

Numerical Simulation of a Wind Turbine with Hydraulic Transmission System

NTNU – Trondheim
Norwegian University of
Science and Technology

Zhiyu Jiang^[1,2,3], Limin Yang^[4], Zhen Gao^[1,2,3], Torgeir Moan,^[1,2,3]

- [1] Department of Marine Technology, Norwegian University of Science and Technology (NTNU), Trondheim, Norway
 [2] Centre for Autonomous Marine Operations and Systems, NTNU, Trondheim, Norway
 [3] Centre for Ships and Ocean Structures, NTNU, Trondheim, Norway
 [4] DNV GL, Høvik, Norway

Abstract

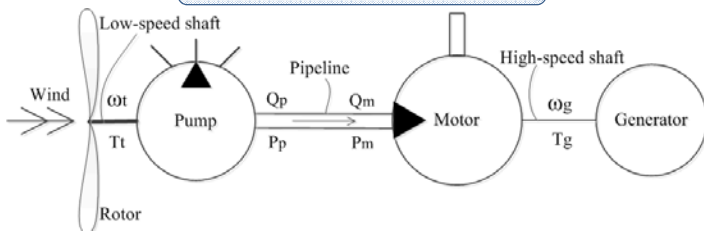
We investigate numerical modeling and analysis of wind turbines with high-pressure hydraulic transmission machinery. A dynamic model of the hydraulic system is developed and coupled with the aeroelastic code HAWC2 through external Dynamic Link Library. The hydraulic transmission system consists of a hydraulic pump, transportation pipelines, a hydraulic motor, and check valves. By use of the Runge-Kutta-Fehlberg method with step size and error control, we solved the Ordinary Differential Equations of the hydraulic system with a time step smaller than the one used in the HAWC2 main program. Under constant and turbulent wind conditions, the performances of a land-based turbine during normal operation are presented.

Objectives

During the study, the research objectives are the following:

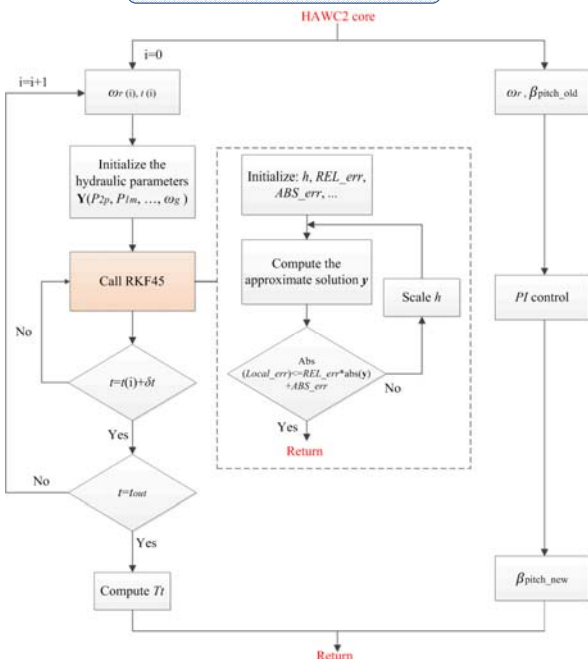
- To model the hydraulic transmission system by Ordinary Differential Equations
- To propose an approach for numerical simulation of hydraulic turbines

Mathematical Modeling

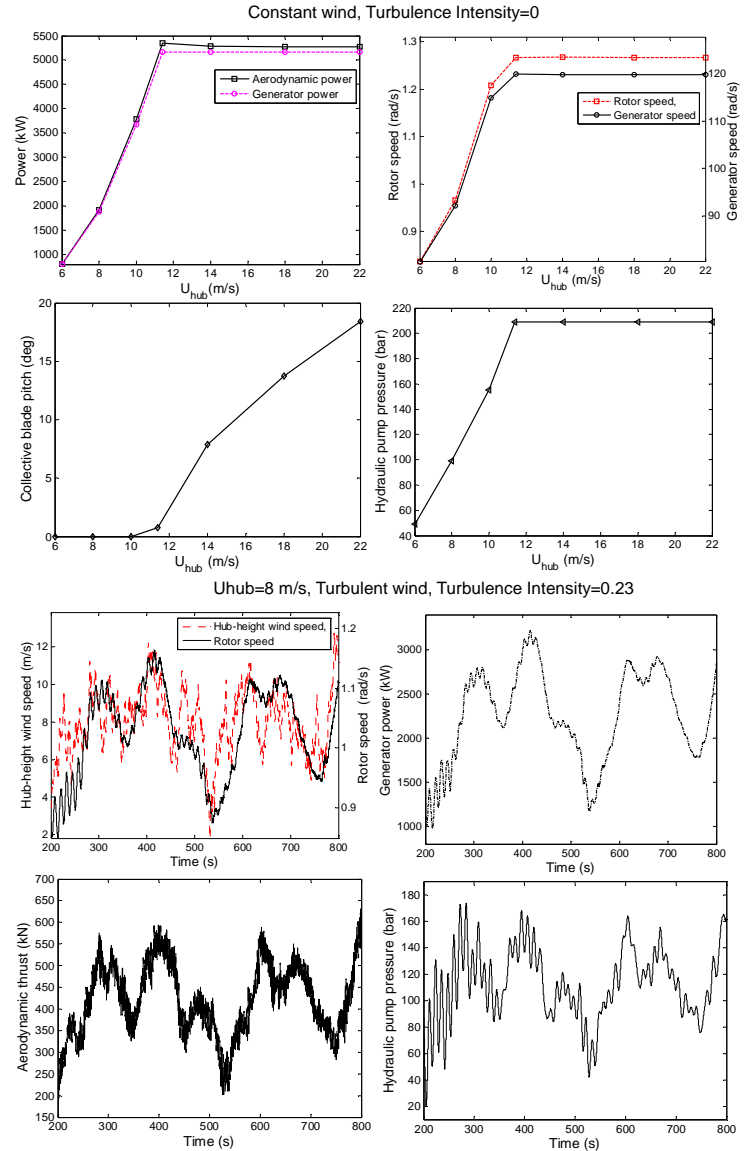


Main shaft $T_t = J_p \dot{\omega}_t + D_p(P_p - P_{low})$
 Pump $\dot{P}_p = \frac{\beta}{V_p} [\omega_t D_p - Q_{ip}(P_p - P_{low}) - Q_{ep} P_p - Q_p]$
 Motor $\dot{P}_m = \frac{\beta}{V_m} [Q_m - Q_{im}(P_m - P_{low}) - Q_{em} P_m - D_m \omega_g]$
 Transmission line $\begin{bmatrix} \dot{Q}_{p0} \\ \dot{Q}_{m0} \end{bmatrix} = \omega_c \begin{bmatrix} A_0 & Q_{p0} \\ Q_{m0} & P_p \end{bmatrix} + B_0 \begin{bmatrix} P_p \\ P_m \end{bmatrix}$ $\begin{bmatrix} \dot{Q}_{p1} \\ \dot{Q}_{m1} \end{bmatrix} = \omega_c \begin{bmatrix} A_1 & Q_{p1} \\ Q_{m1} & P_m \end{bmatrix} + B_1 \begin{bmatrix} P_p \\ P_m \end{bmatrix}$
 Generator $\dot{\omega}_g = \frac{1}{J_m + J_g} [D_m P_m - C_m \omega_g - T_g]$

Numerical Approach



Results



Conclusions

- The presented numerical approach is robust and efficient
- The hydraulic wind turbine has decent performance under constant and turbulent wind conditions

Acknowledgment

The authors gratefully acknowledge the financial support from the European Commission through the 7th Framework Programme (MARINA Platform—Marine Renewable Integrated Application Platform, Grant Agreement 241402).

References

1. Yang, L., Moan, T. (2011). Dynamic analysis of wave energy converter by incorporating the effect of hydraulic transmission lines, *Ocean Engineering*, 38(16) 1089-1102.
2. Skaare, B., Hornsten, B., Nielsens, F.G. (2012). Modeling, simulation and control of a wind turbine with a hydraulic transmission system, *Wind Energy*, 1-19. Refer to the paper for more.