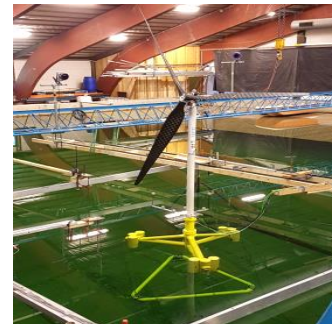
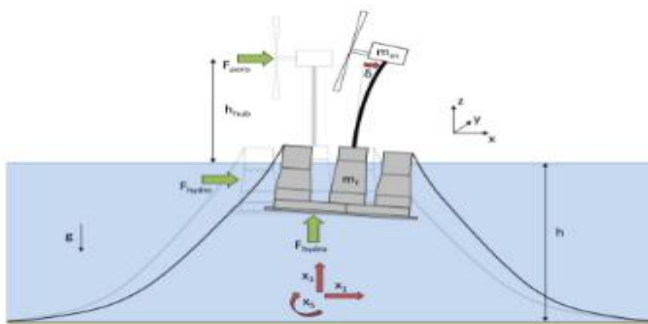


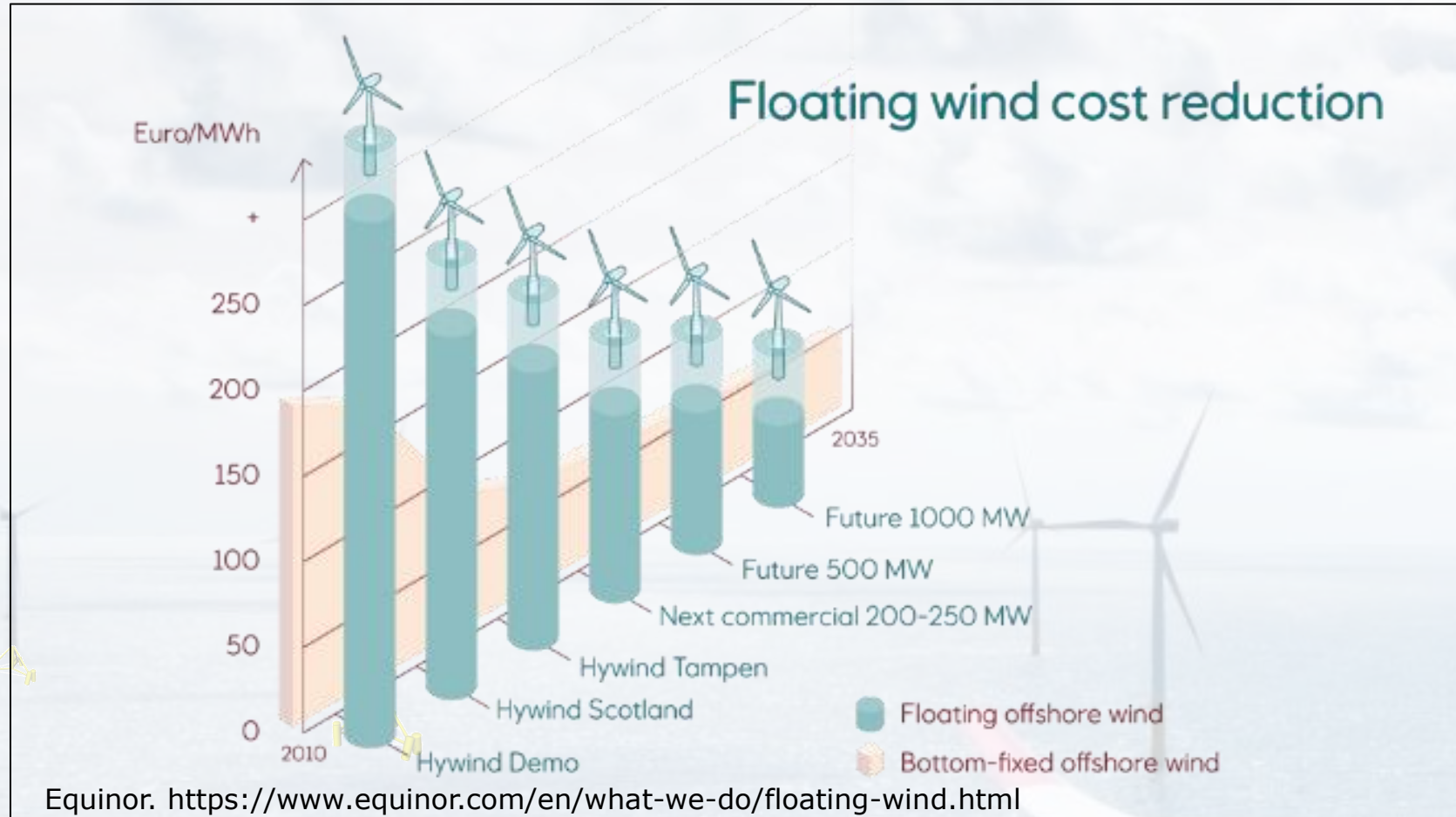


**Accurate and validated design tools
for floating wind turbine substructures.**
Recent results from the FloatStep project



Henrik Bredmose¹, Antonio Pegalajar-Jurado¹, Christine L. Hansen¹, Nicoló Pollini¹, Kasper Laugesen², Bjarne Jensen³, Michael Borg⁴, Johan Rønby⁵, Jana Orszaghova⁶

Floating wind farms and the needed LCOE reduction



The TetraSpar concept

Stiesdal Offshore
Technologies

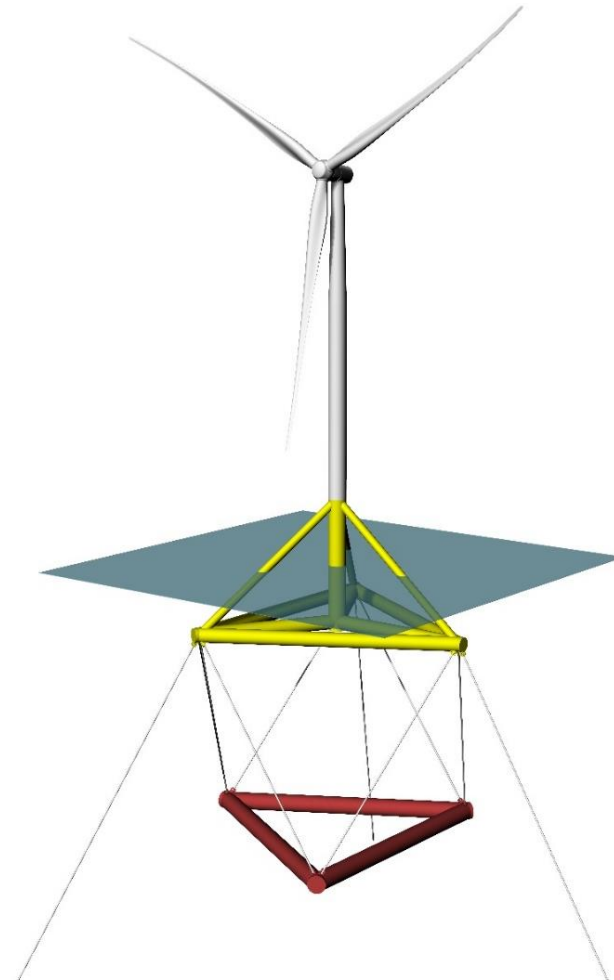
Innovation Fund Denmark

Mindset

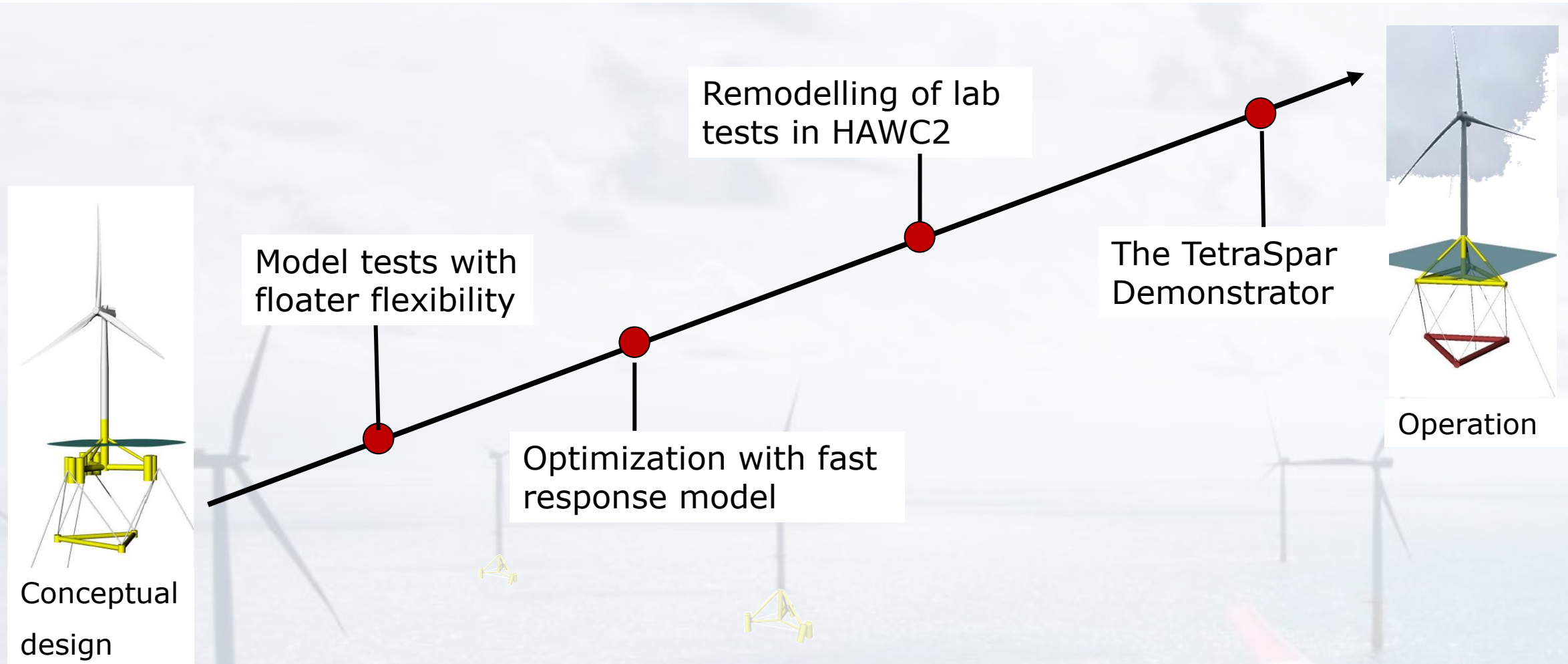
- Conventional thinking
 - We have designed this structure – now, how do we build it?
- TetraSpar thinking
 - We need to manufacture this way – now, how do we design it?

Concept

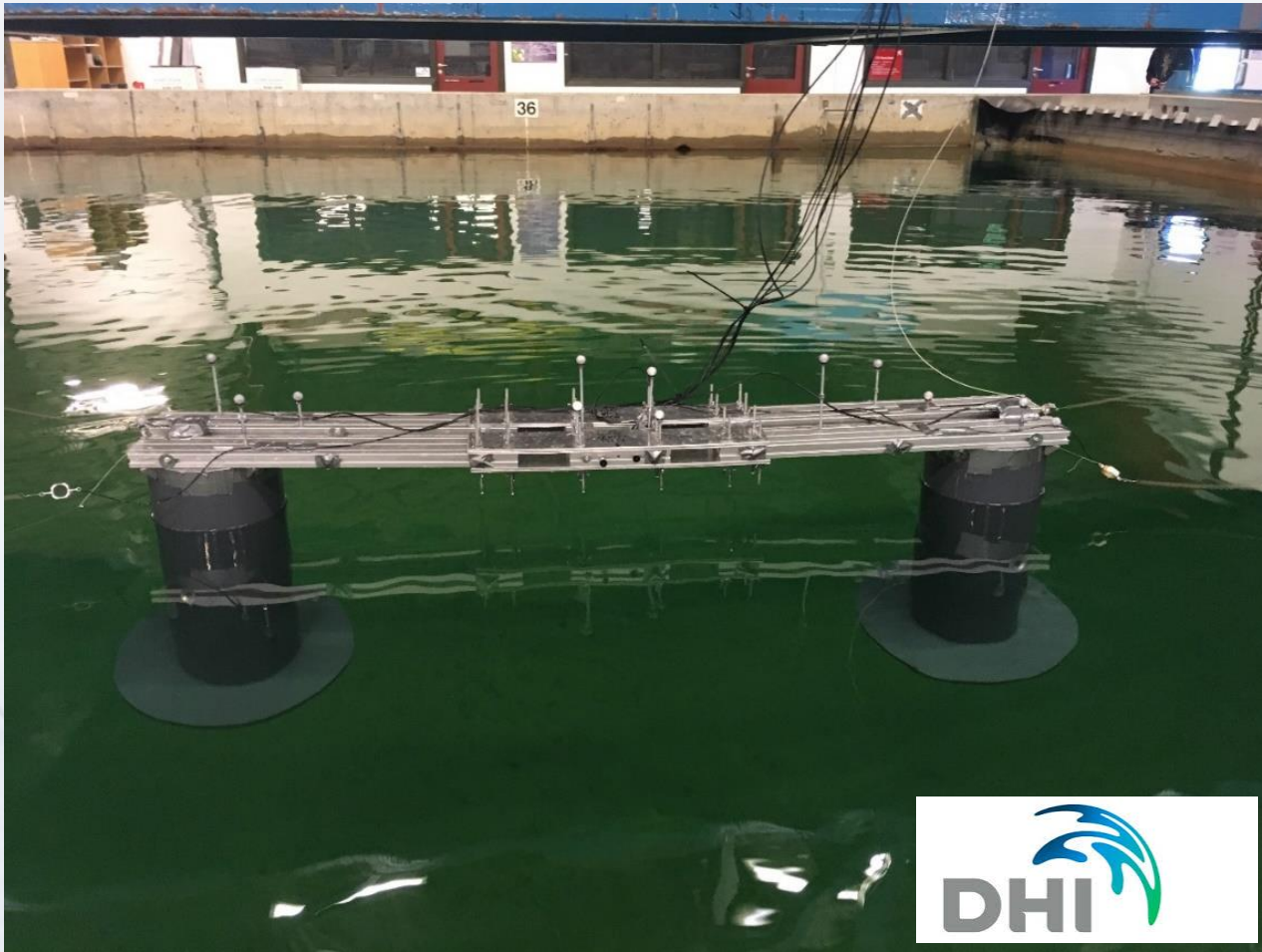
- Modular – all components factory-made, transported by road
- Components assembled at quayside with bolts (not exposed to sea water)
- Turbine mounted in harbor and towed to site, no installation vessels
- Weight 1000-1500 t for 6 MW turbine



FloatStep: Validated and efficient design tools



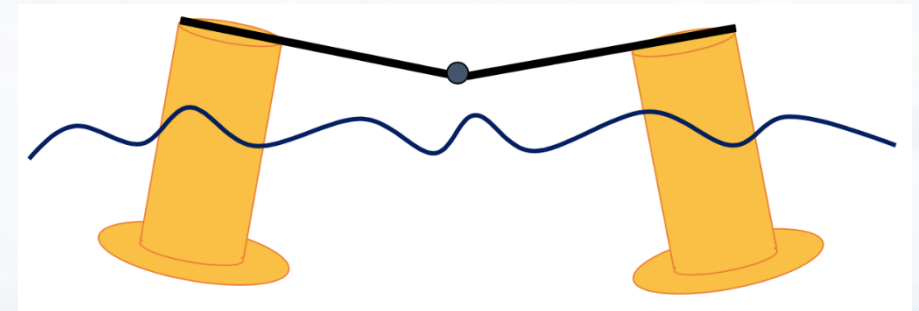
Model tests with flexible floater



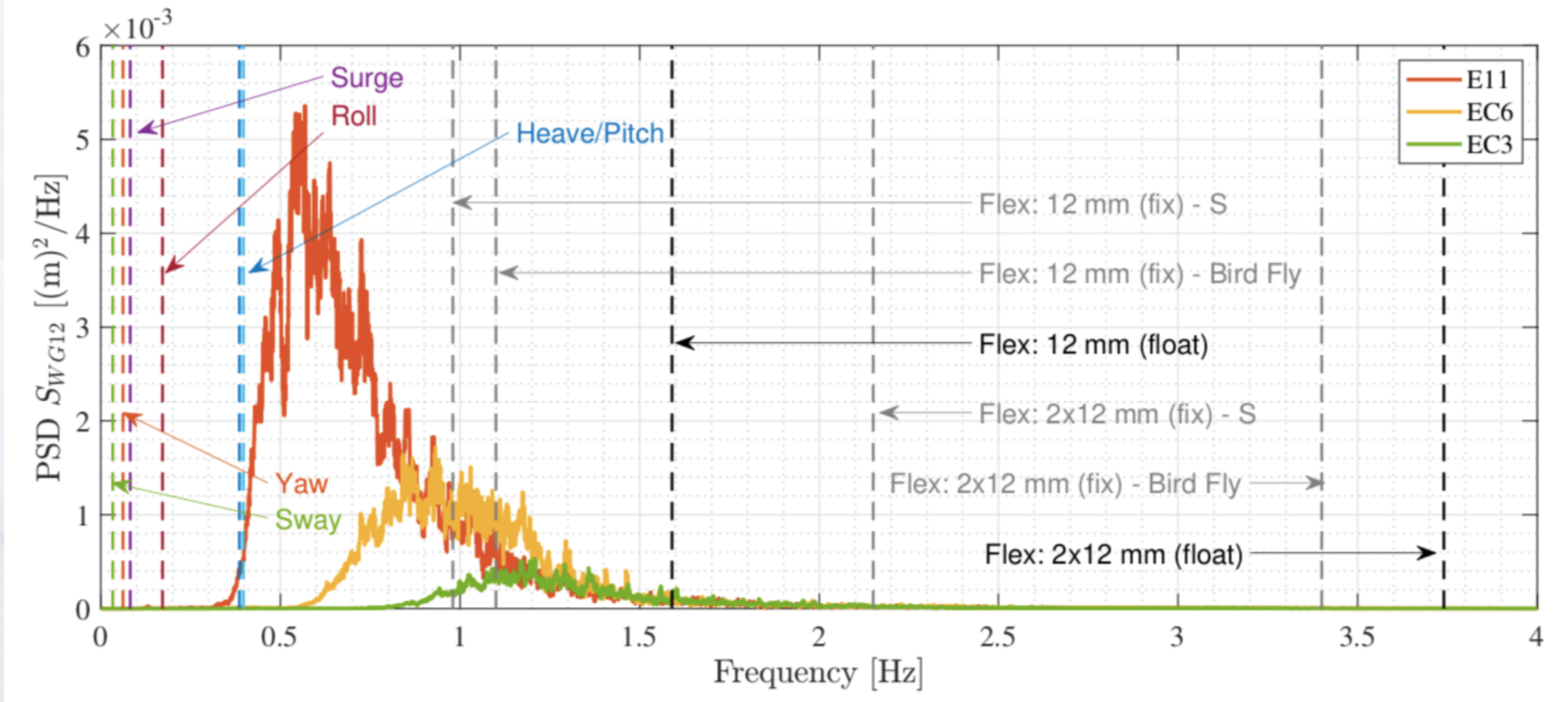
2-cylinder with heave plate structure

Flexible beam

Adjustable flexibility: (12 or 24 mm thickness)

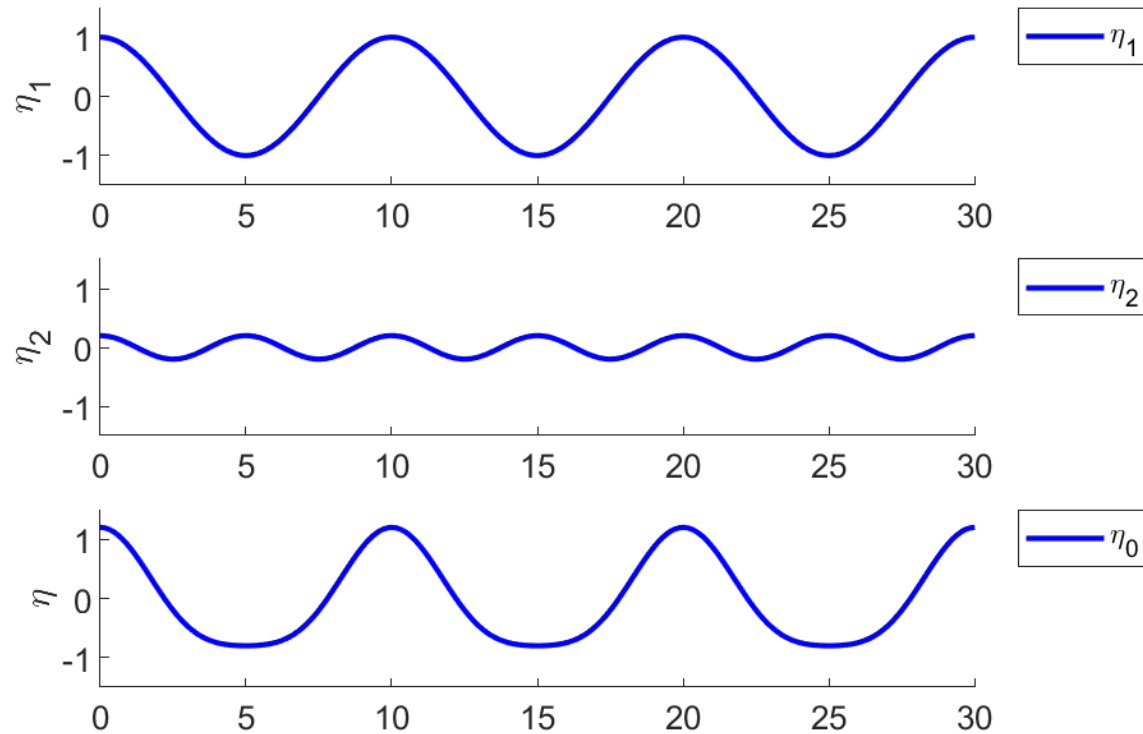


Natural frequencies and wave climates



Methodology – Harmonic separation

Walker, Taylor & Eatock Taylor (2004)



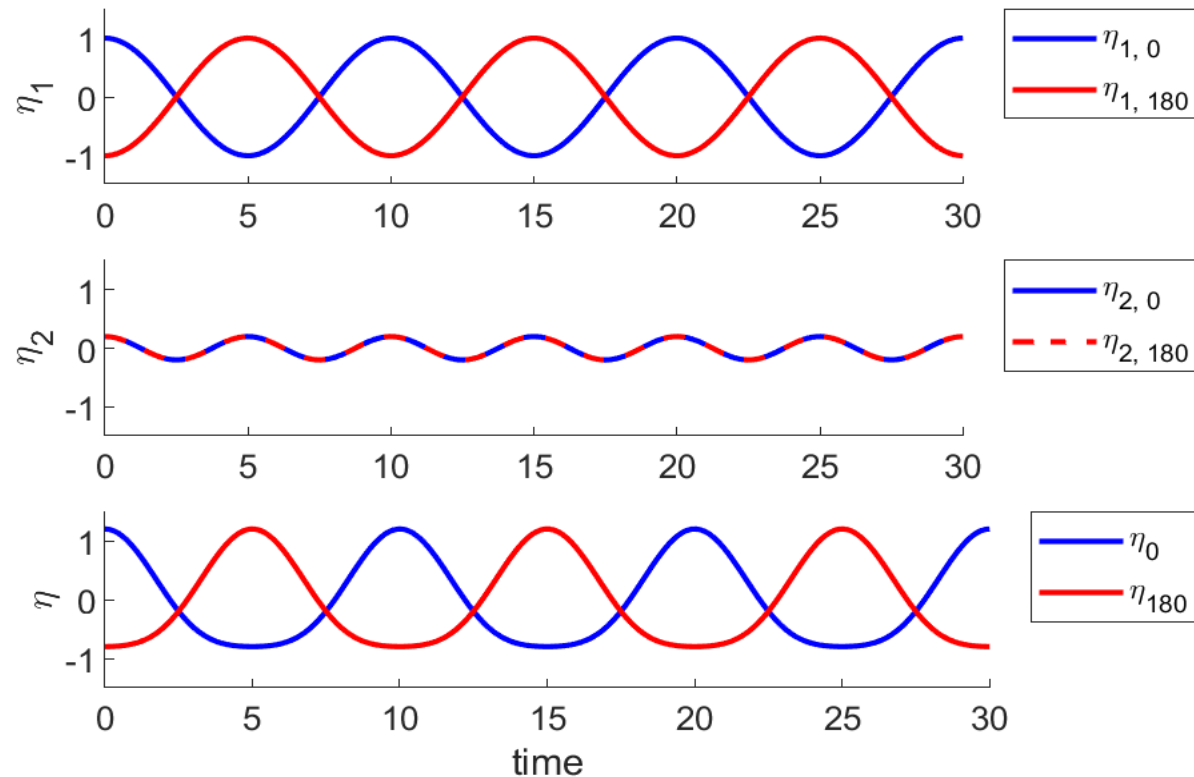
$$\eta_1 = a \cos \omega t$$

$$\eta_2 = c_{11} a^2 \cos 2\omega t$$

$$\eta_0 = \eta_1 + \eta_2$$

Methodology – Harmonic decomposition

Walker, Taylor & Eatock Taylor (2004)



$$\eta_1 = a \cos \omega t$$

$$\eta_2 = c_{11} a^2 \cos 2\omega t$$

$$\eta_0 = \eta_1 + \eta_2$$

$$\eta_{180} = -\eta_1 + \eta_2$$

$$\eta_1 = (\eta_0 - \eta_{180})/2$$

$$\eta_2 = (\eta_0 + \eta_{180})/2$$

Four-phase harmonic separation

- Fitzgerald et al (2014; Method)
- Ghadirian et al (2019; Numerical for monopile loads)
- Orzaghova et al (2021; Experimental, floating, FloatStep)

$$Q^{(1)} = \frac{Q_0 - Q_{90}^H - Q_{180} + Q_{270}^H}{4} = Aq_{11} \cos \phi + A^2 f_D \cos \phi + A^3 q_{31} \cos \phi + O(A^5)$$

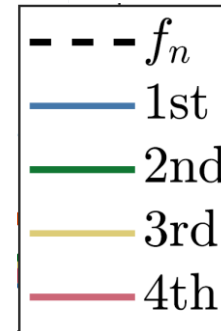
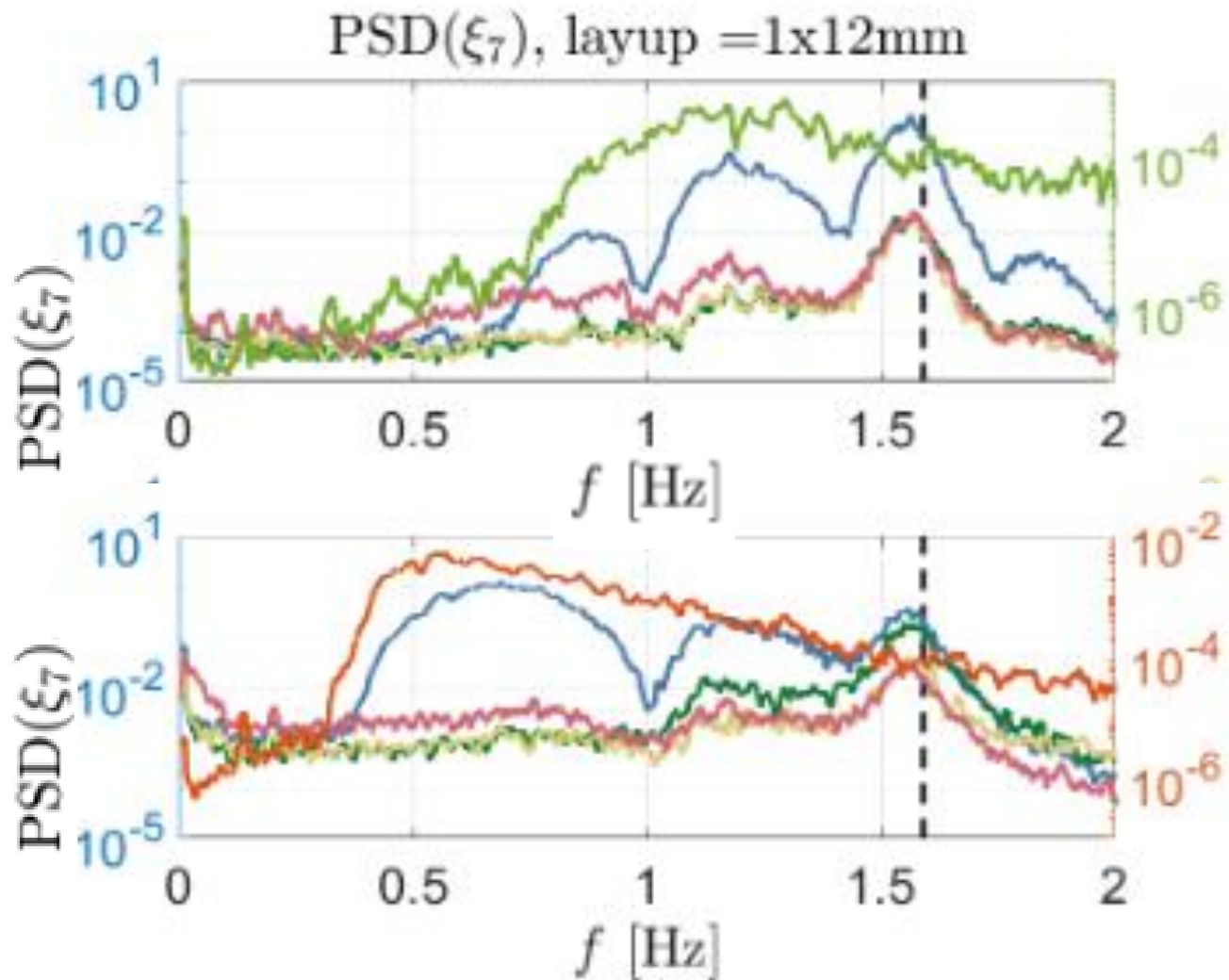
$$Q^{(2)} = \frac{Q_0 - Q_{90} + Q_{180} - Q_{270}}{4} = A^2 q_{22} \cos 2\phi + A^4 q_{42} \cos 2\phi + O(A^6)$$

$$Q^{(3)} = \frac{Q_0 + Q_{90}^H - Q_{180} - Q_{270}^H}{4} = A^2 f_D \cos 3\phi + A^3 q_{33} \cos 3\phi + O(A^5)$$

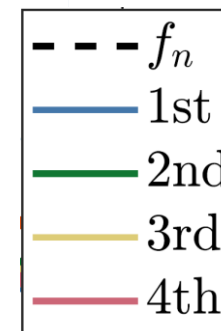
$$Q^{(4)} = \frac{Q_0 + Q_{90} + Q_{180} + Q_{270}}{4} = A^2 q_{20} + A^4 q_{40} + A^4 q_{44} \cos 4\phi + O(A^6)$$

Allows harmonic content to be separated into 1st, 2nd, 3rd and 4th harmonic

Flexible mode - Analysis by harmonic separation

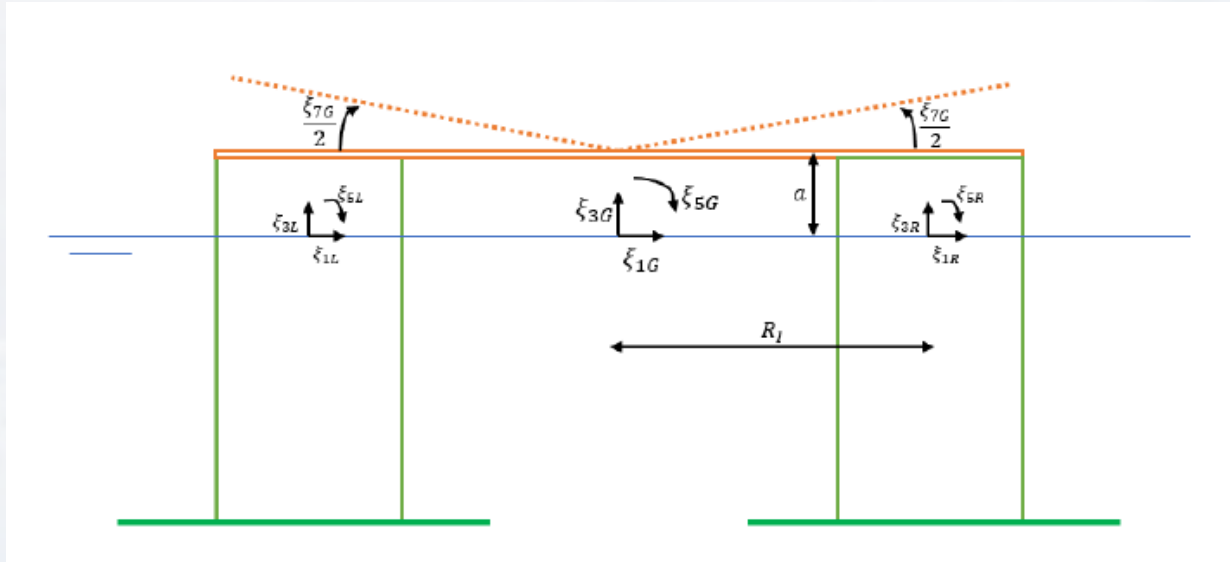


Weak sea state (top green line)
Flexible mode at 1.6 Hz driven linearly



Storm sea state (top red line)
Flexible mode at 1.6 Hz strong 2nd-order contribution

Reproduction in Morison based model



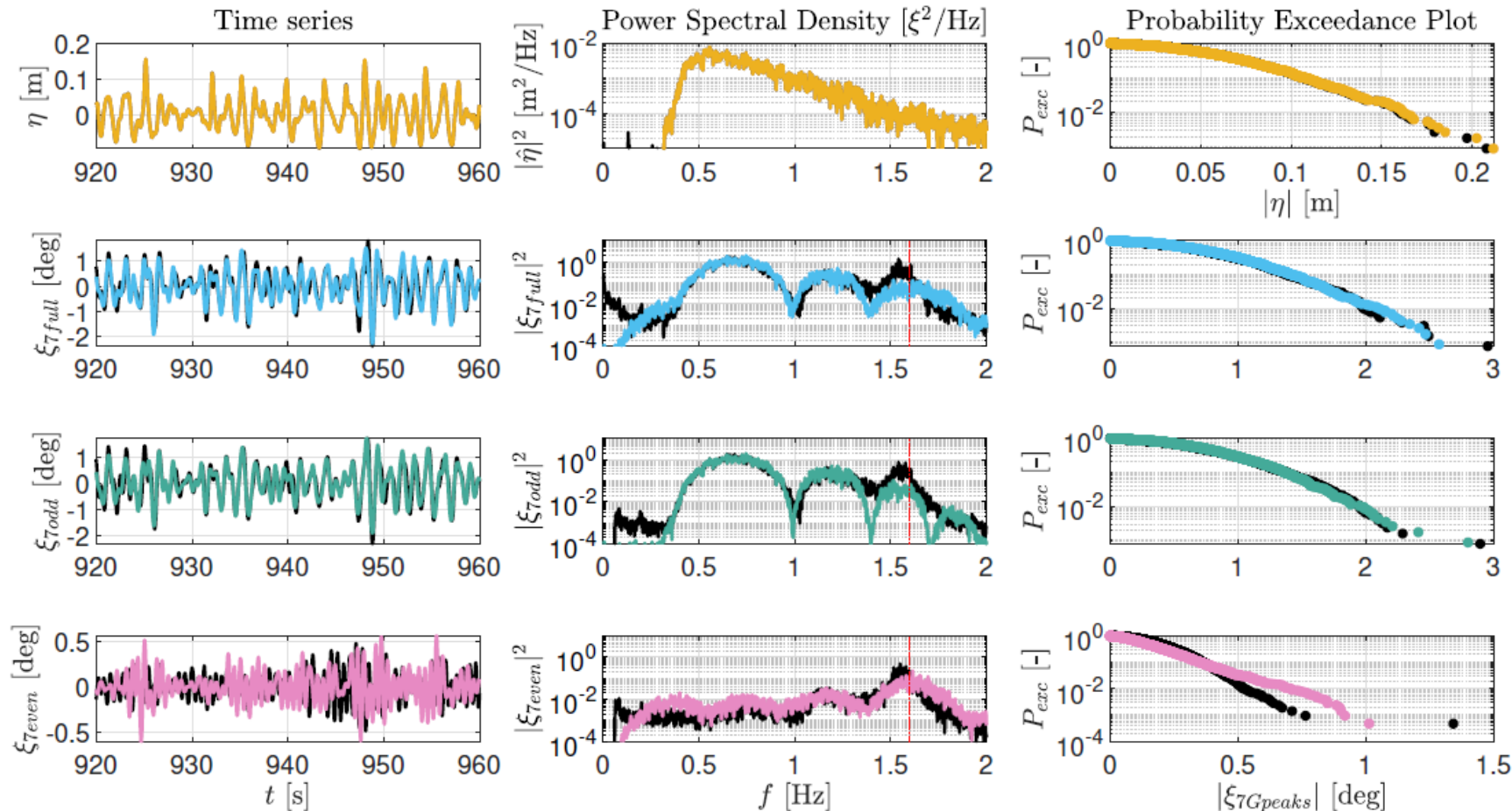
Two stiff beams with torsional spring
 Second-order wave forcing accelerated $O(N \log N)$
 Rainey force model (inertia loads)
 Drag force

Calibrated modal damping

Further info:

- Hansen et al (2022 – in preparation)
- MSc thesis Maude Vincent (2020)
- MSc thesis Christine Lynggaard Hansen (2021)
- Bredmose & Pegalajar-Jurado (2021)

Reproduction: Storm sea state



Measurements in black

Hansen et al (2022 - in preparation)

Free surface elevation

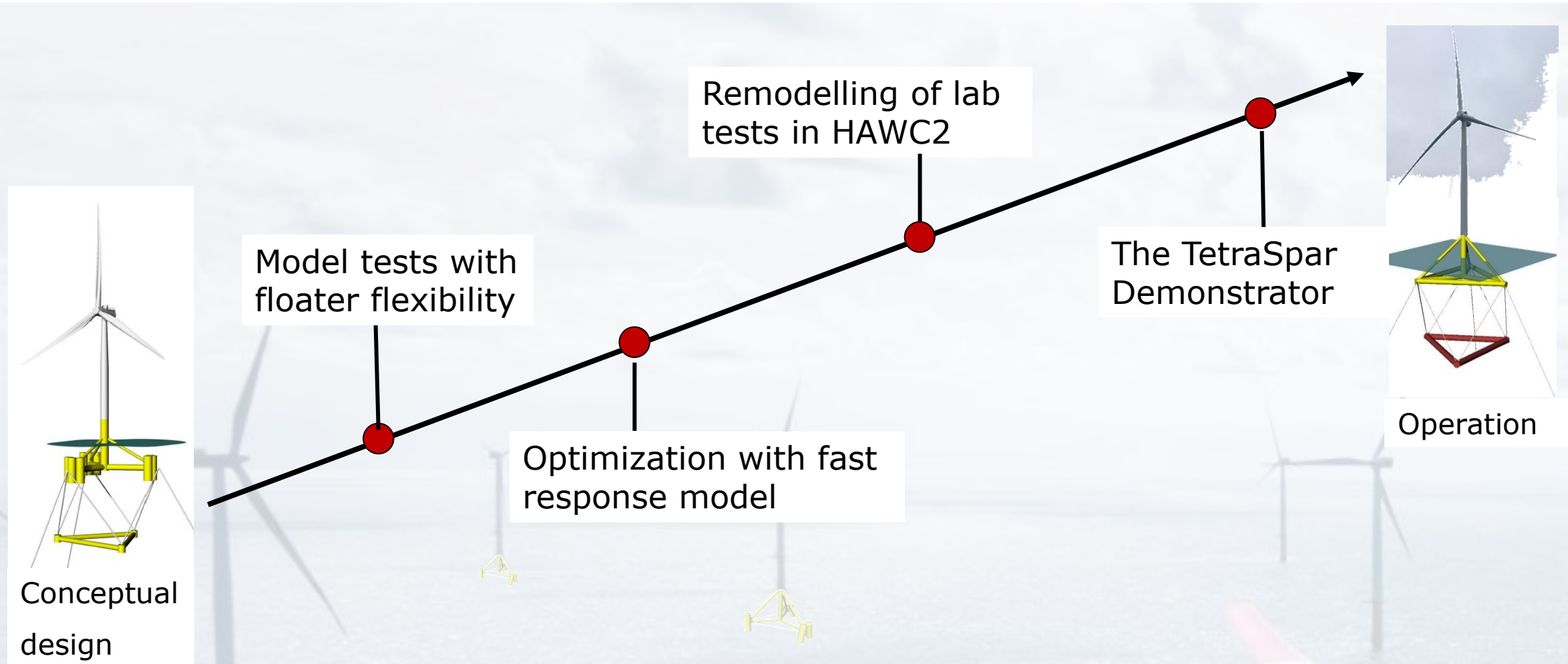
Total flexible motion

Linear + drag

Second-harmonic
(second-order inviscid)

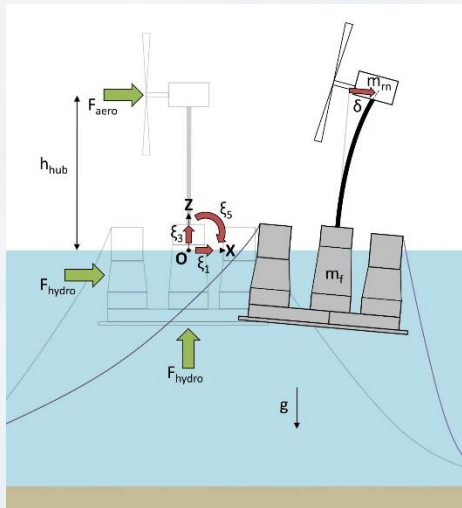
Successful reproduction of flexible motion to second order.

Validated and efficient design tools



15 MW TetraSpar type design by QuLAF and optimization

The QuLAF model



Precompute rotor loads and aero damping

3 floater DOFs
1 tower DOF

WAMIT data for hydro
Linearized mooring

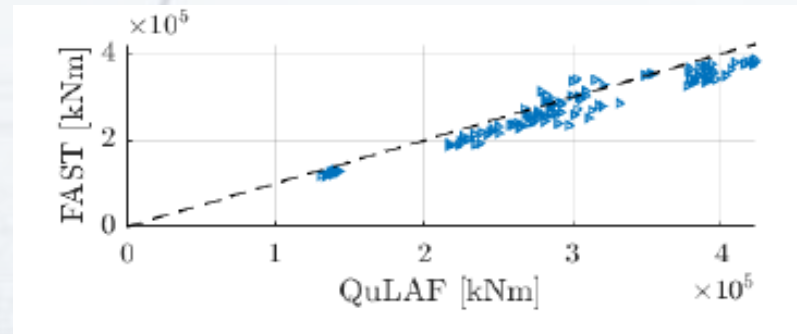
2000 x real time

Pegalajar-Jurado et al (2018)

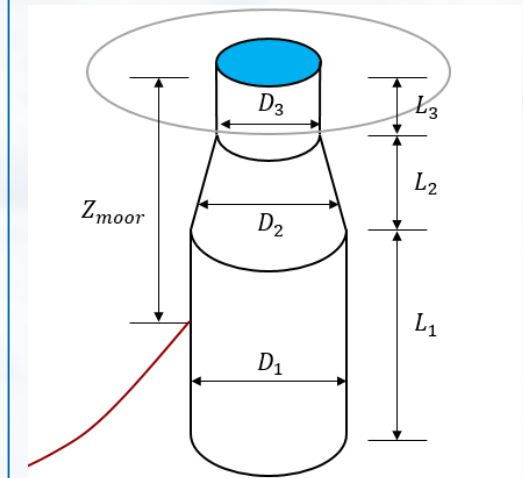


Madsen et al (2019)

Validation study
Tower bottom moment



Spar designs in FloatStep:



Dou et al (2020)
Pollini et al (2021)

See also Hegseth et al (2020)

The IEA WIND 15-MW offshore reference wind turbine

240 m

150 m

15 MW

March 2020
IEA Wind TCP Task 37
Definition of the IEA Wind 15-Megawatt Offshore Reference Wind Turbine
Technical Report
iea wind

Gaertner, Evan, Jennifer Rinker, Latha Sethuraman, Frederik Zahle, Benjamin Anderson, Garrett Barter, Nikhar Abbas, Fanzhong Meng, Pietro Bortolotti, Witold Skrzypinski, George Scott, Roland Feil, Henrik Bredmose, Katherine Dykes, Matt Shields, Christopher Allen, and Anthony Viselli. 2020. *Definition of the IEA 15-Megawatt Offshore Reference Wind Turbine*. Golden, CO: National Renewable Energy Laboratory. NREL/TP-5000-75698. <https://www.nrel.gov/docs/fy20osti/75698.pdf>

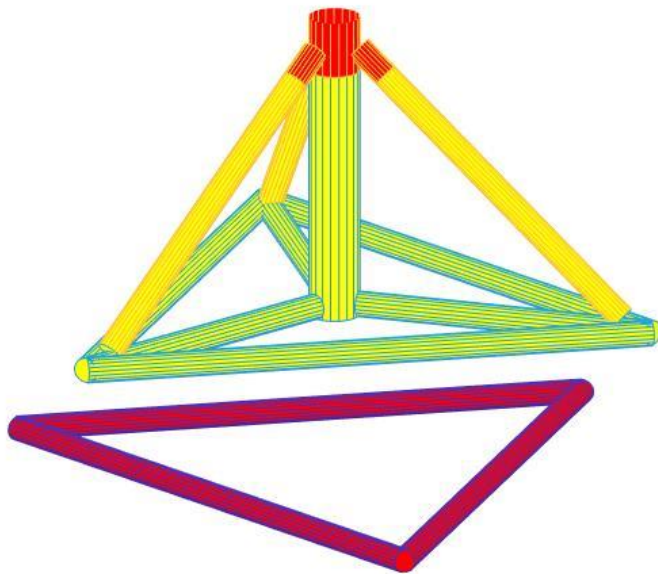
Graphics by NREL



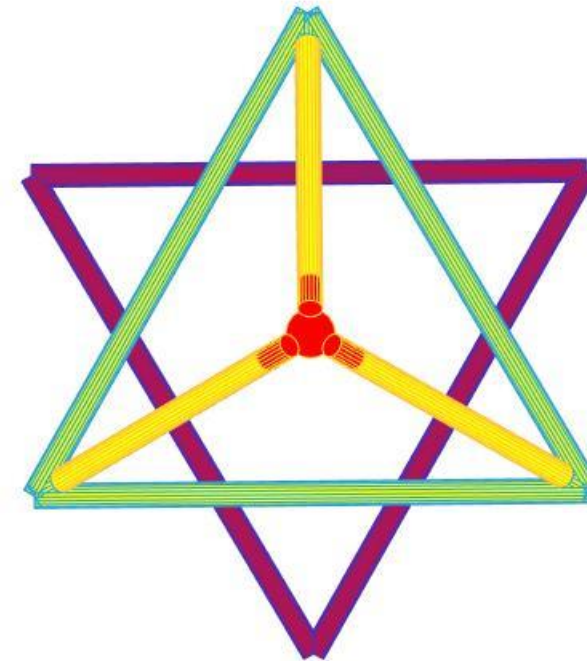
Made by NREL and DTU

Publicly available as FAST and HAWC2 models at <https://github.com/IEAWindTask37/IEA-15-240-RWT>

Extension to optimization with motion 6+1 DOFs



Currently:
6 DOFs of the floater
1 DOFs of the tower



- | | |
|----------------------|--------------------------|
| x1: L central column | x6: D horizontal brace |
| x2: D central column | x7: Depth of keel |
| x3: L pontoon | x8: Depth fairlead |
| x4: D pontoon | x9: Anchor radius |
| x5: D brace | x10: Mooring line length |

Nicoló Pollini ++



Governing equations

- Design variables (\mathbf{v}): geometric properties of the floater and the mooring system.
- The equations of motion for a floating wind turbine in QuLAF are:

$$(-\omega^2 (\mathbf{M}(\mathbf{v}) + \mathbf{A}(\mathbf{v})) + i\omega\mathbf{B}(\mathbf{v}) + \mathbf{C}(\mathbf{v})) \hat{\boldsymbol{\xi}}_j(\omega) = \hat{\mathbf{F}}_j^h(\mathbf{v}, \omega) + \hat{\mathbf{F}}_j^a(\omega)$$

- \mathbf{M} structural mass; \mathbf{A} added mass; \mathbf{B} damping; \mathbf{C} restoring matrix (hydrostatic and mooring); \mathbf{F}^h hydrodynamic loads (Morison's equation); \mathbf{F}^a precomputed aerodynamic loads.
- The optimization problem includes limits on eigenvalues from the eigenvalue problems

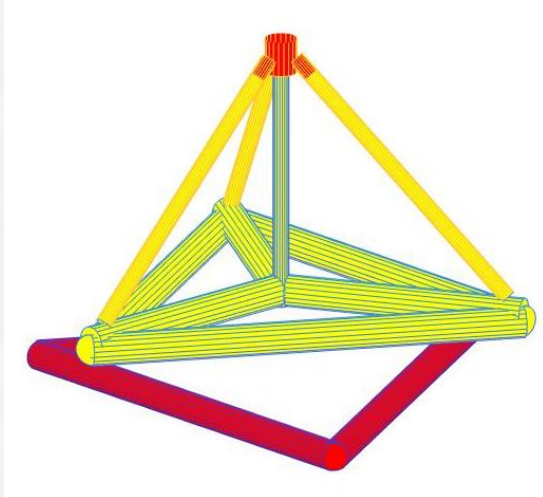
$$(\mathbf{C}(\mathbf{v}) - \lambda_i(\mathbf{M}(\mathbf{v}) + \mathbf{A}(\mathbf{v})))\phi_i = \mathbf{0}$$

- and limits on the maximum static responses

$$\mathbf{C}(\mathbf{v})\mathbf{u} = \mathbf{f}^s$$

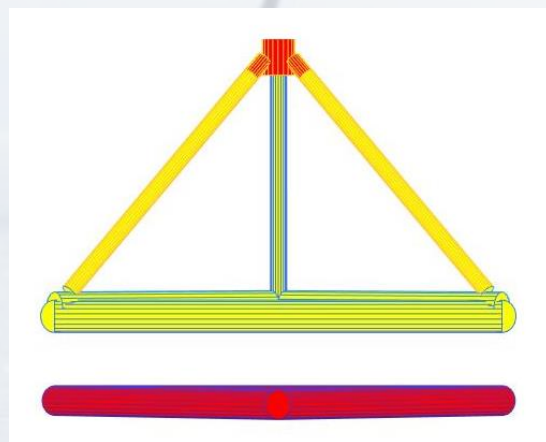
Dou et al (2020)
Pollini et al (2021)

Initial results – no transient dynamic constraints



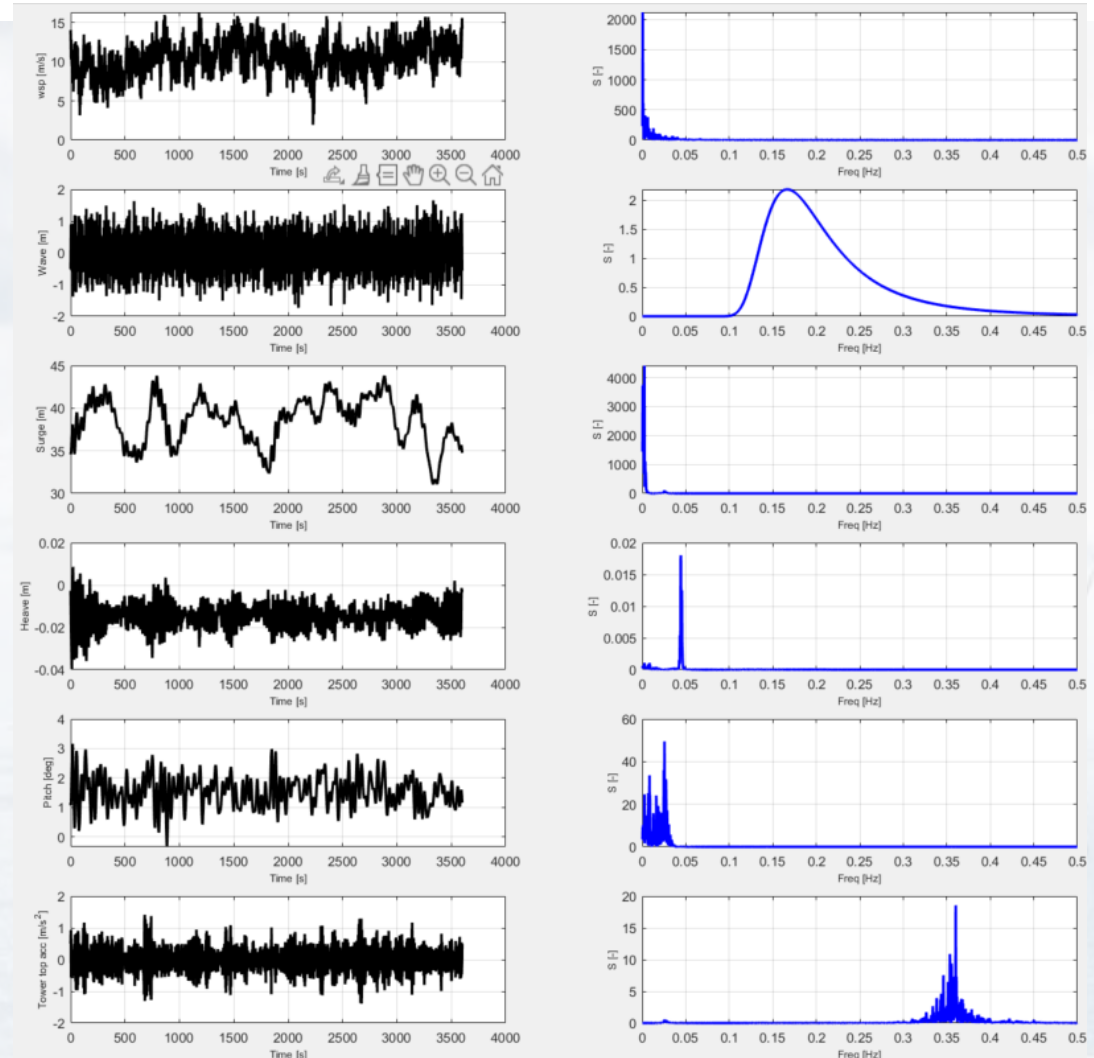
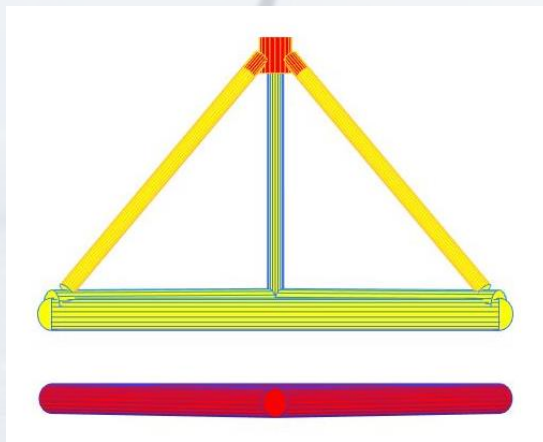
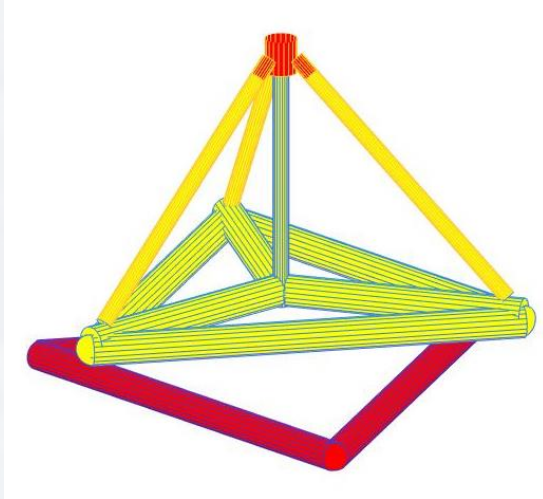
Initial height of central column: 50.000 meter
 Initial diameter of central column: 10.000 meter
 Initial length of pontoon: 65.000 meter
 Initial diameter of pontoon: 5.000 meter
 Initial diameter of brace: 5.000 meter
 Initial diameter of horizontal brace: 5.000 meter
 Initial depth of keel from msl: -80.000 meter
 Initial depth of fairlead: -10.000 meter
 Initial anchor radius: 650.000 meter
 Initial mooring line length: 750.000 meter
 cost: 21.5579

Optimized height of central column: 76.328 meter
 Optimized diameter of central column: 5.005 meter
 Optimized length of pontoon: 79.536 meter
 Optimized diameter of pontoon: 10.000 meter
 Optimized diameter of brace: 5.000 meter
 Optimized diameter of horizontal brace: 10.000 meter
 Optimized depth of keel from msl: -100.602 meter
 Optimized depth of fairlead: -36.720 meter
 Optimized anchor radius: 676.312 meter
 Optimized mooring line length: 772.139 meter
 cost: 84.1235

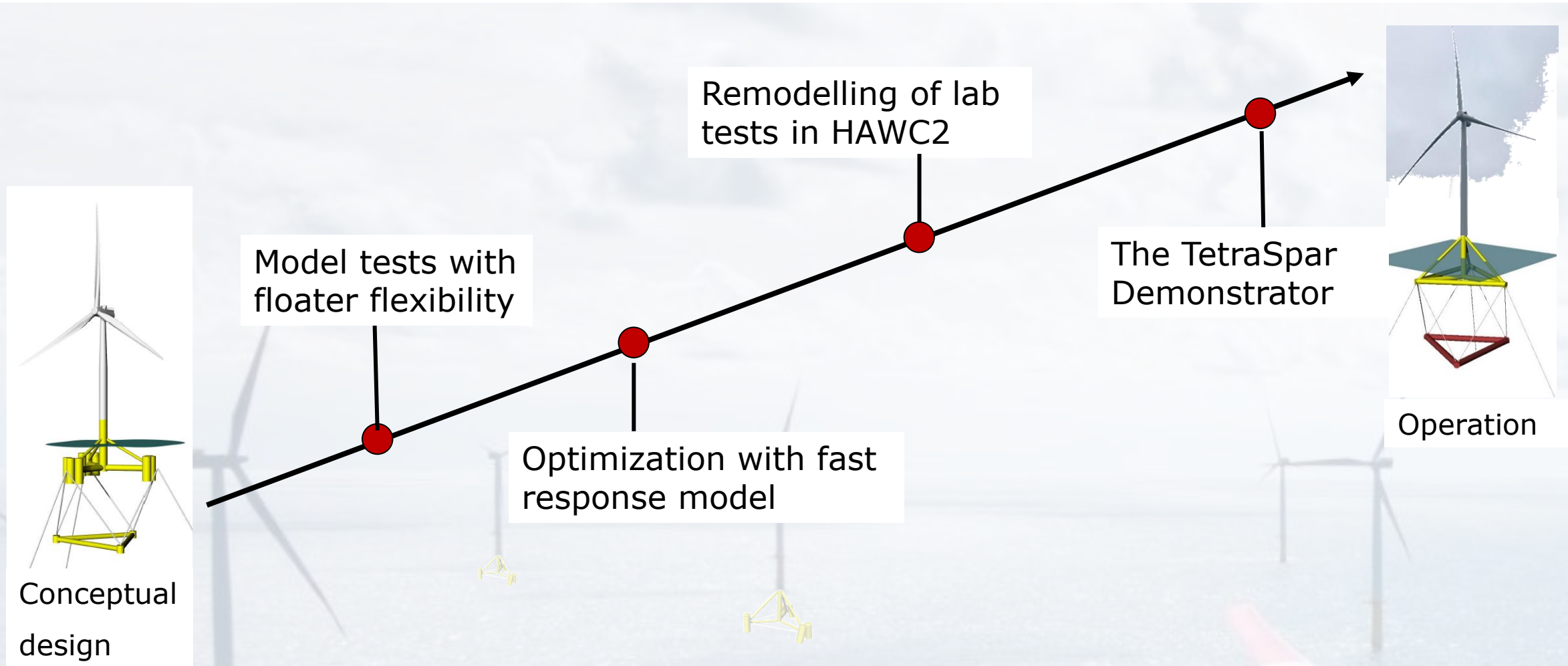


Nicoló Pollini ++

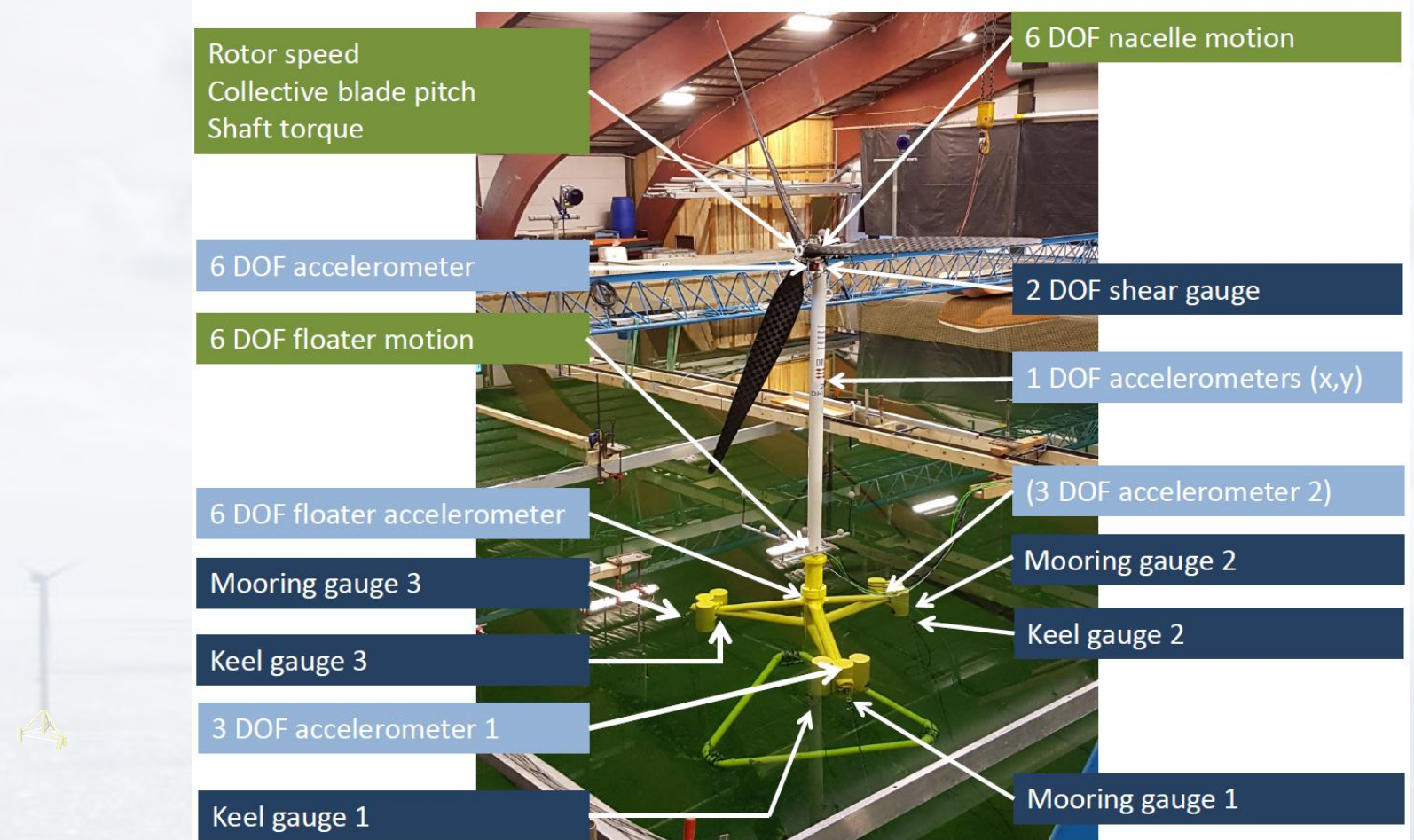
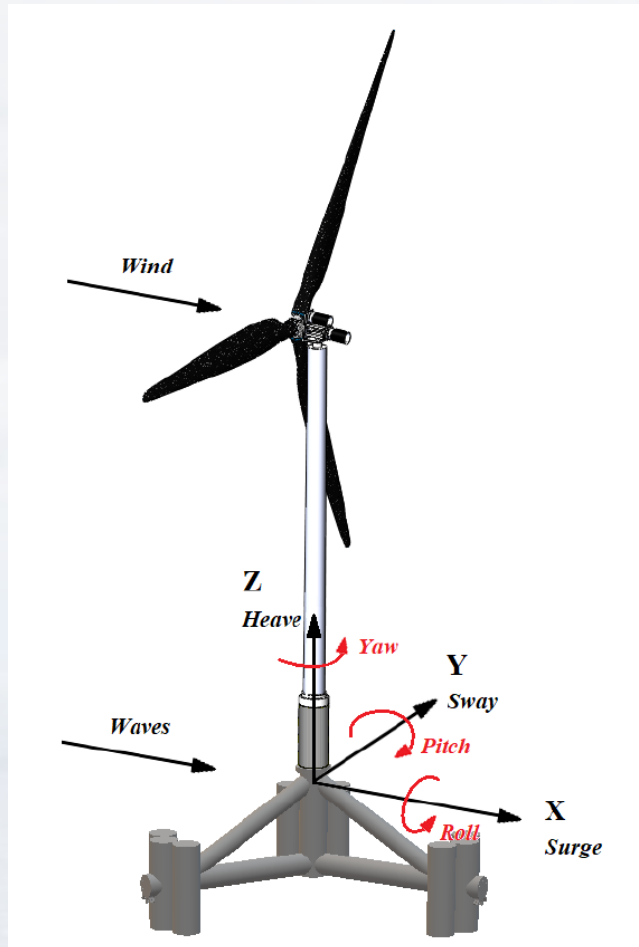
Initial results – no transient dynamic constraints



Validated and efficient design tools

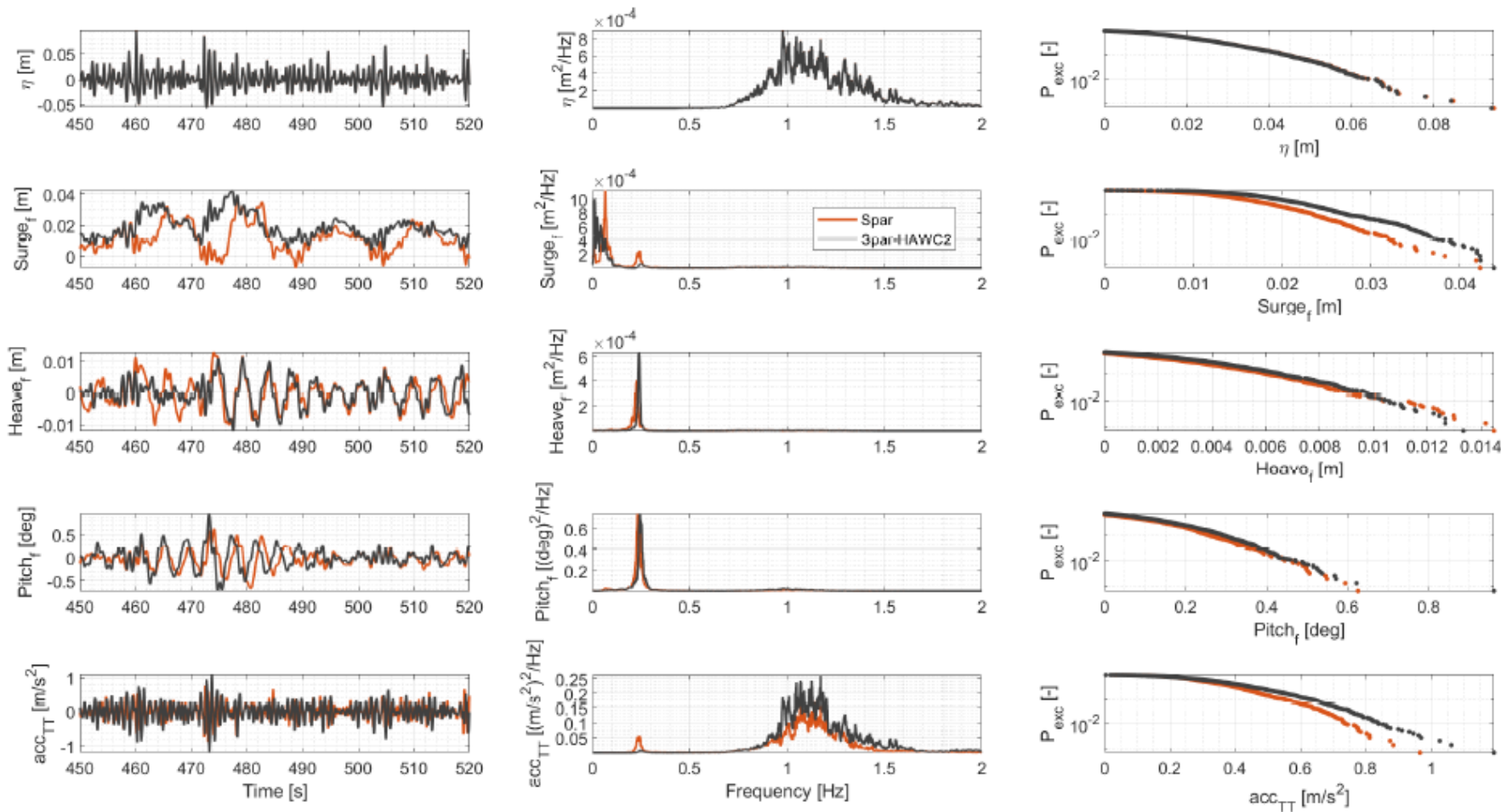


Validation of HAWC2 for 2017 TetraSpar proof of concept tests



Wave state of rated wind speed

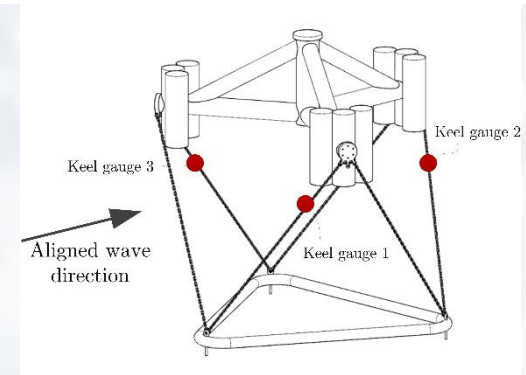
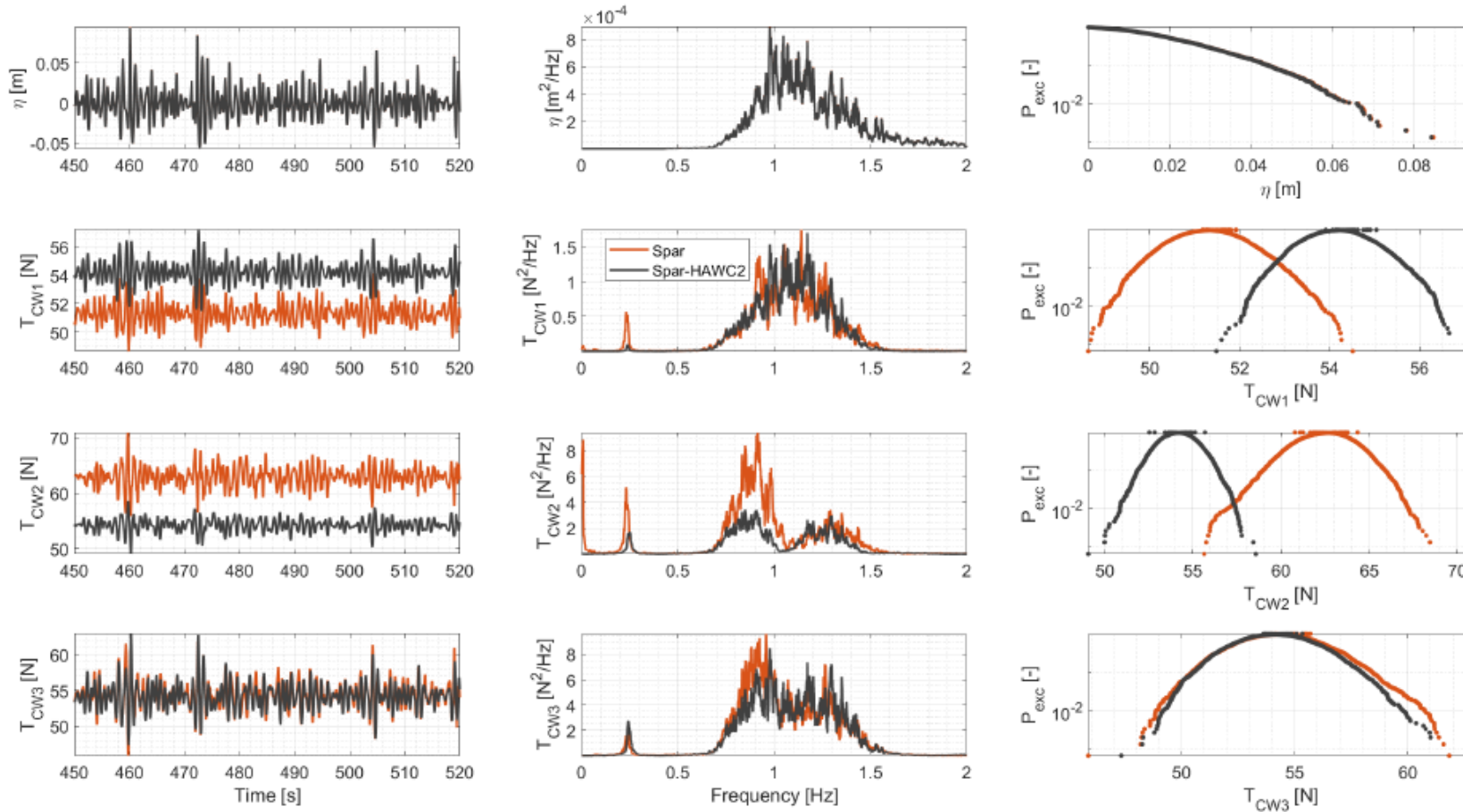
Wamit, Newman, relative Morison drag, calibrated damping



Good reproduction for inline floater motion and tower top acceleration.

Wave state of rated wind speed

Wamit, Newman, relative Morison drag, calibrated damping



Keel line 1 tension OK except offset in pre-tension

Some difference for line 2 (but good in storm sea state, not shown here)

Line 3 perfect

Ongoing: Remodelling of TetraSub concept tests 2021



A variant of the TetraSub concept was tested with the scaled DTU 10 MW turbine in the spring of 2021

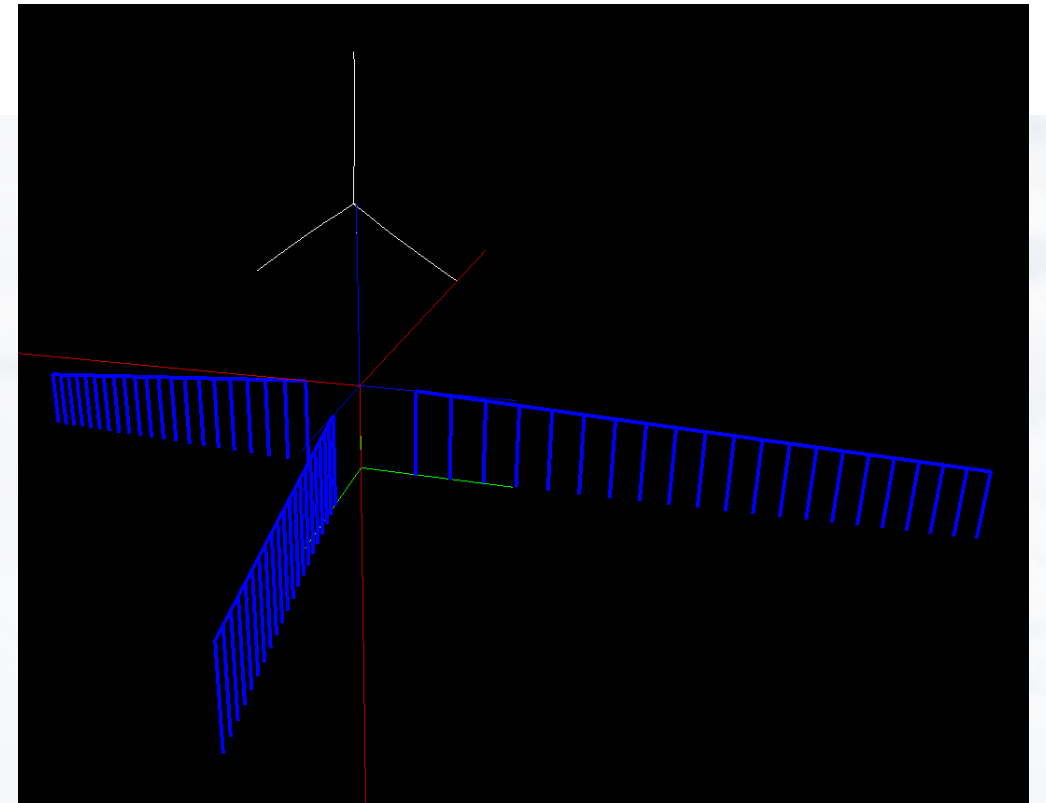


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Re-modelling in HAWC2

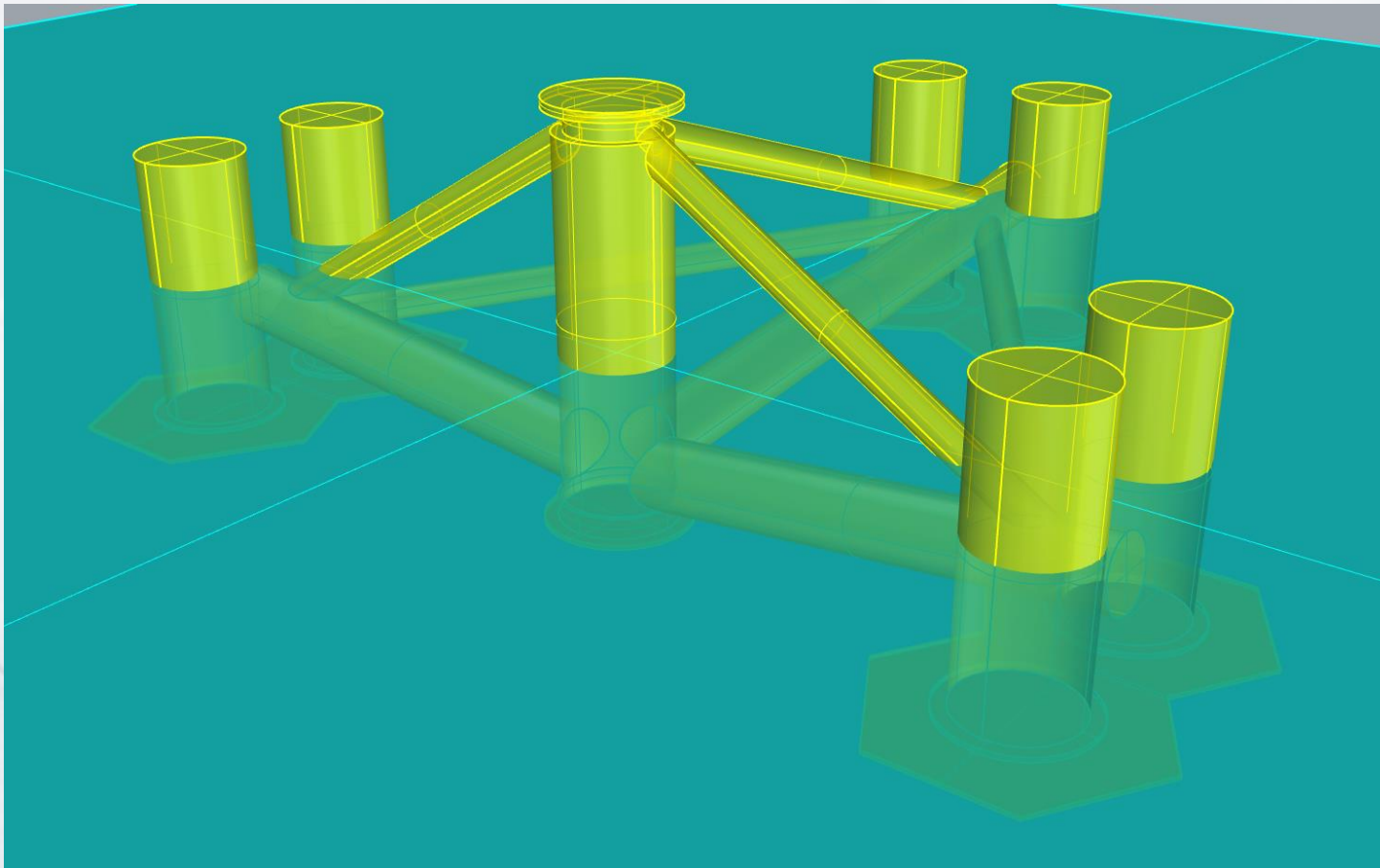
- Model in 1:60 scale
- Turbine model from previous campaigns
- Floater modelled in WAMIT to obtain
 - Added mass and damping
 - Hydrostatic properties
 - Wave load transfer functions
- No viscous effects yet



Antonio Pegalajar-Jurado

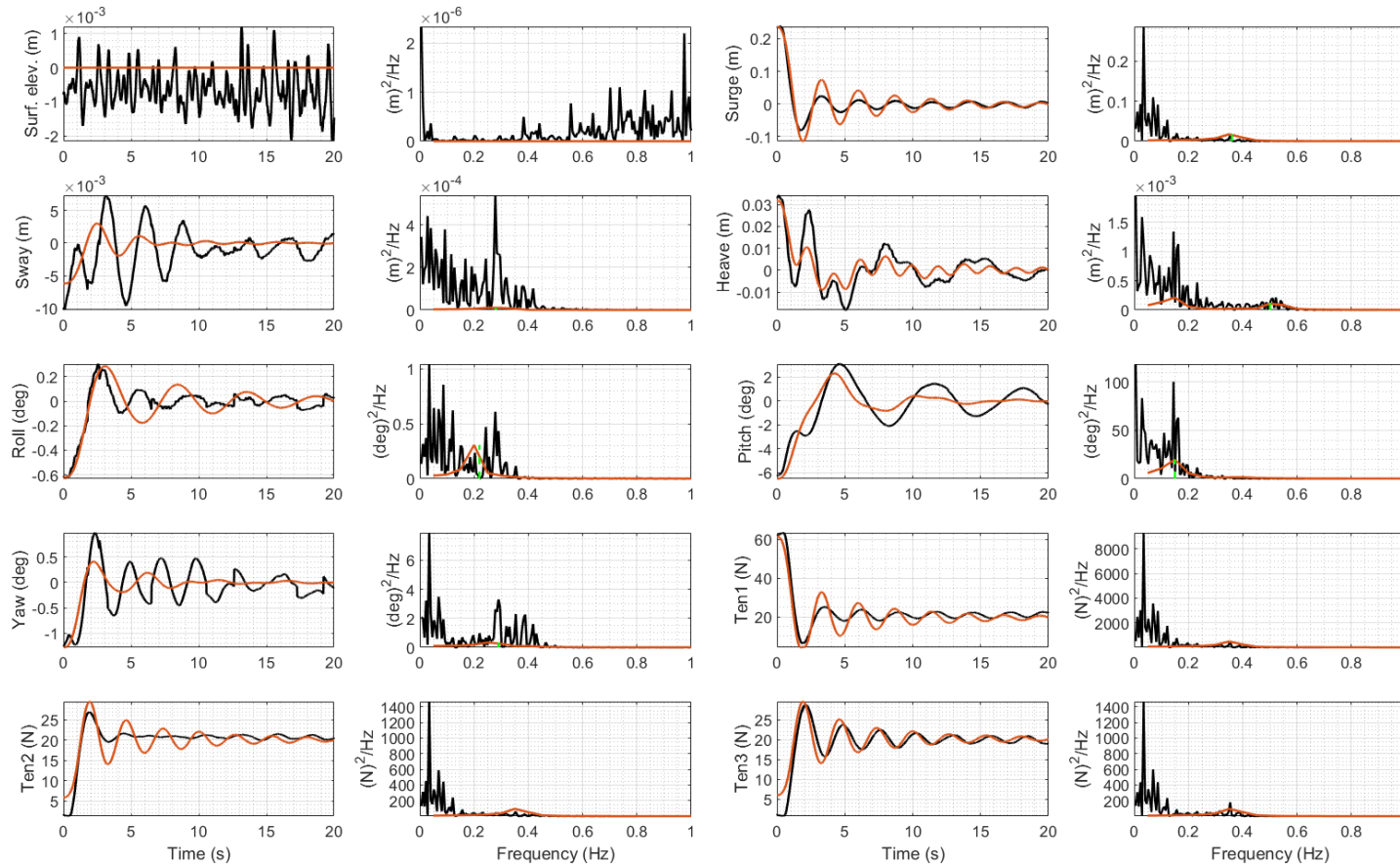
WAMIT analysis of tested TetraSub floater

- CAD geometry v3: right draft

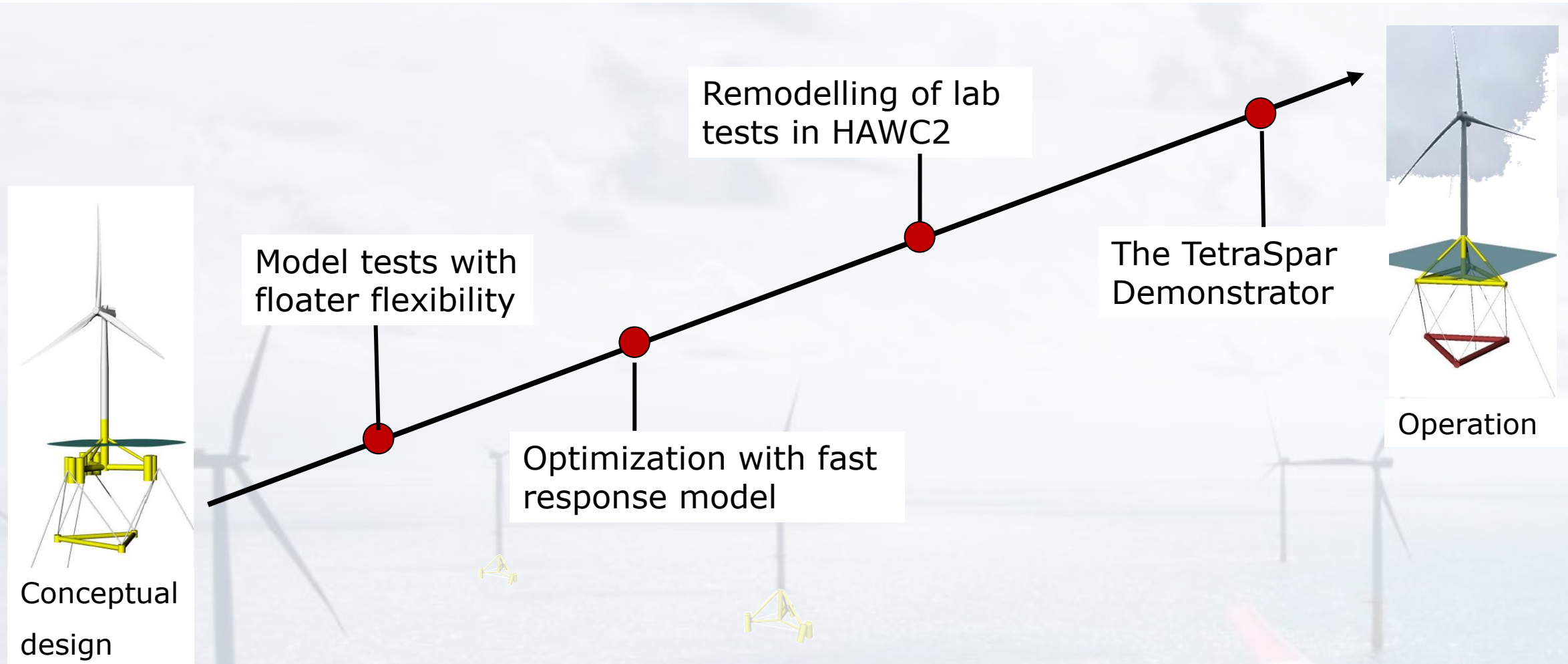


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HAWC2 model – surge decay

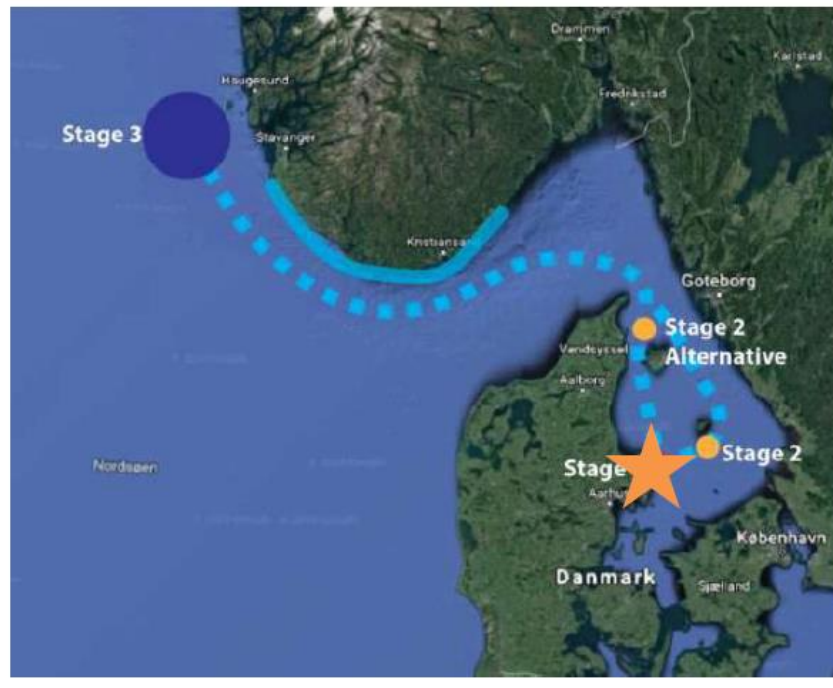


Validated and efficient design tools



TetraSpar Demonstrator installed 2021

Prototype with 3.6 MW SGRE turbine
Installed at the MetCentre, Karmøy, July 2021



Stiesdal Offshore
Technologies

Assembly of the TetraSpar Demonstrator 2021



Photo: TetraSpar Demo / Stiesdal

Rotor lift May 2021

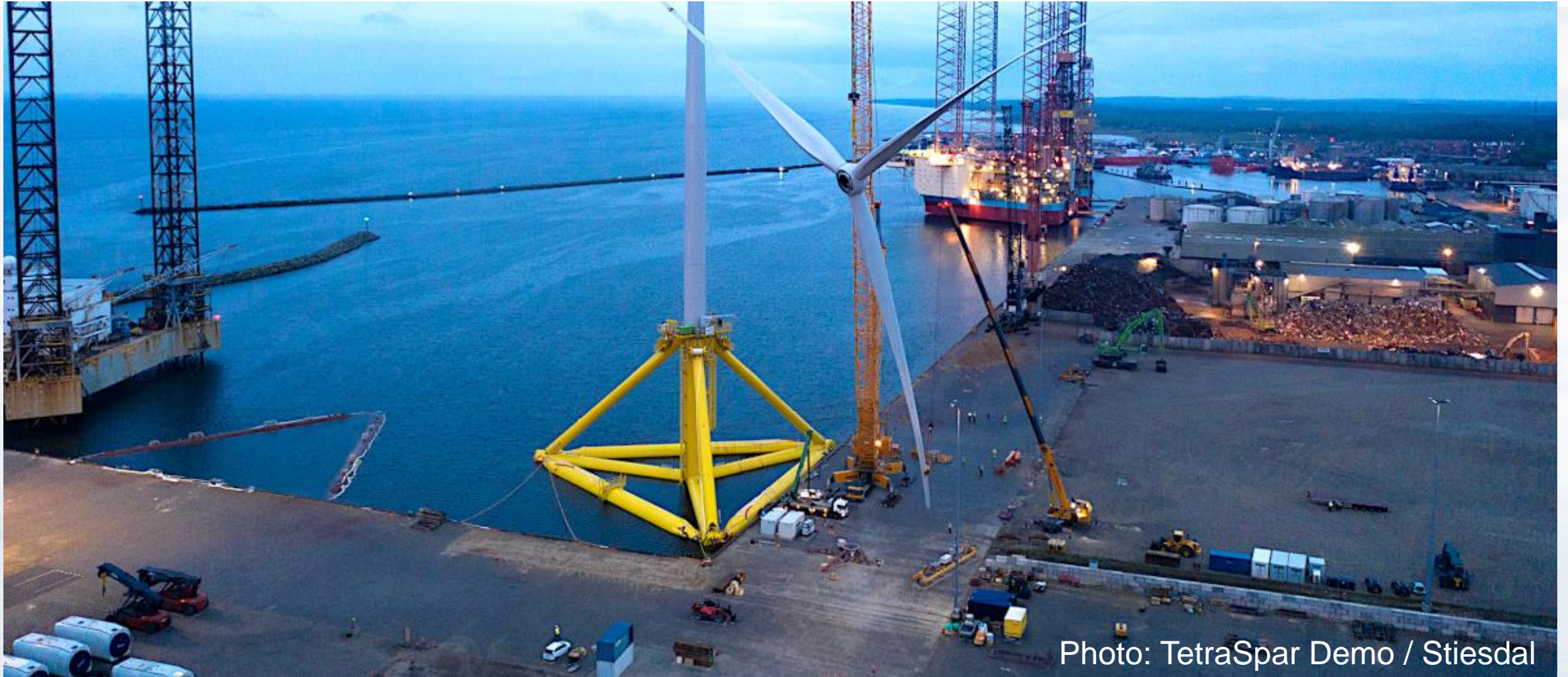


Photo: TetraSpar Demo / Stiesdal

Towing of the TetraSpar Demonstrator July 2021

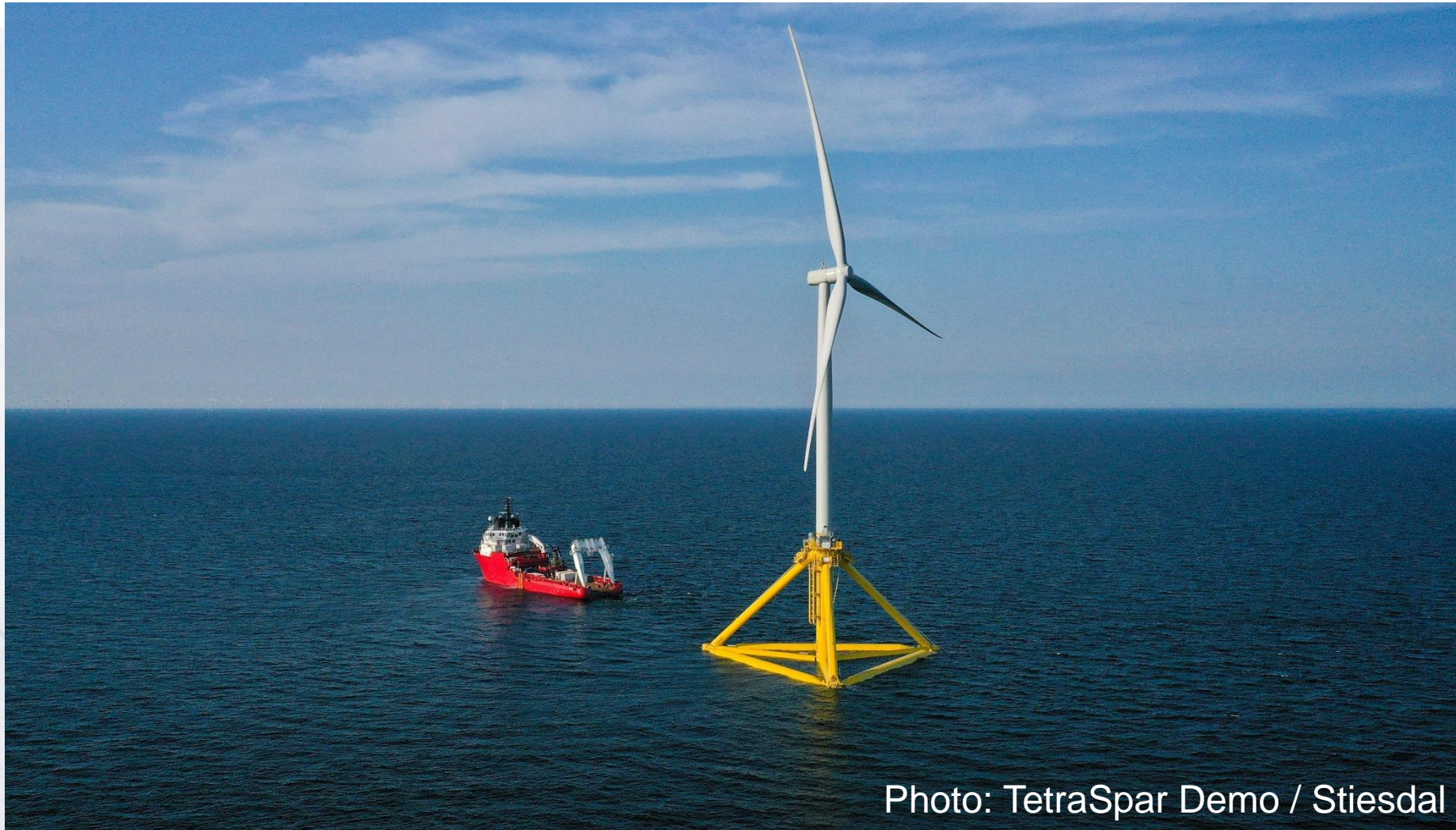


Photo: TetraSpar Demo / Stiesdal

Installation TetraSpar Demonstrator July 2021



Photo: TetraSpar Demo / Stiesdal

Installed! TetraSpar Demonstrator July 2021



Photo: TetraSpar Demo / Stiesdal

Next steps in FloatStep

- Analysis of full scale data
- Analysis of model scale tests for blade pitch control
- OpenFOAM solver for floating structures

Publications of FloatStep

Madsen, Pegalajar-Jurado and Bredmose (2019) 'Performance study of the QuLAF pre-design model for a 10MW floating wind turbine', *Wind Energy Science* 4, pp 527-547.

H. Bredmose and A. Pegalajar-Jurado (2021) 'Second-order monopile wave loads at linear cost'. *Coastal Engineering*. **170** 103952

Orszaghova, J., Taylor, P. H., Wolgamot, H. A., Madsen, F. J., Pegalajar-Jurado, A. M., & Bredmose, H. (2021). Wave-and drag-driven subharmonic responses of a floating wind turbine. *Journal of Fluid Mechanics*, 929, 1402-1413.

Borg, M., Pegalajar-Jurado, A., Bredmose, H., Stiesdal, H., Madsen, F.J., Nielsen, T.R.L., Mikkelsen, R.F, Mirzaei, M. and Lomholt, A.K. (2022), 'Dynamic response analysis of the TetraSpar floater in waves: Experimental and numerical investigations'. In revision.

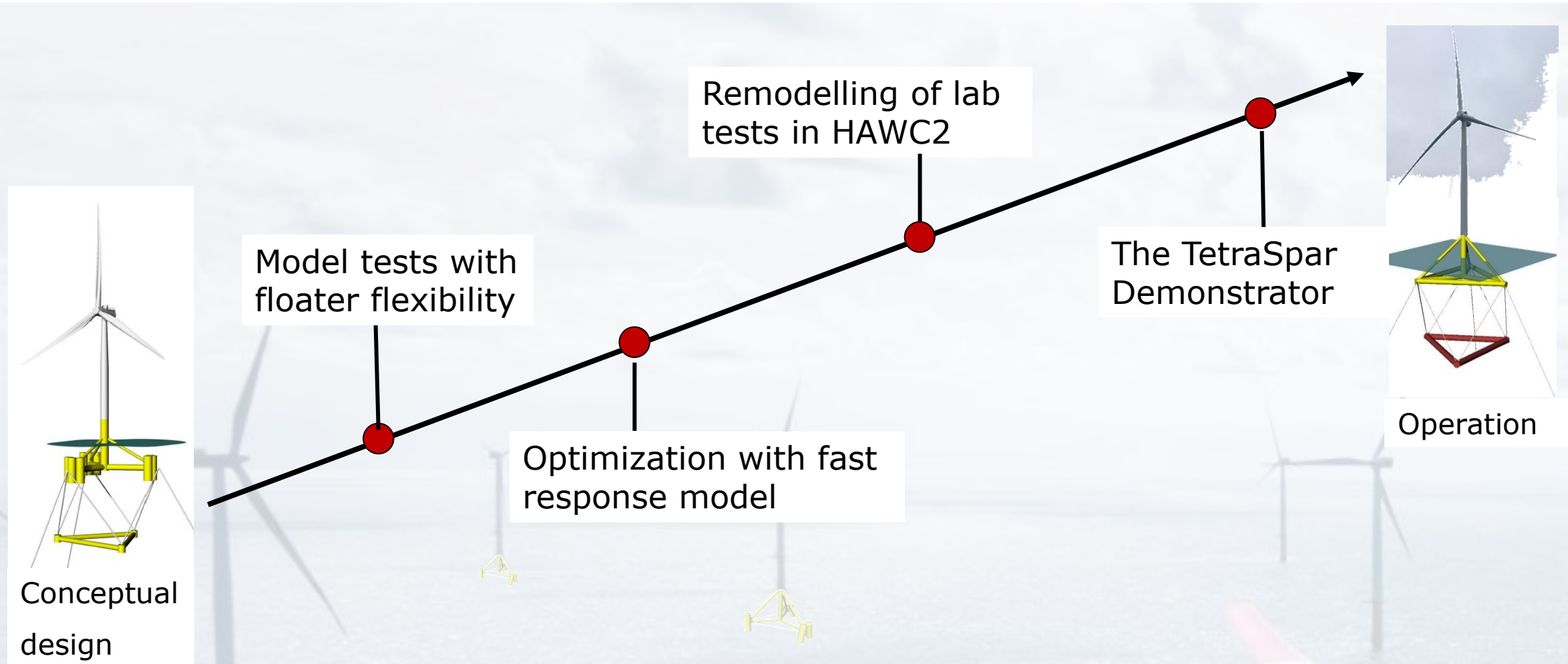
Pegalajar-Jurado, Madsen and Bredmose (2019) 'Damping identification of the TetraSpar floater in two configurations with Operational Modal Analysis'. 2nd Int Offshore Wind Technical Conference, Malta, November 2019. ASME.

Dou, S., Pegalajar-Jurado, A., Wang, S., Bredmose, H., & Stolpe, M. (2020). Optimization of floating wind turbine support structures using frequency-domain analysis and analytical gradients. *Journal of Physics: Conference Series*, 1618(4)

Pegalajar-Jurado, A. and Bredmose, H. (2020) 'Accelerated hydrodynamic analysis for spar buoys with second-order wave excitation'. 39th Int. Conf. Ocean Offshore Arctic Engng, OMAE 2020, Florida, June 2020.

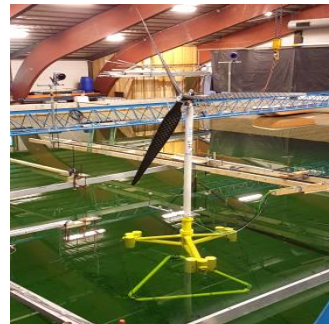
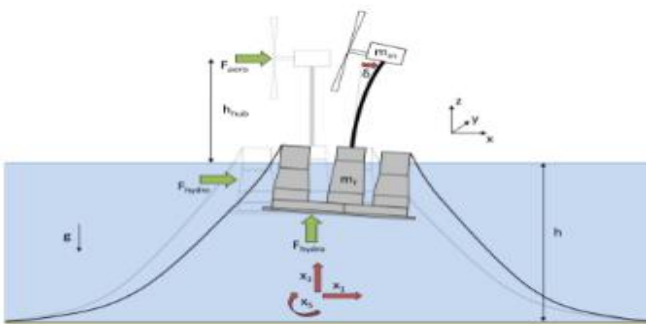
Pollini, N., Pegalajar-Jurado, A., Dou, S., Bredmose, H., & Stolpe, M. (2021). Gradient-based optimization of a 15 MW wind turbine spar floater. *Journal of Physics: Conference Series*, 2018(1), [012032]. <https://doi.org/10.1088/1742-6596/2018/1/012032>

Validated and efficient design tools





**Accurate and validated design tools
for floating wind turbine substructures.**
Recent results from the FloatStep project



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Kasper Laugesen², Bjarne Jensen³, Michael Borg⁴, Johan Rønby⁵, Jana Orszaghova⁶