

#### Installation and substructure

# Nonlinear hydroelastic response of a monopile wind turbine foundation in regular waves

<u>Vincent Leroy</u>, Erin Bachynski, Jean-Christophe Gilloteaux, Aurélien Babarit, Pierre Ferrant

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# Context

#### Hydroelasticity of bottom-fixed wind turbines foundations

> Morison, potential flow theory (FNV, ...) for cylinders, simple geometries

#### Floating wind turbines

> Most of the numerical models are rigid-flexible: rigid hull + elastic tower, blades and drivetrain, ignoring the elasticity of the platform

#### In design phases, current models assume a rigid hull to compute internal loads Hydrodynamic loads are computed with

- Linear potential flow theory possibly multi-body
- Morison equation and linear or 2nd order wave kinematics



Structure internal loads and deformations

Nonlinear hydroelastic response of monopile wind turbine foundation





Floatgen FWT ©Centrale Nantes/Above All



# **Project HeloFOW**

Hydroelasticity of large FWT platforms Financed by WEAMEC Centrale Nantes LHEEA (France) / NTNU IMT (Norway)

#### Numerical

- > How to account for elasticity in hydrodynamic calculations? (coupling)
- → Develop a coupling between non-linear potential flow solver and a FEM "beam" model

#### Experimental

> Experimental testing of flexible/segmented platform models

#### First step: implementation and verification on a monopile foundation





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# **WSCN** solver

## Weak-scatterer theory

Solver developed in Centrale Nantes since 2011

#### **Assumptions**

- > Potential flow  $\rightarrow \Delta \phi = 0$  in the fluid
- > Weakly non linear

Weak-Scatterer hypotheses: 
$$\begin{cases} \phi = \phi^{I} + \phi^{P} \\ \eta = \eta^{I} + \eta^{P} \end{cases}, \text{ with } \begin{cases} \phi^{P} = o(\phi^{I}) \\ \eta^{P} = o(\eta^{I}) \end{cases} \text{ and } \begin{cases} \phi^{P} \xrightarrow{r \to \infty} 0 \\ \eta^{P} \xrightarrow{r \to \infty} 0 \end{cases}$$

- > Free surface boundary conditions are written at incident wave elevation  $\eta^{I}(x, y, t)$
- > Loads

$$F_{hydro} = -\iint p \, \boldsymbol{n} dS \qquad \text{where} \qquad p = -\rho \left( \frac{\partial \phi^{\mathrm{I}}}{\partial t} + \frac{\partial \phi^{\mathrm{P}}}{\partial t} + \frac{1}{2} \nabla \phi^{\mathrm{I}} \cdot \nabla \phi^{\mathrm{I}} + \nabla \phi^{\mathrm{P}} \cdot \nabla \phi^{\mathrm{I}} + gz \right)$$

> Advantages: allows large motions and fully non-linear wave fields

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Δ

n



# WSCN solver In a few lines, for a fixed or floating body

> 1<sup>st</sup> Boundary Value Problem : 2<sup>nd</sup> Green identity for velocity potential and its gradient  $\phi^P(M)$  and  $\frac{\partial \phi^P}{\partial n}(M)$ 



Fluid-structure coupling: node acceleration

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# **Structural solver: FEM analysis**

Python FEM solver for beams: "beampy"

- > Based on Euler-Bernoulli theory
- > Verified with comparison to other models
- > Dynamics solved with modal superposition



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# **Fluid-structure coupling**

- > Hydrodynamic force:  $\mathbf{F}^{WSC} = -\iint p\mathbf{n}dS = \mathbf{F}_0^{WSC} + \iint \rho \frac{\partial \phi^P}{\partial t} \mathbf{n}dS = \mathbf{F}_0^{WSC} + \mathbf{L}\dot{\phi}$ 
  - L represents the projection of the hydrodynamic mesh on the structure mesh  $(N_s \times N_h)$
- > Equation of motion:

BVP2:

Boundary condition (body):

$$\begin{cases} \mathbf{M}\ddot{\boldsymbol{u}} - \mathbf{L}\dot{\boldsymbol{\phi}} = -\mathbf{C}\dot{\boldsymbol{u}} - \mathbf{K}\boldsymbol{u} + \mathbf{F}_{0}^{\mathrm{WSC}} + \mathbf{F}^{ext} \\ \mathbf{G}\dot{\boldsymbol{\phi}} = \mathbf{H}\dot{\boldsymbol{\phi}}_{n} \\ \dot{\boldsymbol{\phi}}_{n} - \mathbf{D}\ddot{\boldsymbol{u}} = -\dot{\boldsymbol{\phi}}_{n}^{I} + \boldsymbol{B} + \boldsymbol{Q} \end{cases}$$

Beam element i

Solved at the same time in a RK4 integration scheme.

> With modal superposition: 
$$\psi^T \mathbf{M} \psi \ddot{y} - \psi^T \mathbf{L} \dot{\phi} = -\psi^T \mathbf{C} \psi \dot{y} - \psi^T \mathbf{K} \psi y + \psi^T \left( \mathbf{F_0}^{WSC} + \mathbf{F}^{ext} \right)$$

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# Verification on a bottom-fixed wind turbine

### Monopile foundation

- > Geometry: uniform beam, embedded at the mudline
  - Length  $L_0 = 100 m$
  - Diameter D = 6 m
  - Thickness  $\epsilon = 7.5 \ cm$
  - Water depth d = 30 m
  - 50 beam elements, 2100 nodes in hydrodynamic mesh



#### > Aims:

- Verify the accuracy of the coupling in linear waves
- Observe non-linear and coupling effects in steep waves

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# Verification on a bottom-fixed wind turbine

#### Reference and load cases

- > Reference models
  - 1. Sima (SINTEF): Morison equation + Stokes 2nd order wave + direct FEM No viscous forces ( $C_d = 0$ ),  $C_m$  chosen from MacCamy-Fuchs
  - 2. "Semi-analytic": analytic modes + Morison with Airy waves
- > Set of 10 regular waves (Airy, Rienecker-Fenton)
  - Waves periods from 3 to 8s, amplitudes from 0.1 to 6 m, with 1.3 to 39% steepness (*kA*)
- > Compare
  - Hydrodynamic forces
  - Mudline bending moment
  - Tower mid-height and top displacement



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## Regular waves (1)

- > Rienecker-Fenton (WSCN) / Stokes 2<sup>nd</sup> order (Sima)
- > Mudline bending moment



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LHEEA CENTRALE

CNTS

BREAKING CRITERION (SOLITARY WAVE)

CNOIDA

WATER WAVE

0.01

0.001 H gT<sup>2</sup>

0.000

DEEP-WATER BREAKING CRITERION

INTERMEDIATE DEPTH WAVES

H = 0.142

LINEAR WAVE TH

DEEP WAT

## Regular waves (2)

- > Rienecker-Fenton (WSCN) / Stokes 2<sup>nd</sup> order (Sima)
- > Mudline bending moment



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DEEP-WATER BREAKING CRITERION

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## Regular waves (1)

- > Rienecker-Fenton (WSCN) / Stokes 2<sup>nd</sup> order (Sima)
- > Mudline bending moment harmonics



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LHEEA CENTRALE

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## Regular waves (2)

- > Rienecker-Fenton (WSCN) / Stokes 2<sup>nd</sup> order (Sima)
- > Mudline bending moment harmonics

> DLCs: (T = 3 s, A = 0.353 m) and

$$(T = 5 s, A = 0.981 m)$$
 and



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CNIS

BREAKING CRITERION (SOLITARY WAVE)

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WATER WAVE

0.01

0.001 H gT<sup>2</sup>

0.000

0.000

DEEP-WATER BREAKING CRITERION

INTERMEDIATE DEPTH WAVES

0.01

(Le Méhauté, 1976)

H = 0.142

LINEAR WAVE TH

(T = 8 s, A = 2.511 m)

d 0.1 aT<sup>2</sup> DEEP WAT



# **Conclusions, future works**

> Implementation of a non-linear hydro-elastic coupling between WSCN and FEM

> Comparison with Morison + Stokes 2<sup>nd</sup> order waves, on the case of a monopile

- Good agreement on 1<sup>st</sup> order and 2<sup>nd</sup> order harmonics
- Differences in steep waves, particularly on high order harmonics

> Comparison with experimental data on a flexible monopile

- > Simulation of Floating Wind Turbines
- > Experimental studies at Centrale Nantes (next year)



## References

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# Thank you for your attention



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