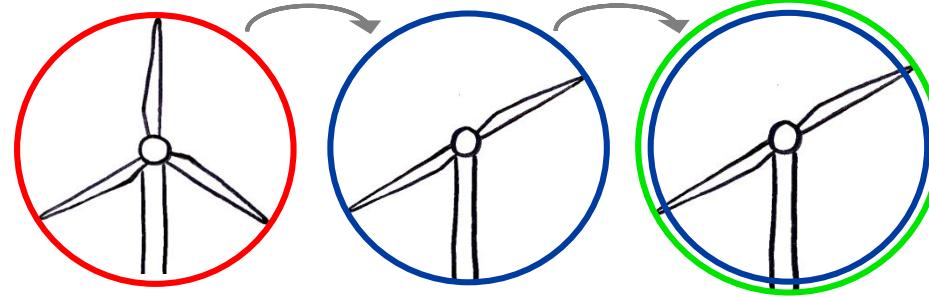
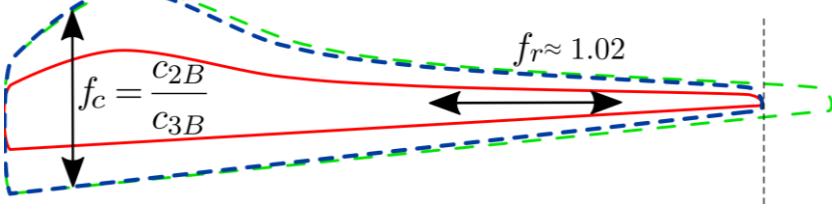
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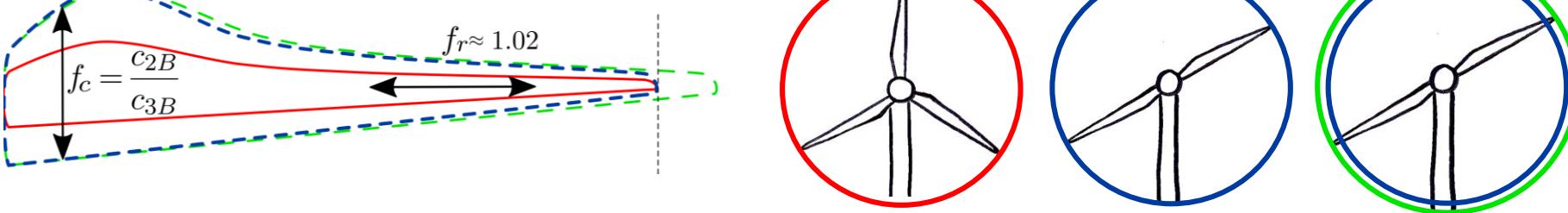
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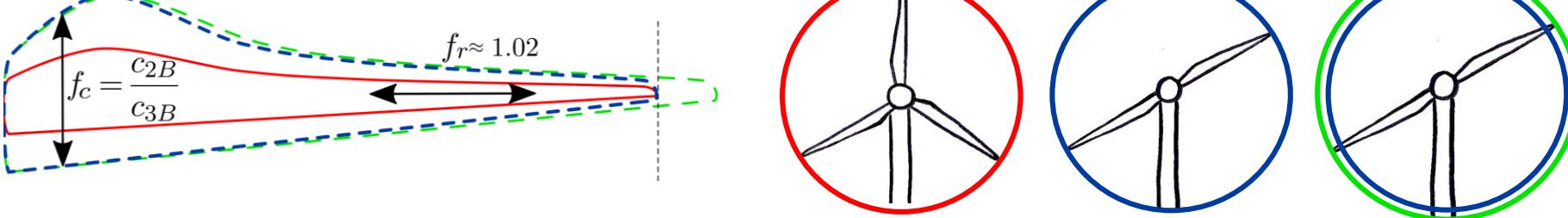
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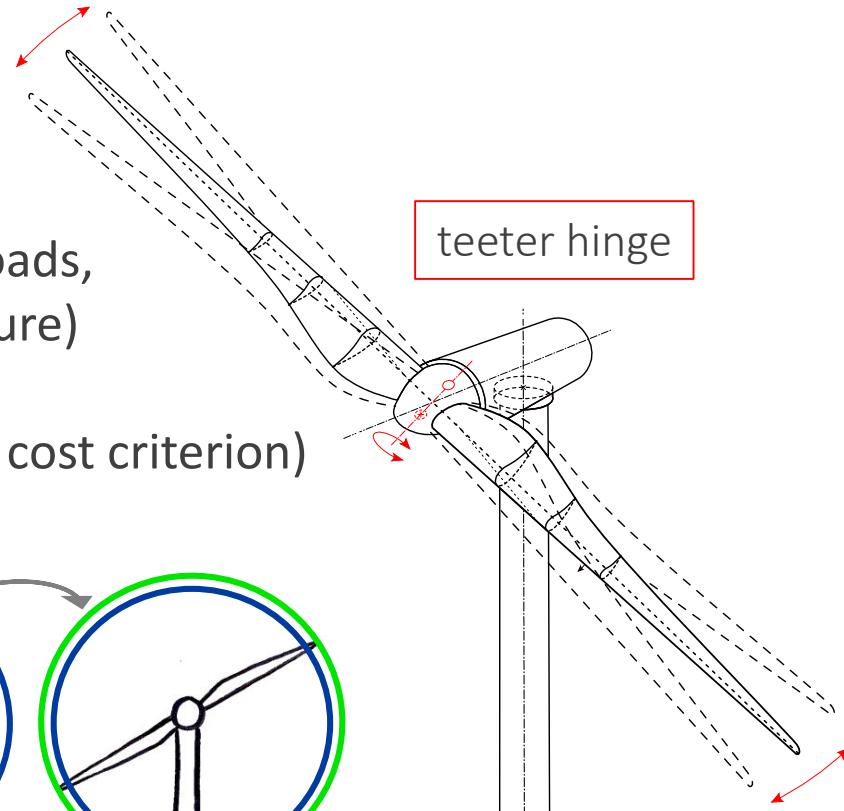
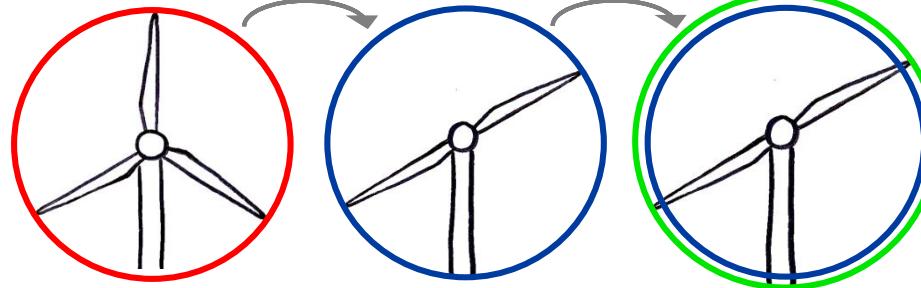
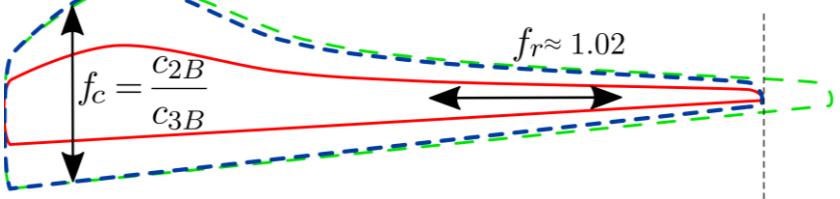
Variants:

2B twin: (two-bladed, fixed-hub, upwind, *controller with speed exclusion zone and active tower dampers*)

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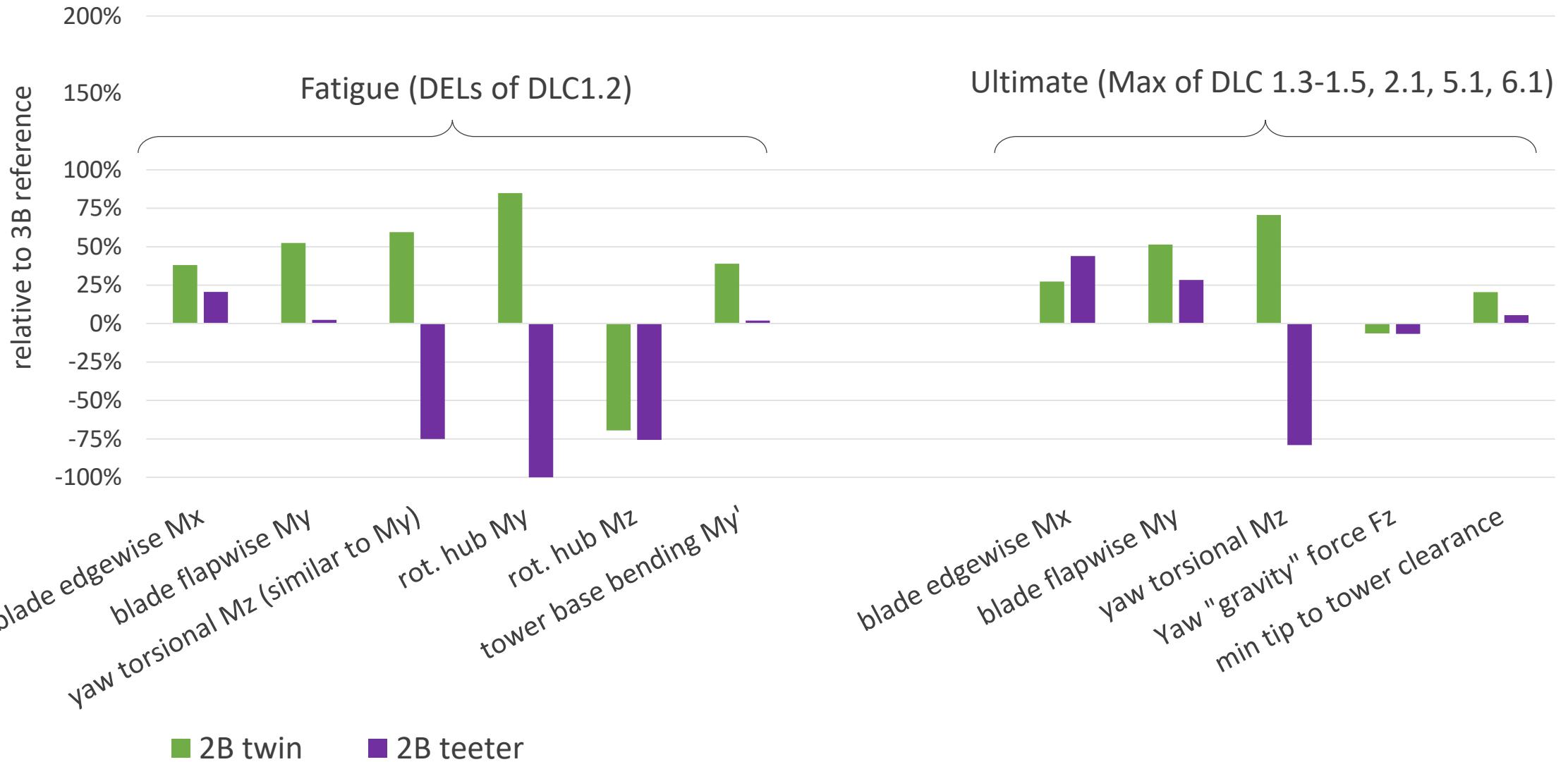


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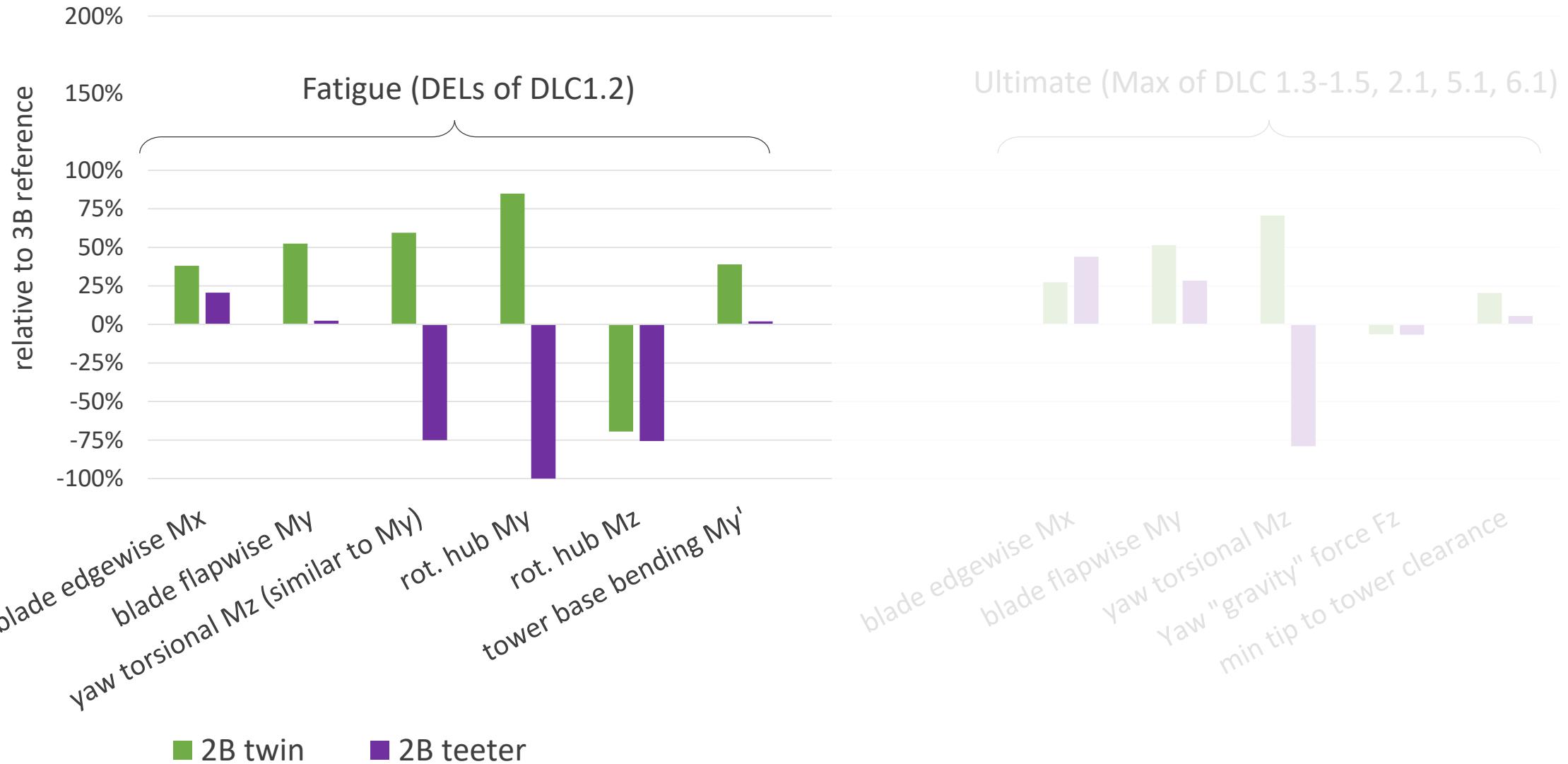
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2B teeter (like 2B twin but with a teeter hinge)

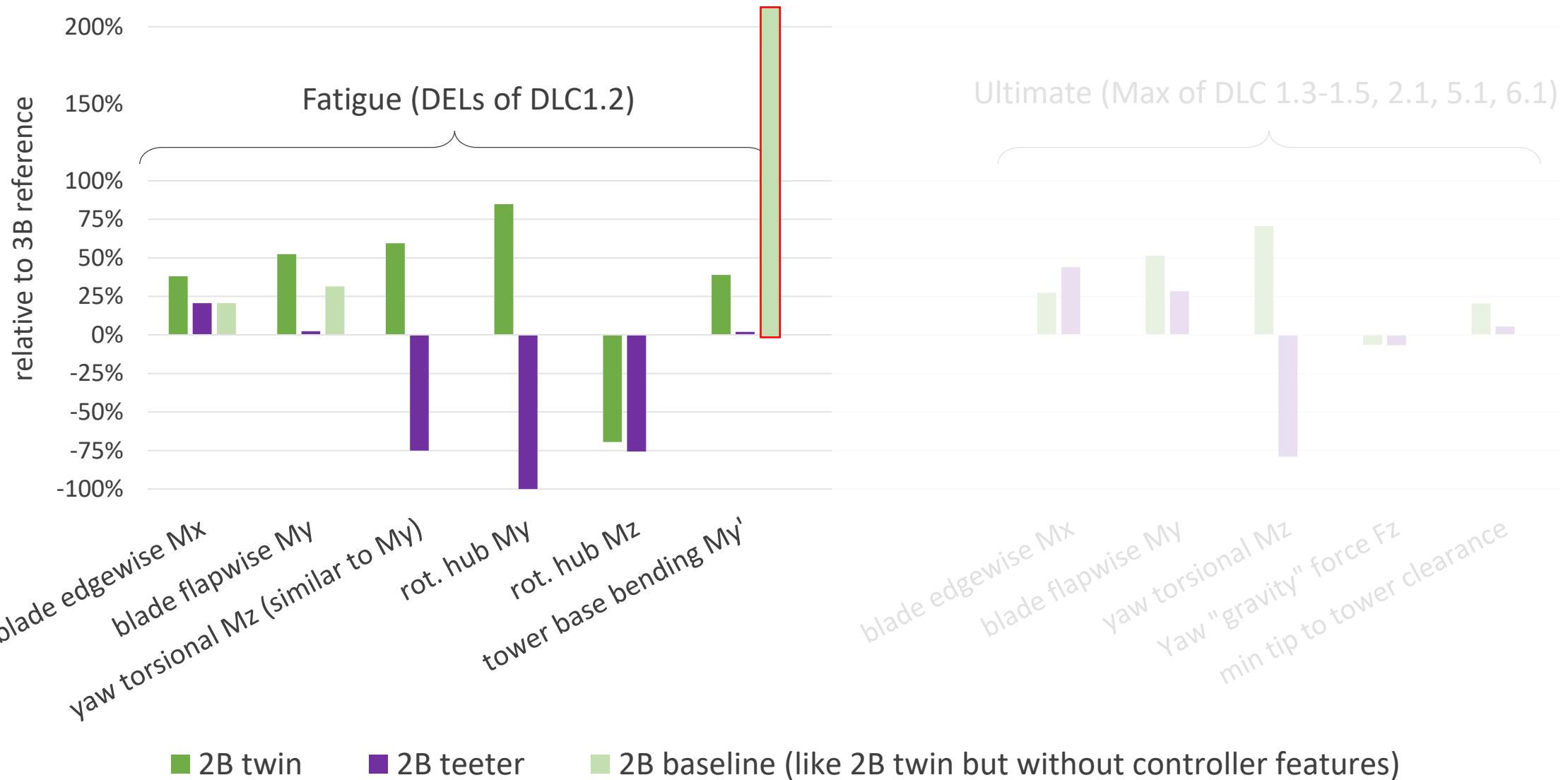
1) Load difference compared to the three-bladed reference (3B ref)



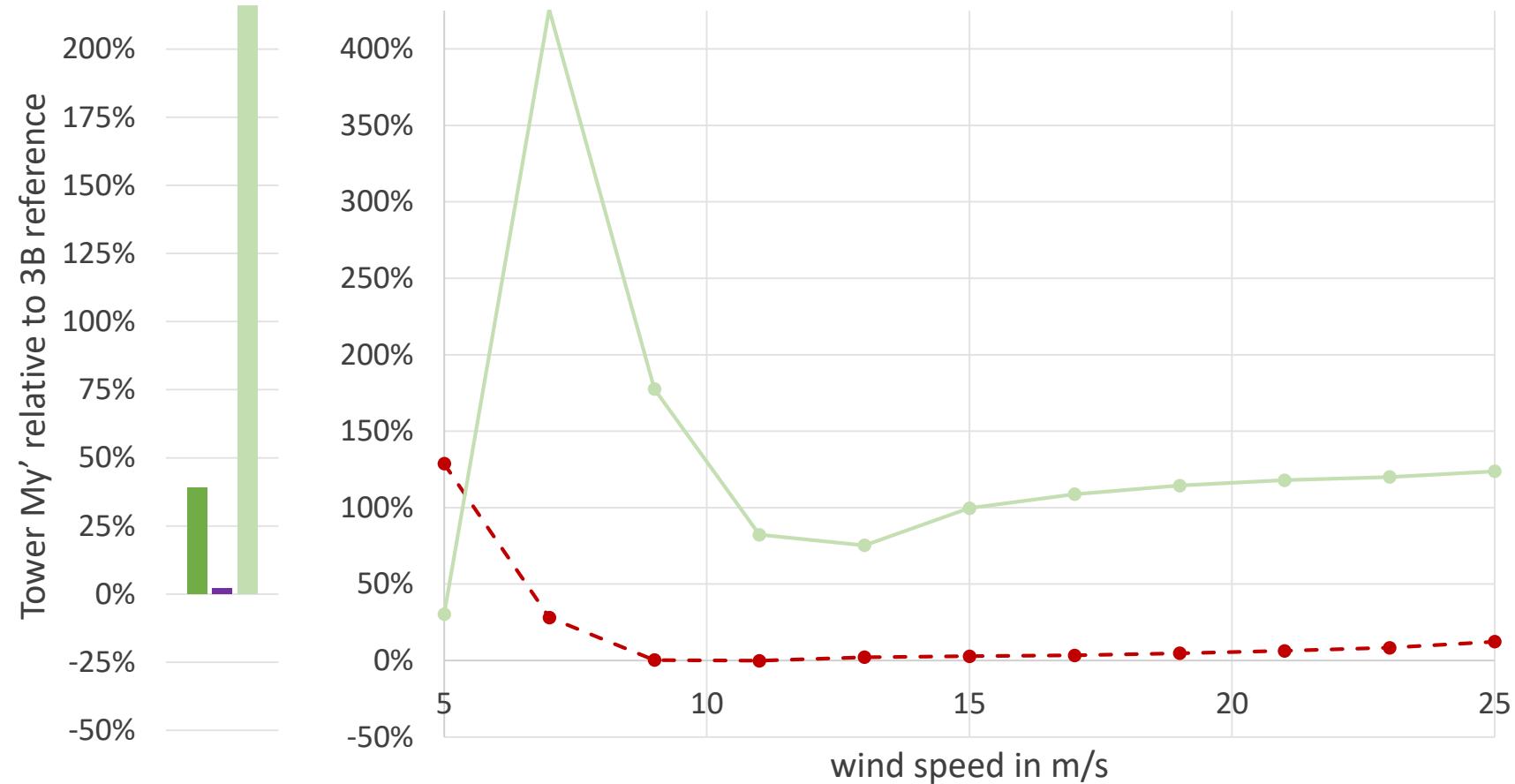
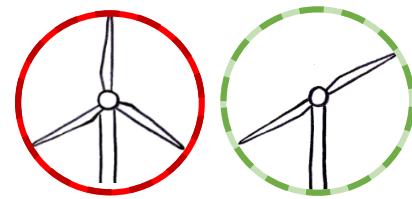
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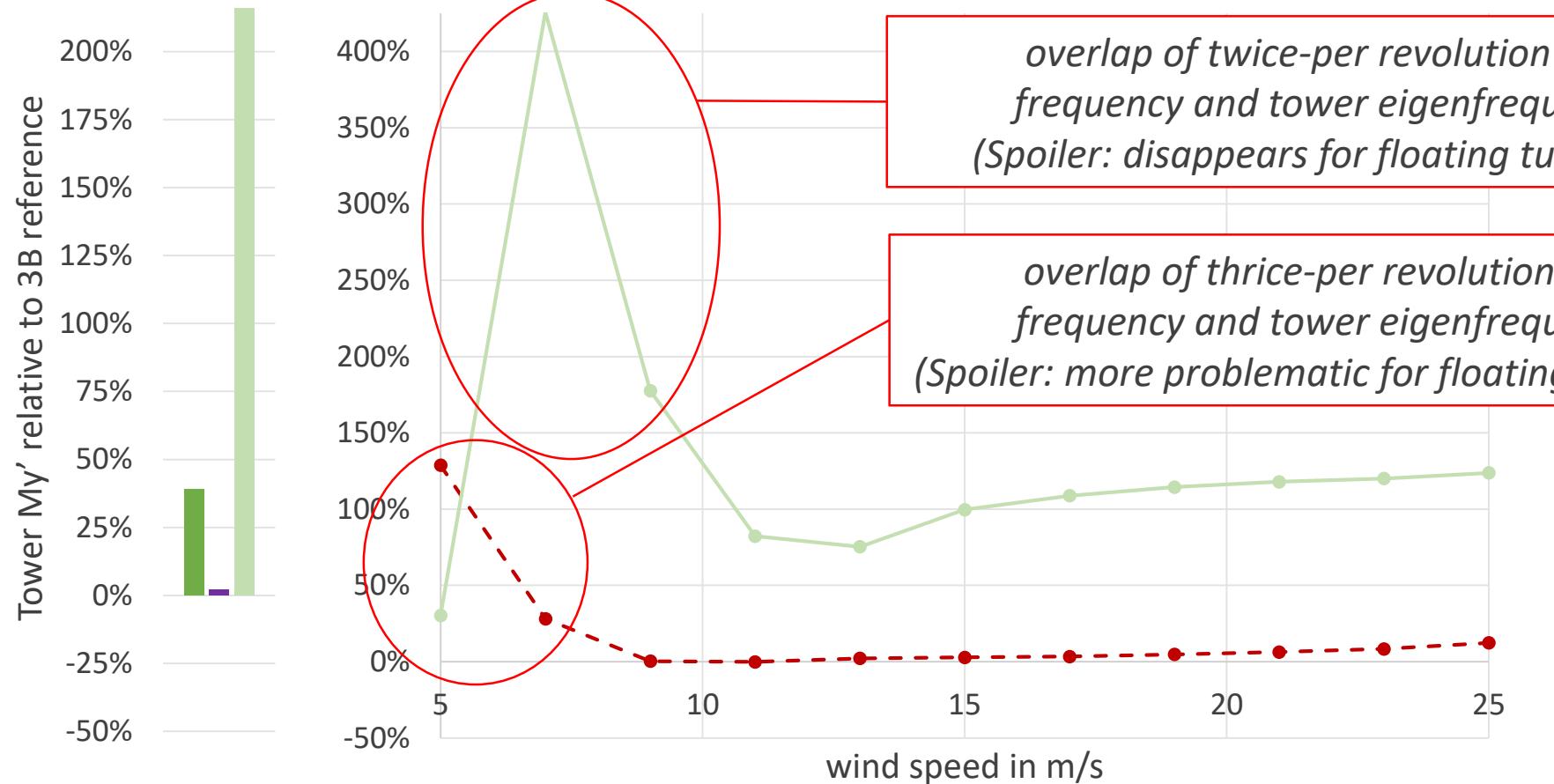
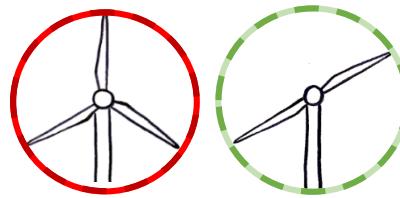


1) Load difference: Tower fatigue 2B vs. 3B in DLC 1.2



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■ 2B101 Twin ■ 2B101 Teeter ■ 2B101 baseline (like 2B twin but without controller features)

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*overlap of twice-per revolution (2P)
frequency and tower eigenfrequency
(Spoiler: disappears for floating turbines.)*

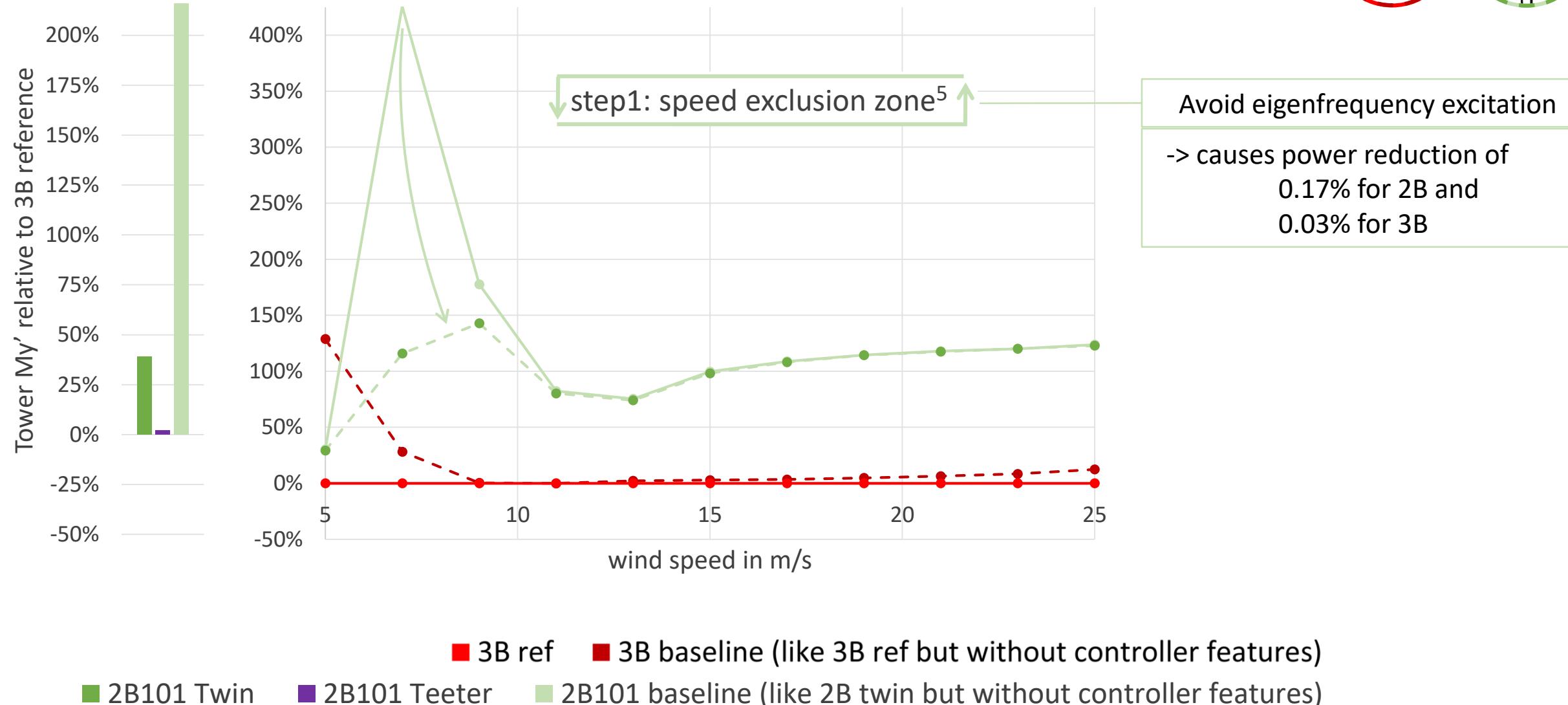
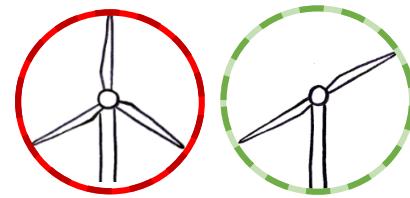
*overlap of thrice-per revolution (3P)
frequency and tower eigenfrequency
(Spoiler: more problematic for floating turbines.)*



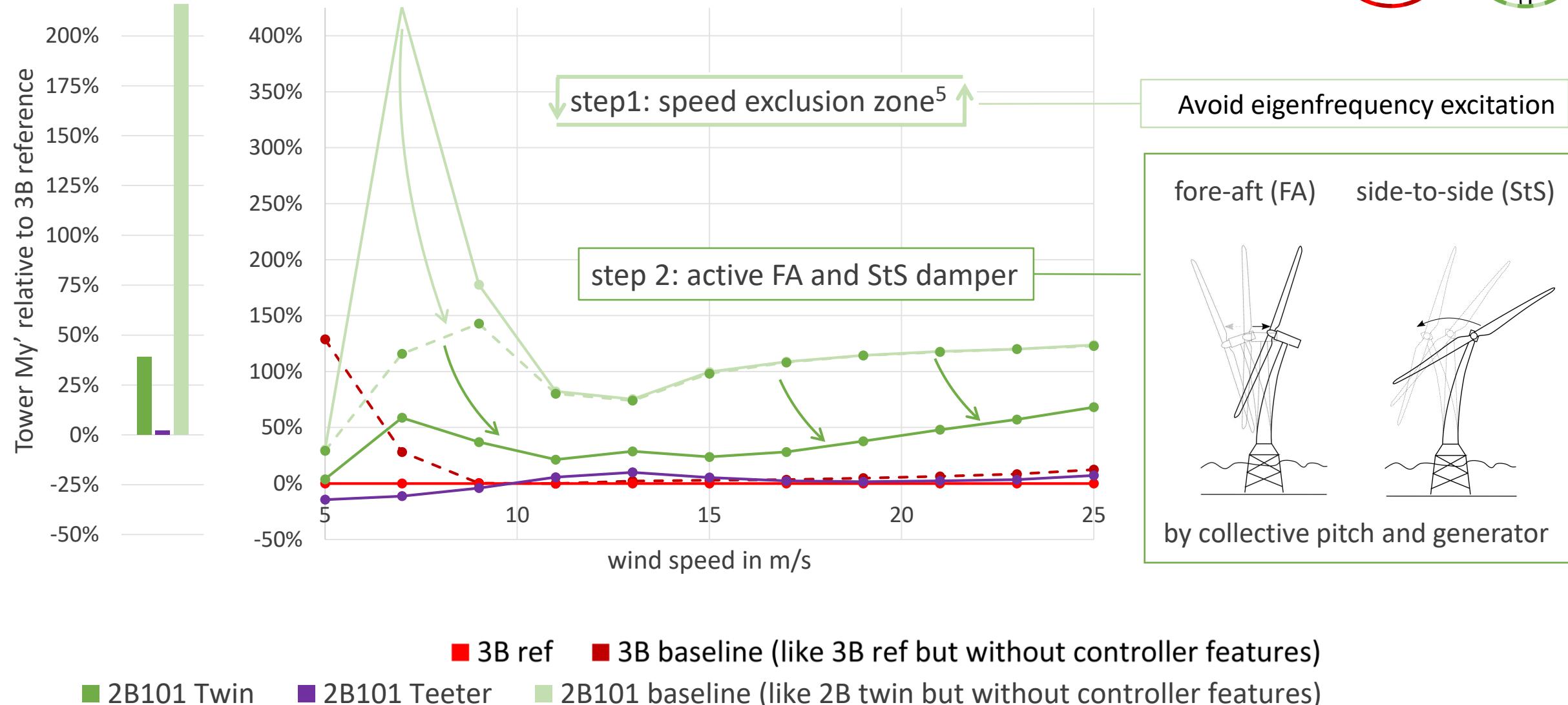
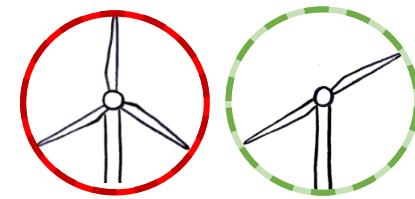
source: www.hausjournal.net

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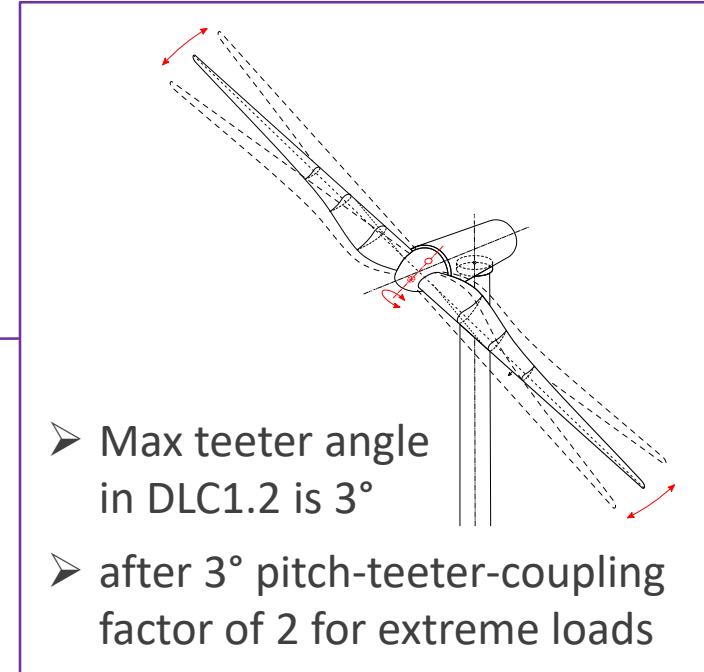
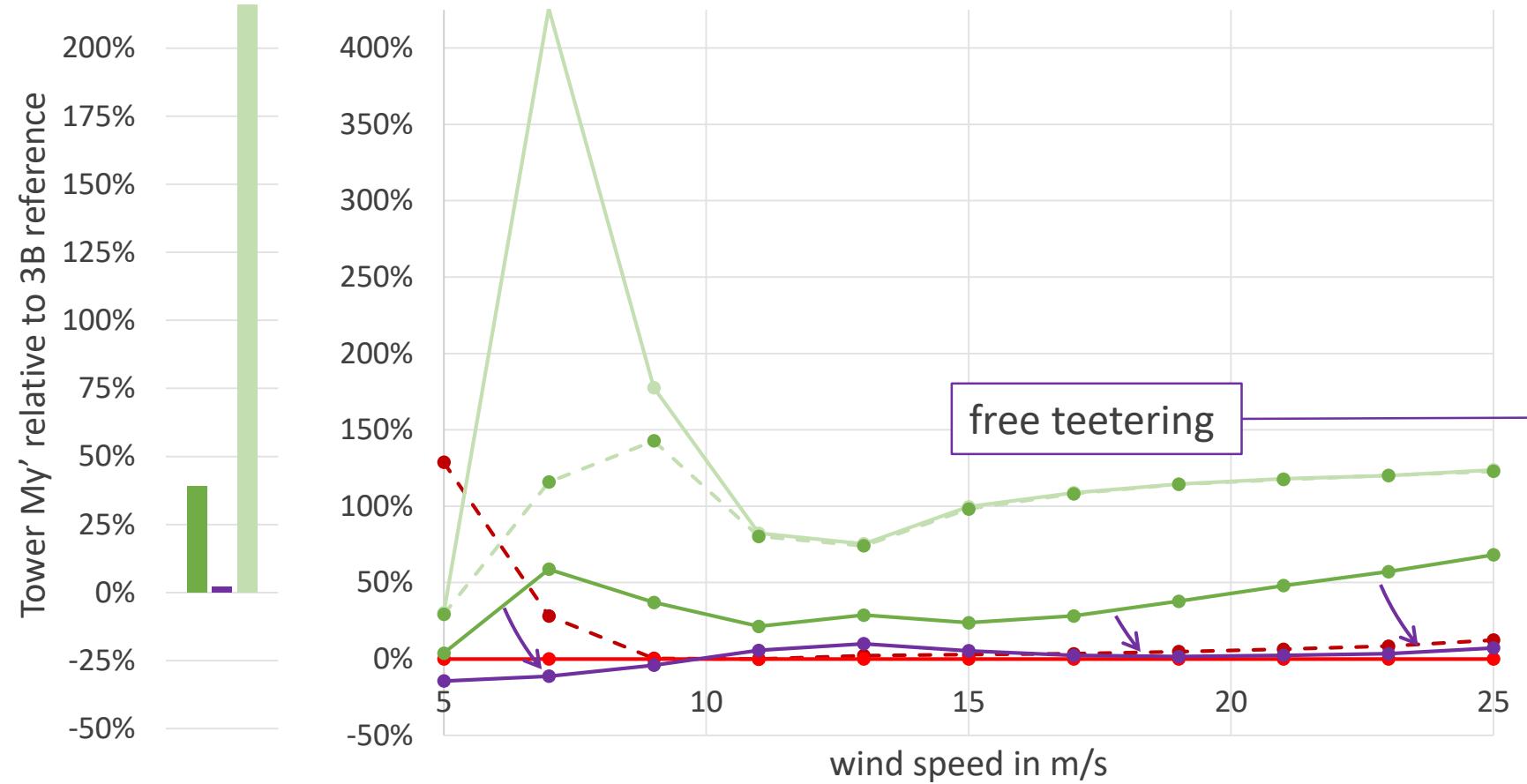
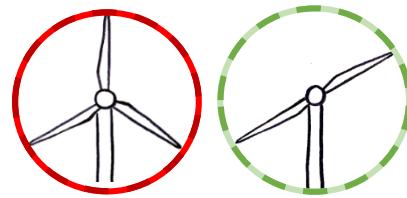
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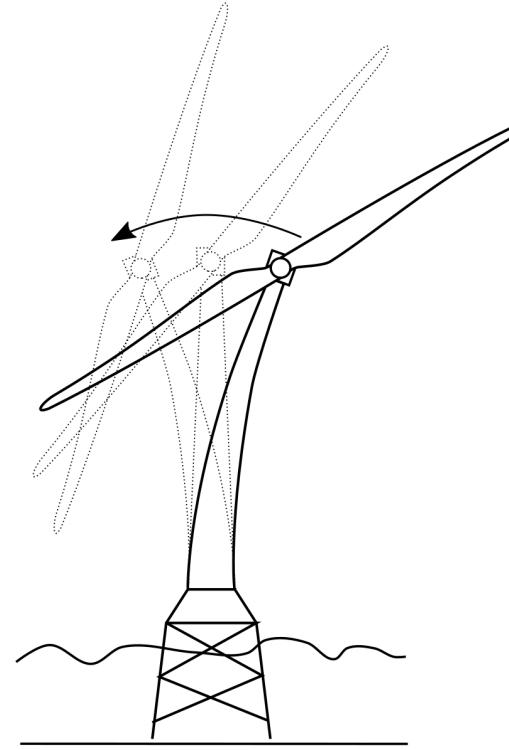
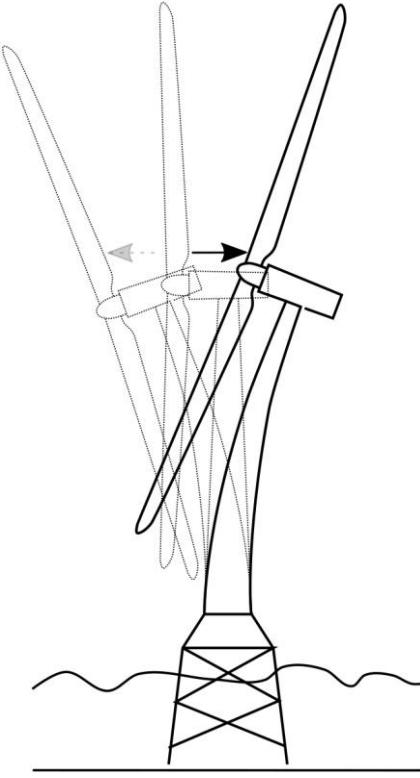
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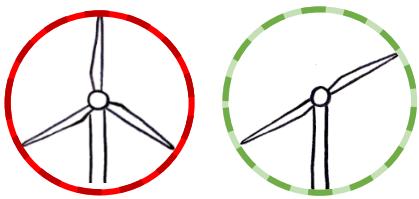


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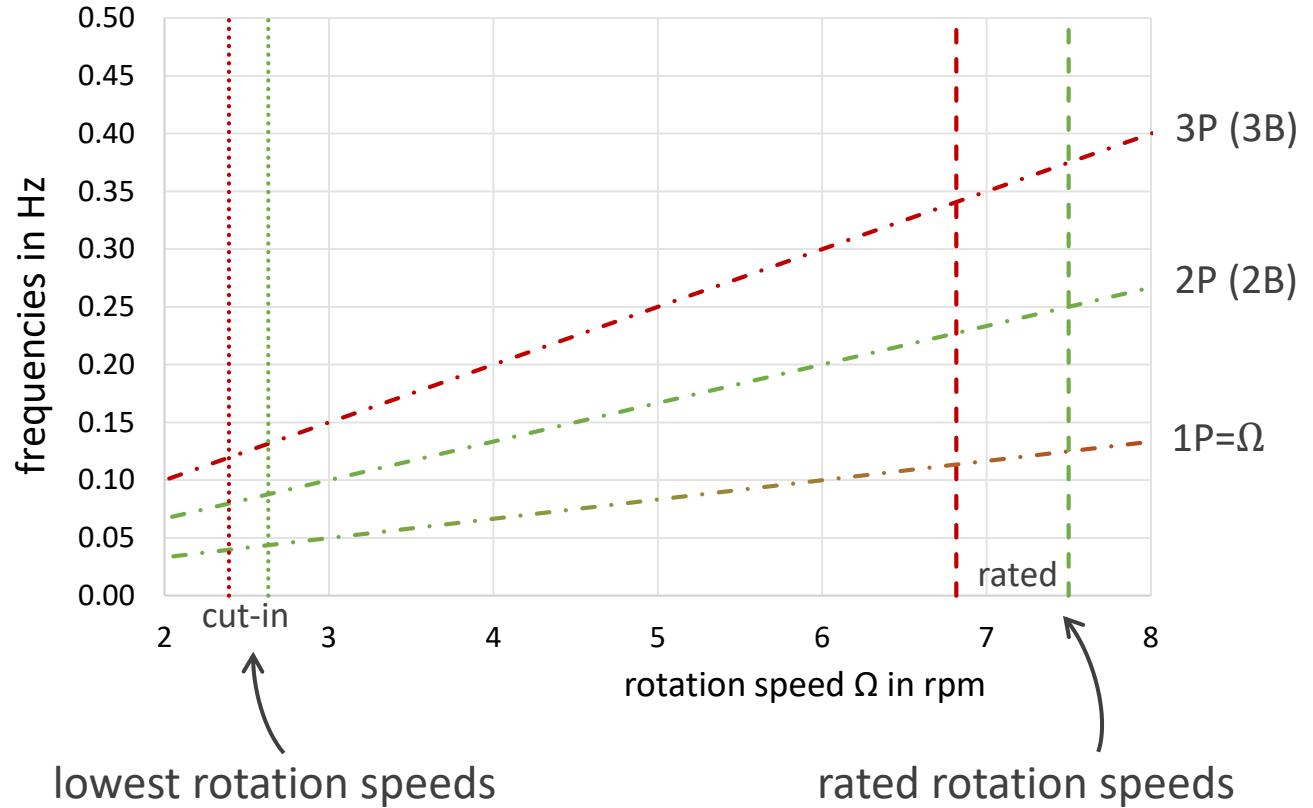


Campbell diagram

2) Campbell diagram – focus on first tower eigenfrequencies

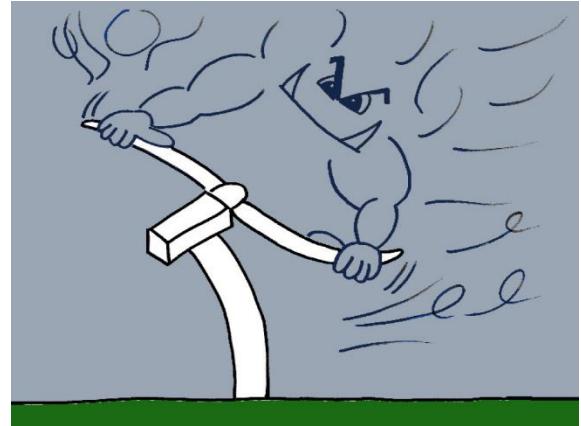
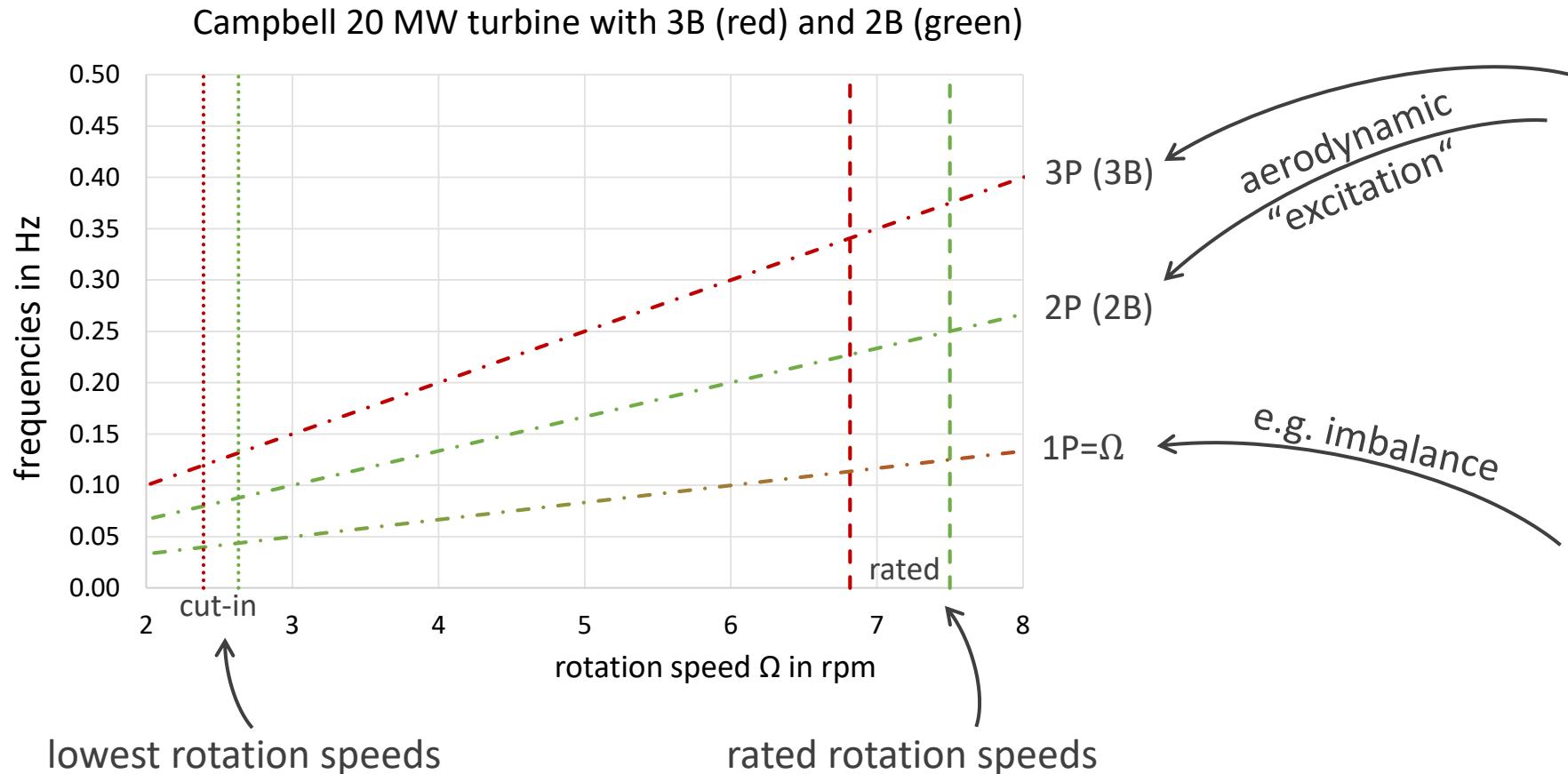
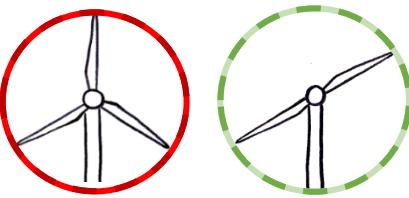


Campbell 20 MW turbine with 3B (red) and 2B (green)



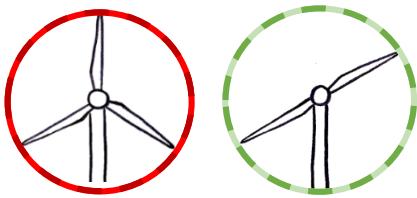
1P, 2P and 3P => once, twice or thrice per revolution

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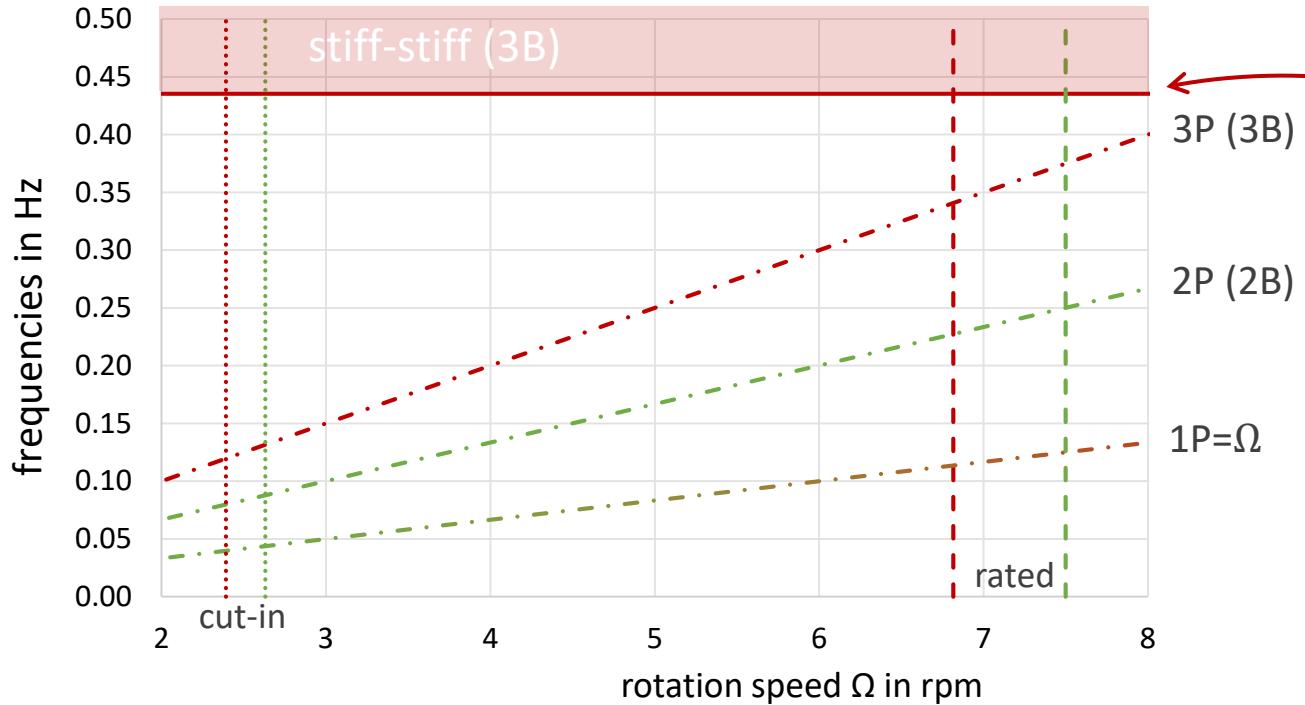


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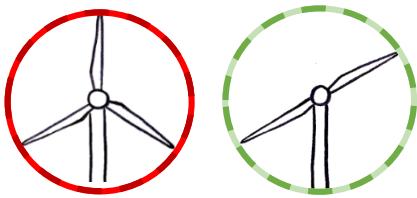


— 3B stiff-stiff tower

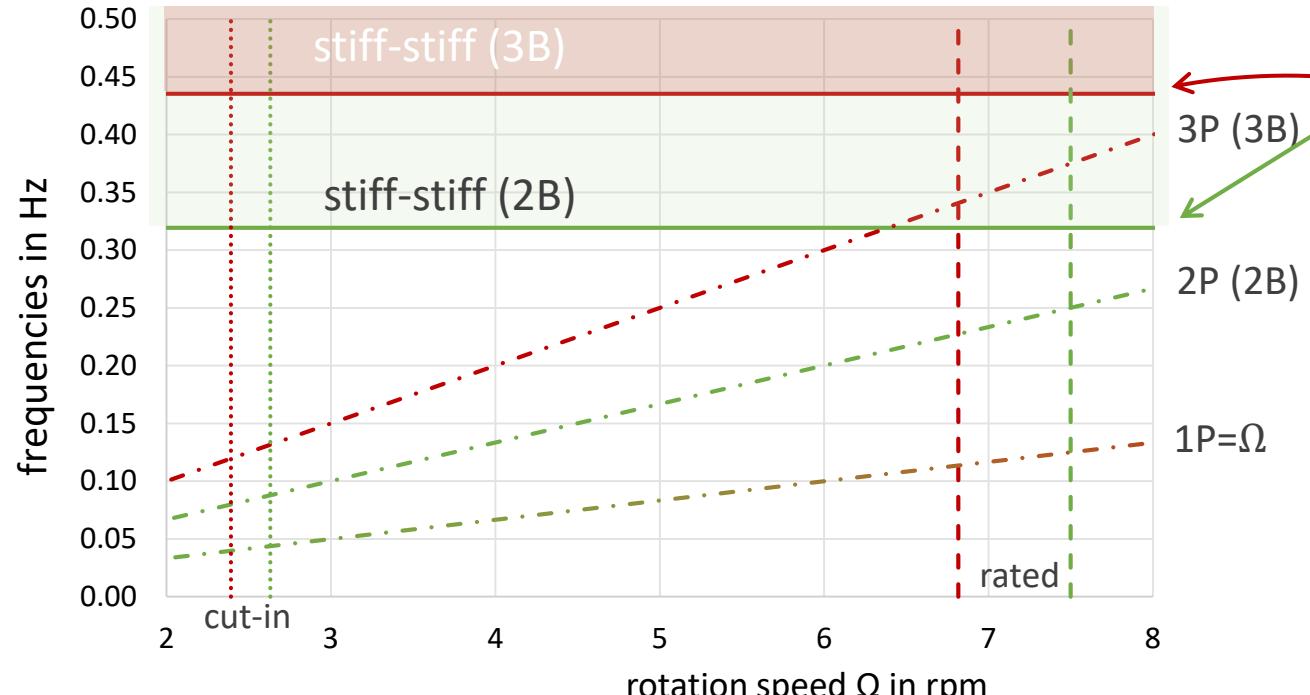
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• tower frequency **above** 2P or 3P

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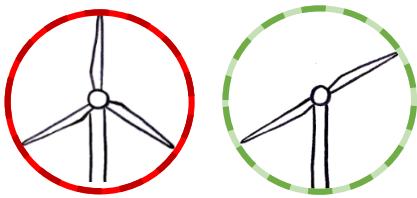


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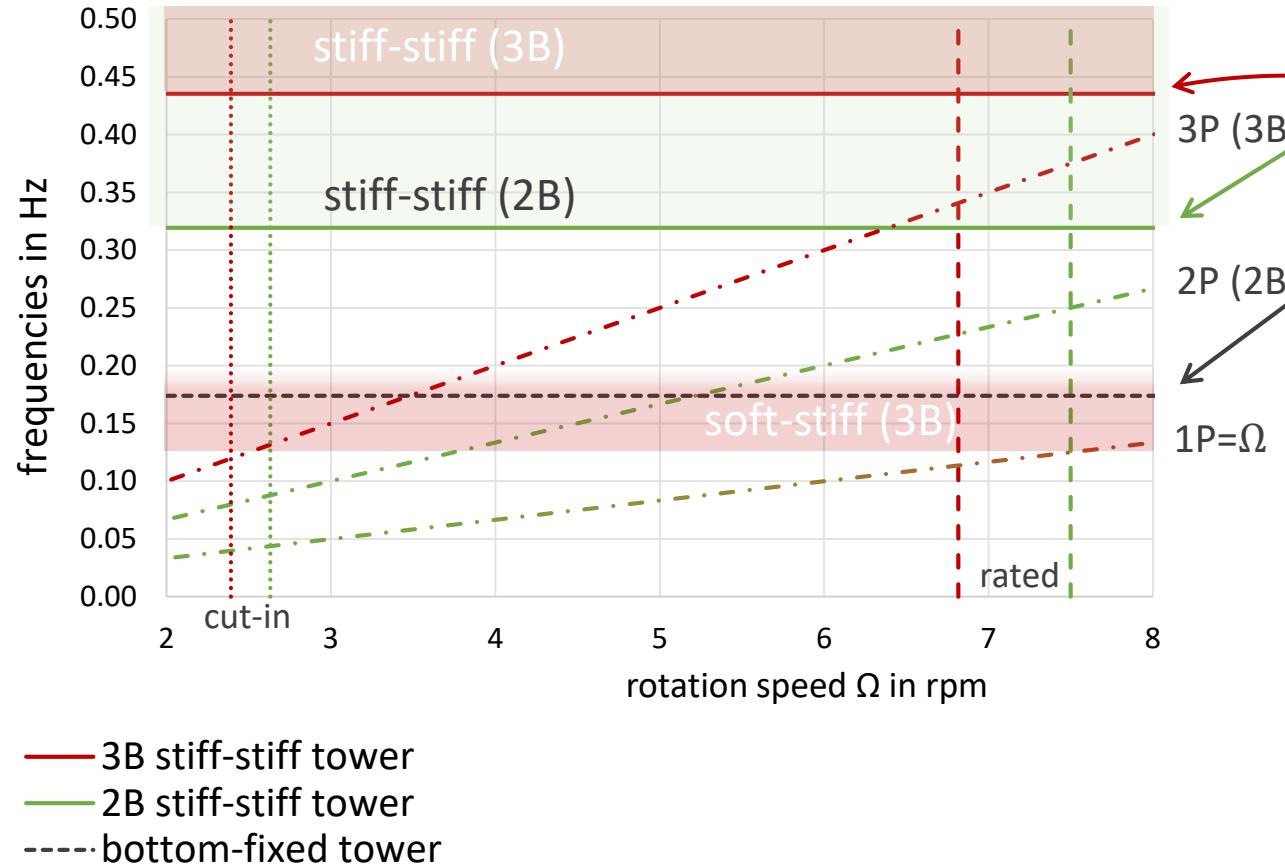
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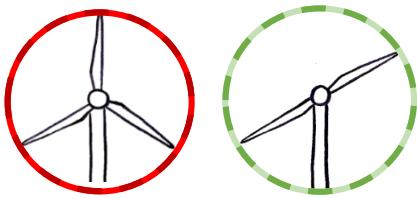
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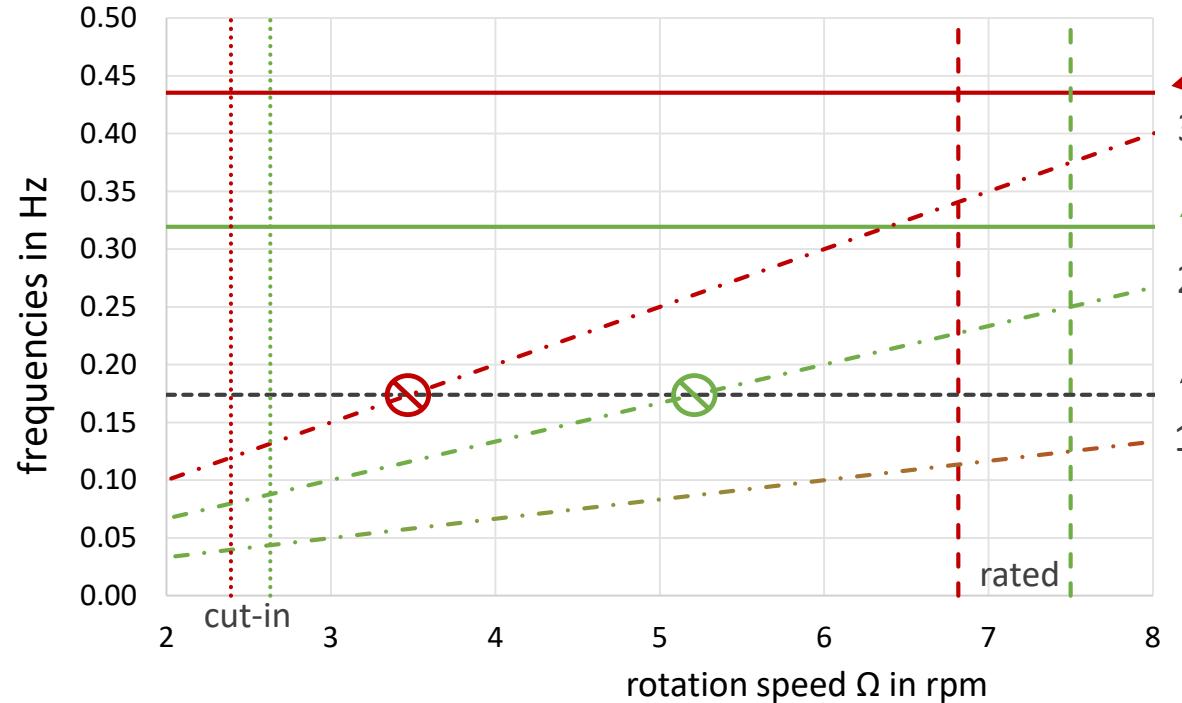
- tower frequency **between** 1P and 2P or 3P

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Campbell 20 MW turbine with 3B (red) and 2B (green)



- 3B stiff-stiff tower
- 2B stiff-stiff tower
- bottom-fixed tower

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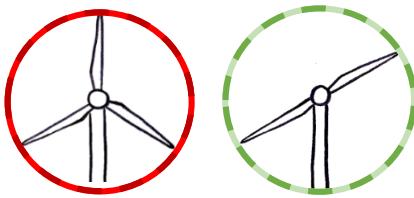
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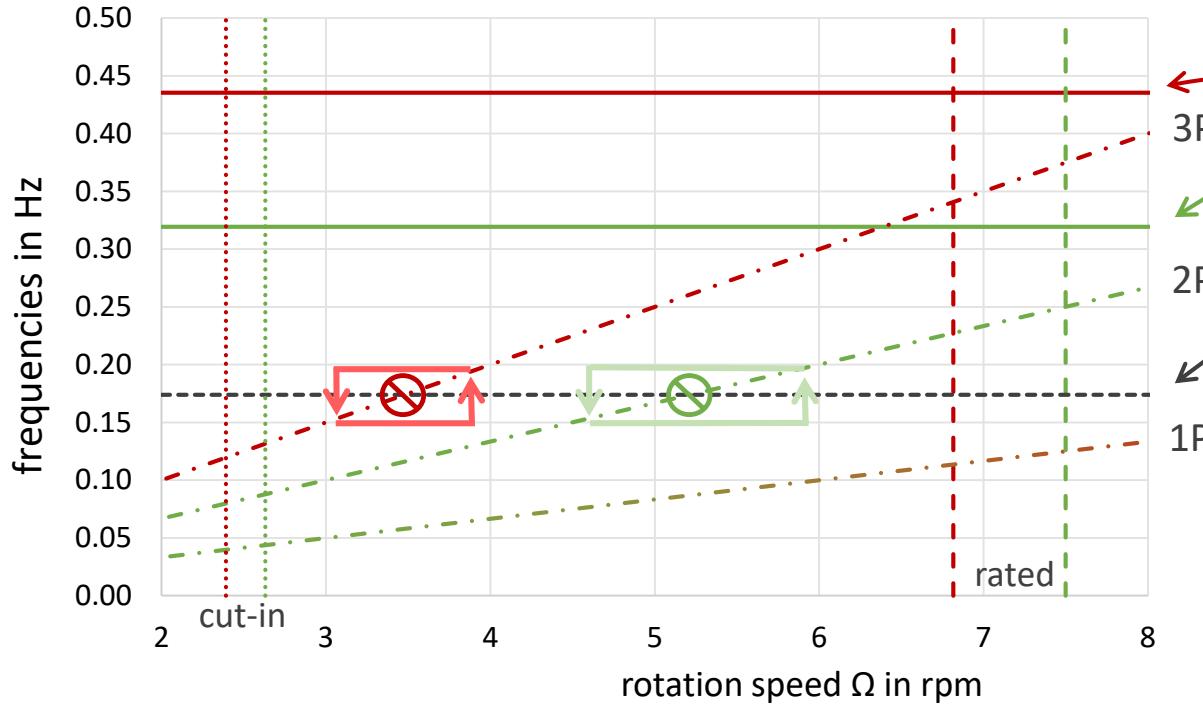
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1P = Ω

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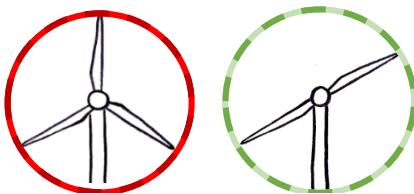
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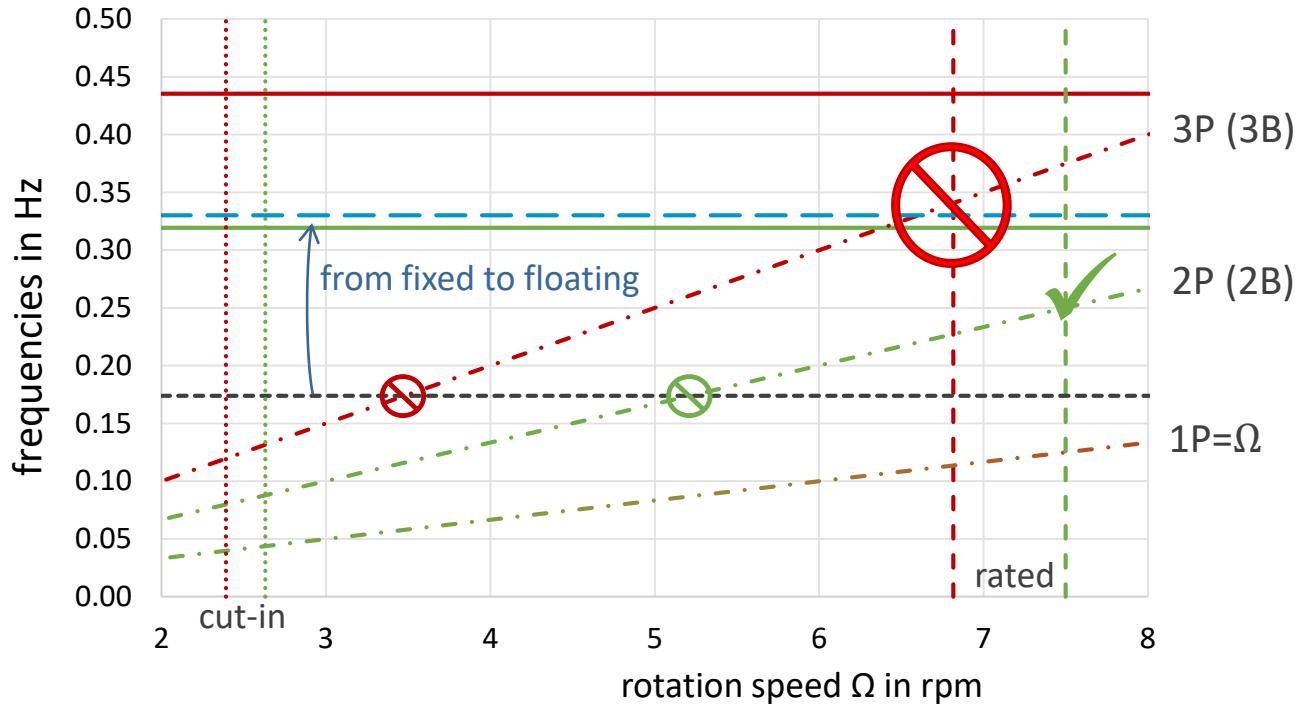
➤ *Very little space between 1P and 2P thus high tower fatigue of two-bladed turbine (with classical bottom-fixed tower).*

➤ „adjust“ tower eigenfrequency is extremely expensive. (speed exclusion zone is cheaper.)

2) Tower eigenfrequency – big two-bladed turbines issue or advantage?

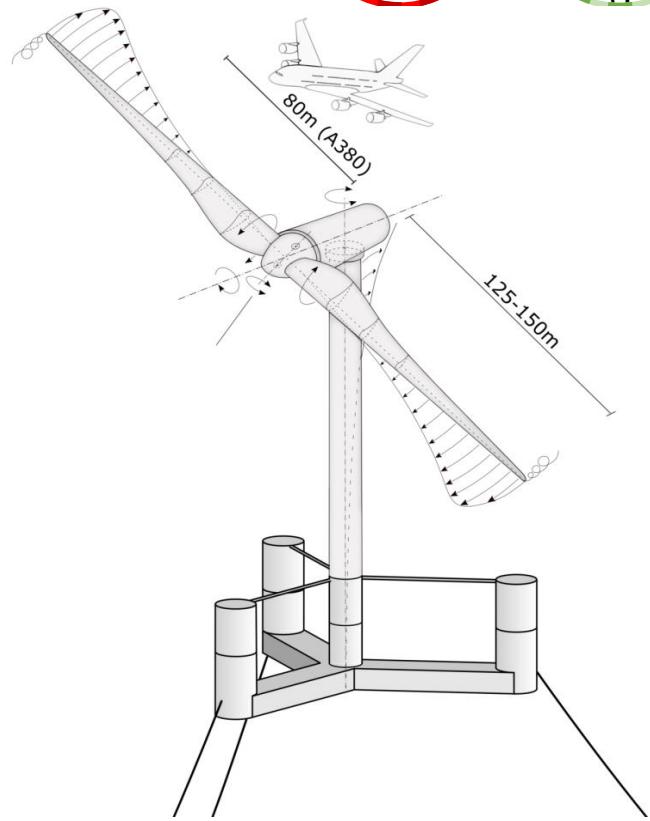


Campbell 20 MW VolturnUS with 3B (red) and 2B (green)



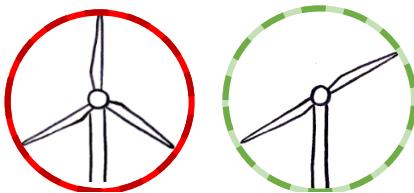
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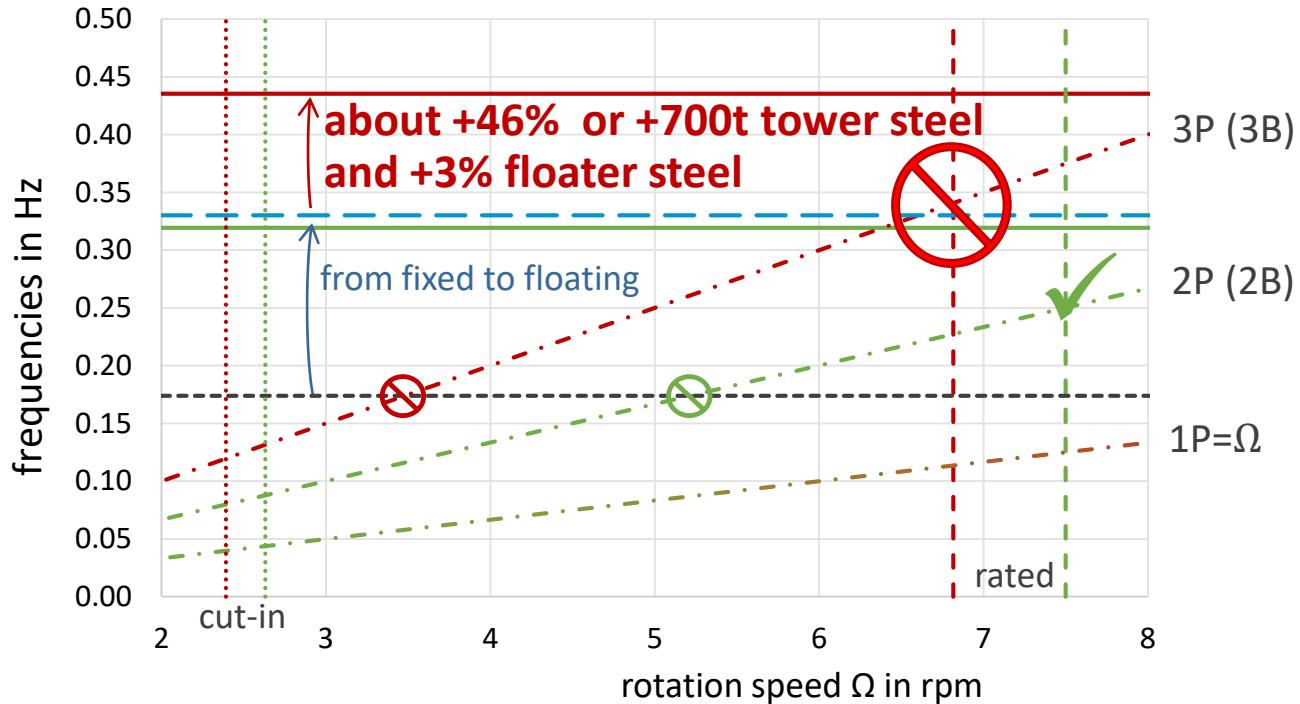


- tower eigenfrequency increases vastly for floating turbines
- problematic for three-bladed turbines due to high 3P-frequency

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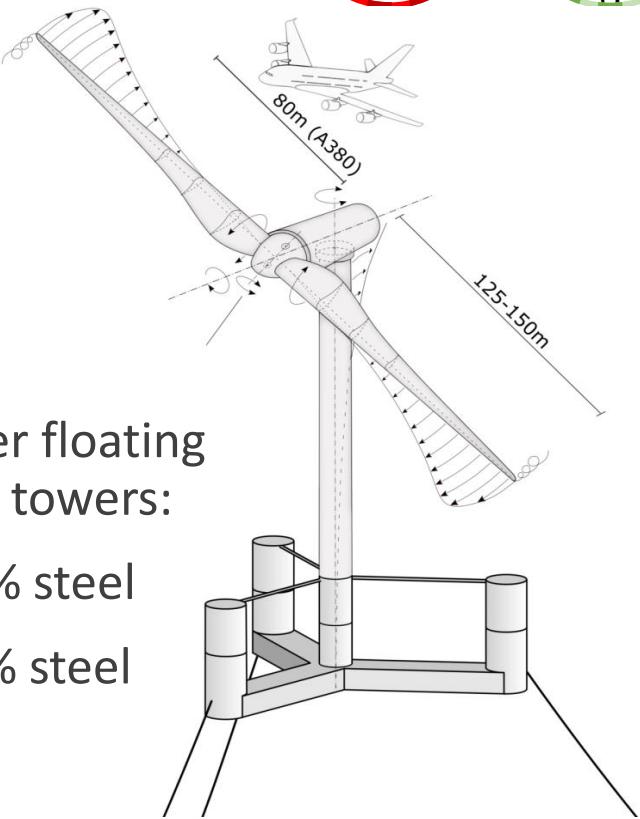


- 3B stiff-stiff tower
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Compared to other floating turbine's stiff-stiff towers:

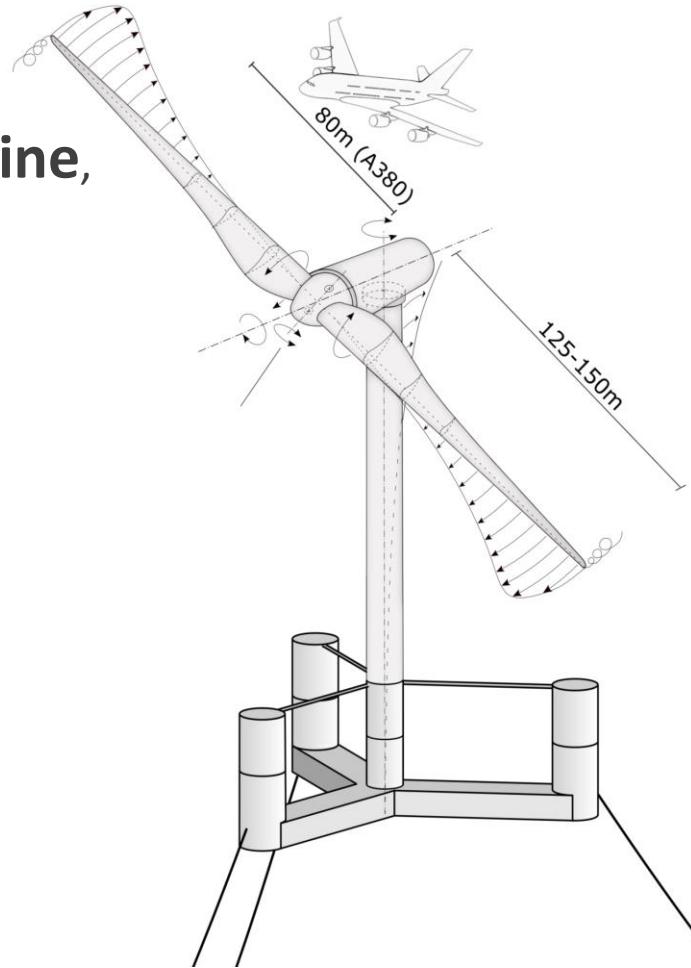
- NREL with +55% steel
- EDF with +72% steel



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4) Summary

- **Bottom-fixed: Tower fatigue issues for a two-bladed 20MW turbine,**
due to naturally poor soft-stiff tower design options.
- **Floating: Tower fatigue issues for a three-bladed 20MW turbine.**
Cost and load advantages for two-bladed towers.
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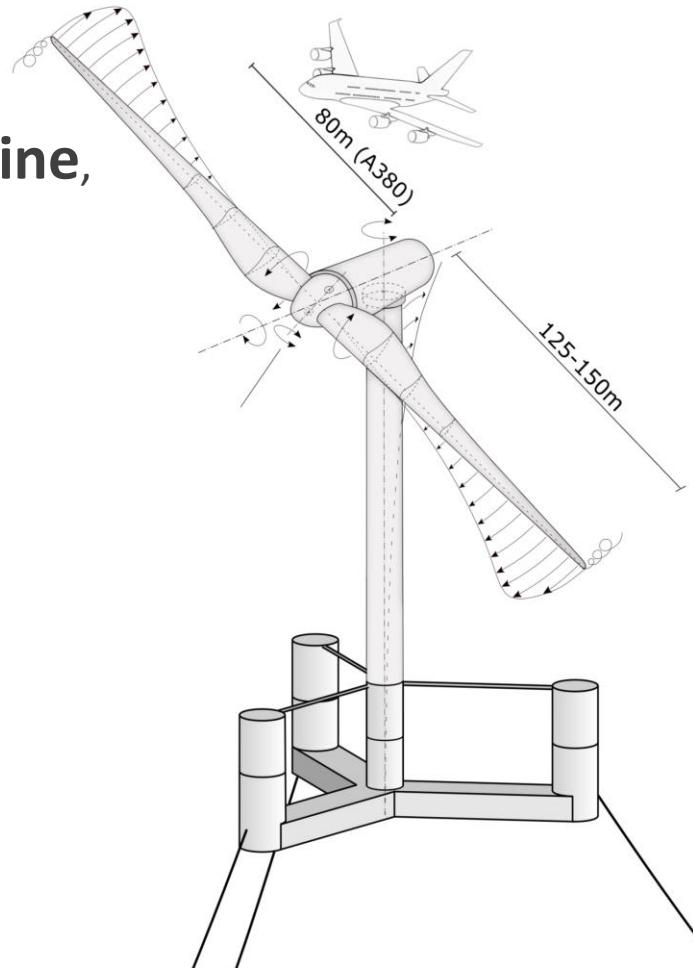


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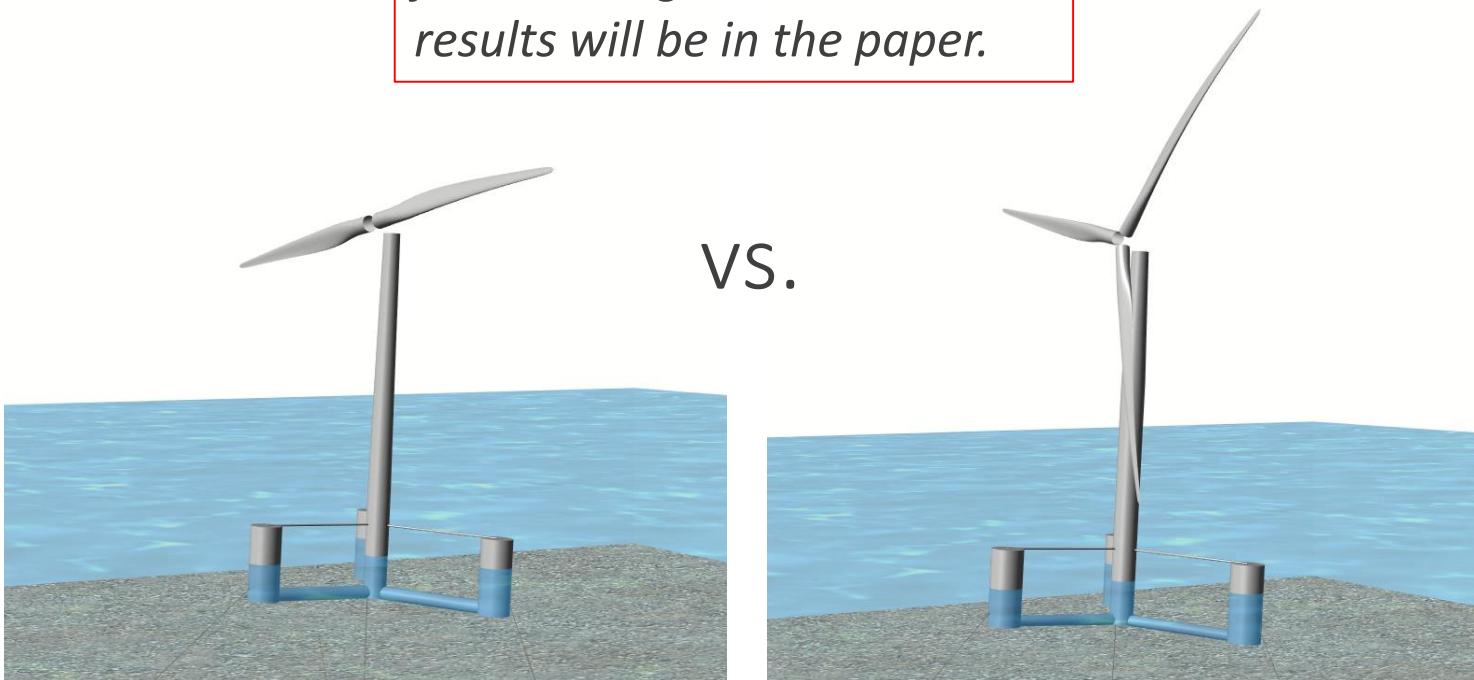
Further two-bladed advantages:

- **Teetering motion (and teeter-issues) decrease with turbine sizes**
- **Two-bladed turbine's blades are easier to upscale**
(larger tip tower clearance and higher strength to weight ratio)
- **Less mass in the rotor and generator thus less rotor-nacelle inertia**
- **Further cost reductions in the whole life-cycle**
(one blade, bearing and actuator less to manufacture, transport,
erect, maintain, decommission + lighter tower and floater)



Thank you for your time

More details on an objective floater design and simulation results will be in the paper.



Many thanks to the funding of



Federal Ministry
for Economic Affairs
and Energy

and

— EnBW

Fabian Anstock, M.Sc.
Research Associate

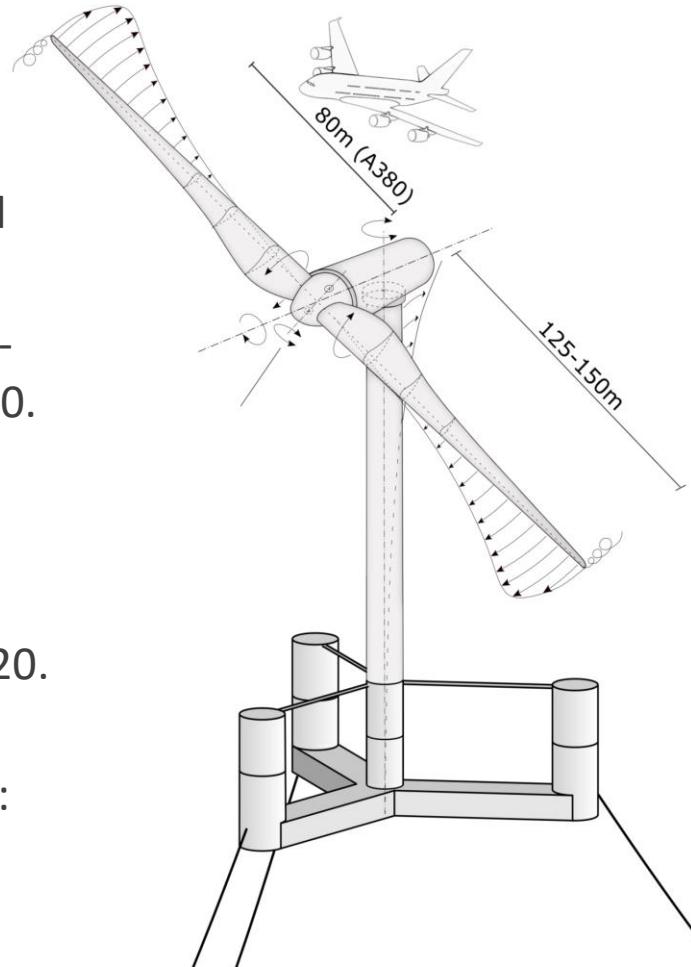
Project: X-Rotor – two-bladed wind turbines

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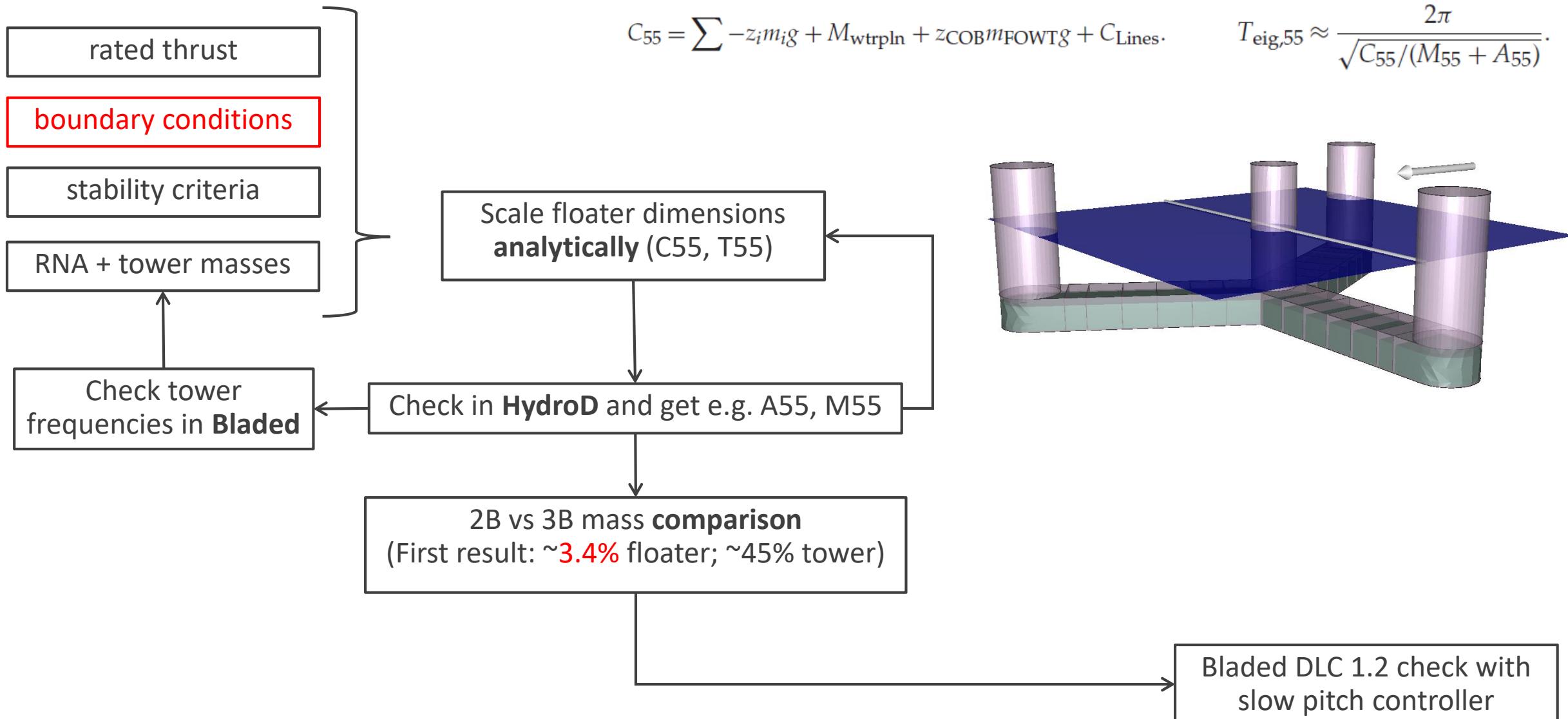
HAMBURG UNIVERSITY OF APPLIED SCIENCES
Competence Center for Renewable Energy
and Energy Efficiency
Berliner Tor 21 / 20099 Hamburg

Own references

- [1] Anstock F., Schütt M., and Schorbach V. A new approach for comparability of two- and three-bladed 20 MW offshore wind turbines. JoP, 2019.
- [2] Schütt M., Anstock F., and Schorbach V. Progressive structural scaling of a 20 MW two-bladed offshore wind turbine rotor blade examined by finite element analyses. JoP, 2020.
- [3] Schütt M., Anstock F., and Schorbach V. A procedure to redesign a comparable blade structure of a two-bladed turbine based on a three-bladed reference. JoP, 2021.
- [4] Anstock F. and Schorbach V., A control cost criterion for controller tuning of two- and three-bladed 20MW offshore wind turbines, Journal of Physics: Conference Series, 2020.
- [5] Anstock F. and Schorbach V. The effect of a speed exclusion zone and active tower dampers on an upwind fixed-hub two-bladed 20 MW wind turbine. Journal of Physics: Conference Series, 2021.



OBJECTIVE (pre-)design steps for a two- and three-bladed floater

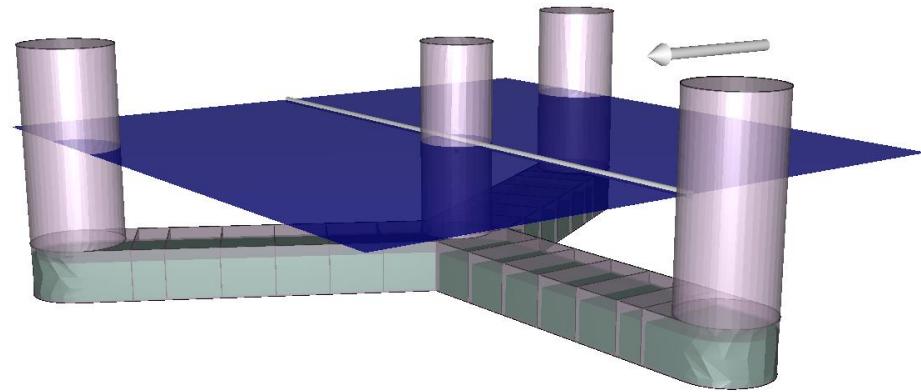


Necessary boundary conditions

- Freeboard: fixed at 15m ✓
- Center column: fixed at 11m (=20MW tower bottom) ✓
- Outer column Diameter: Linear scaled. ✓
- Pontoon width: Linear scaled. ✓
- Pontoon height: Scaled by complete water ballast filling ✓
- 5° platform heeling angle at mean rated rotor thrust ✓
- Hub height constant → $2B = 30\text{m}$ ground clearance, $3B = 33\text{m}$ ✓ **(for dynamic comparison)**
- Heeling eigenfrequency $\geq 25\text{s}$ ✓
- Floater drag: Left at 20m ✓
- Steel wall thickness: Linear scaled ✓

Unsure:

- Mooring line diameter: Scaled proportional to floater mass or thrust? **-> to DLC 6.1 max load and uplift**
- Iron ore concrete thickness: **Check if needed**; otherwise linear scaled
- Max. heeling angle? -> advised to be around 15° at ESS ✓



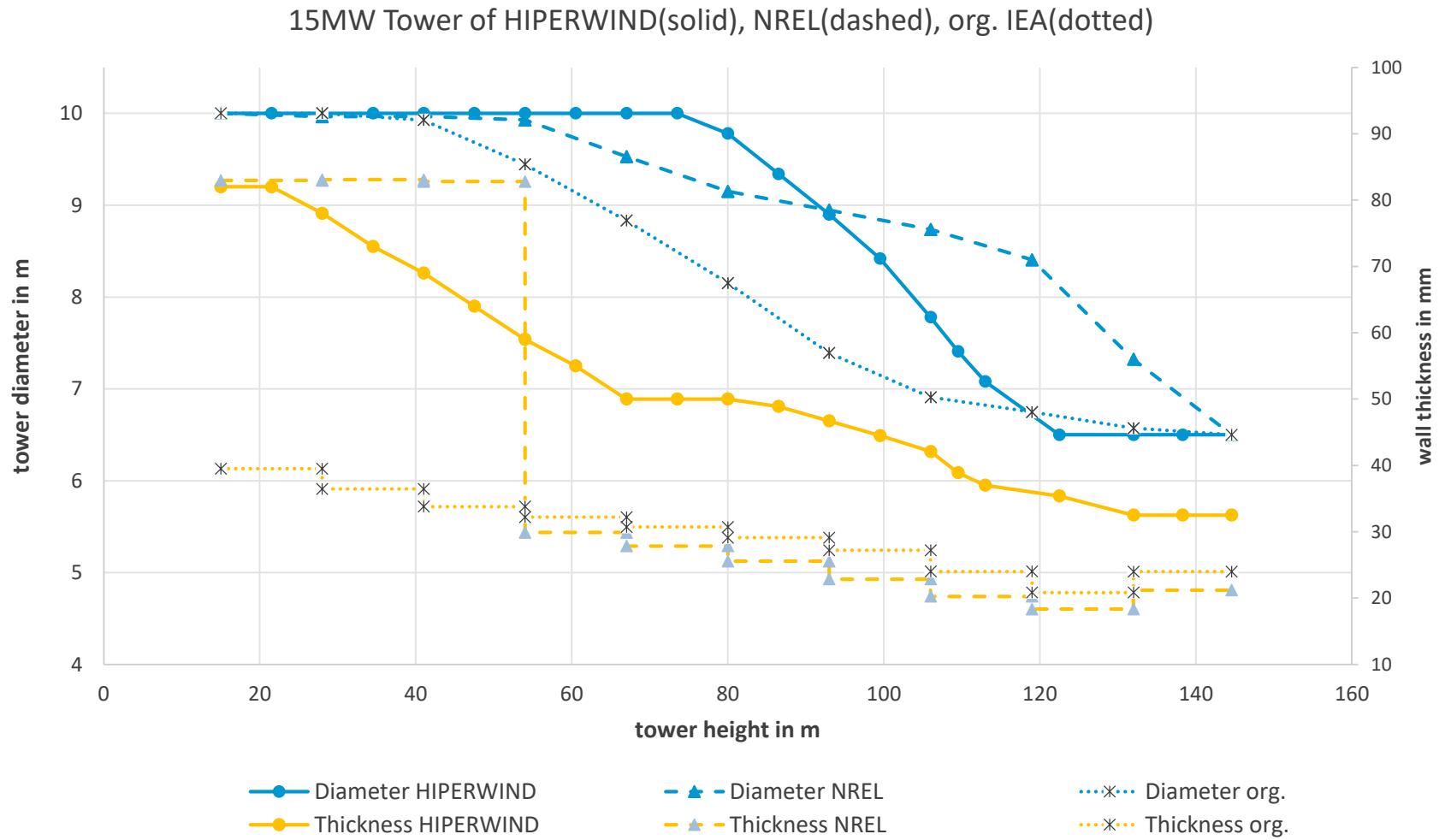
Tower load issue

Old (bottom-fixed) tower load approach:

- 3B-tower is driven by 3B-(fatigue) loads
- Every increase by 2B-loads have to be compensated by thicker tower walls.

Now:

- 3B-tower is driven by its eigenfrequency
- but 3B tower can withstand huge loads (factor of two for 15 MW VolturnUS vs. Monopile at the base)



15 MW Tower weights: Original IEA 803.4t, NREL's Volturn 1250t (+55%), HIPERWIND 1415t (+72%)

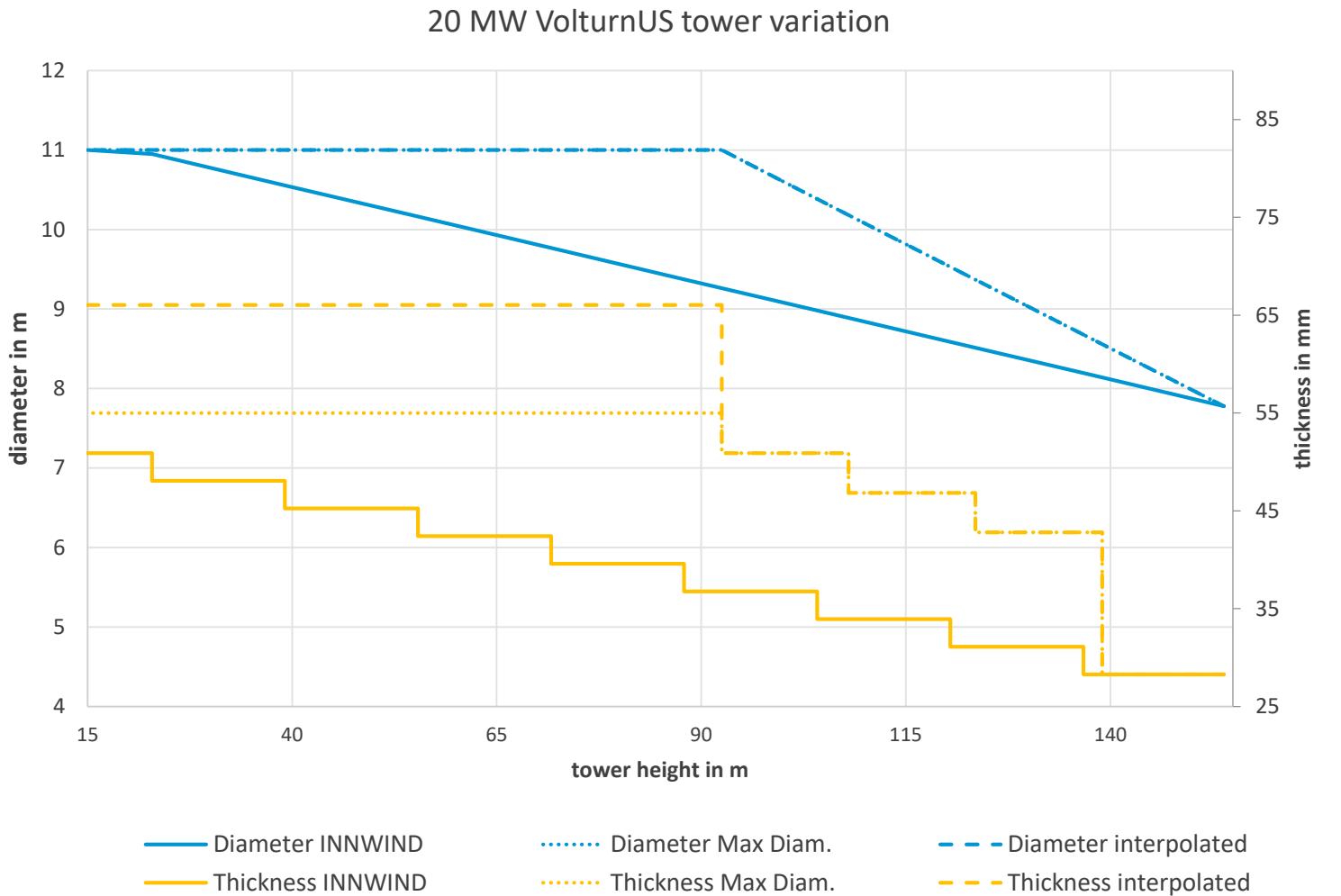
Tower variations for 20 MW

Tower variation with HIPERWIND boundaries:

- max tower wall inclination of 3 deg
- max diameter to thickness ratio of 200

Results:

- INNWIND original: 0.33 Hz, 1409t
- Max Diameter: 0.42 Hz, 1894t (+38%)
- ✓ Max Diam. + interpolated D/t: 0.44 Hz, 2124t, (+56%) (For 15 MW it had been 72%)



15 MW Tower weights: Original IEA 803.4t, NREL's Volturn 1250t (+55%), HIPERWIND 1415t (+72%)

Turbine data: X-Rotors' three-bladed and two-bladed turbines

| | 20 MW 3B Reference (pitch-controlled, fixed hub) | 20 MW 2B Twin (pitch-controlled, fixed hub) | 20 MW 2B Teeter (pitch-controlled, teetering hub) |
|---|--|---|---|
| Rotor diameter | 252.2 m | 257.4 m | 257.4 m |
| Rated tip speed | 90 m/s | 101 m/s | 101 m/s |
| Rated wind speed | 11.4 m/s | 11.4 m/s | 11.4 m/s |
| Rated rotor speed | 6.82 rpm | 7.49 rpm | 7.49 rpm |
| Controller concept | Variable-speed PI-controller, StS- and FA-tower damper | Variable-speed PI-controller, StS- and FA-tower damper | Variable-speed PI-controller, StS- and FA-tower damper |
| Blade mass | 117.9 t | 164.4 t | 138.3 t |
| Total blade mass Σ | 353.7 t | 328.8 t (-7 %) | 276.6 t (-21.8%) |
| Rotor mass (incl. hub) | 636.2 t | 611.2 t | 559.1 t |
| Hub height | 167.9 m | 167.9 m | 167.9 m |
| Tower height | 163.14 m | 163.14 m | 163.14 m |
| Tower mass | 1353.6 t | 2355.3 t (+74 %) | 1371.2 t (+1.3%) |
| 1st tower eigenfrequency | 0.166 Hz | 0.168 Hz | 0.17 Hz |
| Drive train concept | Direct-Drive | Direct-Drive | Direct-Drive |
| Nacelle mass | 1098.0 t | 1050.2 t | 1050.2 t |

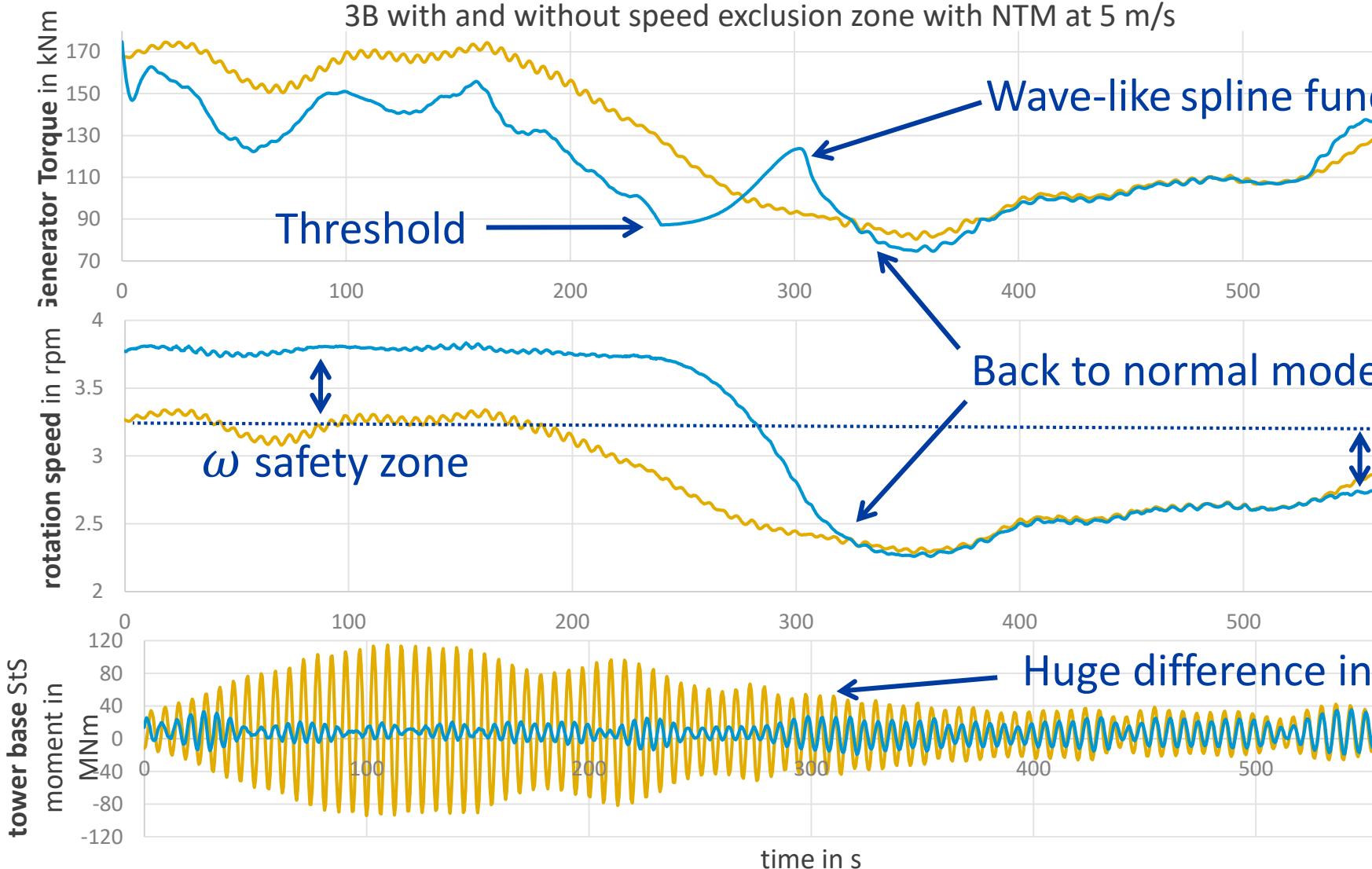
Load Summary

| DLC 1.2 – Fatigue | absolute | | | | relative (2B/3B) | |
|---|------------|------------|--------------|--|------------------|--------------|
| | 3B ref | 2B101 Twin | 2B101 Teeter | | 2B101 Twin | 2B101 Teeter |
| blade root edgewise bending moment Mx DEL in Nm | 71002740 | 105901159 | 92506189 | | 1.49 | 1.30 |
| blade root flapwise bending moment My DEL in Nm | 45376077 | 73257074 | 49189214 | | 1.61 | 1.08 |
| blade root torsional moment Mz DEL in Nm | 737543 | 1295715 | 1104815 | | 1.76 | 1.50 |
| yaw roll moment Mx DEL in Nm | 2473398.18 | 3215212 | 1869201 | | 1.30 | 0.76 |
| yaw nodding moment My DEL in Nm | 21975649 | 35467791 | 7237193 | | 1.61 | 0.33 |
| yaw torsional moment Mz DEL in Nm | 21109974.8 | 33656891 | 5252466 | | 1.59 | 0.25 |
| hub stationary nodding moment My DEL in Nm | 21253726 | 34271970 | 3745416 | | 1.61 | 0.18 |
| hub stationary yawing moment Mz DEL in Nm | 20472812.5 | 33068200 | 3798590 | | 1.62 | 0.19 |
| hub rotating moment My DEL in Nm | 26353208.5 | 48706135 | 3574 | | 1.85 | 0.00 |
| hub rotating moment Mz DEL in Nm | 26290111.1 | 8040764 | 6399148 | | 0.31 | 0.24 |
| main shaft rotating bending DEL at first bearing in worst direction in Nm | 38786740 | 47984555 | 34044479 | | 1.24 | 0.88 |
| tower base bending moment DEL in worst direction in Nm | 42072893 | 58465556 | 42917463 | | 1.39 | 1.02 |
| Pitch mean ADC (actuator duty cycle) in deg/600s | 115 | 568 | 597 | | 4.93 | 5.17 |
| pitch rotations in 25 years | 335149 | 1650827 | 1734346 | | 4.93 | 5.17 |
| mean power in W (if online) | 12252621 | 12196080 | 12187187 | | 1.00 | 0.99 |
| Extreme loads | | | | | | |
| Blade root flapwise moment My in Nm | 143218962 | 216860713 | 183836849 | | 1.51 | 1.28 |
| Blade root edgewise moment Mx in Nm | 70687995 | 90006419 | 101775588 | | 1.27 | 1.44 |
| Yaw nodding My in Nm | -114572990 | 133372140 | -99928797 | | 1.16 | 0.87 |
| Yaw roll Mx in Nm | 41742905 | 45627127 | 38716177 | | 1.09 | 0.93 |
| Yaw torsional Mz in Nm | 113998910 | 194570873 | 23921628 | | 1.71 | 0.21 |
| Yaw "gravity" force Fz in N | -17228985 | -16114630 | -16084946 | | 0.94 | 0.93 |
| minimum tip to tower clearance in m | 5.62 | 6.77 | 5.93 | | 1.20 | 1.06 |
| generator mass [kg] scaled by faster rotation speed | 523,960 | 483,830 | 483,830 | | 0.92 | 0.92 |
| blade mass [kg] | 117,894 | 164,402 | 138,314 | | 1.39 | 1.17 |
| material cost share jacket | 0.59398 | | | | | |

could be a game changer for floating turbines

- Be aware for blade masses and loads that the blades of the two-bladed turbines are ~2% larger.
- All DELs have a reference frequency of 1Hz and all values are Rayleigh distribution weighted with 11.4m/s rated wind speed
- If further values are desired, feel free to contact fabian.anstock@haw-hamburg.de

Speed exclusion zone – how does it work? Example

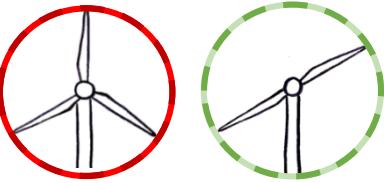


3B: Tower DEL¹ reduced by -9.9% and energy yield¹ by -0.03%

2B101: Tower DEL¹ reduced by -33.8% and energy yield¹ by -0.17%

¹ DLC 1.2 with 0°, +8° yaw and Rayleigh distribution with 11.4 m/s mean wind speed

20 MW VoltturnUS tower eigenfrequency examples



Tower frequency variation:

INNWIND original: 0.33 Hz, 1408823 t

Max Diameter : 0.42 Hz, 1894039 t (+38%)

+ HIPERWIND D/t : 0.48 Hz, 2625289 t, (+95%)

+ interpolated D/t: 0.44 Hz, 2123969 t, (+56%)

