

EERA DeepWind'19

Trondheim - Norway



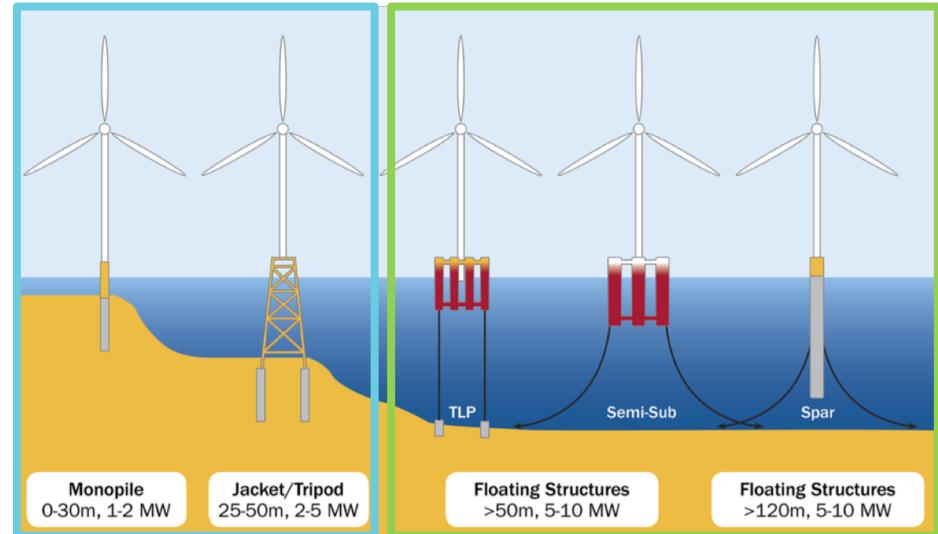
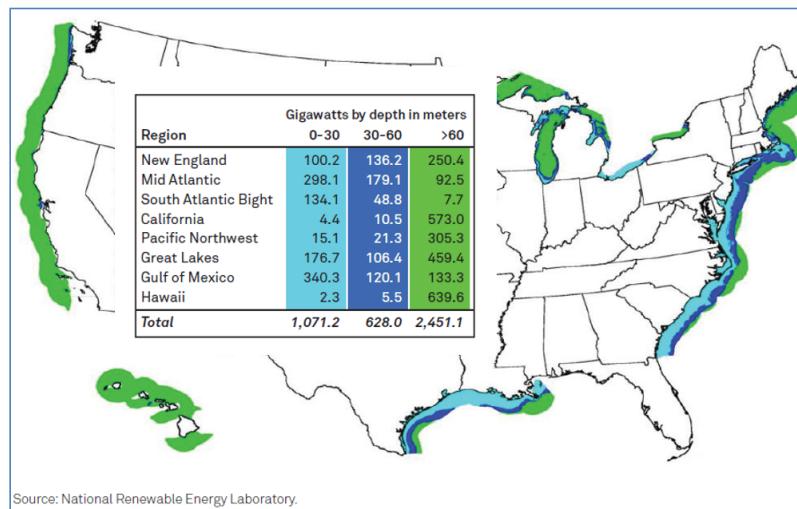
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VARIABLE-SPEED VARIABLE-PITCH CONTROL FOR A
WIND TURBINE SCALE MODEL

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FLOATING OFFSHORE



- Offshore wind energy LCOE is still high
- Floating offshore wind energy is a potential game changer for LCOE reduction
 - Greater energy production
 - Increased range of possible installation sites
 - Lower installation costs
- Deep seas represent a significant fraction of exploitable wind energy in Europe and worldwide

FOWTs WIND TUNNEL TESTING

- Experimental data required to calibrate/validate numerical simulation tools
- Scale model testing:
 - Lower costs than full-scale experiments
 - Control of environmental conditions
 - Lower uncertainties
- Hybrid/HIL testing
 - Rotor loads (including control) reproduced by a wind turbine scale model
 - Hydrodynamic loads and platform motion from numerical computations
 - 6-DOFs robot moves the wind turbine model in real-time



CONTROL SCALING

SCALE MODEL CONTROL

- Required to improve experiment fidelity
- Reproduction of rotor dynamics and control induced loads
- Direct investigation of FOWT control problem



- Non-ideal model scaling
- Low Reynolds flow
- Not possible to achieve target response with a scaled controller



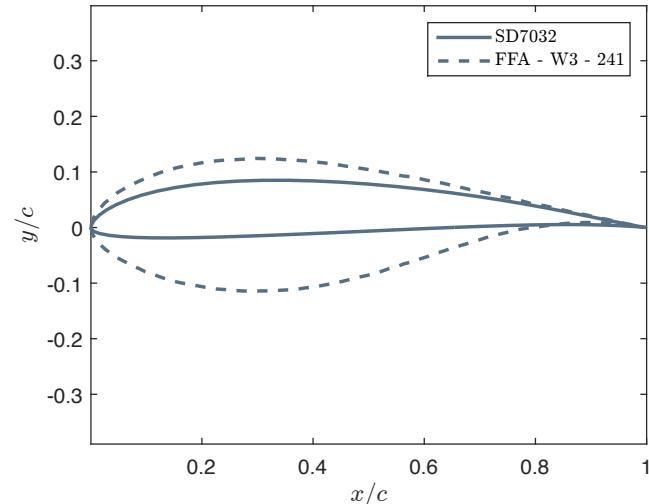
WIND TURBINE SCALE MODEL

Scale	Expression	Value
Length	λ_L	75
Velocity	λ_v	3

ROTOR

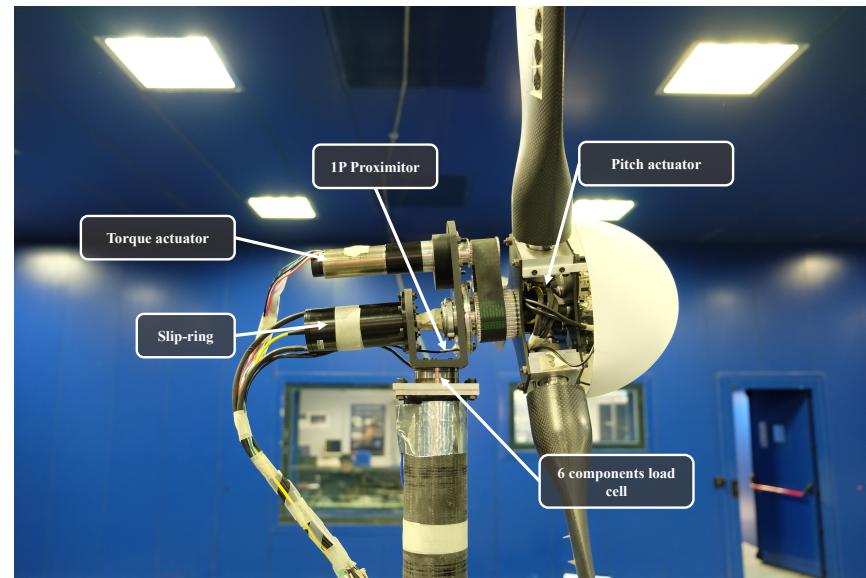
Performance scaling: low-Re blades

- Match thrust coefficient
- Match scaled weight
- Match first flapwise frequency



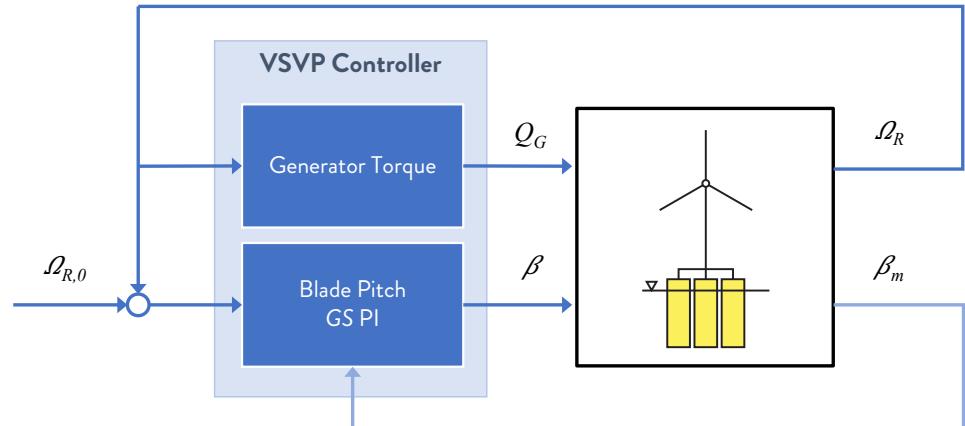
MECHATRONIC CONFIGURATION

- Similar to the full-scale turbine with torque and pitch actuators
- Onboard sensors acquired in real-time
- Embedded Control and Monitoring system



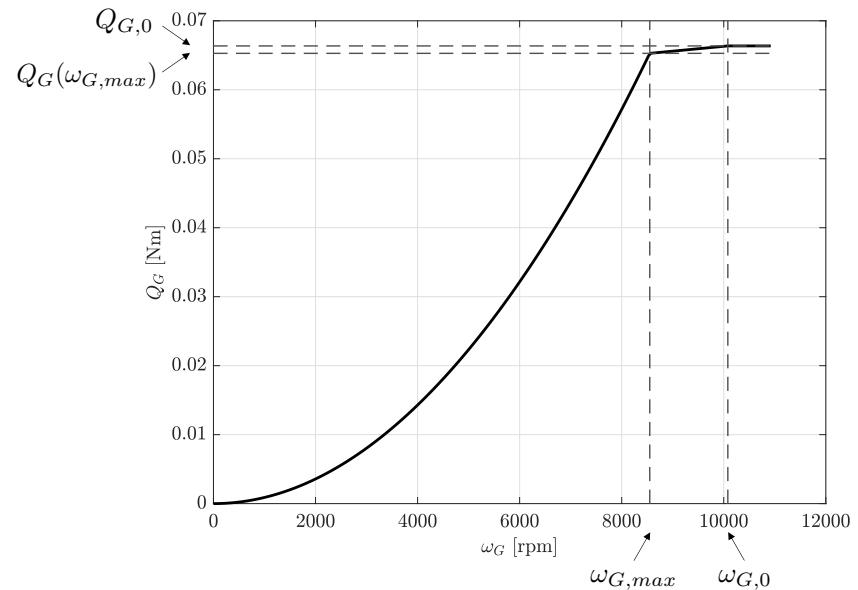
PARTIAL LOAD

- Constant pitch angle $\beta = 5^\circ$
 - Variable generator torque
- $$Q_G = K_G \omega_G^2$$
- K_G chosen to maximize power coefficient



TRANSITIONS

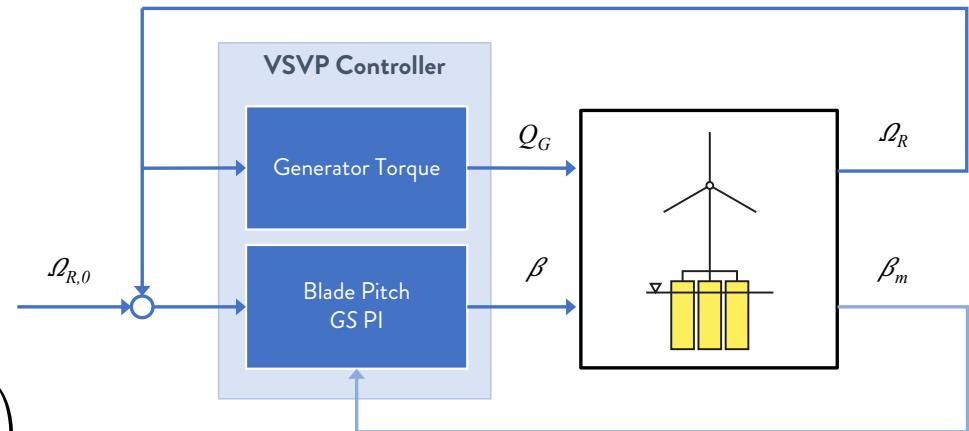
- No region 1.5
- Linear transition to reach rated torque (no-PI torque controller)



FULL LOAD

- Constant torque
- Variable collective pitch angle
- Generator speed and generator power feedback

$$\beta = \left(k_P^\omega e_\omega + k_I^\omega \int e_\omega dt \right) + \left(k_P^P e_P + k_I^P \int e_P dt \right)$$



GAIN SCHEDULING

- Quadratic aerodynamic gains scheduling
- Additional non-linear gain scheduling for large speed excursions

$$\eta_A = \frac{1}{1 + \frac{\beta}{KK_1} + \frac{\beta^2}{KK_2}}$$

$$\eta_{NL} = 1 + \frac{e_\omega^2}{(\omega_2 - \omega_0)^2}$$

DRIVETRAIN NON-IDEALITIES

Largely due to commercially available components and mechatronic design

- Not possible to have scaled generator/transmission
- Technological limits for blades realization

EFFECTS

- WT controller works on HSS feedback
- Drivetrain inertia directly affects rotor dynamics and pitch controller response

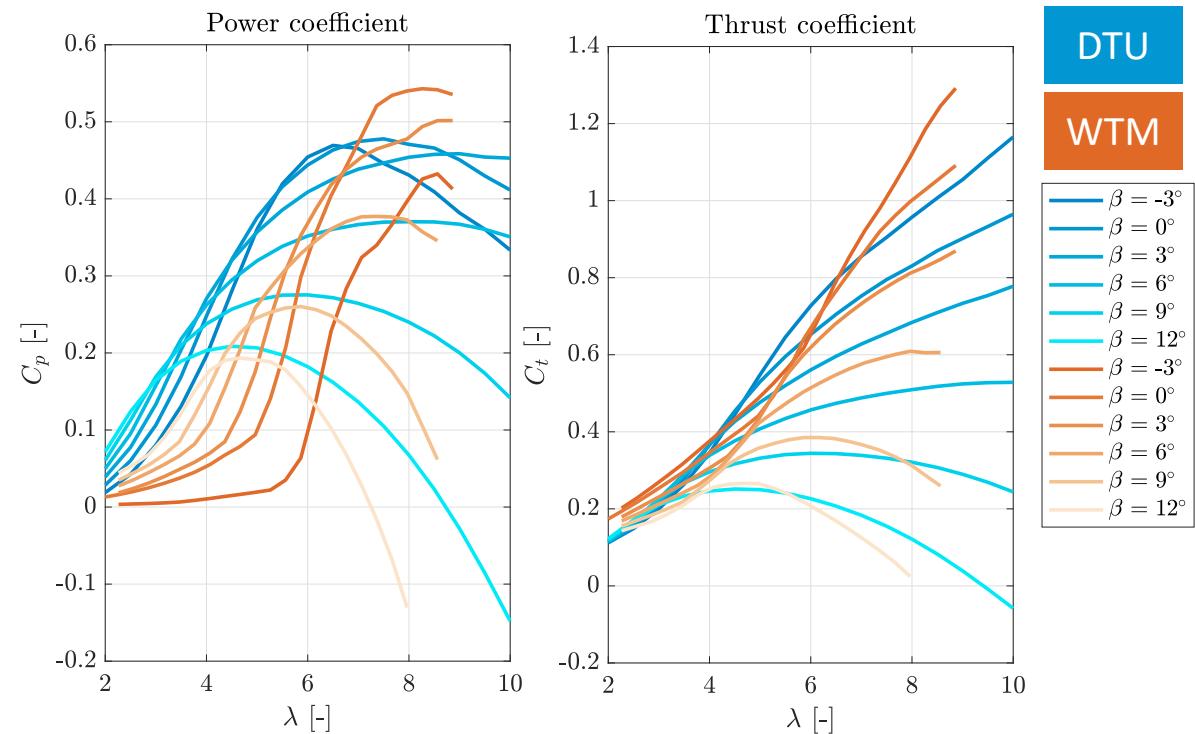
DRIVETRAIN PROPERTIES		
	DTU	WTM
Transmission ratio	50	42
LSS inertia	0.066	0.279
HSS inertia	6.323e-7	6.438e-6
Mechanical efficiency	1	0.735
Electrical efficiency	0.94	0.894

POWER COEFFICIENT

- Lower than target for small β and low values of λ
- Max C_p of 0.54 at $\beta = 0^\circ$ and $\lambda = 8.26$
- Influence on the WT start-up
- Above-rated: lower β to keep power at rated

THRUST COEFFICIENT

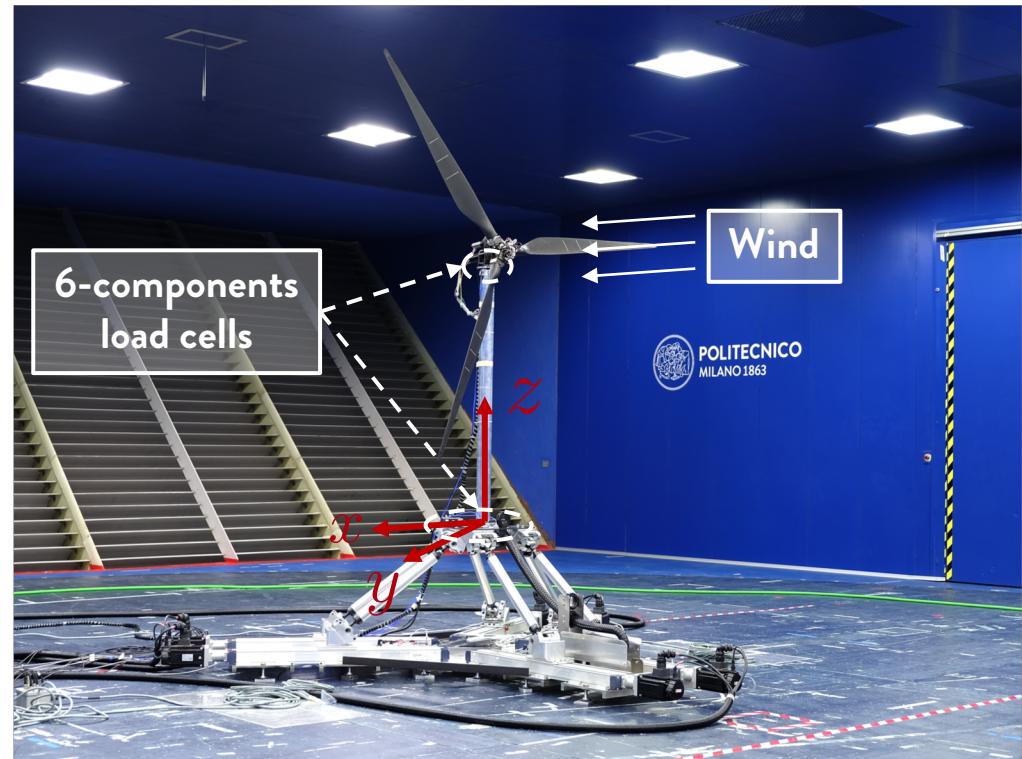
- Closer to target
- Some differences for small β and low values of λ



WIND TUNNEL TESTS

SCALE MODEL TESTING

- Laminar wind conditions
- Load measurements from two load cells
- **Steady-state tests**
 - Full-scale wind speed from 9 to 25 m/s
 - Average loads and control inputs at regime
- **Dynamic tests**
 - Sinusoidal surge motion at different frequencies and amplitudes
 - Below and above rated mean wind speeds



WIND TUNNEL TESTS

CONTROLLER SETTINGS

- Based on the public definition of the LIFES50+ OO-Star Wind Floater Semi 10MW
1. Original parameters were scaled
 2. Parameters referred to HSS were corrected for different efficiency/transmission ratio
 3. Increased below-rated pitch angle
 4. Modified generator torque constant (max C_p for $\beta = 5^\circ$)

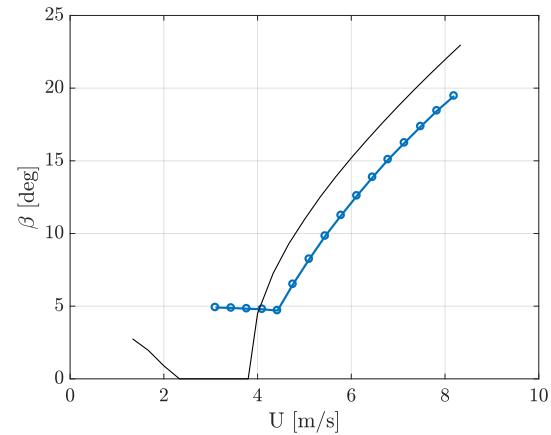
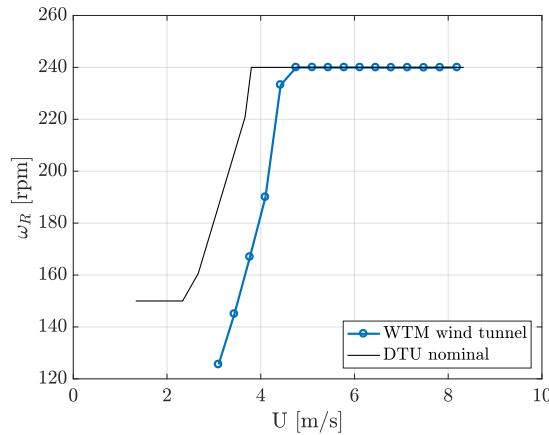
Parameter	Symbol	Unit	Value
Rated generator speed	$\omega_{G,0}$	rpm	10080
Region 2 transition speed	$\omega_{G,max}$	rpm	8550
Rated generator power	$P_{G,0}$	W	70.044
Generator torque constant	K_G	Nm/(rad/s) ²	$8.143 \cdot 10^{-8}$
Minimum pitch angle	β_{min}	deg	5
Proportional speed gain	k_P^ω	s	$1.831 \cdot 10^{-4}$
Integral speed gain	k_I^ω	—	$2.095 \cdot 10^{-4}$
Proportional power gain	k_P^P	rad/W	$8.265 \cdot 10^{-4}$
Integral power gain	k_I^P	rad/(Ws)	$2.070 \cdot 10^{-2}$
Linear gain scheduling factor	KK_1	deg	198.329
Quadratic gain scheduling factor	KK_2	deg ²	693.222
Speed for doubled gains	ω_2	rpm	13104



STEADY-STATE TESTS

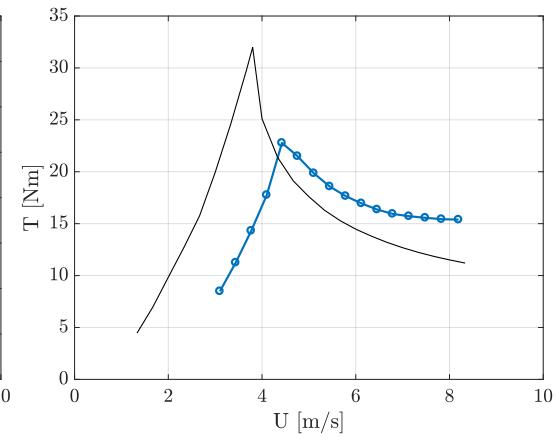
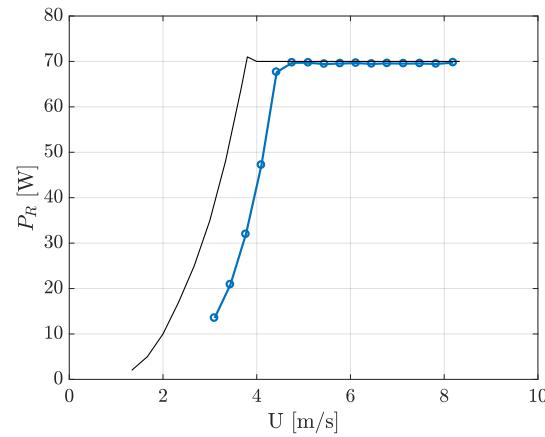
PARTIAL LOAD

- Rated reached at 14 m/s
- Steady-state angular speed lower than target
- Low λ and increased β lead to decreased power and low thrust force



FULL LOAD

- Pitch angle always lower than target
- Increased thrust force: WTM rotor designed to have target thrust at DTU 10MW nominal pitch angles

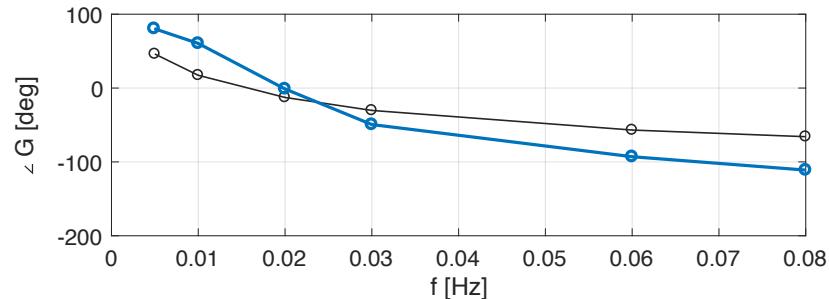
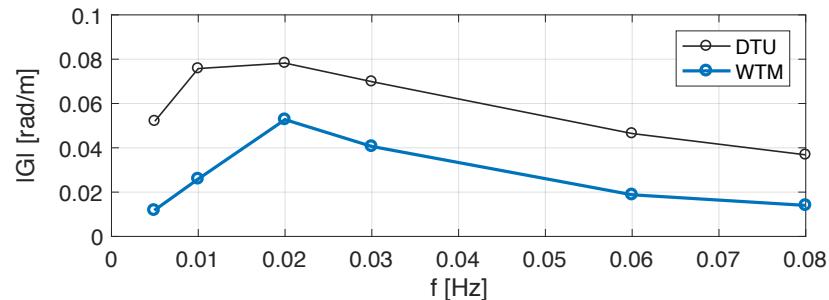
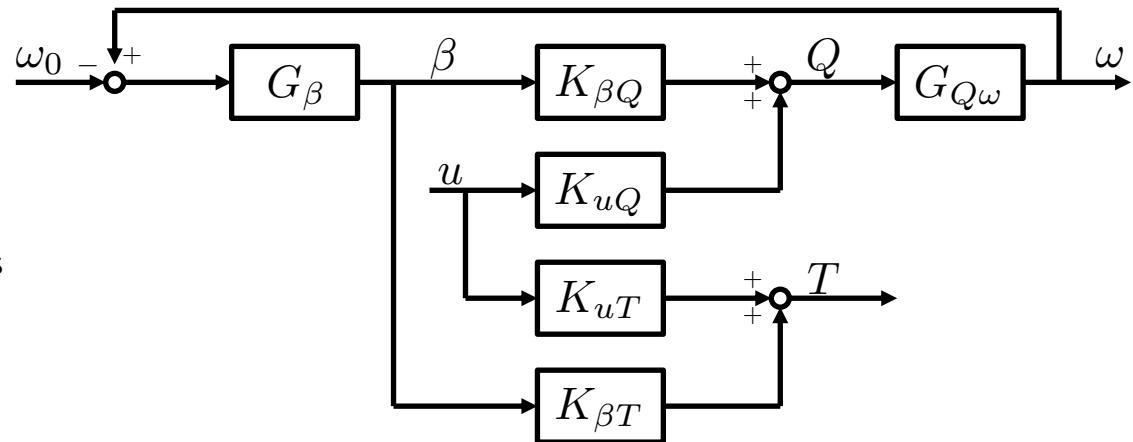


IDEAL CLOSED-LOOP

- Pitch controller disturbance rejection function
- Above-rated mean wind speed: 18 m/s

$$G(f) = \frac{\omega_R(f)}{u(f)} = \frac{G_{u\omega}}{1 - G_\beta G_{\beta\omega}}$$

- $G_{u\omega}$ depends both on the drivetrain mechanical properties and on rotor aerodynamics
- G_β is the PI-pitch controller transfer function
- $G_{\beta\omega}$ depends both on the drivetrain mechanical properties and on rotor aerodynamics



CONCLUSIONS

