

# Frictional pressure drop for two-phase flow of carbon dioxide in a tube: Comparison between models and experimental data

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## Motivation

- In CO<sub>2</sub> transport by pipeline, two-phase flow can occur in several situations:
  - Start-up
- Pressure release
- Due to intermittent supply of  $CO_2$
- In normal operation
- To calculate the flow in such situations, simulators need models for the frictional pressure drop (among other things)
- $\bullet$  We therefore compare some models for frictional pressure drop with steady-state experimental data (six experiments) for pure CO\_2 in tube of 10 mm inner diameter

#### Conclusions

- Overall, the Friedel model performed best for our data
   A large experimental database had a larger impact than CO<sub>2</sub>-specific phenomenological modelling
  - The Friedel model gave a lower standard deviation on our data then on the large data collection employed by Friedel. This may indicate that the Friedel model is as suitable for CO<sub>2</sub> as for other fluids
- The Cheng *et al.* model performed best when only the high-flowing-vapor-fraction data were considered
- The homogeneous model underestimated all our pressure-drop data
- The friction-model-input sensitivity and the sensor uncertainty are small compared to the uncertainty in the friction models themselves
- It would be interesting to include more experimental data in the analysis
  It would be interesting to compare the models for more CCS-relevant conditions, i.e. for larger pipes and including impurities.



The test rig located at the Statoil Research Centre at Rotvoll (Trondheim). Photo: Statoil.





Flow patterns predicted by the Cheng *et al.* model for each experiment. S: Stratified, SW: Stratified-wavy, I: Intermittent, A: Annular, D: Dry-out.





Sensors measuring absolute pressure (PIT and PT), differential pressure (PDT), temperature (TT) and mass flow rates (FE) are placed as shown.  $L_1 = 0.2 \text{ m}$ ,  $L_2 = 50.5 \text{ m}$ ,  $L_3 = 101 \text{ m}$  and  $L_4 = 139 \text{ m}$ .

Experimental conditions		Sensor uncertainties		
Variable	Range	Source	Uncertain	
Mass flux, $G = \dot{m}/A (\text{kg}/(\text{m}^2 \text{s}))$ Flowing vapor fraction, $x = \dot{m}_g/\dot{m}_{tot}$ (-) Saturation temperature, $T$ (°C) Reduced pressure, $p_r$ (-) Heat flux, $q''$ (W/m <sup>2</sup> )	1058-1663 0.099-0.742 3.8-17 0.52-0.72 -91-150.8	Temperature sensor, $T$ Absolute-pressure sensor, $p$ Differential-pressure sensor, $\Delta p$ Gas-flow meter, $\dot{m}_G$ Liquid-flow meter, $\dot{m}_L$	$\pm 0.5 \text{ K}$ $\pm 0.16 \text{ bar}$ $\pm 0.05 \text{ bar}$ $\pm 0.06 \%$ $\pm 0.3 \%$	

### The models

• In the Friedel model, the wall-friction force is calculated as

 $F_{\rm w} = \frac{1}{2} \frac{f_{\ell o} |G| G}{\rho_{\ell} d_{\rm h}} \Phi.$   $\Phi$  is a two-phase frictional multiplicator – an empirical correlation  $\Phi = \Phi(Fr, We, f_{\rm go}, f_{\ell o}, \rho_{\rm h}, \rho_{\rm g}, \rho_{\ell}, \mu_{\rm g}, \mu_{\ell}, x).$ 

# • The *Cheng et al.* model was developed specifially for CO<sub>2</sub> and includes phenomenological models for various flow patterns.

• The *homogeneous flow* is the simplest kind of model, where the quantities are calculated assuming no slip between the phases.

Friction-model error for all the experiments		and for high flowing vapor fraction				
Model s <sub>R</sub>	$R(\%) \ \bar{e}(\%)$		Model	$s_R$ (%) $\bar{e}$ (%)		
Friedel 9. Cheng <i>et al.</i> 57 Homogenous 29	.7 8.13 7.74 19.93 9.18 19.11		Friedel Cheng <i>et al.</i> Homogenous	10.2 8.78 1.85 1.35 20.12 12.92		

(c) Homogeneous-flow model

Comparison between experimental and calculated frictional pressure drop.







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