

Kan behovsstyrt ventilasjon erstatte lokal varme?

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Seminar 19.11.13



Kan behovsstyrt ventilasjon erstatte lokal varme?

- Principles
- Challenges
- VABAT Project

Principles

Increased envelope performances

+

- Low heating demand

Principles

Increased envelope performances

+

- Low heating demand

-

- Indoor air quality
- Overheating

Principles

Increased envelope performances

+

■ Low heating demand

-

■ Indoor air quality

■ Overheating

➔ Energy brought by the ventilation air exclusively



Heating with air: historical perspective

- First used in the 70s:
 - High heating demand
 - High temperature blowing
- Poor indoor climate:
 - Radiative assymetry
 - Stratification
 - Carbonization of particles



Heating with air: historical perspective

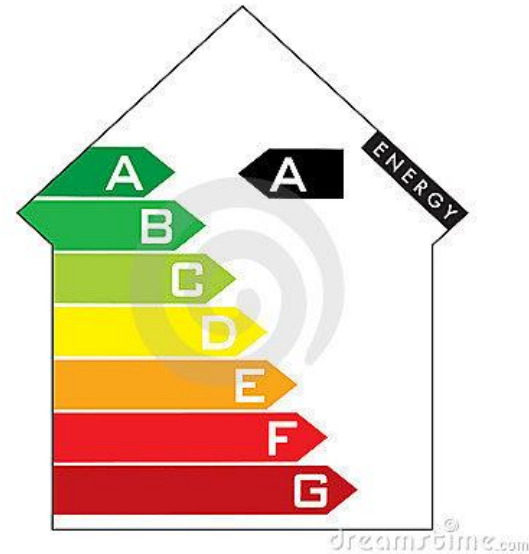
- First used in the 70s:
 - High heating demand
 - High temperature blowing
- Poor indoor climate:
 - Radiative assymetry
 - Stratification
 - Carbonization of particles



➔ Bad reputation

Heating with air: nowadays

- Lower heating demand:
 - Low ΔT , hygienic Q
 - Low thermal inertia of air
- Temperature controlled DCV
- Suited for refurbishment



- ➔ Is it possible?
- ➔ Indoor climate?

Supply temperature required *(Wigenstad et al., 2012)*

$$P = \dot{V} \cdot c_p \cdot \Delta t \text{ [kW]}$$

Example

80m² apartment

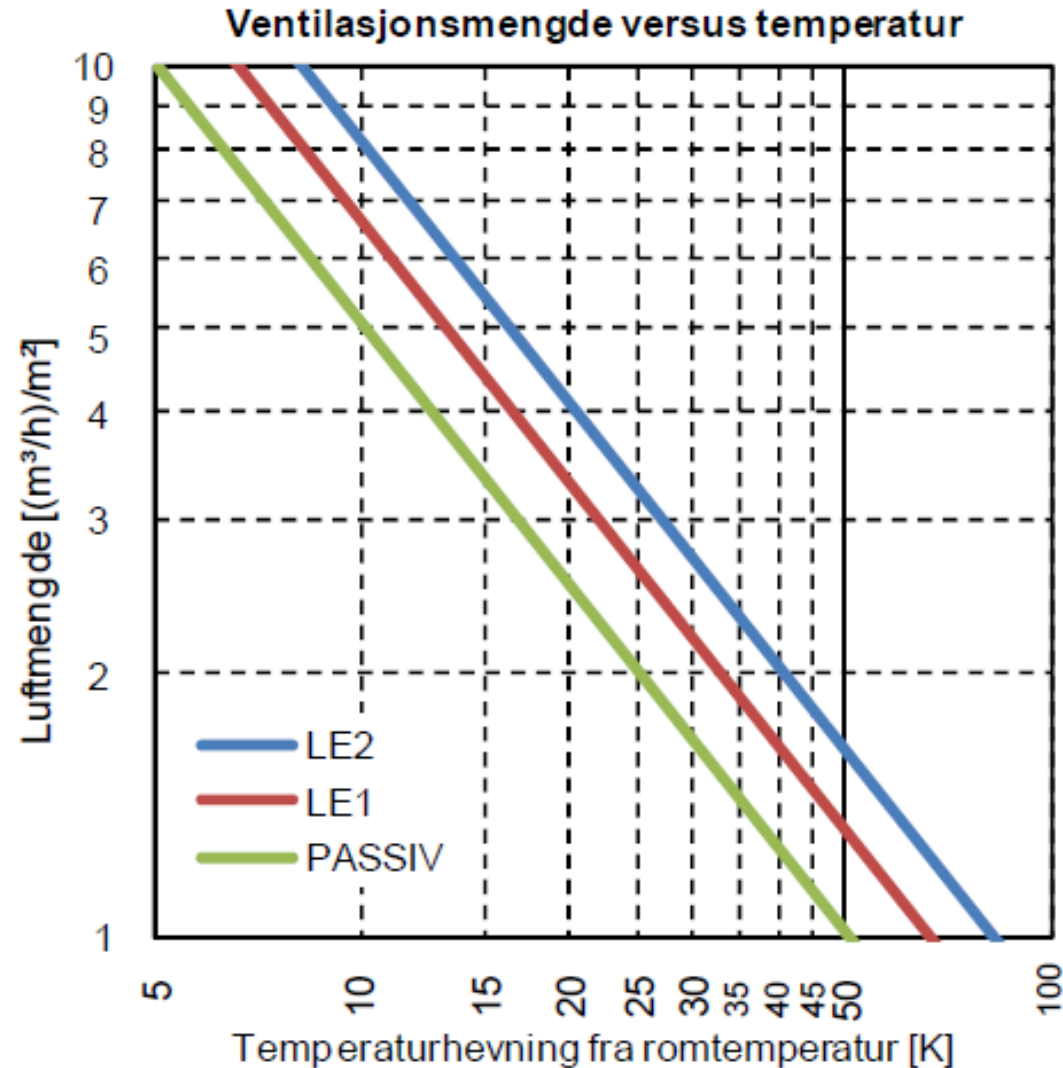
Outside design T: -20°C

Heating demand (NS 3700):

Passiv: 17 W/m²

ACR: ~1,6 m³/h/m²

Required T_{supply}: ~50°C



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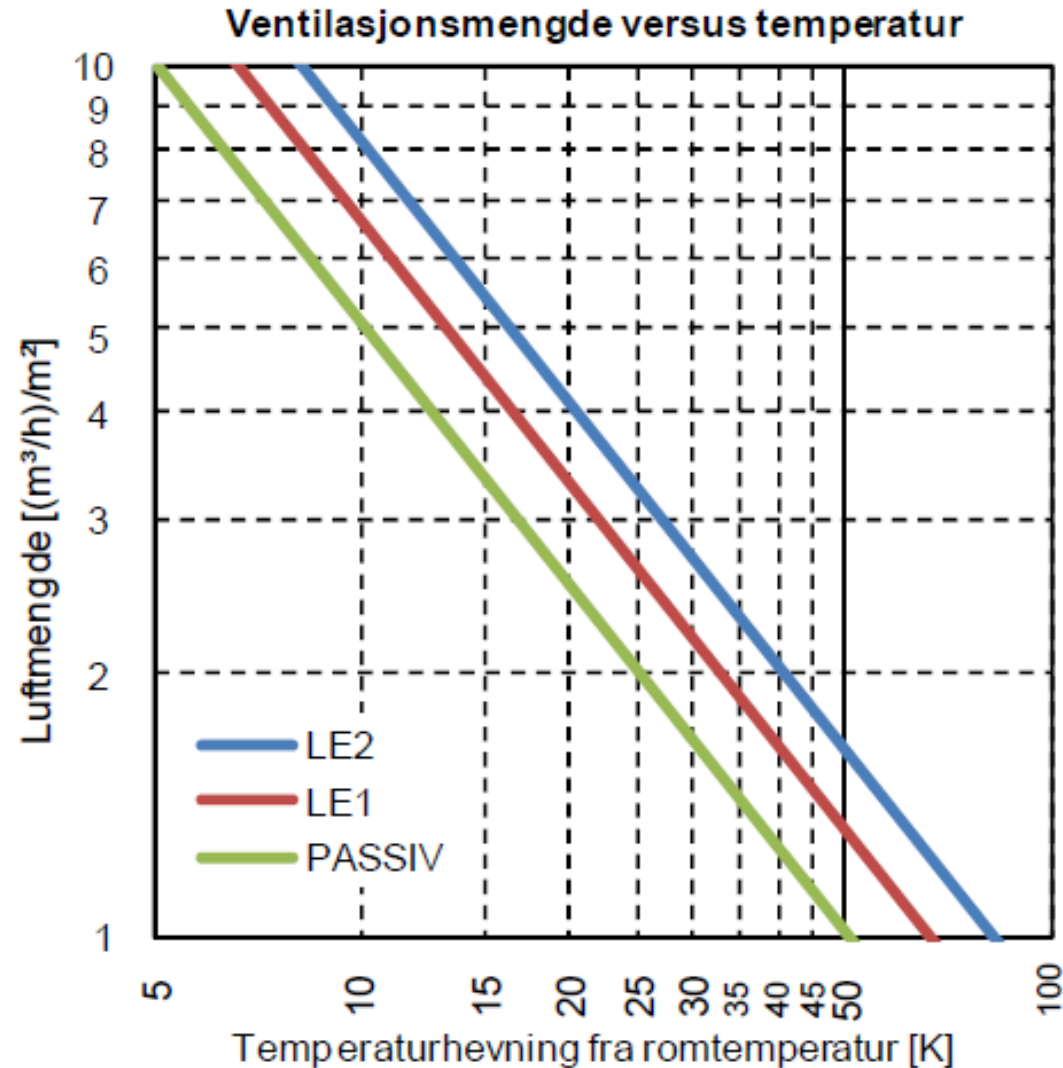
ACR: ~1,6 m³/h/m²

Required T_{supply}: ~50°C

Schools: ACR: ~8 m³/h/m²

Office: ACR: ~6 m³/h/m²

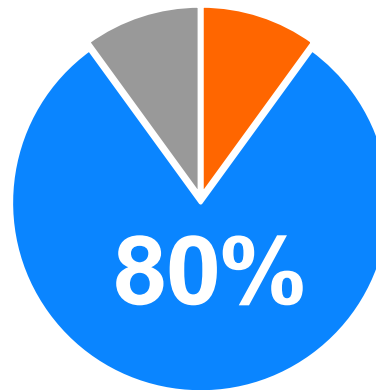
Required T_{supply}: ~26-28°C



Economic impact of Indoor Climate

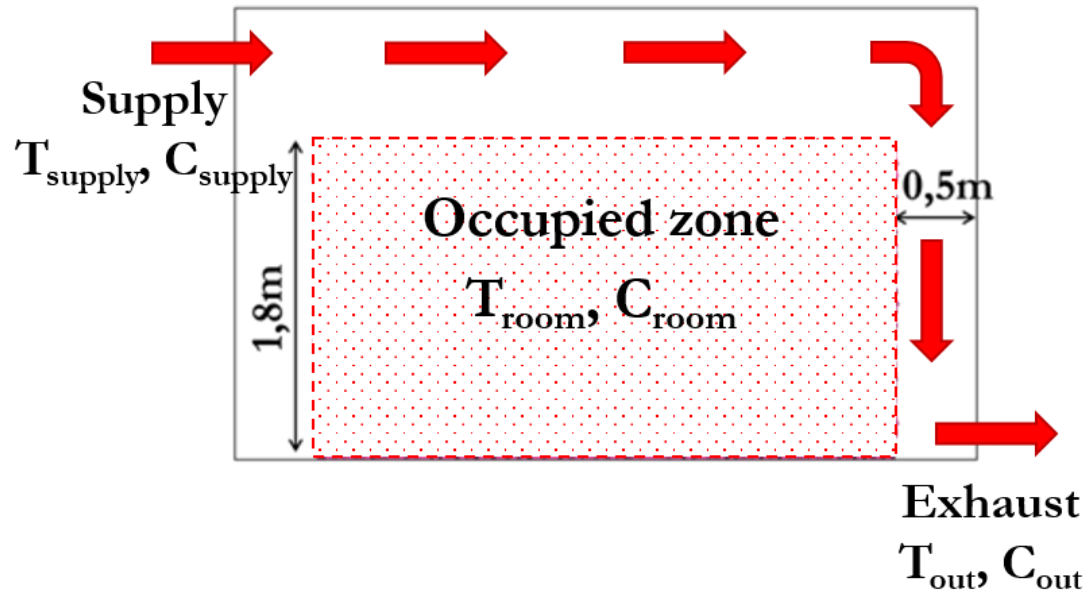
Operating expense in a typical office building (*REHVA Guidebook 17, 2012*)

- Building related cost
- Salary and employee related cost
- Miscellaneous cost



Thermal comfort/IAQ is the priority!

Ventilation efficiency



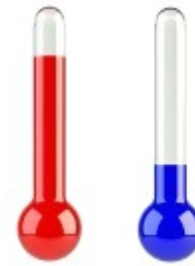
- Contaminant removal effectiveness (*Sandberg, 1981*)

$$\text{CRE} = \frac{C_{\text{out}} - C_{\text{supply}}}{C_{\text{room}} - C_{\text{supply}}}$$

- Temperature efficiency (*Awbi et Gan, 1993*)

$$\epsilon_T = \frac{T_{\text{out}} - T_{\text{supply}}}{T_{\text{room}} - T_{\text{supply}}}$$

Thermal comfort



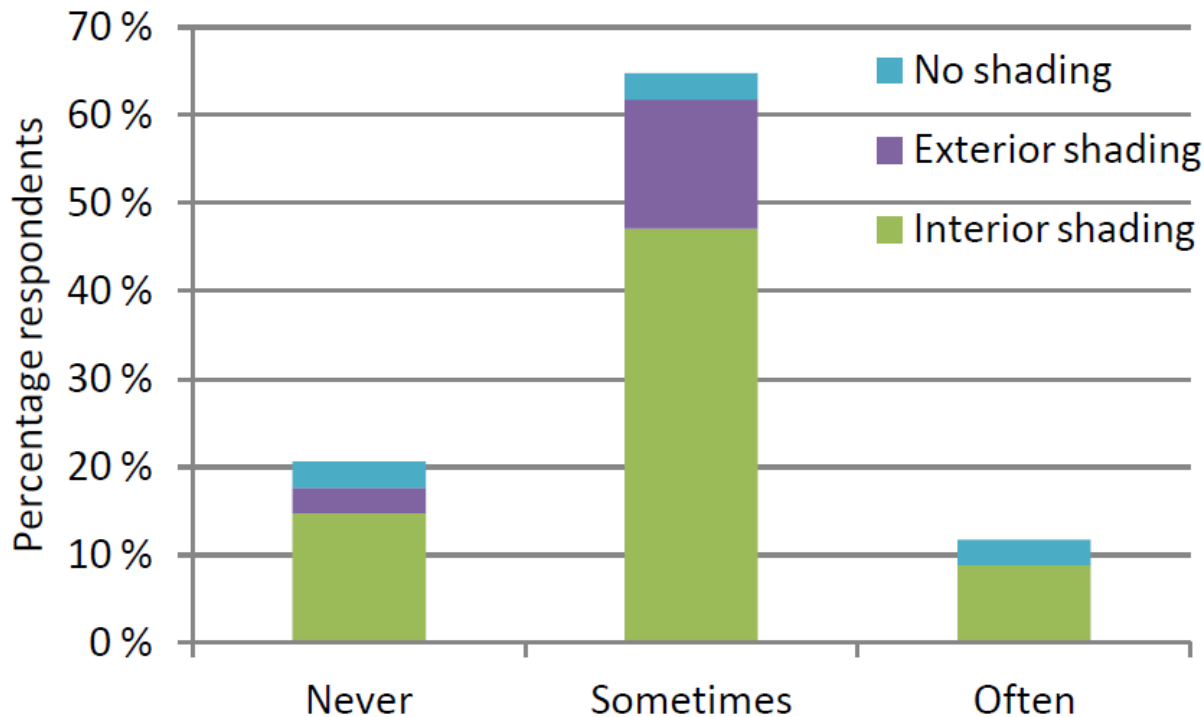
EN15251 / ISO7730 :

- Operative temperature heating: $T_{op} = 21,5^{\circ}\text{C}$
- Operative temperature cooling: $T_{op} = 25,5^{\circ}\text{C}$

- Maximum air velocity heating: $V_{air} < 0,15 \text{ m/s}$
- Maximum air velocity cooling: $V_{air} < 0,25 \text{ m/s}$
- Thermal stratification $< 4,2^{\circ}\text{C}$ between 0,1 and 1,7m

Overheating

Løvåshagen, Bergen: percentage of dissatisfied by overheating
(Berge&Mathisen, 2013)



VABAT Project (2009-2013)



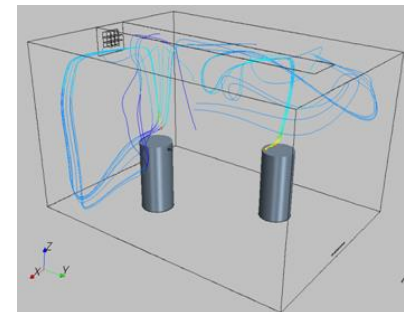
- Low energy buildings
- Is the system relevant?
 - Thermal comfort
 - Ventilation efficiency
 - Airflow patterns
- Energy refurbishment



Methods

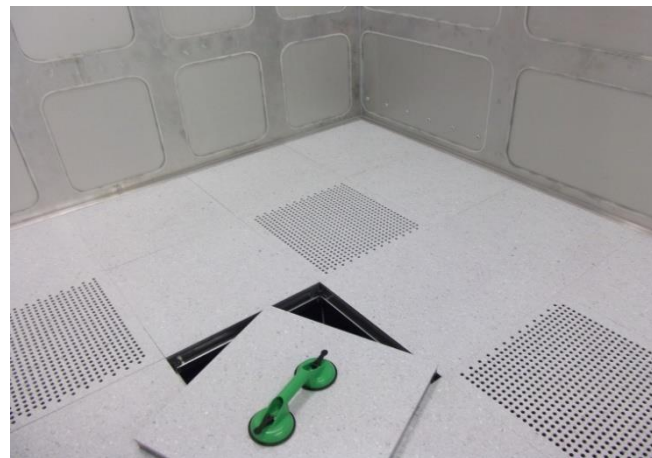
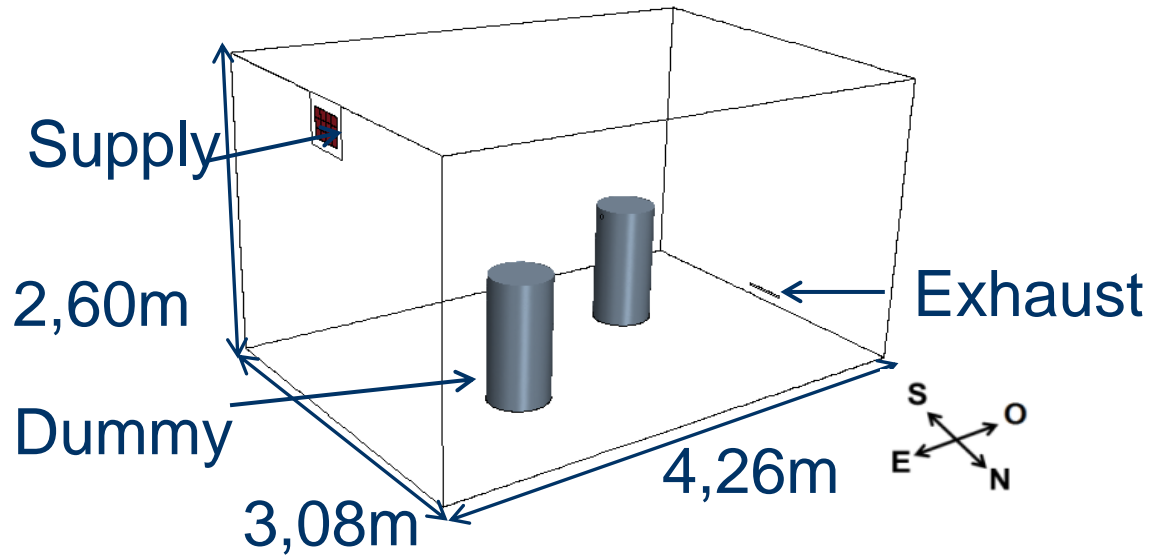


**Measurements
(climate chamber)**



CFD simulations

Test chamber



Sensors

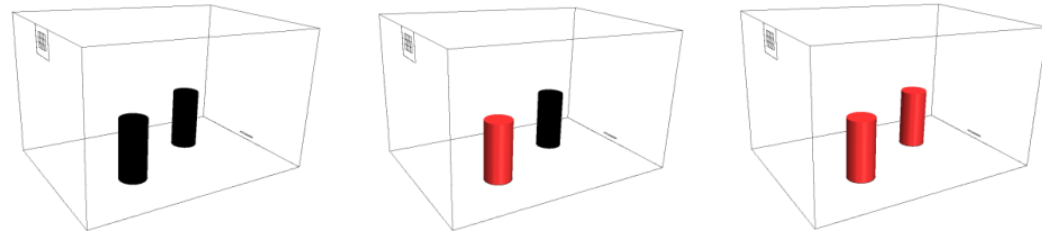
~180 sensors

- Occupied zone: 27 stations
- Boundary conditions (walls, supply, exhaust)

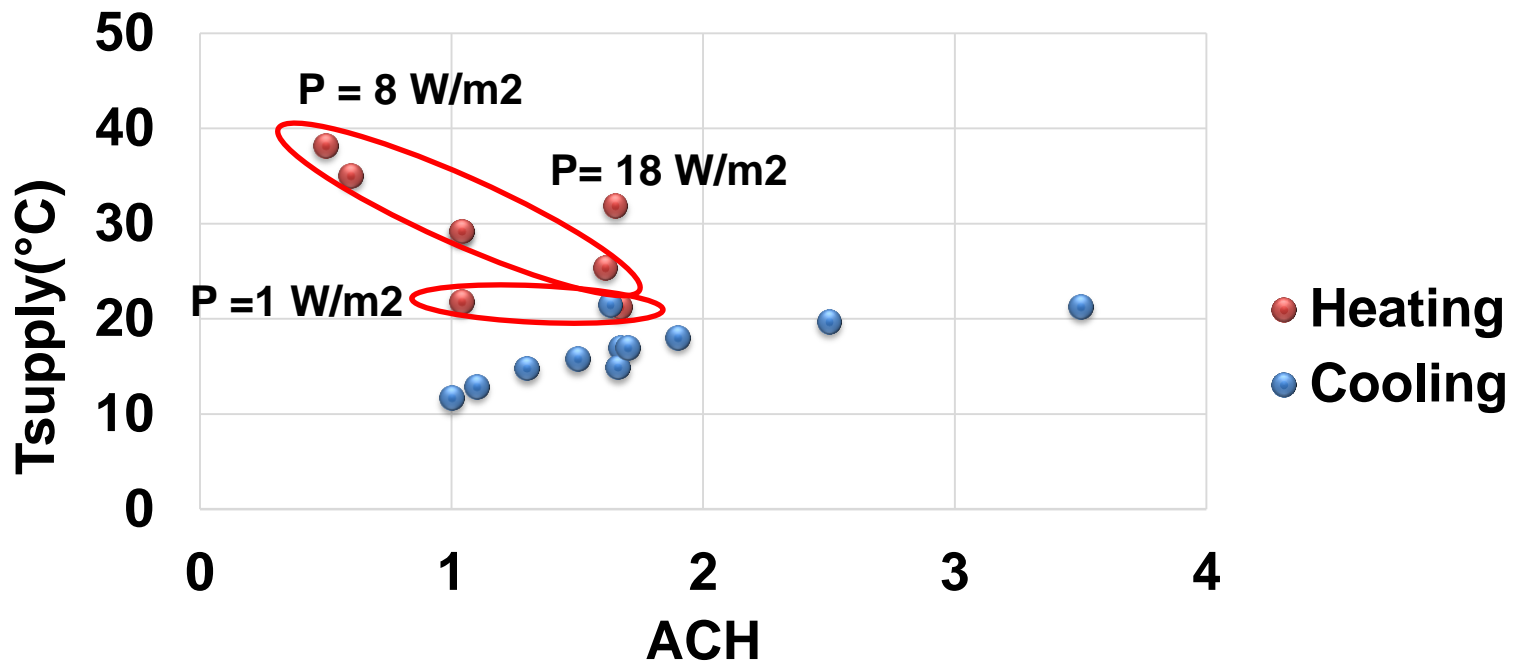


Test cases

- Steady-state
- Heating, Cooling
- 0, 1 or 2 occupants

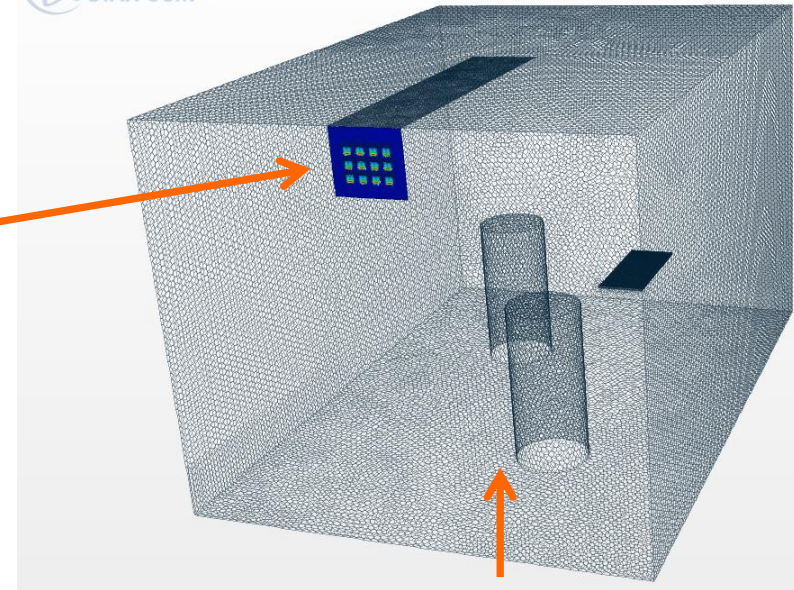
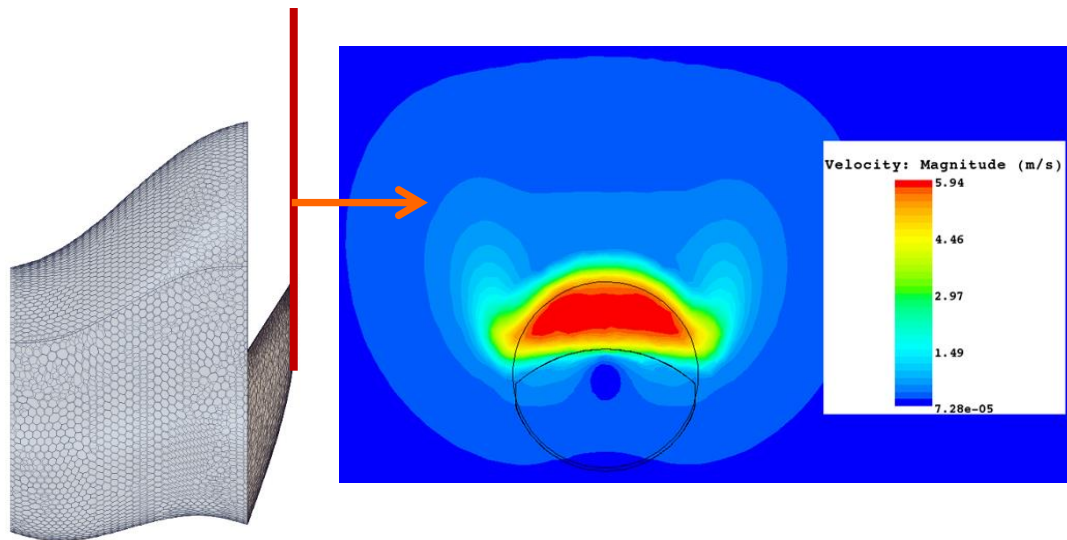


Tested cases



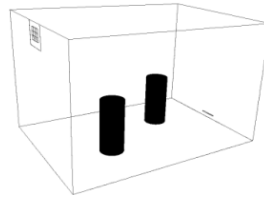
CFD simulations

- STAR-CCM+
- Turbulence modelling
- Air diffuser modelling

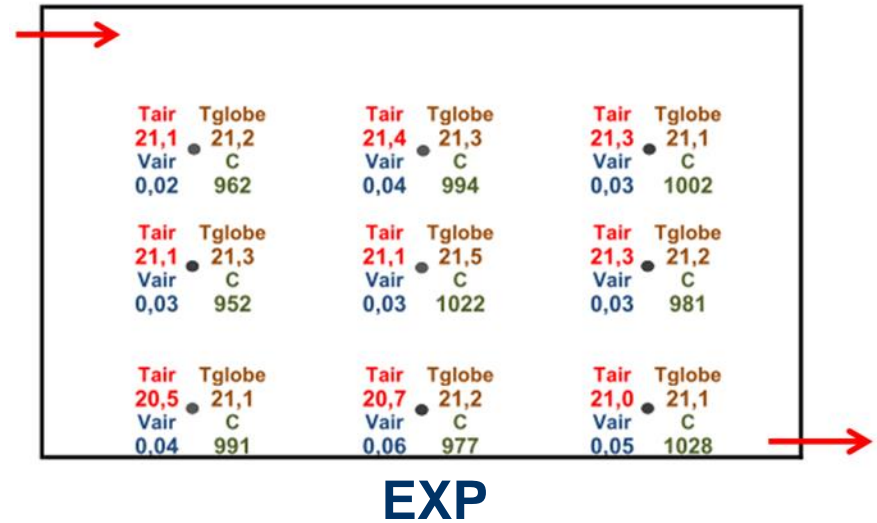
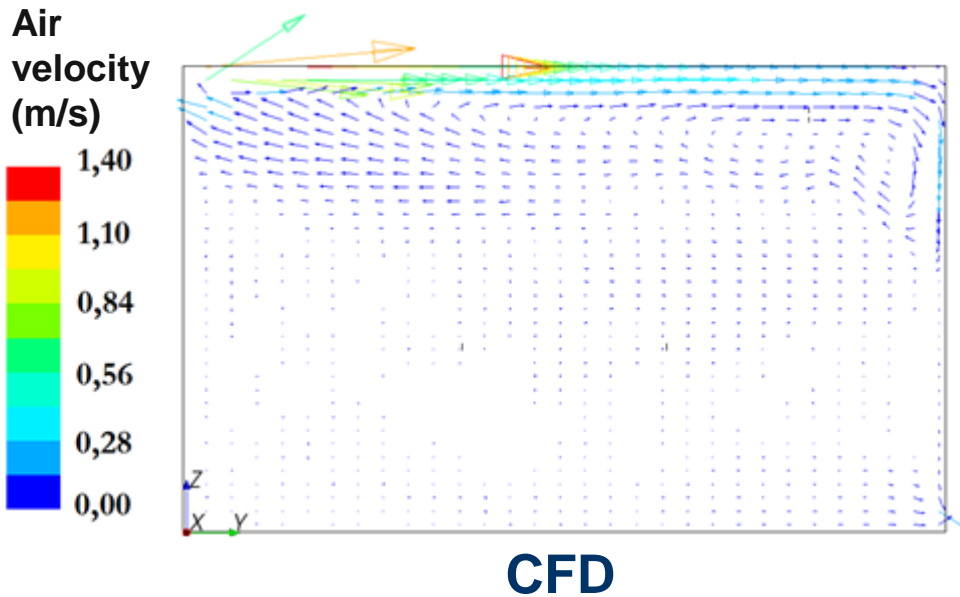


**Boundary conditions
= experiments**

Heating

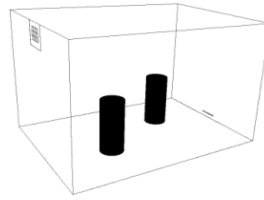


$T_{\text{supply}} = 31,2^{\circ}\text{C}$
 $\text{ACR} = 1,6 \text{ vol/h}$

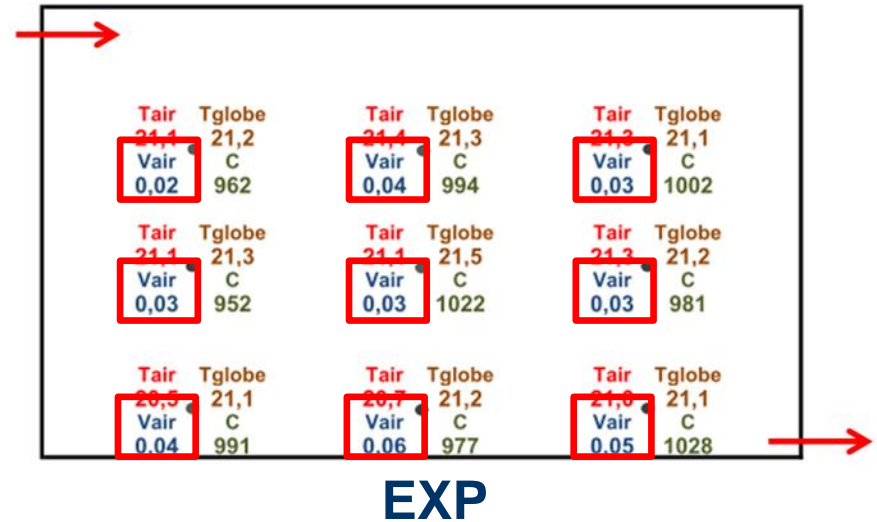
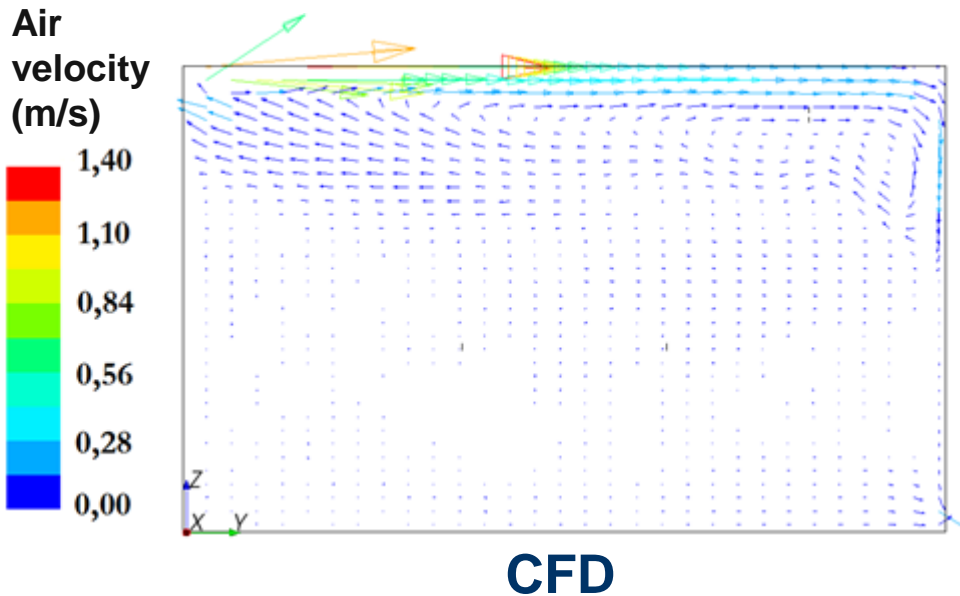


- The warm air jet reaches the opposite wall

Heating



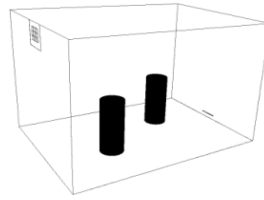
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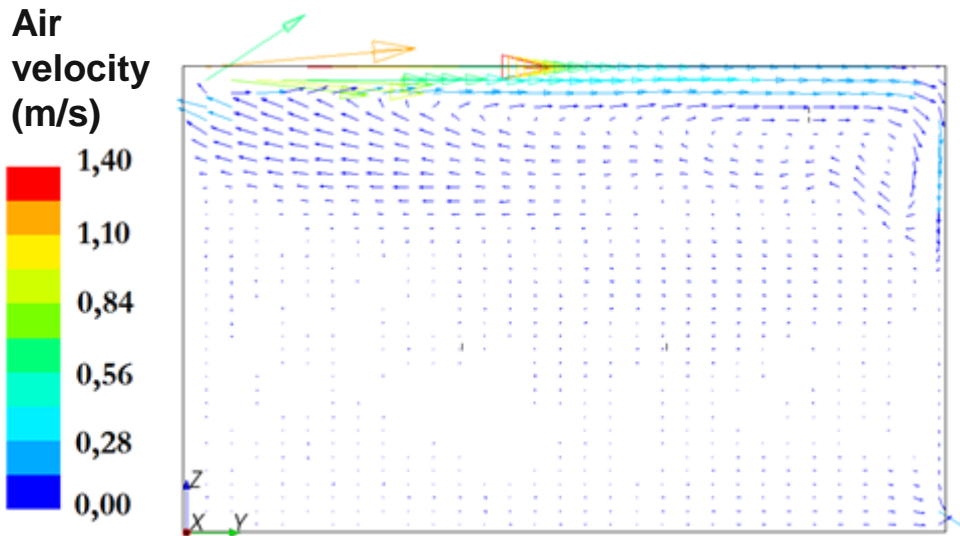
- The warm air jet reaches the opposite wall

$V_{\text{air}} < 0,15 \text{ m/s}$
 No discomfort
 by draught

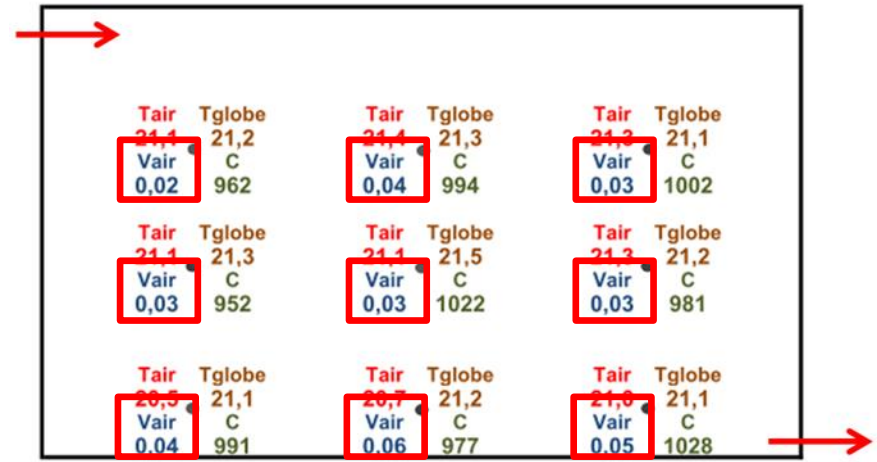
Heating



$T_{\text{supply}} = 31,2^{\circ}\text{C}$
 $\text{ACR} = 1,6 \text{ vol/h}$



CFD

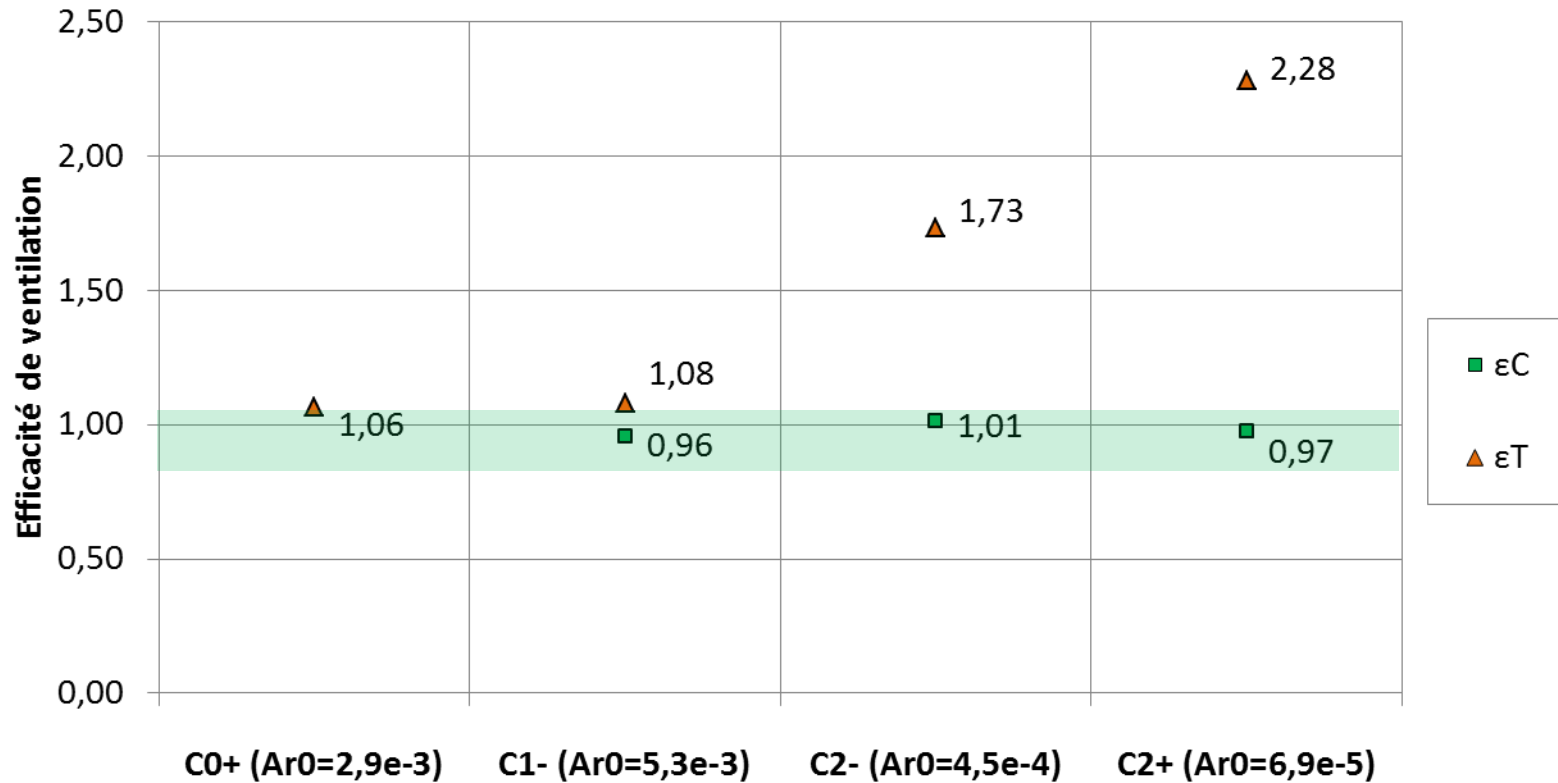


EXP

- The warm air jet reaches the opposite wall
- Short-circuiting?
- Thermal stratification?

$V_{\text{air}} < 0,15 \text{ m/s}$
**No discomfort
 by draught**

Ventilation efficiency

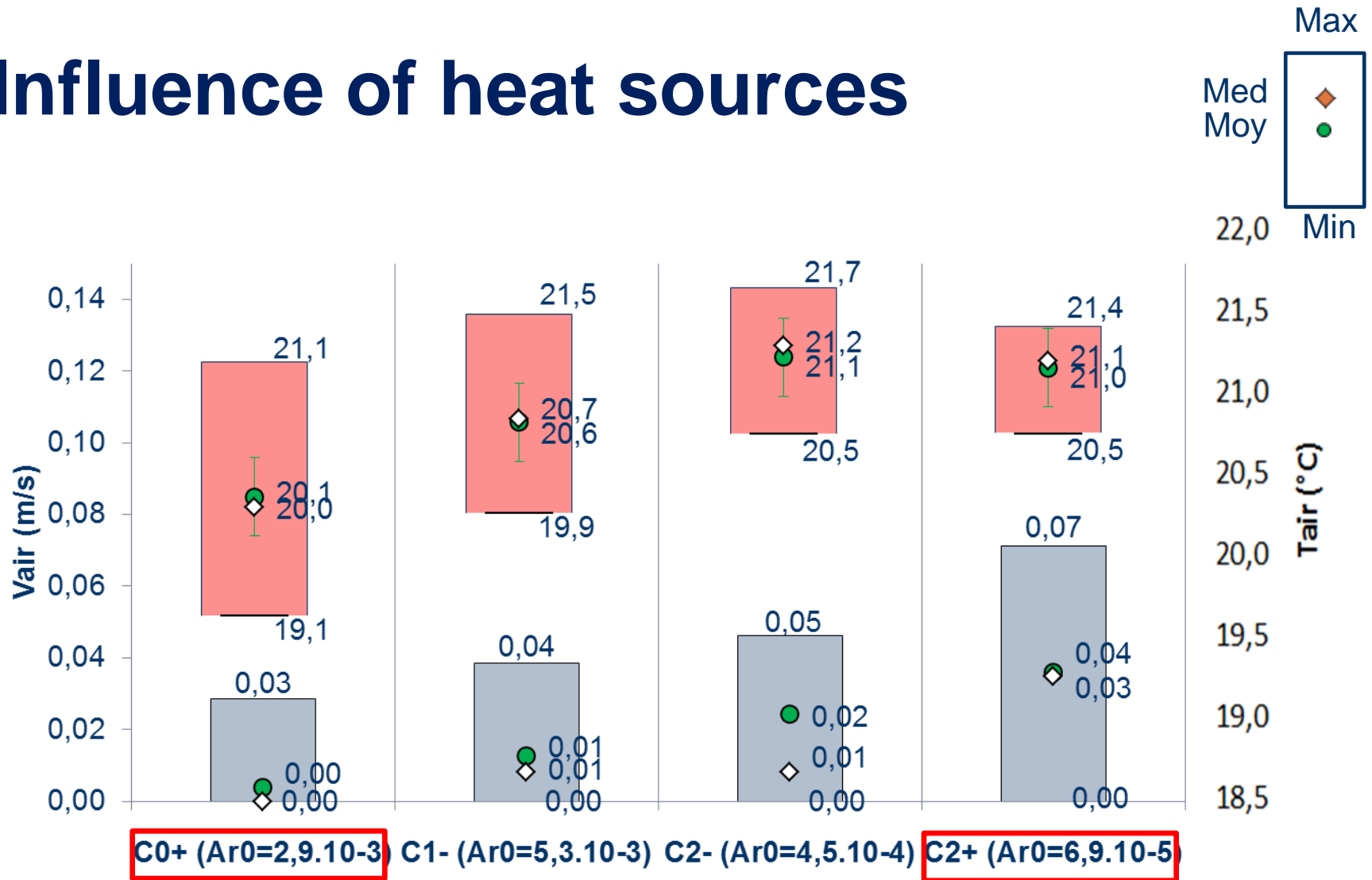


■ CRE close to 1

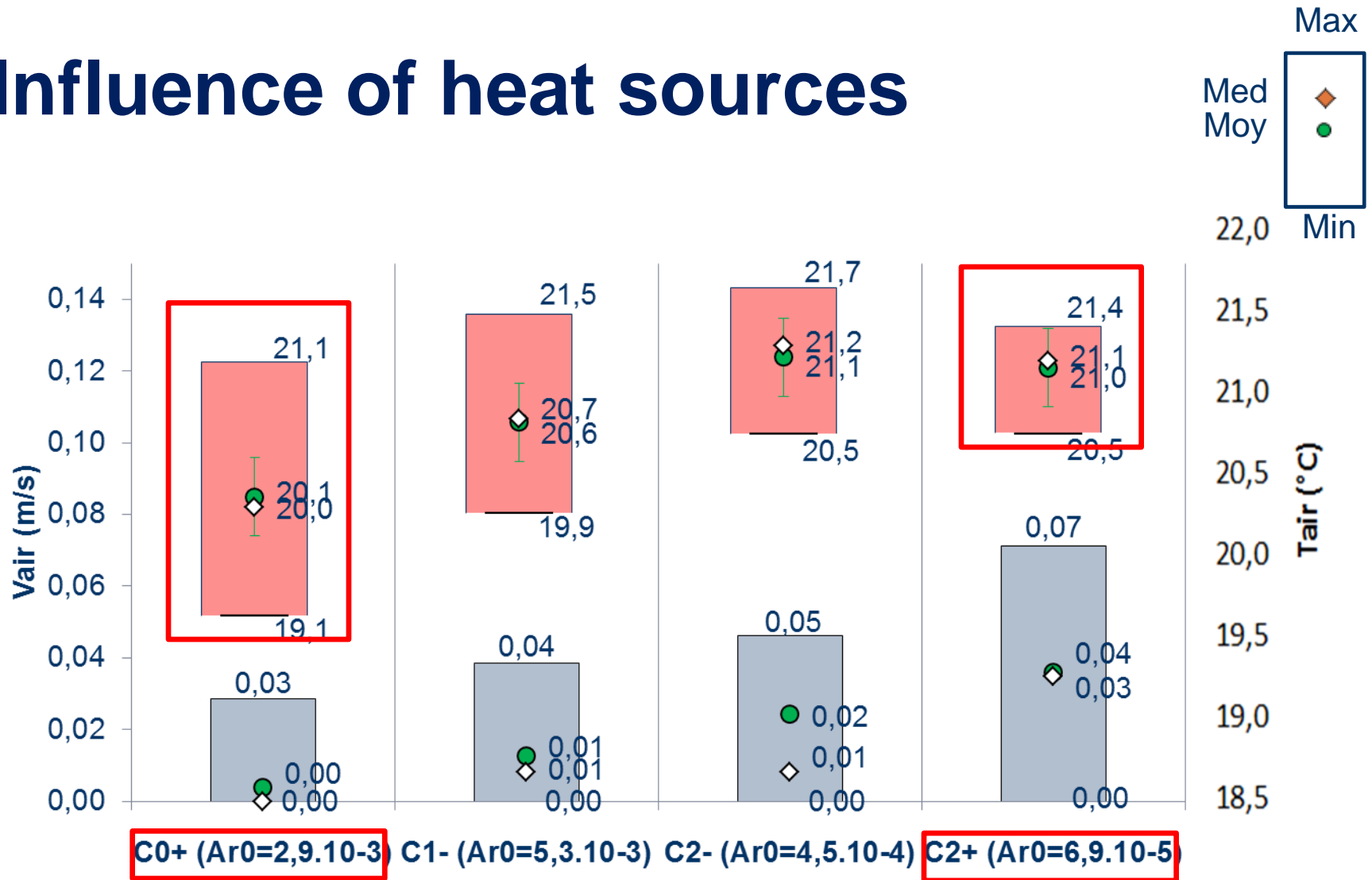
■ $\epsilon_T \geq 1$

Good ventilation efficiency

Influence of heat sources

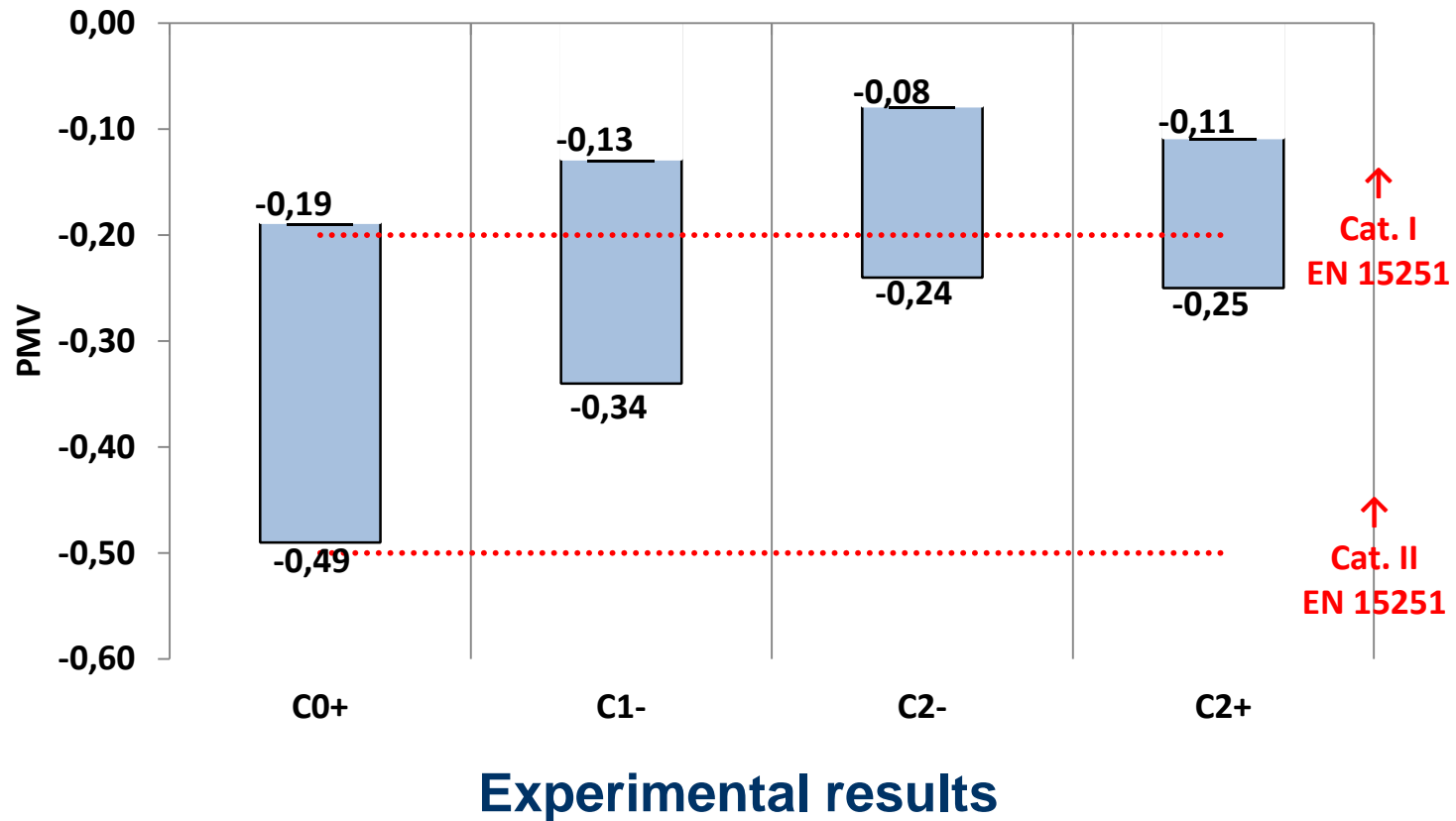


Influence of heat sources



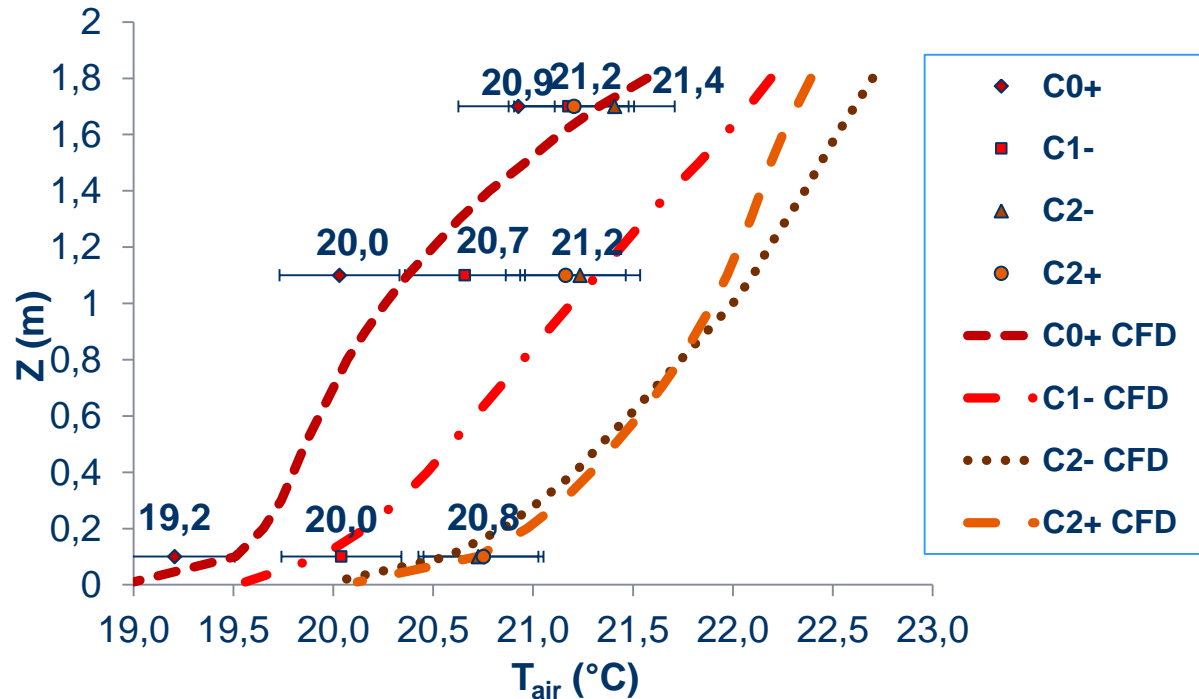
Internal heat sources beneficial

Predicted Mean Vote



- Good values of PMV (EN NS 15251)

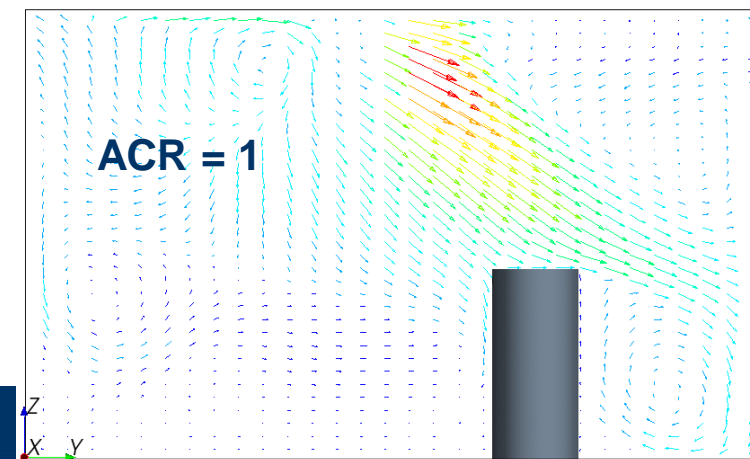
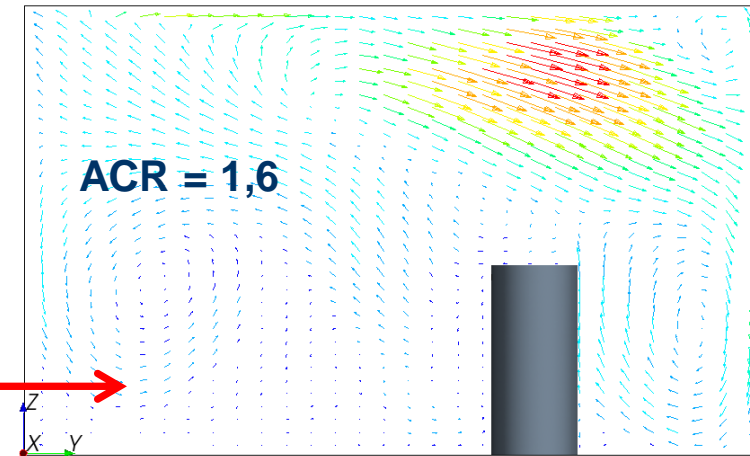
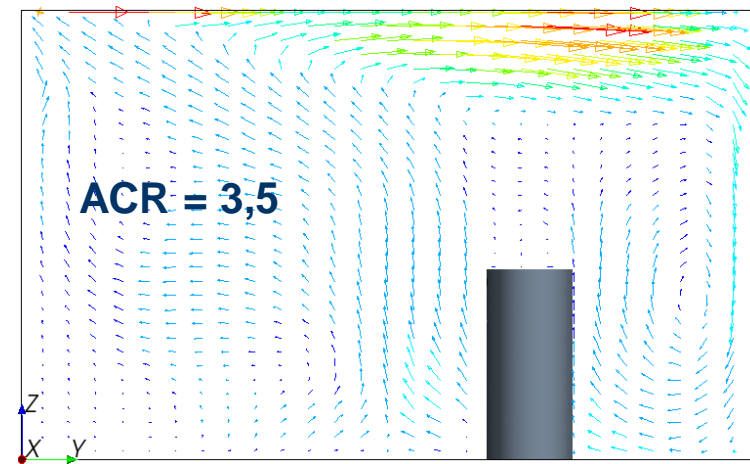
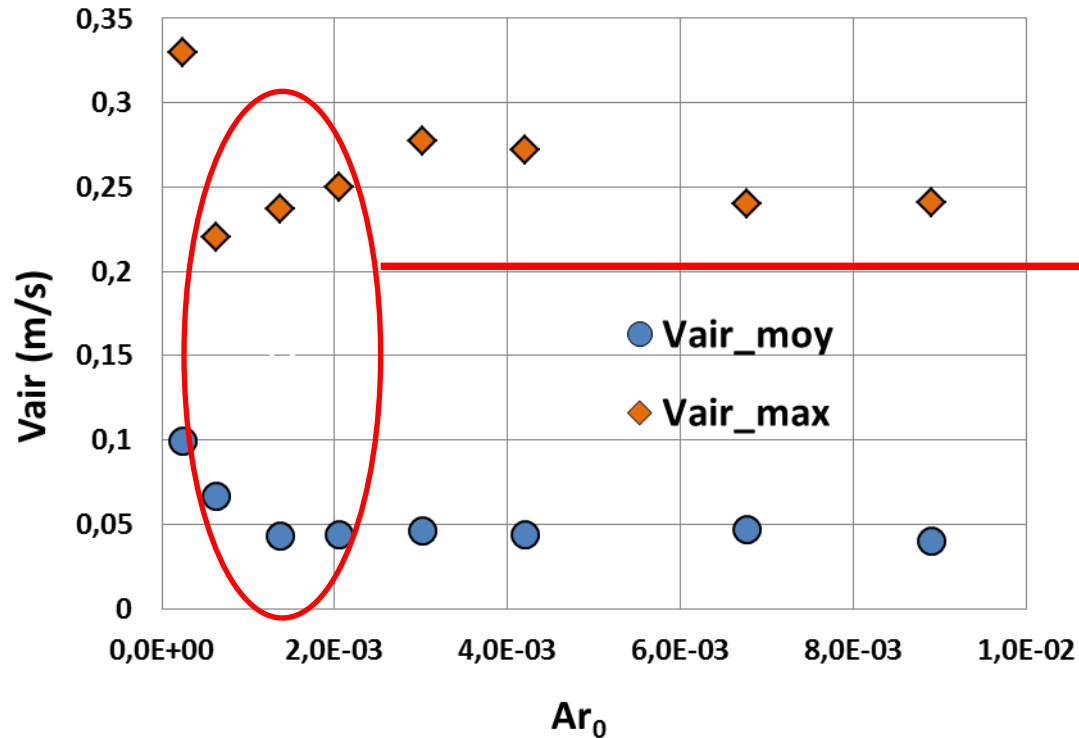
Thermal stratification



$\Delta T \text{ max} = 1,7^\circ\text{C} < 4,2^\circ\text{C}$
No discomfort by thermal stratification

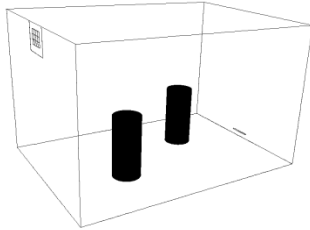
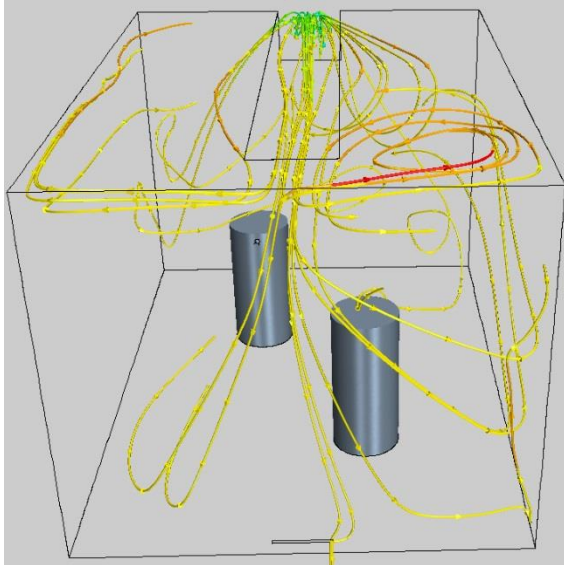
Cooling

- Negative buoyancy forces
- Drop of the jet inside of the occupied zone!

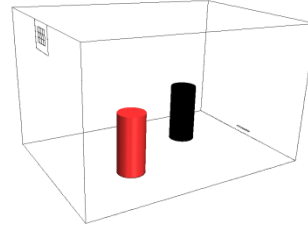
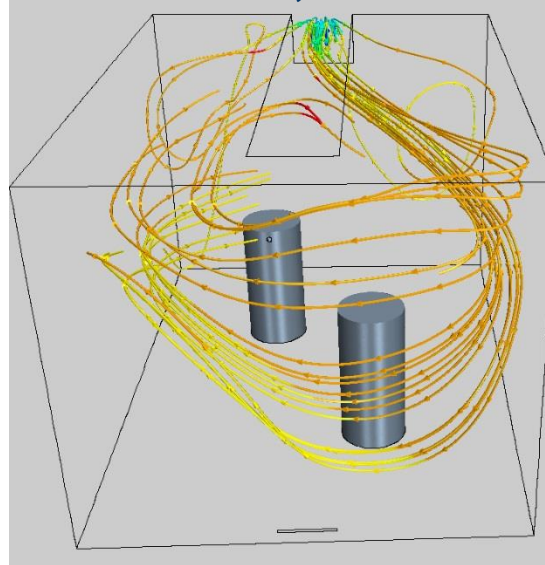


Cooling

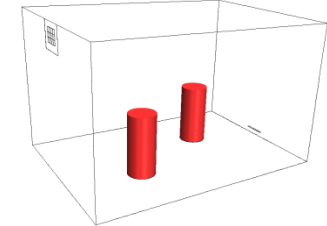
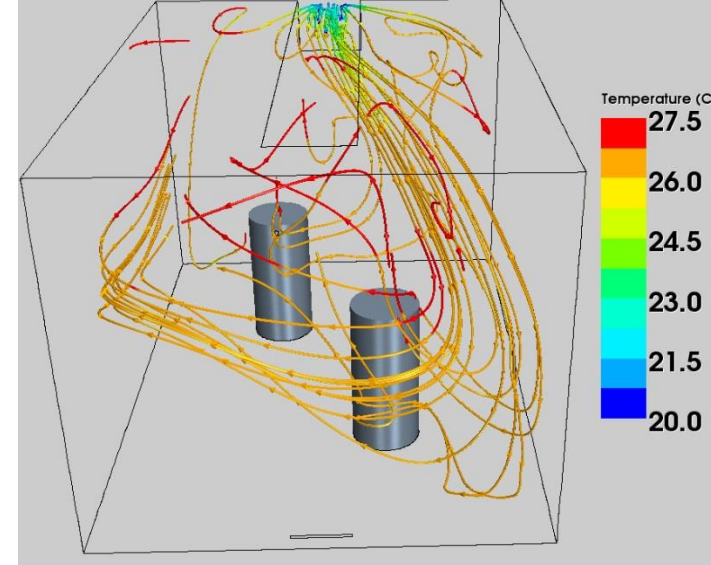
$T_{\text{supply}} = 21,5^{\circ}\text{C}$



$T_{\text{supply}} = 17,0^{\circ}\text{C}$



$T_{\text{supply}} = 14,9^{\circ}\text{C}$



- Deflexion of the air jet
- Tridimensionnall airflow

Negative influence of the heat sources on the airflow

Can DCV replace local heating?

Yes!

If:

- Low heating demand (envelope, climate)
- Educational/office buildings/dwellings
- Well functioning DCV
- Refurbishment
- Overheating (active air diffuser)
- Norway: more measurements needed!

Perspectives

- ForKlima (Forenklet behovsstyrt klimatisering av kontorbygg med svært lavt oppvarmingsbehov) – GK miljøhuset



Perspectives

- Noise level
- Distribution losses in ventilation ducts
- Dry air in winter

Thank you!

Questions?