



# Dynamic Responses Analysis for Initial Design of a 12 MW Floating Offshore Wind Turbine with Semi-Submersible Platform

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## Introduction

- Why do we need 12 MW Floating Offshore Wind turbine (FOWT)?
  - Able to use in Deep Water : the stable and strong wind flows.
  - Improve energy production capacity and reduce construction costs.
  - Solution for noise and insufficient space.
- The purpose with the design of a 12 MW UOU(University of Ulsan) FOWT.
  - Desing of FOWTs must consider both aerodynamics and hydrodynamics.
  - The floating platform has the lowest natural frequencies.
  - Initial dimensional design of tower to avoid buckling and resonances.
  - Solution for unstable coupling between platform motion and pitch controller
- Dynamic responses analysis for initial design of a 12 MW UOU FOWT using fully coupled analysis was performed to determine the suitability.

## Design of 12 MW Floating Offshore Wind Turbine

- The initial design of 12 MW UOU FOWT was performed based on a 5 MW NREL wind turbine for offshore model, using geometric laws of similarity.

### 12MW FOWT Design Process & Properties

Rating	5 MW	12 MW
Control	Variable Speed, Collective Pitch	Variable Speed, Collective Pitch
Drivetrain	High Speed, Multiple-Stage Gearbox	Low Speed, Direct Drive(gearless)
Rotor, Hub Diameter	126 m, 3 m	195.2 m, 4.64 m
Hub Height	90 m	124.6 m
Cut-in, Rated, Cut-Out Wind Speed	3 m/s, 11.4 m/s, 25 m/s	3 m/s, 11.2 m/s, 25 m/s
Cut-in, Rated Rotor Speed	6.9 rpm, 12.1 rpm	3.03 rpm, 8.25 rpm
Overhang, Shaft Tilt, Pre-cone	5 m, 5°, 2.5°	7.78 m, 5°, 3°
Rotor Mass	110,000 kg	297,660 kg
Nacelle Mass	240,000 kg	400,000 kg (Target)
Tower Mass (for offshore)	249,718 kg	781,964 kg

### Geometric Scale Ratio

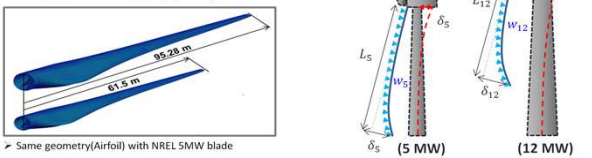
$$P = C_p \cdot \frac{1}{2} \rho A V^3$$

$$\lambda_g = \sqrt{\frac{12MW}{5MW}} = \sqrt{\frac{A_{12}}{A_5}} = \sqrt{\frac{P_{12}}{P_5}} = 1.549$$

- $P$ : Rotor power (kW)
- $C_p$ : Max. power coefficient of rotor
- $\rho$ : Air density, (1.225 kg/m<sup>3</sup>)
- $A$ : Rotor swept area (m<sup>2</sup>)
- $V$ : Wind Speed (m/s)
- $\lambda_g$ : Geometric scale ratio

### Scale-up Blade & Tower properties (Beam deflection)

- 61.5 (m) 5MW glass blade : 17.7 ton
- 95.28 (m) 12MW carbon (spar cap) blade : 42.7 ton

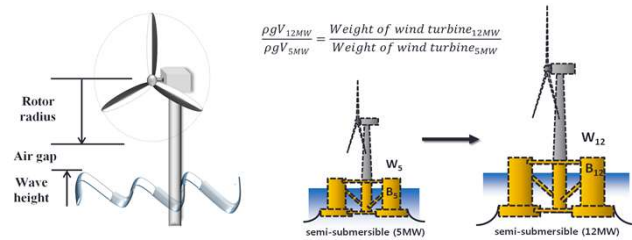


### Hub height

- Rotor radius + Extreme wave height (half) with 50-year occurrence  $\times$  S.F. of 1.8 : 97.6 + 30.0 / 2  $\times$  1.8 = 124.6 m

### Scale-up Platform properties

- Ratio of  $W_{12}$ (1480ton) to  $W_5$ (600ton)
- OC4 semi-submersible "displaced volume" 13,917m<sup>3</sup> (5MW)  $\rightarrow$  34,336m<sup>3</sup> (12MW)



## Tower Buckling Analysis

### Critical load ( $P_{cr}$ ) from Euler equation

$$P_{cr} = \frac{\pi^2 EI}{L_e^2} = \frac{\pi^2 EI}{(KI)^2}$$

$$\sigma_{cr} = \frac{P_{cr}}{A}$$

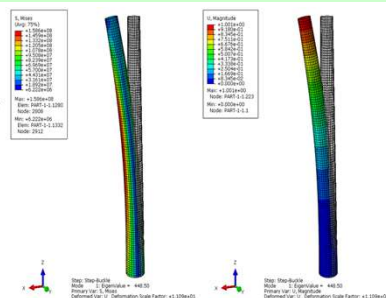
### Effective load ( $P_{eff}$ ) of tower axial weight

- $P_{eff} = \frac{33}{140} \cdot W_{tower} + W_{head}$
- Equivalent weight of tower
- $W_{head}$ : Lumped mass of Rotor & Nacelle

### Analysis Results (ABAQUS)

$\sigma_{cr}$ (N)	$P_{cr}$ (N)	$P_{eff}$ (N)	$\delta$ (m)
1.586.E+08	7.953.E+07	8.649.E+06	1.001

$P_{cr} < P_{eff}$  : the tower is stable



## Tower Resonance Analysis

- A tower design is proposed to avoid the 3P resonance problem due to the direct expansion of the 5 MW wind turbine support.

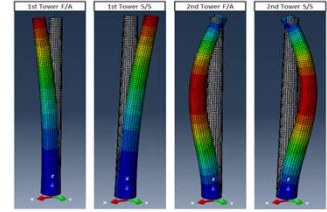
### Tower Redesign

Frequency is inverse proportional to length  $\rightarrow$  Reduced the tower length

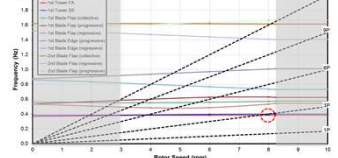
	Tower length	Tower base diameter	Tower base thickness	Tower top diameter	Tower top thickness	Tower weight
Before	110.85 m	9.634 m	0.040 m	5.756 m	0.028 m	781,964 kg
After	104.25 m	9.634 m	0.040 m	5.756 m	0.028 m	755,066 kg

### 12 MW Tower Natural Frequencies (ABAQUS)

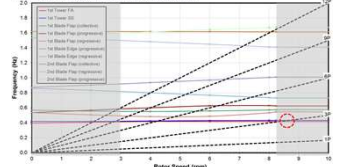
Mode	FAST (MBC3)	ABAQUS
1st Tower Fore-Aft	0.3317	0.3310
1st Tower Side-to-Side	0.3327	0.3310
2nd Tower Fore-Aft	2.7888	2.6997
2nd Tower Side-to-Side	2.8033	2.7103



### 12MW Campbell diagram



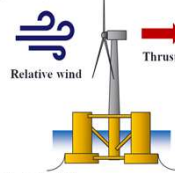
### 12MW Campbell diagram (Tower Redesign)



## Control system of 12MW FOWT

- In the case of a FOWT, the negative damping problem occurs when applying conventional pitch control system of land-based wind turbine.
- The negative damping has the reducing rated power and increasing fatigue load.
- 12 MW FOWT was modified, the PI controller to avoid negative damping problem and the response speed of the blade pitch controller to be lower than the response speed of the platform.

### Negative damping of Floating Offshore Wind Turbine (In Region-III)

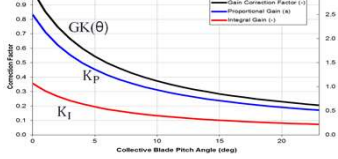


- Tilt out of wind
  - Relative wind speed decrease  $\rightarrow$  Thrust increase
- Tilt into wind
  - Relative wind speed increase  $\rightarrow$  Thrust decrease

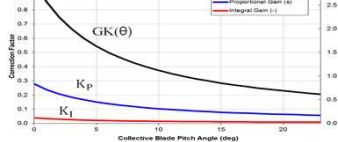
### Change the gains of PI controller, $K_p$ and $K_i$

- Adjusting the response speed of the blade pitch controller to be lower than the response speed of the platform
- Natural Frequency of Platform pitch : 0.21 rad/s
- Natural Frequency of PI Controller : 0.6 rad/s  $\rightarrow$  0.2 rad/s

### Gain-scheduling law (Land based)



### Gain-scheduling law (Offshore)

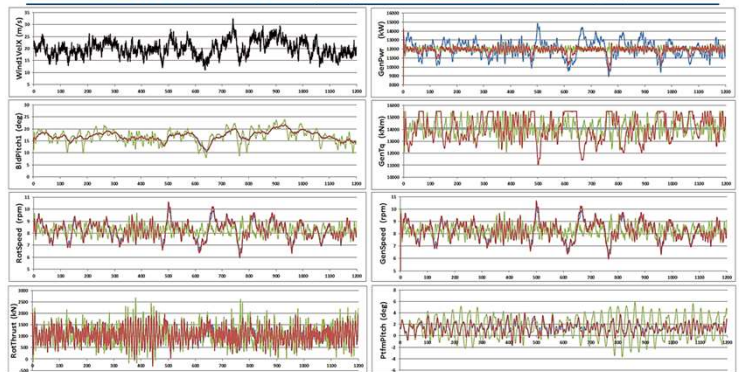


## Numerical simulation

### Simulation results

NTM: 20(m/s)  
Hs: 15.14(m)  
Tp: 9.17(s)

Land-based control: 0.6(rad/s) & Power Const.  
Offshore control: 0.2(rad/s) & Torque Const.  
Offshore control: 0.2(rad/s) & Power Const.



## Conclusion

- Initial design of a 12 MW UOU FOWT using fully coupled analysis was performed to determine the suitability.
- Dimensions of tower was approved by buckling analysis.
- 3P Resonance avoided through the redesign of the tower.
- Negative damping was solved through the response speed control of the blade-pitch controller.

### ACKNOWLEDGEMENT

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